

Markings such as "ASME," "ASME Standard," or any other marking including "ASME," ASME logos, or the Certification Mark shall not be used on any item that is not constructed in accordance with all of the applicable requirements of the Code or Standard. Use of ASME's name, logos, or Certification Mark requires formal ASME certification; if no certification program is available, such ASME markings may not be used. (For Certification and Accreditation Programs, see https://www.asme.org/shop/certification-accreditation.)

Items produced by parties not formally certified by ASME may not be described, either explicitly or implicitly, as ASME certified or approved in any code forms or other document.

Joed Service of Servic

AN INTERNATIONAL CODE

# 2017 ASME Boiler & Pressure Vessel Code

2017 Edition

# ME BRYC.III. 1. N.F. (ASME BRYC Section 3 Division 1 N.F.) ASME Boiler and Preon Construction ASME Boiler and Preon Construc RULES FOR CONSTRUCTION



Date of Issuance: July 1, 2017

This international code or standard was developed under procedures accredited as meeting the criteria for American National Standards and it is an American National Standard. The Standards Committee that approved the code or standard was balanced to assure that individuals from competent and concerned interests have had an opportunity to participate. The proposed code or standard was made available for public review and comment that provides an opportunity for additional public input from industry, academia, regulatory agencies, and the public-at-large.

ASME does not "approve," "rate," or "endorse" any item, construction, proprietary device, or activity.

ASME does not take any position with respect to the validity of any patent rights asserted in connection with any items mentioned in this document, and does not undertake to insure anyone utilizing a standard against liability for infringement of any applicable letters patent, nor assume any such liability. Users of a code or standard are expressly advised that determination of the validity of any such patent rights, and the risk of infringement of such rights, is entirely their own responsibility.

Participation by federal agency representative(s) or person(s) affiliated with industry is not to be interpreted as government or industry endorsement of this code or standard.

ASME accepts responsibility for only those interpretations of this document issued in accordance with the established ASME procedures and policies, which precludes the issuance of interpretations by individuals.

The endnotes and preamble in this document (if any) are part of this American National Standard.



ASME collective membership mark



Certification Mark

The above ASME symbol is registered in the U.S. Patent Office.

"ASME" is the trademark of The American Society of Mechanical Engineers.

No part of this document may be reproduced in any form, in an electronic retrieval system or otherwise, without the prior written permission of the publisher.

Library of Congress Catalog Card Number: 56-3934 Printed in the United States of America

Adopted by the Council of The American Society of Mechanical Engineers, 1914; latest edition 2017.

The American Society of Mechanical Engineers Two Park Avenue, New York, NY 10016-5990

Copyright © 2017 by
THE AMERICAN SOCIETY OF MECHANICAL ENGINEERS
All rights reserved

# **TABLE OF CONTENTS**

Statement of Policy on the Use of th	xive Certification Mark and Code Authorization in Advertising xive SME Marking to Identify Manufactured Items
	the Boiler and Pressure Vessel Standards Committees x
	xvii
Organization of Section III	
Summary of Changes	x
	0rder xlii
	nges in the Boiler and Pressure Vessel Code xliv
Article NF-1000	Introduction
NF-1100	Scope and General Requirements
NF-1110	Aspects of Construction Covered by These Rules
NF-1120	Rules for Supports and Their Classification
NF-1130	Boundaries of Jurisdiction
NF-1200	Types of Supports and Attachments
NF-1210	Types of Supports
NF-1230	Welding Between Types of Supports
NF-1240	Attachments
NI-1240	
Article NF-2000	Material
NF-2100	General Requirements for Material
NF-2110	Scope of Principal Terms Employed
NF-2120	Support Material
NF-2130	Certification of Material
NF-2140	Welding Material
NF-2150	Material Identification
NF-2160	Deterioration of Material in Service
NF-2170	Heat Treatment to Enhance Impact Properties
NF-2180	Procedures for Heat Treatment of Material
NF-2200	Material Test Coupons and Specimens for Ferritic Steel Material 12
NF-2210	Heat Treatment Requirements
NF-2170 NF-2180 NF-2200 NF-2210 NF-2220 NF-2300 NF-2310 NF-2320 NF-2330 NF-2330	Procedure for Obtaining Test Coupons and Specimens for Quenched and
0	Tempered Material
NF-2300	Fracture Toughness Requirements for Material
NF-2310	Material to Be Impact Tested
NF-2320	Impact Test Procedures
NF-2330	Test Requirements and Acceptance Standards 16
NF-2340	Number of Impact Tests Required
NF 2350	Retests
NF-2360	Calibration of Instruments and Equipment
NF-2400	Welding Material
NF-2410	General Requirements
NF-2420	Required Tests
NF-2430	Weld Metal Tests
NF-2440	Storage and Handling of Welding Material
NF-2500	Examination and Repair of Material
NF-2510	Examination and Repair of Material Other Than Bolting
NF-2520	Required Examination and Acceptance Standards for Class 1 Springs 24

NF-2530	Required Examination and Acceptance Standards for Coiled Wire Rope	
	1 0	4
NF-2580	Examination of Bolting Material and Rods	4
NF-2600		6
NF-2610	Documentation and Maintenance of Quality System Programs 2	6
Article NF-3000	<b>Design</b>	7
NF-3100		7
NF-3110		7
NF-3120	Design Considerations	
NF-3130		9
NF-3140		0
NF-3200	Design Rules for Plate, and Shell-Type Supports	1
NF-3210	31	1
NF-3210 NF-3220	Design by Analysis for Class 1	1
NF-3250		8
NF-3260	Design by Analysis for Class 2 dilu MC	1
NF-3270	r	1
NF-3280		1
NF-3300	Design Rules for Linear-Type Supports 4	3
NF-3310		3
NF-3320	Design by Linear Elastic Analysis for Class 4	6
NF-3330		9
NF-3340	Limit Analysis Design for Class 1 7	2
NF-3350	Design by Analysis for Class 2 and MC	7
NF-3360		8
NF-3370		8
NF-3380		8
NF-3400		9
NF-3410		9
NF-3420		0
NF-3450		2
NF-3460		2
NF-3470	Design by Experimental Stress Analysis	2
NF-3480		2
NF-3500	Design Rules for Component Supports 8	2
	Design Rules for Component Supports	2
NF-3500	Design Rules for Component Supports	2
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2, 3, and MC8	2 2 2 2 3
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2, 3, and MC8	2 2 2
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8	2 2 2 2 3
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8 Design by Load Rating 8	2 2 2 3 3
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8 Design by Load Rating 8 Design Rules for Piping Supports 8	2 2 2 3 3 3
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2, 3, and MC8Experimental Stress Analysis8Design by Load Rating8Design Rules for Piping Supports8General Requirements8	2 2 2 2 3 3 3 3
NF-3500 NF-3510 NF-3520 NE-3550	Design Rules for Component Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2, 3, and MC8Experimental Stress Analysis8Design by Load Rating8Design Rules for Piping Supports8General Requirements8Design by Analysis for Class 18	2 2 2 3 3 3 3 3 3
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620	Design Rules for Component Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2, 3, and MC8Experimental Stress Analysis8Design by Load Rating8Design Rules for Piping Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2 and 38	2 2 2 3 3 3 3 4
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650	Design Rules for Component Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2, 3, and MC8Experimental Stress Analysis8Design by Load Rating8Design Rules for Piping Supports8General Requirements8Design by Analysis for Class 18Design by Analysis for Class 2 and 38Experimental Stress Analysis8	2 2 2 3 3 3 3 4 4
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3670	Design Rules for Component Supports General Requirements Design by Analysis for Class 1  Design by Analysis for Class 2, 3, and MC  Experimental Stress Analysis  Design by Load Rating  Besign Rules for Piping Supports General Requirements  Design by Analysis for Class 1  Besign by Analysis for Class 2 and 3  Experimental Stress Analysis  Design by Load Rating  Besign by Analysis for Class 2 and 3  Experimental Stress Analysis  Besign by Load Rating	2 2 2 3 3 3 3 4 4 5
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3670 NF-3680	Design Rules for Component Supports General Requirements Design by Analysis for Class 1  Design by Analysis for Class 2, 3, and MC  Experimental Stress Analysis  Design by Load Rating  Besign Rules for Piping Supports  General Requirements  Design by Analysis for Class 1  Besign by Analysis for Class 1  Besign by Analysis for Class 2 and 3  Experimental Stress Analysis  Besign by Load Rating	2 2 2 3 3 3 3 4 4 5 5
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3650 NF-3670 NF-3680 Article NF-4000	Design Rules for Component Supports General Requirements Besign by Analysis for Class 1 Besign by Analysis for Class 2, 3, and MC Experimental Stress Analysis Besign by Load Rating Besign Rules for Piping Supports General Requirements Besign by Analysis for Class 1 Besign by Analysis for Class 1 Besign by Analysis for Class 2 and 3 Experimental Stress Analysis Besign by Load Rating Besign Bes	2 2 2 3 3 3 3 4 4 5 5 6
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3650 NF-3670 NF-3680 Article NF-4000 NF-4100	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8 Design by Load Rating 8 Design Rules for Piping Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2 and 3 8 Experimental Stress Analysis 8 Design by Load Rating 8 Fabrication and Installation 8 General Requirements 8 Introduction 8	2 2 2 3 3 3 3 4 4 5 5 6 6
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3670 NF-3670 NF-3680  Article NF-4000 NF-4100 NF-4110	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8 Design by Load Rating 8 Design Rules for Piping Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2 and 3 8 Experimental Stress Analysis 8 Design by Load Rating 8 Fabrication and Installation 8 General Requirements 8 Introduction 8 Certification of Materials and Fabrication by Support Certificate Holder 8	2 2 2 2 3 3 3 3 3 4 4 5 5 6 6 6
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3670 NF-3670 NF-3680  Article NF-4000 NF-4100 NF-4110 NF-4120	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8 Design by Load Rating 8 Design Rules for Piping Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2 and 3 8 Experimental Stress Analysis 9 Experimental Stress Analysis 9 Design by Load Rating 8  Fabrication and Installation 8 General Requirements 8 Introduction 8 Certification of Materials and Fabrication by Support Certificate Holder 8 Repair of Material 8	2 2 2 2 3 3 3 3 3 4 4 5 5 6 6 6 6
NF-3500 NF-3510 NF-3520 NF-3550 NF-3570 NF-3580 NF-3600 NF-3610 NF-3620 NF-3650 NF-3650 NF-3680  Article NF-4000 NF-4110 NF-4120 NF-4130	Design Rules for Component Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2, 3, and MC 8 Experimental Stress Analysis 8 Design by Load Rating 8 Design Rules for Piping Supports 8 General Requirements 8 Design by Analysis for Class 1 8 Design by Analysis for Class 2 and 3 8 Experimental Stress Analysis 9 Design by Load Rating 8 Design by Analysis for Class 2 and 3 8 Experimental Stress Analysis 9 Design by Load Rating 8 Fabrication and Installation 8 General Requirements 8 Introduction 8 Certification of Materials and Fabrication by Support Certificate Holder 8 Repair of Material 8 Forming, Fitting, and Aligning 8	2 2 2 2 3 3 3 3 4 4 5 5 6 6 6 6 6
NF-3500 NF-3510 NF-3520 NF-3550 NF-3550 NF-3570 NF-3680 NF-3610 NF-3620 NF-3650 NF-3670 NF-3680  Article NF-4000 NF-4110 NF-4120 NF-4130 NF-4200	Design Rules for Component Supports General Requirements Design by Analysis for Class 1 Design by Analysis for Class 2, 3, and MC Experimental Stress Analysis Design by Load Rating Design Rules for Piping Supports General Requirements Design by Analysis for Class 1 Design by Analysis for Class 1 Design by Analysis for Class 2 and 3 Experimental Stress Analysis Design by Load Rating  Fabrication and Installation General Requirements  Introduction Certification of Materials and Fabrication by Support Certificate Holder Repair of Material Forming, Fitting, and Aligning Cutting, Forming, and Bending	2 2 2 2 3 3 3 3 3 4 4 5 5 6 6 6 6 6 7

NF-4240	Requirements for Welded Joints	89
NF-4300	Welding Qualifications	89
NF-4310	General Requirements	89
NF-4320	Welding Qualifications and Records	89
NF-4330	General Requirements for Welding Procedure Qualification Tests	90
NF-4400	Rules Governing Making and Repairing Welds	92
NF-4410	Precautions to Be Taken Before Welding	92
NF-4420	Rules for Making Welded Joints	92
NF-4430	Welding of Attachments	93
NF-4440	Special Requirements for Welding	93
NF-4450	Special Requirements for Welding	93
NF-4500	Brazing	95
NF-4510	Rules for Brazing	95
NF-4600	Heat Treatment	96
NF-4610	Welding Preheat and Interpass Requirements	96
NF-4620	Postweld Heat Treatment	96
NF-4660	Heat Treatment of Electroslag Welds	101
NF-4700	Requirements for Bolted Construction	101
NF-4710	Bolting and Threading	101
NF-4720	Bolting	101
	Dokumg	
Article NF-5000	Examination	104
NF-5100	General Requirements for Examination	104
NF-5110	Procedures, Qualifications, and Evaluations	104
NF-5120	Time of Examination of Welds	104
NF-5200	Required Examination of Welds*	104
NF-5210	Examination of Class 1 Support Welds	104
NF-5220	Examination of Class 2 and MC Support Welds	105
NF-5230	Examination of Class 3 Support Welds	105
NF-5240	Inertia and Continuous Drive Friction Welds	105
NF-5300	Acceptance Standards	106
NF-5320	Radiographic Acceptance Standards	106
NF-5330	Ultrasonic Acceptance Standards	106
NF-5340	Magnetic Particle Acceptance Standards	106
NF-5350	Liquid Penetrant Acceptance Standards	107
NF-5360	Visual Acceptance Standards	107
NF-5500	Qualifications and Certification of Nondestructive Examination	
	Personnel	108
NF-5510	General Requirements	108
	Personnel Qualification, Certification, and Verification	108
NF-5520 NF-5530	Records	109
1		
Article NF-8000	Certificates of Authorization and Certification Documents	110
NF-8100	General Requirements	110
Mandatory Appendix NF-I	Additional Material for Supports	111
Article NF-I-1000	Introduction	111
NF-1-1100	Scope	111
NF-I-1200	•	
HF-1-1200	General Requirements	111
Article NF-I-2000	Materials	112
NF-I-2100	General	112
NF-I-2200	Requirements	112
NF-I-2210	*	112
NF-I-2220		112
NF-I-2230		112
NF-I-2240		112

Article NF-I-3000	Welding	145
NF-I-3100	General	145
NF-I-3110		145
NF-I-3200	Welding Qualifications	145
NF-I-3210		145
NF-I-3220		145
NF-I-3300	Additional Requirements	145
NF-I-3310	·	145
NF-I-3320		145
NF-I-3330		145
NF-I-3340		145
NF-I-3400	Requirements for S-Number Materials	145
NF-I-3410		145
NF-I-3500	Specific Requirements for S-Number 12 Group Number 1 Materials	145
NF-I-3510	Preheat	145
NF-I-3520	Postweld Heat Treatment	145
NF-I-3530	Impact Test Requirements	146
Mandatory Appendix NF-II	Design of Single Angle Members	147
Article NF-II-1000	Introduction Scope	147
NF-II-1100	Scano	147
NF-II-1200	General Requirements	147
Article NF-II-2000	Nomenclature	148
Article NF-II-3000	Design Requirements General	149
NF-II-3100	General	149
NF-II-3200	Tension	149
NF-II-3300	Shear	149
NF-II-3400	Compression	149
NF-II-3500	Flexure	150
NF-II-3500 NF-II-3510		150
	Allowable Bending Stress	
NF-II-3520	Bending About Geometric Axes	151 152
NF-II-3530	Bending About Principal Axes	
NF-II-3600	Combined Stresses	152
NF-II-3610	Axial Compression and Flexure	152
NF-II-3620	Axial Tension and Bending	153
Mandatory Appendix NF-III	Energy-Absorbing Support Material	154
Article NF-III-1000	General Requirements	154
Article NF-III-2000	Material Requirements	155
Article NF-III-3000	Design	156
Article NF-III-4000	Fabrication	157
Nonmandatory Appendix NF-A	Structural Bolt Preloading	158
Article NF-A-1000	Introduction	158
NF-A-1100		158
NF A 1200	Scope	
NF-A-1300	Discussion	158
1-	Bolt Tightening (Preload Methods)	159
✓ NF-A-1310	Torque Wrenches	159
NF-A-1320	Turn-of-the-Nut	160
NF-A-1330	Load-Indicating Washers	160
NF-A-1340	Strain Gages	161
NF-A-1350	Fastener Elongation/Extension	161

Nonmandatory Appendix NF-B	Design Allowable Stresses for Plate and Shell and Linear-Type Supports	162
Article NF-B-1000	Introduction	162
NF-B-1100	Objective	162
NF-B-1200	Scope	162
Article NF-B-2000	Linear Supports — Method for Establishing Allowable Stresses	163
NF-B-2100	Responsibility for Assigning Yield and Ultimate Strength Values	163
NF-B-2200	Tension Member Allowable Stress	163
NF-B-2300	Beam Bending Allowable Stress	163
NF-B-2400	Compression Member Allowable Stress	163
NF-B-2500	Beam-Column Allowable Stress	164
NF-B-2600	Bolting Allowable Stress	164
Article NF-B-3000	Plate and Shell Supports — Allowable Stresses	166
Article NF-B-4000	Materials With Low Ratios of Tensile to Yield Strength	167
Article NF-B-5000	Special Consideration	168
NF-B-5100	Copper-Based Alloys	168
NF-B-5200	Stainless Steel	168
NF-B-5300	Nonductile Behavior and Stress Corrosion Cracking	168
Nonmandatory Appendix NF-C	Design Basis for Linear-Type Supports	169
Article NF-C-1000	Introduction	169
NF-C-1100	Objective	169
NF-C-1200	Scope	169
Article NF-C-2000	Material Availability and Requirements Differences	170
Article NF-C-3000	AISC Paragraph Selections	171
Article NF-C-4000	Cross-Reference to AISC	172
Nonmandatory Appendix NF-D	Tolerances	175
Article NF-D-1000	Introduction	175
NF-D-1100	Objective	175
NF-D-1200	Scope	175
NF-D-1300	Seneral Requirements	175
NF-D-1310	Material	175
NF-D-1320 NF-D-1330	Fabrication Tolerances	175
C.W	Installation Tolerances	175
Nonmandatory Appendix NF-E	Dampers, Energy Absorbers, and Snubbers	181
Article NF-E-1000	15.6.1	181
NF-E-1100	Scope and Definitions	181
NF-E-1110	Scope	181
NF-E-1120 NF-E-1200	Definitions	181 181
NF-E-1210	Dampers Description	181
NF-E-1220	Function	181
NF-E-1230	Applications	181
NF-E-1240	Modeling	181
NF-E-1300	Energy Absorbers	181
NF-E-1310	Description	181
NF-E-1320	Function	182
NF-E-1330	Applications	182
NF-E-1340	Modeling	182

NF-E-1400 NF-E-1410	Snubbers  Description	182 182	
NF-E-1420	Function	182	
NF-E-1430	Applications	182	
NF-E-1440	Modeling		
Nr-E-1440	Modeling	182	
FIGURES			
NF-1132-1	Typical Examples of Jurisdictional Boundaries Between Supports and the Building		
	Structure	3 <sup>3</sup> 7	
NF-1214-1	Typical Standard Support Units	7	
NF-2311(b)-1	Impact Test Exemption Curves for Materials for Class 2, 3, and MC Supports	15	
NF-2331(a)-1	Required C <sub>v</sub> Energy Values for Class 2 and MC Support Material Other Than Bolting	19	
NF-2331(a)-2	Required C <sub>v</sub> Energy Values for Class 3 Support Material Other Than Bolting	20	
NF-2433.1-1	Delta Ferrite Content	25	
NF-3226.1-1	Permissible Welded Joints for Class 1 Plate- and Shell-Type Supports	35	
NF-3226.1(a)-1	Typical Weld Details Referenced	36	
NF-3226.1(a)-2	Butt Welding of Plates of Unequal Thickness	37	
NF-3256.1-1	Permissible Welded Joints for Class 2, 3, and MC Plate- and Shell-Type Supports	40	
NF-3324.1(a)(1)	Typical Friction-Type Connections Using High-Strength Bolts	63	
NF-3332.3-1	Illustrative Examples	73	
NF-3426.1-1	Additional Permissible Welded Joints for Class 1 Standard Supports	81	
NF-3626(a)-1	Additional Permissible Welded Joints for Class 1 Piping Supports	84	
NF-4427-1	Fillet Weld Dimensions	94	
NF-4441-1	Weld Joint Producing Through-Thickness Loading	95	
NF-II-3510	Equal Leg Single Angle Limits for $F_b = 0.66S_y$	151	
NF-A-1311		159	
NF-B-2400-1	Effective Length Factor for Various End Conditions	164	
NF-B-2400-2	Alignment Chart for Effective Length of Columns in Continuous Frames	165	
TADI FC			
TABLES	Matarial Tables Descrived for Constitute	10	
NF-2121(a)-1	Material Tables Required for Supports Exemptions From Impact Testing Under NF-2311(b)(9) and NF-2311(b)(10) for Ma-	10	
NF-2311(b)-1	terial Thickness to $2^{1}/2$ in. (64 mm), Inclusive	14	
NF-2331(a)-1	Required C <sub>v</sub> Lateral Expansion Values for Class 1 Support Material Other Than Bolting	17	
NF-2331(a)-1	Required C <sub>v</sub> Lateral Expansion Values for Class 1 support Material Other Than Bolting Required C <sub>v</sub> Lateral Expansion Values for Class 2 and MC Support Material Other Than	17	
N1-2331(a)-2	Bolting	17	
NF-2331(a)-3	Required C <sub>v</sub> Lateral Expansion Values for Class 3 Support Material Other Than Bolting	17	
NF-2331(a)-4	Required C <sub>v</sub> Energy Values for Class 1 Support Material Other Than Bolting	18	
NF-2333-1	Required C <sub>v</sub> Values for Bolting Material	21	
NF-2432.1-1	Sampling of Welding Materials for Chemical Analysis	23	
NF-2432.2(a)-1	Welding Material Chemical Analysis	24	
NF-3131(a)-1	Reference Paragraphs for Procedures for Design of Component Supports, Piping Sup-		
0101(a) 1	ports, and Standard Supports	30	
NF-3221.2-1	Elastic Analysis Stress Categories and Stress Limit Factors for Class 1 Plate- and Shell-		
C),	Type Supports Designed by Analysis	32	
NF-3225.21	Stress Limit Factors for Class 1, 2, 3, and MC Bolt Design by Analysis	34	
NF-3251.2-1	Elastic Analysis Stress Categories and Stress Limit Factors for Class 2, 3, and MC Plate-		
ر کا ک	and Shell-Type Supports Designed by Analysis	39	
NF-3312.1(b)-1	Elastic Analysis Stress Categories and Stress Limit Factors for Class 1, 2, 3, and MC		
$\sim$	Linear-Type Supports Designed by Analysis	44	
NF-3322.2(e)(2)-1	Limiting Proportions for Channels and Tees	55	
NF-3324.5(a)-1	Allowable Stress Limits for Class 1 Linear-Type Support Welds	65	
NF-3324.6(a)(4)-1	Effective Slip Coefficient Versus Surface Condition	68	
NF-3324.6(b)(1)-1	Minimum Edge Distances	68	
NF-3332.2-1	Classification of Loading Conditions	69	
NF-3332.3-1	Stress Categories	70	

	NF-3332.4-1	Allowable Stress Ranges	75
	NF-3611-1	Suggested Piping Support Spacing	83
	NF-4232-1	Maximum Allowable Offset in Final Butt-Welded Joints	89
	NF-4622.1-1	Mandatory Requirements for Postweld Heat Treatment (PWHT) of Welds	97
	NF-4622.4(c)-1	Alternative Holding Temperatures and Times	98
	NF-4622.7(b)-1	Exemptions to Mandatory PWHT	99
	NF-4622.7(b)-2	Additional Exemptions to Mandatory PWHT	100
	NF-4721(a)-1	Bolt Hole Sizes	102
	NF-4721(b)-1	Tolerances on Oversized Holes	102
	NF-I-2100-1	Design Stress Intensity $(S_m)$ , Maximum Allowable Stress $(S)$ , Yield Strength $(S_y)$ , and	
		Tensile Strength $(S_u)$ Values	113
	NF-I-2100-1M	Design Stress Intensity $(S_m)$ , Maximum Allowable Stress $(S)$ , Yield Strength $(S_y)$ and	
		Tensile Strength $(S_u)$ Values	130
	NF-I-2100-2	ASTM Specifications $B_w$ Values for Angles $B_w$ Values $B_w$ Values for Angles $B_w$ Values $B_w$ V	144
	NF-II-3532-1	$B_w$ Values for Angles	152
	NF-B-4000	Factors of Safety for a Material With a Low Value of $S_u/S_y$ Ratio $\dots$	167
	NF-C-4000	Design Basis for Linear-Type Supports	173
	NF-D-1320-1	Clamp Tolerances	176
	NF-D-1330-1	Local Installation Tolerances for Supports	177
	ENDNOTES		183
		<b>, ♦</b> `	
		K Y The second of the second o	
		· · · · · · · · · · · · · · · · · · ·	
		the state of the s	
		N	
ţ.	, <b>x</b>		
	45.		
	cillo		
	$\cdot$		
	· N ·		
į	Ola.		
	C		
· C	$\sim$		
.0			
COULT .			
A. T.		Additional Exemptions to Mandatory PWHT Bolt Hole Sizes Tolerances on Oversized Holes Design Stress Intensity (S <sub>m</sub> ), Maximum Allowable Stress (S), Yield Strength (S <sub>y</sub> ), and Tensile Strength (S <sub>y</sub> ) Values Design Stress Intensity (S <sub>m</sub> ), Maximum Allowable Stress (S), Yield Strength (S <sub>y</sub> ), and Tensile Strength (S <sub>y</sub> ) Values ASTM Specifications B <sub>w</sub> Values for Angles Factors of Safety for a Material With a Low Value of S <sub>w</sub> /S <sub>y</sub> , Ratio Design Basis for Linear-Type Supports Clamp Tolerances Local Installation Tolerances for Supports  ORDER TO SUPPORT OF THE STREET OF THE	
70.			
12			
		ix	

# **(17**)

# **LIST OF SECTIONS**

# **SECTIONS**

- I Rules for Construction of Power Boilers
- II Materials
  - Part A Ferrous Material Specifications
  - Part B Nonferrous Material Specifications
  - Part C Specifications for Welding Rods, Electrodes, and Filler Metals
  - Part D Properties (Customary)
  - Part D Properties (Metric)
- III Rules for Construction of Nuclear Facility Components
  - Subsection NCA General Requirements for Division 1 and Division 2
  - Appendices
  - Division 1
    - Subsection NB Class 1 Components
    - Subsection NC Class 2 Components
    - Subsection ND Class 3 Components
    - Subsection NE Class MC Components
    - Subsection NF Supports
    - Subsection NG Core Support Structures
  - Division 2 Code for Concrete Containments
  - Division 3 Containment Systems for Transportation and Storage of Spent Nuclear Fuel and High-Level Radioactive Material
  - Division 5 High Temperature Reactors
- IV Rules for Construction of Heating Boilers
- V Nondestructive Examination
- VI Recommended Rules for the Care and operation of Heating Boilers
- VII Recommended Guidelines for the Care of Power Boilers
- VIII Rules for Construction of Pressure Vessels
  - Division 1
  - Division 2 Alternative Rules
  - Division 3 Alternative Rules for Construction of High Pressure Vessels
- IX Welding, Brazing, and Fusing Qualifications
- X Fiber-Reinforced Plastic Pressure Vessels
- XI Rules for Inservice Inspection of Nuclear Power Plant Components
- XII Rules for Construction and Continued Service of Transport Tanks

<sup>\*</sup> The 2015 Edition of Section III was the last edition in which Section III, Division 1, Subsection NH, *Class 1 Components in Elevated Temperature Service*, was published. The requirements located within Subsection NH were moved to Section III, Division 5, Subsection HB, Subpart B for the elevated temperature construction of Class A components.

# INTERPRETATIONS

torical BPVC interpretations may also be found in the Database.

CODE CASES

The Boiler and Pressure Vessel Code committees meet regularly to consider proposed additions and revisions to the ode and to formulate Cases to clarify the intent of existing requirements or provide, when the need is urgent, rules for aterials or constructions not covered by existing Code rules. Those Cases that have been adapted propriate 2017 Code Cases book: "Boilers and Pressure Vessele" or "Example automatically to the purel". agplements atton of the state o Code and to formulate Cases to clarify the intent of existing requirements or provide, when the need is urgent, rules for materials or constructions not covered by existing Code rules. Those Cases that have been adopted will appear in the appropriate 2017 Code Cases book: "Boilers and Pressure Vessels" or "Nuclear Components." Supplements will be sent or made available automatically to the purchasers of the Code Cases books up to the publication of the 2019 Code.

# **FOREWORD**\*

In 1911, The American Society of Mechanical Engineers established the Boiler and Pressure Vessel Committee to formulate standard rules for the construction of steam boilers and other pressure vessels. In 2009, the Boiler and Pressure Vessel Committee was superseded by the following committees:

- (a) Committee on Power Boilers (I)
- (b) Committee on Materials (II)
- (c) Committee on Construction of Nuclear Facility Components (III)
- (d) Committee on Heating Boilers (IV)
- (e) Committee on Nondestructive Examination (V)
- (f) Committee on Pressure Vessels (VIII)
- (g) Committee on Welding, Brazing, and Fusing (IX)
- (h) Committee on Fiber-Reinforced Plastic Pressure Vessels (X)
- (i) Committee on Nuclear Inservice Inspection (XI)
- (j) Committee on Transport Tanks (XII)
- (k) Technical Oversight Management Committee (TOMC)

Where reference is made to "the Committee" in this Foreword, each of these committees is included individually and collectively.

The Committee's function is to establish rules of safety relating only to pressure integrity, which govern the construction of boilers, pressure vessels, transport tanks, and nuclear components, and the inservice inspection of nuclear components and transport tanks. The Committee also interprets these rules when questions arise regarding their intent. The technical consistency of the Sections of the Code and coordination of standards development activities of the Committees is supported and guided by the Technical Oversight Management Committee. This Code does not address other safety issues relating to the construction of boilers, pressure vessels, transport tanks, or nuclear components, or the inservice inspection of nuclear components or transport tanks. Users of the Code should refer to the pertinent codes, standards, laws, regulations, or other relevant documents for safety issues other than those relating to pressure integrity. Except for Sections XI and XII, and with a few other exceptions, the rules do not, of practical necessity, reflect the likelihood and consequences of deterioration in service related to specific service fluids or external operating environments. In formulating the rules, the Committee considers the needs of users, manufacturers, and inspectors of pressure vessels. The objective of the rules is to afford reasonably certain protection of life and property, and to provide a margin for deterioration in service to give a reasonably long, safe period of usefulness. Advancements in design and materials and evidence of experience have been recognized.

This Code contains mandatory requirements, specific prohibitions, and nonmandatory guidance for construction activities and inservice inspection and testing activities. The Code does not address all aspects of these activities and those aspects that are not specifically addressed should not be considered prohibited. The Code is not a handbook and cannot replace education, experience, and the use of engineering judgment. The phrase *engineering judgment* refers to technical judgments made by knowledgeable engineers experienced in the application of the Code. Engineering judgments must be consistent with Code philosophy, and such judgments must never be used to overrule mandatory requirements or specific prohibitions of the Code.

The Committee recognizes that tools and techniques used for design and analysis change as technology progresses and expects engineers to use good judgment in the application of these tools. The designer is responsible for complying with Code rules and demonstrating compliance with Code equations when such equations are mandatory. The Code neither requires nor prohibits the use of computers for the design or analysis of components constructed to the

<sup>\*</sup> The information contained in this Foreword is not part of this American National Standard (ANS) and has not been processed in accordance with ANSI's requirements for an ANS. Therefore, this Foreword may contain material that has not been subjected to public review or a consensus process. In addition, it does not contain requirements necessary for conformance to the Code.

<sup>\*\*</sup> Construction, as used in this Foreword, is an all-inclusive term comprising materials, design, fabrication, examination, inspection, testing, certification, and pressure relief.

requirements of the Code. However, designers and engineers using computer programs for design or analysis are cautioned that they are responsible for all technical assumptions inherent in the programs they use and the application of these programs to their design.

The rules established by the Committee are not to be interpreted as approving, recommending, or endorsing any proprietary or specific design, or as limiting in any way the manufacturer's freedom to choose any method of design or any form of construction that conforms to the Code rules.

The Committee meets regularly to consider revisions of the rules, new rules as dictated by technological development. Code Cases, and requests for interpretations. Only the Committee has the authority to provide official interpretations of this Code. Requests for revisions, new rules, Code Cases, or interpretations shall be addressed to the Secretary in writing and shall give full particulars in order to receive consideration and action (see Submittal of Technical Inquiries to the Boiler and Pressure Vessel Standards Committees). Proposed revisions to the Code resulting from inquiries will be presented to the Committee for appropriate action. The action of the Committee becomes effective only after confirmation by ballot of the Committee and approval by ASME. Proposed revisions to the Code approved by the Committee are submitted to the American National Standards Institute (ANSI) and published at http://go.asme.org/BPVCPublicReview to invite comments from all interested persons. After public review and final approval by ASME, revisions are published at regular intervals in Editions of the Code.

The Committee does not rule on whether a component shall or shall not be constructed to the provisions of the Code. The scope of each Section has been established to identify the components and parameters considered by the Committee in formulating the Code rules.

Questions or issues regarding compliance of a specific component with the Code rules are to be directed to the ASME Certificate Holder (Manufacturer). Inquiries concerning the interpretation of the Code are to be directed to the Committee. ASME is to be notified should questions arise concerning improper use of an ASME Certification Mark.

AND ENTROCK COM. CICK to view the full published for Assure Bryco. When required by context in this Section, the singular shall be interpreted as the plural, and vice versa, and the feminine, masculine, or neuter gender shall be treated as such other gender as appropriate.

xiii

# STATEMENT OF POLICY ON THE USE OF THE CERTIFICATION MARK AND CODE AUTHORIZATION IN ADVERTISING

ASME has established procedures to authorize qualified organizations to perform various activities in accordance with the requirements of the ASME Boiler and Pressure Vessel Code. It is the aim of the Society to provide recognition of organizations so authorized. An organization holding authorization to perform various activities in accordance with the requirements of the Code may state this capability in its advertising literature.

Organizations that are authorized to use the Certification Mark for marking items or constructions that have been constructed and inspected in compliance with the ASME Boiler and Pressure Vessel Code are issued Certificates of Authorization. It is the aim of the Society to maintain the standing of the Certification Mark for the benefit of the users, the enforcement jurisdictions, and the holders of the Certification Mark who comply with all cequirements.

Based on these objectives, the following policy has been established on the usage in advertising of facsimiles of the Certification Mark, Certificates of Authorization, and reference to Code construction. The American Society of Mechanical Engineers does not "approve," "certify," "rate," or "endorse" any item, construction, or activity and there shall be no statements or implications that might so indicate. An organization holding the Certification Mark and/or a Certificate of Authorization may state in advertising literature that items, constructions, or activities "are built (produced or performed) or activities conducted in accordance with the requirements of the ASME Boiler and Pressure Vessel Code," or "meet the requirements of the ASME Boiler and Pressure Vessel Code." An ASME corporate logo shall not be used by any organization other than ASME.

The Certification Mark shall be used only for stamping and nameplates as specifically provided in the Code. However, facsimiles may be used for the purpose of fostering the use of such construction. Such usage may be by an association or a society, or by a holder of the Certification Mark who may also use the facsimile in advertising to show that clearly specified items will carry the Certification Mark. General usage is permitted only when all of a manufacturer's items are constructed under the rules.

# STATEMENT OF POLICY ON THE USE OF ASME MARKING TO IDENTIFY MANUFACTURED ITEMS

The ASME Boiler and Pressure Vessel Code provides rules for the construction of boilers, pressure vessels, and nuclear components. This includes requirements for materials, design, fabrication, examination, inspection, and stamping. Items constructed in accordance with all of the applicable rules of the Code are identified with the official Certification Mark described in the governing Section of the Code.

Markings such as "ASME," "ASME Standard," or any other marking including "ASME" or the Certification Mark shall not be used on any item that is not constructed in accordance with all of the applicable requirements of the Code.

Items shall not be described on ASME Data Report Forms nor on similar forms referring to ASME that tend to imply that all Code requirements have been met when, in fact, they have not been. Data Report Forms covering items not fully complying with ASME requirements should not refer to ASME or they should clearly identify all exceptions to the ASME requirements.

# SUBMITTAL OF TECHNICAL INQUIRIES TO THE BOILER AND PRESSURE VESSEL STANDARDS COMMITTEES

# 1 INTRODUCTION

(a) The following information provides guidance to Code users for submitting technical inquiries to the applicable Boiler and Pressure Vessel (BPV) Standards Committee (hereinafter referred to as the Committee). See the guidelines on approval of new materials under the ASME Boiler and Pressure Vessel Code in Section II, Part D for requirements for requests that involve adding new materials to the Code. See the guidelines on approval of new welding and brazing materials in Section II, Part C for requirements for requests that involve adding new welding and brazing materials ("consumables") to the Code.

Technical inquiries can include requests for revisions or additions to the Code requirements, requests for Code Cases, or requests for Code Interpretations, as described below:

- (1) Code Revisions. Code revisions are considered to accommodate technological developments, to address administrative requirements, to incorporate Code Cases, or to clarify Code intent.
- (2) Code Cases. Code Cases represent alternatives or additions to existing Code requirements. Code Cases are written as a Question and Reply, and are usually intended to be incorporated into the Code at a later date. When used, Code Cases prescribe mandatory requirements in the same sense as the text of the Code. However, users are cautioned that not all regulators, jurisdictions, or Owners automatically accept Code Cases. The most common applications for Code Cases are as follows:
  - (-a) to permit early implementation of an approved ode revision based on an urgent need
  - (-b) to permit use of a new material for Code construction
- (-c) to gain experience with new materials or atternative requirements prior to incorporation directly into the Code
  - (3) Code Interpretations
- (-a) Code Interpretations provide clarification of the meaning of existing requirements in the Code and are presented in Inquiry and Reply format. Interpretations do not introduce new requirements.
- (-b) If existing Code text does not fully convey the meaning that was intended, or conveys conflicting requirements, and revision of the requirements is required to support the Interpretation, an Intent Interpretation will be issued in parallel with a revision to the Code.
- (b) Code requirements, Code Gases, and Code Interpretations established by the Committee are not to be considered as approving, recommending, certifying, or endorsing any proprietary or specific design, or as limiting in any way the freedom of manufacturers, constructors, or Owners to choose any method of design or any form of construction that conforms to the Code requirements.
- (c) Inquiries that do not comply with the following guidance or that do not provide sufficient information for the Committee's full understanding may result in the request being returned to the Inquirer with no action.

# 2 INQUIRY FORMAT

Submittals to the Committee should include the following information:

- (a) Purpose. Specify one of the following:
  - (1) request for revision of present Code requirements
  - (2) request for new or additional Code requirements
  - (3) request for Code Case
  - (4) request for Code Interpretation
- (b) Background. The Inquirer should provide the information needed for the Committee's understanding of the Inquiry, being sure to include reference to the applicable Code Section, Division, Edition, Addenda (if applicable), paragraphs, figures, and tables. Preferably, the Inquirer should provide a copy of, or relevant extracts from, the specific referenced portions of the Code.

(c) Presentations. The Inquirer may desire to attend or be asked to attend a meeting of the Committee to make a formal presentation or to answer questions from the Committee members with regard to the Inquiry. Attendance at a BPV Standards Committee meeting shall be at the expense of the Inquirer. The Inquirer's attendance or lack of attendance at a meeting will not be used by the Committee as a basis for acceptance or rejection of the Inquiry by the Committee. However, if the Inquirer's request is unclear, attendance by the Inquirer or a representative may be necessary for the Committee to understand the request sufficiently to be able to provide an Interpretation. If the Inquirer desires to make a presentation at a Committee meeting, the Inquirer should provide advance notice to the Committee Secretary, to ensure time will be allotted for the presentation in the meeting agenda. The Inquirer should consider the need for additional audiovisual equipment that might not otherwise be provided by the Committee. With sufficient advance notice to the Committee Secretary, such equipment may be made available.

# 3 CODE REVISIONS OR ADDITIONS

Requests for Code revisions or additions should include the following information:

- (a) Requested Revisions or Additions. For requested revisions, the Inquirer should identify those requirements of the Code that they believe should be revised, and should submit a copy of, or relevant extracts from the appropriate requirements as they appear in the Code, marked up with the requested revision. For requested additions to the Code, the Inquirer should provide the recommended wording and should clearly indicate where they believe the additions should be located in the Code requirements.
  - (b) Statement of Need. The Inquirer should provide a brief explanation of the need for the revision or addition.
- (c) Background Information. The Inquirer should provide background information to support the revision or addition, including any data or changes in technology that form the basis for the request, that will allow the Committee to adequately evaluate the requested revision or addition. Sketches, tables, figures, and graphs should be submitted, as appropriate. The Inquirer should identify any pertinent portions of the Code that would be affected by the revision or addition and any portions of the Code that reference the requested revised or added paragraphs.

# 4 CODE CASES

Requests for Code Cases should be accompanied by a statement of need and background information similar to that described in 3(b) and 3(c), respectively, for Code revisions or additions. The urgency of the Code Case (e.g., project underway or imminent, new procedure) should be described. In addition, it is important that the request is in connection with equipment that will bear the Certification Mark, with the exception of Section XI applications. The proposed Code Case should identify the Code Section and Division, and should be written as a Question and a Reply, in the same format as existing Code Cases. Requests for Code Cases should also indicate the applicable Code Editions and Addenda (if applicable) to which the requested Code Case applies.

# 5 CODE INTERPRETATIONS

- (a) Requests for Code Interpretations should be accompanied by the following information:
- (1) Inquiry. The Inquirer should propose a condensed and precise Inquiry, omitting superfluous background information and, when possible, composing the Inquiry in such a way that a "yes" or a "no" Reply, with brief limitations or conditions, if needed, can be provided by the Committee. The proposed question should be technically and editorially correct.
- (2) Reply. The Inquirer should propose a Reply that clearly and concisely answers the proposed Inquiry question. Preferably, the Reply should be "yes" or "no," with brief limitations or conditions, if needed.
- (3) Buckground Information. The Inquirer should provide any need or background information, such as described in 3(b) and 3(c), respectively, for Code revisions or additions, that will assist the Committee in understanding the proposed Inquiry and Reply.
- If the Inquirer believes a revision of the Code requirements would be helpful to support the Interpretation, the Inquirer may propose such a revision for consideration by the Committee. In most cases, such a proposal is not necessary.
- (b) Requests for Code Interpretations should be limited to an Interpretation of a particular requirement in the Code or in a Code Case. Except with regard to interpreting a specific Code requirement, the Committee is not permitted to consider consulting-type requests such as the following:
- (1) a review of calculations, design drawings, welding qualifications, or descriptions of equipment or parts to determine compliance with Code requirements

- (2) a request for assistance in performing any Code-prescribed functions relating to, but not limited to, material selection, designs, calculations, fabrication, inspection, pressure testing, or installation
  - (3) a request seeking the rationale for Code requirements

# **SUBMITTALS**

(a) Submittal. Requests for Code Interpretation should preferably be submitted through the online Interpretation Sub mittal Form. The form is accessible at http://go.asme.org/InterpretationRequest. Upon submittal of the form, the inquirer will receive an automatic e-mail confirming receipt. If the Inquirer is unable to use the online form, the Inquirer may mail the request to the following address:

Secretary ASME Boiler and Pressure Vessel Committee Two Park Avenue New York, NY 10016-5990

All other Inquiries should be mailed to the Secretary of the BPV Committee at the address above. Inquiries are unlikely to receive a response if they are not written in clear, legible English. They must also include the name of the Inquirer and the company they represent or are employed by, if applicable, and the Inquirer's address, telephone number, fax number, and e-mail address, if available.

(b) Response. The Secretary of the appropriate Committee will provide a written response, via letter or e-mail, as appropriate, to the Inquirer, upon completion of the requested action by the Committee. Inquirers may track the status of

xvii

# **(17)**

# **PERSONNEL** ASME Boiler and Pressure Vessel Standards Committees, **Subgroups, and Working Groups**

January 1, 2017

## TECHNICAL OVERSIGHT MANAGEMENT COMMITTEE (TOMC)

J. F. Henry
R. S. Hill III
G. G. Karcher
W. M. Lundy
G. C. Park
M. D. Rana
R. F. Reedy, Sr.
B. W. Roberts
F. J. Schaaf, Jr.
B. F. Shelley
W. J. Sperko
D. Srnic
R. W. Swayne
C. Withers
J. E. Batey, Contributing Member

# HONORARY MEMBERS (MAIN COMMITTEE)

F. P. Barton	W. G. Knecht
T. M. Cullen	J. LeCoff
G. E. Feigel	T. G. McCarty
O. F. Hedden	G. C. Millman
M. H. Jawad	R. A. Moen 💉
A. J. Justin	R. F. Reedy, Sr.

## ADMINISTRATIVE COMMITTEE

T. P. Pastor, Chair	J. F. Henry
S. C. Roberts, Vice Chair	R. S. Hill III
J. S. Brzuszkiewicz, Staff Secretary	G. C. Park
R. W. Barnes	
T. L. Bedeaux	M. D. Rana
D. L. Berger	B. F. Shelley
G. W. Hembree	W. J. Sperko

H. N. Patel, Chair

# CONFERENCE COMMITTEE

D. A. Douin — Ohio, Secretary	J. LeSage, Jr. — Louisiana
M. J. Adams — Ontario, Canada	A. M. Lorimor — South Dakota
J. T. Amato — Minnesota	M. Mailman Northwest
W. Anderson — Mississippi	Territories, Canada
R. D. Austin — Arizona	D. E. Mallory — New Hampshire
R. J. Brockman — Missouri	W. McGivney — City of New York,
J. H. Burpee — Maine	New York
M. Byrum — Alabama	S. V. Nelson — Colorado
C. B. Cantrell — Nebraska	A. K. Oda — Washington
S. Chapman — Tennessee	M. Poehlmann — Alberta, Canada
D. C. Cook — California	J. F. Porcella — West Virginia
B. J. Crawford — Georgia	C. F. Reyes — City of Los Angeles,
E. L. Creaser — New Brunswick,	California
Canada	M. J. Ryan — City of Chicago,
J. J. Dacanay — Hawaii C. Dautrich — North Carolina	Illinois
R. Delury — Manitoba, Canada	D. Sandfoss — Nevada
P. L. Dodge — Nova Scotia, Canada	M. H. Sansone — New York
D. Eastman — Newfoundland and	A. S. Scholl — British Columbia,
Labrador, Canada	Canada
y. J. Esch — Delaware	T. S. Seime — North Dakota
A. G. Frazier — Florida	C. S. Selinger — Saskatchewan,
T. J. Granneman II — Oklahoma	Canada
D. R. Hannon — Arkansas	J. E. Sharier — Ohio
E. G. Hilton — Virginia	N. Smith — Pennsylvania
C. Jackson — City of Detroit,	R. Spiker — North Carolina
Michigan	D. J. Stenrose — Michigan
M. L. Jordan — Kentucky	R. J. Stimson II — Kansas
E. Kawa, Jr. — Massachusetts	R. K. Sturm — Utah
A. Khssassi — Quebec, Canada	S. R. Townsend — Prince Edward
J. Klug — City of Milwaukee, Wisconsin	Island, Canada
	R. D. Troutt — Texas
K. J. Kraft — Maryland K. S. Lane — Alaska	M. C. Vogel — Illinois
L. C. Leet — City of Seattle,	T. Waldbillig — Wisconsin
L. C. Leet — City of Seattle,	1. Transbilling Wildering

# INTERNATIONAL INTEREST REVIEW GROUP

M. Washington — New Jersey

### V. Felix C. Minu MARINE CONFERENCE GROUP Y.-G. Kim T. S. G. Narayannen G. Pallichadath S. H. Leong Y.-W. Park J. S. Brzuszkiewicz, Staff Secretary N. Prokopuk W. Lin A. R. R. Nogales J. G. Hungerbuhler, Jr. J. D. Reynolds O. F. Manafa P. Williamson

Washington

## COMMITTEE ON POWER BOILERS (BPV I)

D. L. Berger, <i>Chair</i>	Y. Oishi
R. E. McLaughlin, Vice Chair	E. M. Ortman
U. D'Urso, Staff Secretary	J. T. Pillow
J. L. Arnold	M. Slater
D. A. Canonico	J. M. Tanzosh
K. K. Coleman	D. E. Tompkins
P. D. Edwards	D. E. Tuttle
J. G. Feldstein	J. Vattappilly
G. W. Galanes	R. V. Wielgoszinski
T. E. Hansen	F. Zeller
J. F. Henry	Y. Li, Delegate
J. S. Hunter	H. Michael, Delegate
G. B. Komora	B. W. Roberts, Contributing

W. L. Lowry

Member F. Massi D. N. French, Honorary Member L. Moedinger T. C. McGough, Honorary Member P. A. Molvie R. L. Williams, Honorary Member

# Subgroup on Design (BPV I)

J. Vattappilly, Chair	P. A. Molvie
D. I. Anderson, Secretary	L. S. Tsai
D. Dewees	M. Wadkinso

H. A. Fonzi, Jr. C. F. Jeerings, Contributing Member J. P. Glaspie S. V. Torkildson, Contributing

G. B. Komora Member

# Subgroup on Fabrication and Examination (BPV I)

J. L. Arnold, <i>Chair</i>	T. E. Hansen
P. Becker	C. T. McDaris
D. L. Berger	R. E. McLaughlin
S. Fincher	R. J. Newell
G. W. Galanes	Y. Oishi
P. F. Gilston	J. T. Pillow
J. Hainsworth	R. V. Wielgoszins

# Subgroup on General Requirements and Piping (BPV I)

E. M. Ortman, Chair	R. E. McLaughlin
D. Tompkins, Vice Chair	B. J. Mollitor
F. Massi, Secretary	J. T. Pillow
P. Becker	D. E. Tuttle
D. L. Berger	M. Wadkinson
P. D. Edwards	R. V. Wielgoszinsl
C. W. Colones	C E Jaarings Con

C. F. Jeerings, Contributing Member T. E. Hansen S. V. Torkildson, Contributing

M. Lemmons Member

R. Uebel, Contributing Member

# Subgroup on Locomotive Boilers (BPV I)

_	-
L. Moedinger, Chair	S. D. Jackson
S. M. Butler, Secretary	M. A. Janssen
P. Boschan	S. A. Lee
J. R. Braun	G. M. Ray
R. C. Franzen, Jr. G. W. Galanes	R. B. Stone
D. W. Griner	M. W. Westland
2	in w. westand

# Subgroup on Materials (BPV I)

G. W. Galanes, Chair	F. Masuyama
J. F. Henry, Vice Chair	D. W. Rahoi
M. Lewis, Secretary	J. M. Tanzosh
S. H. Bowes	•
D. A. Canonico	J. Vattappilly
K. K. Coleman	F. Zeller
V I Haves	M Gold Contr

M. Gold, Contributing Member K. L. Hayes J. S. Hunter B. W. Roberts, Contributing Member O. X. Li

# Subgroup on Solar Boilers (BPV I)

E. M. Ortman <i>, Chair</i>	P. Jennings
R. E. Hearne, Secretary	D. J. Koza F. Massi
H. A. Fonzi, Jr.	F. Massi

G. W. Galanes S. V. Torkildson, Contributing

J. S. Hunter

# Germany International Working Group (BPV I)

H. Michael, Chair	T. Ludwig
H. P. Schmitz, Secretary	R. A. Meyers
M. Bremicker	F. Miunske
P. Chavdarov	P. Paluszkiewicz
B. Daume	H. Schroeder
J. Fleischfresser	A. Spangenberg
E. Helmholdt	M. Sykora
P Kauer	I Hanrichemayo

J. Henrichsmeyer, Contributing R. Kauer

S. Krebs Member

# India International Working Group (BPV I)

U. Revisanakaran, Chair	G. V. S. Rao
A. J. Patil, Vice Chair	M. G. Rao
H. Dalal, Secretary	N. Satheesan
K. Asokkumar	G. U. Shanker
M. R. Kalahasthi	D. Shrivastava
I. Kalyanasundaram A. R. Patil	
A. K. Paul	S. Venkataramana

# Task Group on Modernization of BPVC Section I

D. I. Anderson, Chair R. E. McLaughlin U. D'Urso, Staff Secretary P. A. Molvie J. L. Arnold E. M. Ortman D. Dewees J. T. Pillow G. W. Galanes B. W. Roberts J. P. Glaspie D. E. Tuttle T. E. Hansen J. F. Henry J. Vattappilly

# **COMMITTEE ON MATERIALS (BPV II)**

J. F. Henry, Chair	
J. F. Grubb, Vice Chair	
C. E. O'Brien, Staff Secretary	
F. Abe	
A. Appleton	
J. Cameron	
D. A. Canonico	
A. Chaudouet	
D. B. Denis	
J. R. Foulds	
D. W. Gandy	
M. H. Gilkey	
J. A. Hall	
K. M. Hottle	
M. Ishikawa	
O. X. Li	
F. Masuyama	
R. K. Nanstad	
K. E. Orie	
D. W. Rahoi	
E. Shapiro	

M. J. Slater

R. C. Sutherlin

R. W. Swindeman

# J. M. Tanzosh R. G. Young F. Zeller

O. Oldani, Delegate

H. D. Bushfield, Contributing

Member

M. Gold, Contributing Member W. Hoffelner, Contributing Member

M. Katcher, Contributing Member M. L. Nayyar, Contributing Member

E. G. Nisbett, Contributing Member D. T. Peters, Contributing Member

B. W. Roberts, Contributing Member

E. Thomas, Contributing Member

E. Upitis, Contributing Member T. M. Cullen, Honorary Member W. D. Edsall, Honorary Member

G. C. Hsu, Honorary Member R. A. Moen, Honorary Member

C. E. Spaeder, Jr., Honorary

Member

A. W. Zeuthen, Honorary Member

# **Executive Committee (BPV II)**

J. F. Henry <i>, Chair</i>	J. F. Grubb
C. E. O'Brien, Staff Secretary	R. W. Mikitka
A. Appleton	B. W. Roberts
A. Chaudouet	M. J. Slater
J. R. Foulds	R. C. Sutherlin
M. Gold	R. W. Swindem

## Subgroup on External Pressure (BPV H)

R. W. Mikitka, Chair	J. R. Harris III
D. L. Kurle, Vice Chair	M. H. Jawad
J. A. A. Morrow, Secretary	C. R. Thomas
L. F. Campbell	M. Wadkinson
H. Chen	
D. S. Griffin	M. Katcher, Contributing Member

J. F. Grubb C. H. Sturgeon, Contributing S. Guzey Member

Subgroup on Ferrous Specifications (BPV II)

A. Appleton, <i>Chair</i>	C. Hyde
K. M. Hottle Vice Chair	D. S. Janikowski
P. Wittenbach <i>, Secretary</i>	L. J. Lavezzi
H. Chen	S. G. Lee
B. M. Dingman	W. C. Mack
M. J. Dosdourian	A. S. Melilli
O. Elkadim	K. E. Orie
J. D. Fritz	J. Shick
M. Gold	E. Upitis
T. Graham	*
J. M. Grocki	J. D. Wilson
J. F. Grubb	R. Zawierucha
J. Gundlach	E. G. Nisbett, Contributing Member

# Subgroup on International Material Specifications (BPV II)

A. Chaudouet, Chair	M. Ishikawa
A. R. Nywening, Vice Chair	O. X. Li
T. F. Miskell, Secretary	W. M. Lundy
D. A. Canonico	E. Upitis
H. Chen	F. Zeller
A. F. Garbolevsky	0. Oldani, Delegate

D. O. Henry H. Lorenz, Contributing Member

# Subgroup on Nonferrous Alloys (BPV II)

R. C. Sutherlin, Chair	D. W. Rahoi
M. H. Gilkey, Vice Chair	W. Ren
J. Calland	J. Robertson
D. B. Denis	E. Shapiro
J. F. Grubb	M. H. Skillingberg
T. Hartman	
A. Heino	J. Weritz
M. Katcher	R. Wright
J. A. McMaster	S. Yem
. D 1	7 D

D. T. Peters, Contributing Member L. Paul

# Subgroup on Physical Properties (BPV II)

J. F. Grubb, Chair	H. D. Bushfield, Contributing
D. B. Denis, Vice Chair	Member
E. Shapiro	

## Subgroup on Strength, Ferrous Alloys (BPV II)

M. J. Slater, Chair	M. Ortolani
S/W. Knowles, Secretary	D. W. Rahoi
F. Abe	M. S. Shelton
D. A. Canonico	R. W. Swindeman
A. Di Rienzo	J. M. Tanzosh
J. R. Foulds	R. G. Young
J. A. Hall	F. Zeller
J. F. Henry	M. Gold, Contributing Member
K. Kimura	M. Nair, Contributing Member
F. Masuyama	B. W. Roberts, Contributing
T. Ono	Member

# Subgroup on Strength of Weldments (BPV II & BPV IX)

W. F. Newell, Jr., Chair	J. F. Henry
S. H. Bowes	E. Liebl
K. K. Coleman	J. Penso
M. Denault	D. W. Rahoi
P. D. Flenner	B. W. Roberts
J. R. Foulds	W. J. Sperko
D. W. Gandy	J. P. Swezy, Jr.
M. Ghahremani	J. M. Tanzosh
V I Harras	M Cold Contributing A

K. L. Hayes M. Gold, Contributing Member

# Working Group on Materials Database (BPV II)

R. W. Swindeman, Chair J. L. Arnold, Contributing Member C. E. O'Brien, Staff Secretary J. Grimes, Contributing Member F. Abe W. Hoffelner, Contributing Member J. R. Foulds T. Lazar, Contributing Member J. F. Henry D. T. Peters, Contributing Member M. J. Slater W. Ren, Contributing Member B. W. Roberts, Contributing R. C. Sutherlin D. Andrei, Contributing Member Member

# Working Group on Creep Strength Enhanced Ferritic Steels (BPV II)

J. F. Henry, Chair	W. F. Newell, Jr.
J. A. Siefert, Secretary	M. Ortolani
F. Abe	J. Parker
S. H. Bowes	W. J. Sperko
D. A. Canonico	R. W. Swindeman
K. K. Coleman	J. M. Tanzosh
P. D. Flenner	R. H. Worthington
J. R. Foulds	R. G. Young
G. W. Galanes	F. Zeller

M. Gold G. Cumino, Contributing Member F. Masuyama B. W. Roberts, Contributing

T. Melfi Member

## Working Group on Data Analysis (BPV II)

J. F. Grubb <i>, Chair</i>	W. Ren
F. Abe	M. Subanovic
J. R. Foulds M. Gold	M. J. Swindeman
J. F. Henry	R. W. Swindeman
M Katcher	R W Poherts Contr

M. Katcher B. W. Roberts, Contributing

F. Masuyama Member

# China International Working Group (BPV II)

B. Shou, Chair	X. Wang
A. T. Xu, Secretary	F. Yang
W. Fang	G. Yang
Q. C. Feng	HC. Yang
S. Huo	R. Ye
F. Kong	L. Yin
H. Li	D. Zhang
J. Li	H. Zhang
S. Li	XH. Zhang
Z. Rongcan	Yingkai Zhang
S. Tan	Yong Zhang
C. Wang	Q. Zhao
J. Wang	S. Zhao
QJ. Wang	J. Zou

# COMMITTEE ON CONSTRUCTION OF NUCLEAR FACILITY COMPONENTS (BPV III)

R. S. Hill III, Chair	M. Morishita
R. B. Keating, Vice Chair	D. K. Morton
J. C. Minichiello, Vice Chair	T. Nagata
A. Byk, Staff Secretary	R. F. Reedy, Sr.
T. M. Adams	I. Saito
A. Appleton	S. Sham
R. W. Barnes	C. T. Smith
W. H. Borter	W. K. Sowder, Jr.
C. W. Bruny	W. J. Sperko
T. D. Burchell	J. P. Tucker
R.P. Deubler	K. R. Wichman
A. C. Eberhardt	C. S. Withers
R. M. Jessee	Y. H. Choi, Delegate
R. I. Jetter	T. Ius, Delegate
C. C. Kim	HT. Wang, Delegate
G. H. Koo	M. Zhou, Contributing Member
V. Kostarev	E. B. Branch, Honorary Member
K. A. Manoly	G. D. Cooper, Honorary Member
D. E. Matthews	D. F. Landers, Honorary Member
R. P. McIntyre	R. A. Moen, Honorary Member

M. N. Mitchell

# **Executive Committee (BPV III)**

R. S. Hill III, Chair A. Byk, Staff Secretary	J. C. Minichiello M. Morishita
T. M. Adams	D. K. Morton
C. W. Bruny P. R. Donavin	J. A. Munshi
R. M. Jessee	C. A. Sanna
R. B. Keating	S. Sham
R. P. McIntyre	W. K. Sowder, Jr.

P. R. Donavin, Chair

# Subcommittee on Design (BPV III)

M. N. Mitchell

D. E. Matthews, Vice Chair	W. J. O'Donnell, Sr.
G. L. Hollinger, Secretary	E. L. Pleins
T. M. Adams	S. Sham
R. L. Bratton	P. Tucker
C. W. Bruny	
R. P. Deubler	W. F. Weitze
R. I. Jetter	K. Wright
C. Jonker	T. Yamazaki
R. B. Keating	J. Yang
K. A. Manoly	R. S. Hill III, Contributing Member
R. I. Masterson	M. H. Jawad. Contributing Member

# Subgroup on Component Design (SC-D) (BPV III)

	0 ( ) (
T. M. Adams, Chair	T. M. Musto
R. B. Keating, Vice Chair	T. Nagata
S. Pellet, Secretary	A. N. Nguyen
G. A. Antaki	E. L. Pleins
S. Asada	I. Saito
J. F. Ball	G. C. Slagis
C. Basavaraju	J. R. Stinson
R. P. Deubler	G. Z. Tokarski
P. Hirschberg	J. P. Tucker
OS. Kim	P. Vock
R. Klein	
H. Kobayashi	C. Wilson
K. A. Manoly	J. Yang
D. I. M +	C W Bruny Co

R. J. Masterson C. W. Bruny, Contributing Member
D. E. Matthews A. A. Dermenjian, Contributing

J. C. Minichiello Member

D. K. Morton K. R. Wichman, Honorary Member

# Working Group on Core Support Structures (SG-CD) (BPV III)

M. Nakajima
M. D. Snyder
A. Tsirigotis
R. Vollmer
R. Z. Ziegler

H. S. Mehta J. T. Land, Contributing Member

# Working Group on Design of Division 3 Containment Systems (SG-CD) (BPV III)

D. K. Morton, Chair
D. J. Ammerman
G. Bjorkman
V. Broz
E. L. Pleins
C. J. Temus
X. Zhai

S. Horowitz
D. W. Lewis
I. D. McInnes, Contributing Member
H. P. Shrivastava, Contributing

J. C. Minichiello Member

C. J. Pieper, Honorary Member

# Working Group on HDPE Design of Components (SG-CD) (BPV III)

T. M. Musto, Chair	P. Krishnaswamy
J. Ossmann, Secretary	K. A. Manoly
T. M. Adams	M. Martin
T. A. Bacon	J. C. Minichiello
M. Brandes	D. P. Munson
D. Burwell	F. J. Schaaf, Jr.
S. Choi	R. Stakenborghs
J. R. Hebeisen	H. E. Svetlik

# Working Group on Valves (SG-CD) (BPV III)

Working Group on Vessels (SG-CD) (BPV III)

T. J. Schriefer

M. C. Scott

P. K. Shah

J. Shupert

C. Turylo

D. Vlaicu

W. F. Weitze

T. Yamazaki

R. Z. Ziegler

A. Kalnins, Contributing Member

P. Vock, Chair	C. A. Mizer
S. Jones, Secretary	J. O'Callaghan
M. C. Buckley	H. O'Brien
R. Farrell	K. E. Reid II
G. A. Jolly	J. Sulley
J. Klein	I. H. Tseng
T. Lippucci	J. P. Tucker

D. E. Matthews, *Chair* C. Wilson, *Secretary* 

C. Basavaraju

J. V. Gregg, Jr.

R. B. Keating

M. Kassar

D. Keck

O.-S. Kim T. Mitsuhashi

M. Nair

J. Kim

# Working Group on Piping (SG-CD) (BPV III)

G. A. Antaki, Chair	R. B. Keating
G. Z. Tokarski, Secretary	V. Kostarev
T. M. Adams	D. Lieb
T. A. Bacon	T. B. Littleton
C. Basavaraju	Y. Liu
J. Catalano	J. F. McCabe
F. Claeys	J. C. Minichiello
C. M. Faidy	IK. Nam
R. G. Gilada	M. S. Sills
N. M. Graham	G. C. Slagis
M. A. Gray	N. C. Sutherland
R. I. Gurdal	CI. Wu

R. J. Gurdal
R. W. Haupt
A. N. Nguyen, Contributing Member
A. Hirano
N. J. Shah, Contributing Member
P. Hirschberg
E. A. Wais, Contributing Member
M. Kassar
E. C. Rodabaugh, Honorary

J. Kawahata Member

# Subgroup on Design Methods (SC-D) (BPV III)

C. W. Bruny, <i>Chair</i>	D. Keck
S. McKillop, Secretary	M. N. Mitchell
K. Avrithi	W. J. O'Donnell, Sr.
W. Culp	P. J. O'Regan
P. R. Donavin	W. D. Reinhardt
V. Gregg, Jr.	P. Smith
H. T. Harrison III	S. D. Snow
K. Hsu	
C. Jonker	W. F. Weitze
M. Kassar	K. Wright

# Working Group on Pressure Relief (SG-CD) (BPV III)

J. F. Ball <i>, Chair</i>	A. L. Szeglin
K. R. May	D. G. Thibault
D. Miller	I. H. Tseng

# Working Group on Pumps (SGCD) (BPV III)

M. Higuchi
R. Ladefian
W. Lienau
K. J. Noel
R. J. NOCI
R. A. Patrick
J. Sulley
R. Udo
A. G. Washbur

# Working Group on Design Methodology (SG-DM) (BPV III)

S. D. Snow, Chair	J. F. McCabe
C. F. Heberling II, Secretary	S. McKillop
K. Avrithi	S. Ranganath
C. Basavaraju	W. D. Reinhardt
D. L. Caldwell	D. H. Roarty
D. Dewees	P. K. Shah
C. M. Faidy	R. Vollmer
R. Farrell	S. Wang
H. T. Harrison III	W. F. Weitze
P. Hirschberg	J. Wen
M. Kassar	T. M. Wiger
R. B. Keating	K. Wright
J. Kim	J. Yang
II IZ-11-:	D D D1

H. Kobayashi R. D. Blevins, Contributing Member
T. Liszkai M. R. Breach, Contributing Member

# Working Group on Supports (SG-CD) (BPV III)

J. R. Stinson, Chair
U. S. Bandyopadhyay, Secretary
I. Saito
K. Avrithi
H. P. Srivastava
T. H. Baker
F. J. Birch
R. P. Deubler
N. M. Graham
R. J. Masterson
S. Pellet
H. P. Srivastava
C. Stirzel
G. Z. Tokarski
P. Wiseman
C.-I. Wu

# Working Group on Environmental Effects (SG-DM) (BPV III)

C. Jonker, Chair J. Kim
B. D. Frew, Secretary J. E. Nestell
W. Culp M. Osterfoss
P. J. Dobson T. J. Schriefer

# Working Group on Environmental Fatigue Evaluation Methods (SG-DM) (BPV III)

K. Wright, Chair	S. R. Gosselin
M. A. Gray, Vice Chair	Y. He
W. F. Weitze, Secretary	P. Hirschberg
T. M. Adams	H. S. Mehta
S. Asada	T. Metais
K. Avrithi	JS. Park
R. C. Cipolla	D. H. Roarty
T. M. Damiani	I. Saito
C. M. Faidy	D. Vlaicu
T. D. Gilman	R. Z. Ziegler

# Working Group on Fatigue Strength (SG-DM) (BPV III)

P. R. Donavin, Chair	S. H. Kleinsmitl
M. S. Shelton, Secretary	S. Majumdar
T. M. Damiani	S. N. Malik
D. Dewees	S. Mohanty
C. M. Faidy	D. H. Roarty
S. R. Gosselin	,
R. J. Gurdal	A. Tsirigotis
C. F. Heberling II	K. Wright
C. E. Hinnant	H. H. Ziada

P. Hirschberg W. J. O'Donnell, Sr., Contributing

K. Hsu Member

# Subgroup on Elevated Temperature Design (SC-D) (BPV III)

S. Sham, <i>Chair</i>	G. H. Koo
T. Asayama	S. Majumdar
C. Becht IV	J. E. Nestell
F. W. Brust	W. J. O'Donnell, Sr.
P. Carter	R. W. Swindeman
B. F. Hantz	D. S. Griffin, Contributing Mem
A. B. Hull	W. J. Koves, Contributing Mem
M. H. Jawad	D. L. Marriott, Contributing

R. I. Jetter

# Working Group on Allowable Stress Criteria (SG-ETD) (BPV III)

Member

R. W. Swindeman, Chair	D. Maitra
R. Wright, Secretary	S. N. Malik
J. R. Foulds	J. E. Nestell
C. J. Johns	W Ren
K. Kimura	B. W. Roberts
T. Le	M. Sengupta
M. Li	S. Sham

# Working Group on Analysis Methods (SG-ETD) (BPV III)

P. Carter, Chair
M. J. Swindeman, Secretary
T. Le
M. R. Breach
S. Sham
M. E. Cohen
D. K. Williams
R. L. Jetter

# Working Group on Graphite and Composites Design (SG-DM) (BPV III)

M. N. Mitchell, Chair	S. T. Gonczy
M. W. Davies, Vice Chair	M. G. Jenkins
T. D. Burchell, Secretary	Y. Katoh
A. Appleton	J. Ossmann
S. R. Cadell	M. Roemmler
SH. Chi	S. Yu
W. J. Geringer	G. L. Zeng

# Working Group on Probabilistic Methods in Design (SG-DM) (BPV III)

M. Golliet, Chair	M. Morishita
T. Asayama	P. J. O'Regan
K. Avrithi	N. A. Palm
D. O. Henry	I. Saito
R. S. Hill III.	

# Working Group on Creep-Fatigue and Negligible Creep (SG-ETD) (BPV III)

T. Asayama <i>, Chair</i>	T. Le
F. W. Brust	BL. Lyow
P. Carter	S. N. Malik
R. I. Jetter	H. Qian
G. H. Koo	S. Sham

# Working Group on Elevated Temperature Construction (SG-ETD) (BPV III)

M. H. Jawad, Chair	M. N. Mitchell
A. Mann, Secretary	B. J. Mollitor
D. I. Anderson	C. Nadarajah
R. G. Brown	,
D. Dewees	P. Prueter
B. F. Hantz	M. J. Swindeman
R I letter	I. P. Glaspie, Contribut

R. I. Jetter J. P. Glaspie, Contributing Member
S. Krishnamurthy D. L. Marriott, Contributing

T. Le Member

# Special Working Group on Computational Modeling for Explicit Dynamics (SG-DM) (BPV III)

G. Bjorkman, <i>Chair</i>	W. D. Reinhardt
D. J. Ammerman, Vice Chair	P. YK. Shih
V. Broz, Secretary	S. D. Snow
M. R. Breach	
J. M. Jordan	CF. Tso
S. Kuehner	M. C. Yaksh
D. Molitoris	U. Zencker

# Working Group on High Temperature Flaw Evaluation (SG-ETD) (BPV III)

H. Qian
D. L. Rudland
P. J. Rush
DJ. Shim
S. X. Xu

# Special Working Group on Inelastic Analysis Methods (SG-ETD) (BPV III)

S. Sham, Chair	T. Hassan
S. X. Xu, Secretary	G. H. Koo
R. W. Barnes	BL. Lyow
J. A. Blanco	M. J. Swindeman
B. R. Ganta	G. L. Zeng

# Subgroup on General Requirements (BPV III)

R. P. McIntyre, Chair	E. C. Renaud
L. M. Plante, Secretary	J. Rogers
V. Apostolescu	D. J. Roszman
A. Appleton	C. T. Smith
S. Bell	W. K. Sowder, Jr.
J. R. Berry	R. Spuhl
J. DeKleine	G. E. Szabatura
J. V. Gardiner	D. M. Vickery
J. W. Highlands	C. S. Withers
E. V. Imbro	H. Michael, Delegate
K. A. Kavanagh	G. L. Hollinger, Contributing
YS. Kim	Member

# Working Group on Duties and Responsibilities (SG-GR) (BPV III)

J. V. Gardiner <i>, Chair</i>	Y. Diaz-Castillo
G. L. Hollinger, Secretary	K. A. Kavanagh
D. Arrigo	J. M. Lyons
S. Bell	L. M. Plante
J. R. Berry	D. J. Roszman
P. J. Coco	,
M. Cusick	B. S. Sandhu
J. DeKleine	E. M. Steuck
N. DeSantis	J. L. Williams

# Working Group on Quality Assurance, Certification, and Stamping (SG-GR) (BPV III)

	<b>() v</b>
C. T. Smith, Chair	D. T. Meisch
C. S. Withers, Secretary	R. B. Patel
V. Apostolescu	E. C. Renaud
A. Appleton	T. Rezk
O. Elkadim	J. Rogers
S. M. Goodwin	W.K. Sowder, Jr.
J. Grimm	R. Spuhl
J. W. Highlands	J. F. Strunk
YS. Kim	G. E. Szabatura
B. McGlone	D. M. Vickery
R. P. McIntyre	C. A. Spletter, Contributing Membe

# Special Working Group on General Requirements Consolidation (SG-GR) (RPV III)

~O,	(SG-GR) (BPV III)
J. V. Gardiner, Chair	J. Rogers
C. T. Smith, Vice Chair	D. J. Roszman
S. Bell	B. S. Sandhu
M. Cusick	G. J. Solovey
Y. Diaz-Castillo	R. Spuhl
J. Grimm J. M. Lyons	G. E. Szabatura
B. McGlone	J. L. Williams
R. Patel	C. S. Withers
E. C. Renaud	S. F. Harrison, Contributing
T. Rezk	Member

# Subgroup on Materials, Fabrication, and Examination (BPV III)

R. M. Jessee, Chair	J. Johnston, Jr.
B. D. Frew, Vice Chair	C. C. Kim
S. Hunter, Secretary	M. Lashley
W. H. Borter	T. Melfi
T. D. Burchell	H. Murakami
G. R. Cannell	J. Ossmann
P. J. Coco	J. E. O'Sullivan
M. W. Davies	M. C. Scott
R. H. Davis	W. J. Sperko
D. B. Denis	J. R. Stinson
G. B. Georgiev	J. F. Strunk
S. E. Gingrich	R. Wright
M. Golliet	S. Yee
J. Grimm	H. Michael, Delegate
L. S. Harbison	R. W. Barnes, Contributing Memb

# Working Group on Graphite and Composite Materials (SG-MFE) (BPV III)

	<b>•</b>
T. D. Burchell, Chair	W. J. Geringer
M. W. Davies, Vice Chair	S. T. Gonzcy
M. N. Mitchell, Secretary	M. G. Jenkins
A. Appleton	Y. Katoh
R. L. Bratton	J. Ossmann
S. R. Cadell	M. Roemmler
SH. Chi	N. Salstrom
A. Covac	T. Shibata
S. W. Doms	S. Yu
S.F. Duffy	G. L. Zeng

# Working Group on HDPE Materials (SG-MFE) (BPV III)

M. Golliet, Chair	D. P. Munson
M. A. Martin, Secretary	T. M. Musto
W. H. Borter	S. Patterson
G. Brouette	S. Schuessler
M. C. Buckley	R. Stakenborghs
J. Hakii	O
J. Johnston, Jr.	M. Troughton
P. Krishnaswamy	B. Hauger, Contributing Member

# Joint ACI-ASME Committee on Concrete Components for Nuclear Service (BPV III)

J. A. Munshi, <i>Chair</i>	N. Orbovic
J. McLean, Vice Chair	C. T. Smith
A. Byk, Staff Secretary	J. F. Strunk
K. Verderber, Staff Secretary	T. Tonyan
C. J. Bang	S. Wang
L. J. Colarusso	T. J. Ahl, Contributing Member
A. C. Eberhardt	J. F. Artuso, Contributing Member
F. Farzam	JB. Domage, Contributing Member
P. S. Ghosal	J. Gutierrez, Contributing Member
B. D. Hovis	T. Kang, Contributing Member
T. C. Inman	T. Muraki, Contributing Member
C. Jones	B. B. Scott, Contributing Member
O. Jovall	M. R. Senecal, Contributing
NH. Lee	Member

## Working Group on Design (BPV III-2)

N.-H. Lee, Chair J. A. Munshi M. Allam T. Muraki S. Bae S. Wang

M. Diaz, Contributing Member L. J. Colarusso A. C. Eberhardt S. Diaz, Contributing Member F. Farzam A. Istar, Contributing Member P. S. Ghosal B. R. Laskewitz, Contributing B. D. Hovis Member T. C. Inman

B. B. Scott, Contributing Member C. Jones Z. Shang, Contributing Member 0. Jovall M. Sircar, Contributing Member

# Working Group on Materials, Fabrication, and Examination (BPV III-2)

P. S. Ghosal, Chair N. Lee T. Tonyan, Vice Chair C. T. Smith M. Allam I. F. Strunk C. J. Bang D. Ufuk

J.-B. Domage J. F. Artuso, Contributing Member A. C. Eberhardt J. Gutierrez, Contributing Member B. B. Scott, Contributing Member C. Jones T. Kang Z. Shang, Contributing Member

## Special Working Group on Modernization (BPV III-2)

J. McLean, Chair S. Wang

N. Orbovic, Vice Chair S. Diaz, Contributing Member A. Adediran J.-B. Domage, Contributing Member 0. Jovall F. Lin, Contributing Member C. T. Smith M. A. Ugalde N. Stoeva, Contributing Member

# Subgroup on Containment Systems for Spent Nuclear Fuel and High-Level Radioactive Material (BPV III)

D. K. Morton, Chair C. J. Temus W. H. Borter, Contributing Member D. J. Ammerman, Vice Chair

G. R. Cannell, Secretary R. S. Hill III, Contributing Member G. Bjorkman P. E. McConnell, Contributing

Member V. Broz

S. Horowitz A. B. Meichler, Contributing

D. W. Lewis Member

E. L. Pleins T. Saegusa, Contributing Member

R. H. Smith N. M. Simpson, Contributing

G. J. Solovey Member

# Subgroup on Fusion Energy Devices (BPV III)

W. K. Sowder, Jr., Chair D. Andrei, Staff Secretary X. Li D. J. Roszman, Secretary P. Mokaria L. C. Cadwallader T. R. Muldoon B. R. Doshi M. Porton M. Higuchi F. J. Schaaf, Jr. G. Holtmeier P. Smith M. Kalsev Y. Song K. A. Kavanagh M. Trosen K. Kim C. Waldon I. Kimihiro

I. J. Zatz S. Lee R. W. Barnes, Contributing Member

# Working Group on General Requirements (BPV III-4)

D. J. Roszman, Chair W. K. Sowder, Jr.

## Working Group on In-Vessel Components (BPV III-4)

M. Kalsey, Chair Y. Carin

# Working Group on Magnets (BPV III-4)

K. Kim, Vice Chair S. Lee, Chair

# Working Group on Materials (BPV-IN-4)

M. Porton, Chair

P. Mummery

## Working Group on Vacuum Vessels (BPV III-4)

I. Kimihiro, Chair B. R. Doshi

L. C. Cadwallader

## Subgroup on High Temperature Reactors (BPV III)

M. Morishita, Chair G. H. Koo R. I. Jetter, Vice Chair D. K. Morton S. Sham, Secretary J. E. Nestell N. Broom G. L. Zeng T. D. Burchell

X. Li, Contributing Member M. W. Davies S. Downey L. Shi, Contributing Member

# **Working Group on High Temperature Gas-Cooled Reactors** (BPV III-5)

J. E. Nestell, Chair T. Le T. R. Lupold M. Sengupta, Secretary N. Broom S. N. Malik T. D. Burchell D. L. Marriott M. W. Davies D. K. Morton R. S. Hill III S. Sham E. V. Imbro G. L. Zeng

R. I. Jetter X. Li, Contributing Member Y. W. Kim L. Shi, Contributing Member

## **Working Group on High Temperature Liquid-Cooled Reactors** (BPV III-5)

S. Sham, Chair R. I. Jetter G. H. Koo T. Asayama, Secretary M. Arcaro T. Le R. W. Barnes S. Majumdar P. Carter M. Morishita J. E. Nestell M. E. Cohen

A. B. Hull G. Wu, Contributing Member

# Argentina International Working Group (BPV III)

-	
O. Martinez, Staff Secretary	M. M. Gamizo
A. Acrogliano	A. Gomez
W. Agrelo	I. M. Guerreiro
G. O. Anteri	I. A. Knorr
M. Anticoli	M. F. Liendo
C. A. Araya	L. R. Miño
J. P. Balbiani	I. Monte
A. A. Betervide	R. L. Morard
D. O. Bordato	A. E. Pastor
G. Bourguigne	E. Pizzichini
M. L. Cappella	A. Politi
A. Claus	J. L. Racamato
R. G. Cocco	H. C. Sanzi
A. Coleff	
A. J. Dall'Osto	G. J. Scian
L. M. De Barberis	G. G. Sebastian
D. P. Delfino	M. E. Szarko
D. N. Dell'Erba	P. N. Torano
F. G. Diez	A. Turrin
A. Dominguez	<ol><li>A. Verastegui</li></ol>
S. A. Echeverria	M. D. Vigliano

# China International Working Group (BPV III)

P. Yamamoto

M. Zunino

J. Yan, <i>Chair</i>	G. Sun
W. Tang, Vice Chair	Z. Sun
C. A. Sanna, Staff Secretary	G. Tang
Y. He, Secretary	L. Ting
L. Guo	Y. Tu
Y. Jing	Y. Wang
D. Kang	H. Wu
Y. Li	X. Wu
B. Liang	S. Xue
H. Lin	Z. Yin
S. Liu	G. Zhang
W. Liu	W. Zhang
J. Ma	W. Zhao
K. Mao	Y. Zhong
W. Pei	Z. Zhong

J. Fernández

E. P. Fresquet

# Germany International Working Group (BPV III)

C. Huttner <i>, Chair</i>	C. Krumb
HR. Bath, Secretary	W. Mayinger
B. Arndt	D. Moehring
M. Bauer)	D. Ostermann
G. Daum	G. Roos
R. Doring	J. Rudolph
L. Gerstner	C. A. Sanna
G. Haenle	H. Schau
KH. Herter	R. Trieglaff
R. E. Hueggenberg	P. Völlmecke
E. Iacopetta	J. Wendt
U. Jendrich	F. Wille
D. Koelbl	M. Winter
G. Kramarz	N. Wirtz

# India International Working Group (BPV III)

B. Basu, Chair	S. Kovalai
G. Mathivanan, Vice Chair	D. Kulkarni
C. A. Sanna, Staff Secretary	M. Ponnusamy
S. B. Parkash, Secretary	R. N. Sen
A. D. Bagdare	K. R. Shah
V. Bhasin	A. Sundararajan

India International Working Group (BPV III)		
B. Basu, <i>Chair</i>	S. Kovalai	
G. Mathivanan, Vice Chair	D. Kulkarni	
C. A. Sanna, Staff Secretary	M. Ponnusamy	
S. B. Parkash, Secretary	R. N. Sen	
A. D. Bagdare	K. R. Shah	
V. Bhasin	A. Sundararajan	
	M. Ponnusamy R. N. Sen K. R. Shah A. Sundararajan	
	, o	
Korea International Wo	rking Group (BPV III)	
G. H. Koo, Chair	D. Kwon B. Lee D. Lee	
S. S. Hwang, Vice Chair	B. Lee	
OS. Kim, Secretary	D. Lee	
H. S. Byun	Sanghoon Lee	
GS. Choi	Sangil Lee	
S. Choi	SG. Lee	
J. Y. Hong	H. Lim	
NS. Huh	K-K. Nam	
JK. Hwang	B. Noh	
C. Jang	CK. Oh	
I. I. Jeong	C. Park	
H. J. Kim J. Kim JS. Kim K. Kim	H. Park	
J. Kim		
JS. Kim	JS. Park	
	T. Shin	
MW. Kim	S. Song	
YB. Kim	J. S. Yang	
YS. Kim	0. Yoo	

# Special Working Group on Editing and Review (BPV III)

D. E. Matthews, <i>Chair</i>	D. K. Morton
R. L. Bratton	L. M. Plante
R. P. Deubler	R. F. Reedy, Sr.
A. C. Eberhardt	C. Wilson
I C Minichiello	

# Special Working Group on HDPE Stakeholders (BPV III)

D. Burwell, <i>Chair</i>	D. Keller
S. Patterson, Secretary	M. Lashley
T. M. Adams	K. A. Manoly
M. Brandes	D. P. Munson
S. Bruce	T. M. Musto
S. Choi	J. E. O'Sullivan
C. M. Faidy	V. Rohatgi
M. Golliet	F. J. Schaaf, Jr.
J. Grimes	, , ,
R. M. Jessee	R. Stakenborghs
J. Johnston, Jr.	M. Troughton

# Special Working Group on Honors and Awards (BPV III)

R. M. Jessee, Chair	D. E. Matthews
A. Appleton	J. C. Minichiello
R. W. Barnes	

# Special Working Group on Industry Experience for New Plants (BPV III & BPV XI)

J. T. Lindberg, <i>Chair</i>	OS. Kim
E. L. Pleins, <i>Chair</i>	YS. Kim
J. Ossmann, Secretary	K. Matsunaga
T. L. Chan	D. E. Matthews
H. L. Gustin	R. E. McLaughlin
P. J. Hennessey	O
D. O. Henry	D. W. Sandusky
J. Honcharik	T. Tsuruta
E. V. Imbro	R. M. Wilson
C. G. Kim	S. M. Yee

# Special Working Group on International Meetings (BPV III)

C. T. Smith, Chair	R. S. Hill III
A. Byk, Staff Secretary	M. N. Mitchell
T. D. Burchell	R. F. Reedy, Sr.
S. W. Cameron	C. A. Sanna

# R. L. Crane

# Special Working Group on New Plant Construction Issues (BPV III)

E. L. Pleins, Chair	M. Kris
M. C. Scott, Secretary	J. C. Minichiello
A. Byk	D. W. Sandusky
A. Cardillo	R. R. Stevenson
P. J. Coco	R. Troficanto
J. Honcharik E. V. Imbro	M. L. Wilson
OS Kim	J. Yan
O5 Killi	j. Tan

# Special Working Group on Regulatory Interface (BPV III)

E. V. Imbro, Chair	K. Matsunaga
P. Malouines, Secretary	D. E. Matthews
S. Bell	B. McGlone
A. Cardillo	A. T. Roberts III
P. J. Coco	R. R. Stevenson
,	
J. Honcharik	M.L. Wilson
J. Grimm J. Honcharik	R. R. Stevenson M.L. Wilson

# COMMITTEE ON HEATING BOILERS (BPV IV)

The second secon	•
J. A. Hall <i>, Chair</i>	G. Scribner
T. L. Bedeaux, Vice Chair	R. D. Troutt
G. Moino, Staff Secretary	M. Wadkinson
B. Calderon	R. V. Wielgoszinski
J. Calland	H. Michael, Delegate
J. P. Chicoine	D. Picart, Delegate
J. M. Downs	A. Heino, Contributing Member
B. J. Iske	S. V. Voorhees, Contributing
J. Klug	Member
P. A. Molvie	J. L. Kleiss, Alternate

# Subgroup on Care and Operation of Heating Boilers (BPV IV)

M. Wadkinson, Chair	J. A. Hall
T. L. Bedeaux	P. A. Molvie

J. Calland C. Lasarte, Contributing Member

J. M. Downs

# Subgroup on Cast Boilers (BPV IV)

J. P. Chicoine, <i>Chair</i>	J. A. Hall
T. L. Bedeaux, Vice Chair	J. L. Kleiss

## J. M. Downs

## Subgroup on Materials (BPV IV)

M. Wadkinson, <i>Chair</i>	J. A. Hall
J. Calland	A. Heino
J. M. Downs	B. J. Iske

## Subgroup on Water Heaters (BPV IV

J. Calland <i>, Chair</i>	R. E. Olson
L. Badziagowski	M. A. Taylor
J. P. Chicoine	T. E. Trant
C. Dinic	R. D. Troutt
B. J. Iske	

## Subgroup on Welded Boilers (BPV IV

P. A. Molvie, Chair	J. L. Kleiss
L. Badziagowski	R. E. Olson
T. L. Bedeaux	G. Scribner
B. Calderon	R. D. Troutt
J. Calland	M. Wadkinson
C. Dinic	R. V. Wielgoszinski

W. Hembree, Chair

# COMMITTEE ON NONDESTRUCTIVE EXAMINATION (BPV V)

R. W. Kruzic C. May

J. S. Brzuszkiewicz, Staff Secretary	A. B. Nagel
S. J. Akrin	T. L. Plasek
J. E. Batey	F. J. Sattler
P. L. Brown	P. B. Shaw
M. A. Burns	
B. Caccamise	G. M. Gatti, Delegate
C. Emslander	X. Guiping, <i>Delegate</i>
N. Y. Faransso	A. S. Birks, Contributing Member
N. A. Finney	J. Bennett, Alternate
A. F. Garbolevsky	H. C. Graber, Honorary Member
•	O. F. Hedden, Honorary Member
J. F. Halley	
J. W. Houf	J. R. MacKay, Honorary Member
S. A. Johnson	T. G. McCarty, Honorary Member

# Executive Committee (BPV V)

Exceutive de	minitiee (Bi V V)
F. B. Kovacs, Chair	N. Y. Faransso
G. W. Hembree, Vice Chair	N. A. Finney
J. S. Brzuszkiewicz, Staff Secretary	S. A. Johnson
J. E. Batey	A. B. Nagel
B Caccamise	_

# Subgroup on General Requirements/Personnel Qualifications and Inquiries (BPV V)

C. Emslander, <i>Chair</i>	S. A. Johnson
J. W. Houf <i>, Vice Chair</i>	F. B. Kovacs
S. J. Akrin	D. I. Morris
J. E. Batey	
N. Carter	A. B. Nagel
N. Y. Faransso	A. S. Birks, Contributing Member

N. A. Finney J. P. Swezy, Jr., Contributing

G. W. Hembree Member

# Special Working Group on NDE Resource Support (SG-GR/PQ & I) (BPV V)

N. A. Finney, Chair	R. Kelso
D. Adkins	C. Magruder
J. Anderson	J. W. Mefford, Jr.
D. Bajula	K. Page
J. Bennett	D. Tompkins
C. T. Brown	D. Van Allen
T. Clausing	T. Vidimos
J. L. Garner	R. Ward
K. Hayes	M. Wolf

# Subgroup on Surface Examination Methods (BPV V)

S. A. Johnson, <i>Chair</i>	G. W. Hembre
J. Halley, Vice Chair	R. W. Kruzic
S. J. Akrin	B. D. Laite
J. E. Batey	C. May
P. L. Brown	L. E. Mullins
B. Caccamise	A. B. Nagel
N. Carter	F. J. Sattler
N. Y. Faransso	P. B. Shaw
N. Earanhaugh	C M Catti Da

N. Farenbaugh G. M. Gatti, *Delegate* 

N. A. Finney A. S. Birks, Contributing Member

# Subgroup on Volumetric Methods (BPV V)

A D Namel Chain	C M Hambus
A. B. Nagel, Chair	G. W. Hembree
N. A. Finney, Vice Chair	S. A. Johnson
S. J. Akrin	F. B. Kovacs
J. E. Batey	R. W. Kruzic
P. L. Brown	C. May
B. Caccamise	•
J. M. Davis	L. E. Mullins
N. Y. Faransso	T. L. Plasek
A. F. Garbolevsky	F. J. Sattler
I. F. Halley	C. Vorwald
,	C.M.C. III. D.I.
R. W. Hardy	G. M. Gatti. Delegate

# Working Group on Acoustic Emissions (SG-VM) (BPV V)

N. Y. Faransso, *Chair* S. R. Doctor J. E. Batey, *Vice Chair* R. K. Miller

# Working Group on Radiography (SG-VM) (BPV V)

	. 0.
B. Caccamise, Chair	G. W. Hembree
F. B. Kovacs, Vice Chair	S. A. Johnson
S. J. Akrin	R. W. Kruzic
J. E. Batey	B. D. Laite
P. L. Brown	C. May
C. Emslander	R. J. Mills
N. Y. Faransso	A. B. Nagel
A. F. Garbolevsky	T. L. Plasek
R. W. Hardy	B. White

# Working Group on Ultrasonics (SG-VM) (BPV V)

( )	
N. A. Finney <i>, Chair</i>	R. W. Kruzic
J. F. Halley <i>, Vice Chair</i>	B. D. Laite
B. Caccamise	C. May
J. M. Davis	L. E. Mullins
C. Emslander	
N. Y. Faransso	A. B. Nagel
P. T. Hayes	F. J. Sattler
S. A. Johnson	C. Vorwald

## Working Group on Guided Wave Ultrasonic Testing (SG-VM) (BPV V)

N. Y. Faransso, Chair	S. A. Johnson
J. E. Batey, Vice Chair	G. M. Light
D. Alleyne	P. Mudge
N. Amir	M. J. Quarry
J. F. Halley	J. Vanvelsor

# Italy International Working Group (BPV V)

P. L. Dinelli <i>, Chair</i>	M. A. Grimoldi
A. Veroni, Secretary	G. Luoni
R. Bertolotti	O. Oldani
F. Bresciani	P. Pedersoli
G. Campos	
N. Caputo	A. Tintori
M. Colombo	M. Zambon
F. Ferrarese	G. Gobbi, Contributing Member

E. Ferrari

# COMMITTEE ON PRESSURE VESSELS (BPV VIII)

G. Pontiggia, Contributing Member

COMMITTEE ON FRESSO	NE VESSELS (BFV VIII)
R. J. Basile, Chair	G. B. Rawls, Jr.
S. C. Roberts, Vice Chair	F. L. Richter
E. Lawson, Staff Secretary	C. D. Rodery
S. J. Rossi, Staff Secretary	E. Soltow
G. Aurioles, Sr.	J. C. Sowinski
J. Cameron	D. B. Stewart
A. Chaudouet	D. A. Swanson
D. B. DeMichael	J. P. Swezy, Jr.
J. P. Glaspie	S. Terada
J. F. Grubb	E. Upitis
L. E. Hayden, Jr.	R. Duan, Delegate
G. G. Karcher	P. A. McGowan, Delegate
D. L. Kurle K. T. Lau	H. Michael, Delegate
M. D. Lower	K. Oyamada, Delegate
R. Mahadeen	M. E. Papponetti, Delegate
R. W. Mikitka	X. Tang, Delegate
U. R. Miller	M. Gold, Contributing Member
B. R. Morelock	W. S. Jacobs, Contributing Member
T. P. Pastor	K. Mokhtarian, Contributing
	,

D. T. Peters Member
M. J. Pischke C. C. Neely, Contributing Member
M. D. Rana K. K. Tam, Honorary Member

# Subgroup on Design (BPV VIII)

D. A. Swanson, <i>Chair</i>	M. D. Rana
J. C. Sowinski, Vice Chair	G. B. Rawls, Jr.
M. Faulkner, Secretary	S. C. Roberts
G. Aurioles, Sr.	C. D. Rodery
S. R. Babka	T. G. Seipp
O. A. Barsky	D. Srnic
R. J. Basile	S. Terada
M. R. Breach	J. Vattappilly
F. L. Brown	R. A. Whipple
D. Chandiramani	K. Xu
B. F. Hantz	K. Oyamada, Delegate
C. E. Hinnant	
C. S. Hinson	M. E. Papponetti, Delegate
M. H. Jawad	W. S. Jacobs, Contributing Member
D. L. Kurle	P. K. Lam, Contributing Member
M. D. Lower	K. Mokhtarian, Contributing
R. W. Mikitka	Member
U. R. Miller	S. C. Shah, Contributing Member

K. K. Tam, Contributing Member

T. P. Pastor

## Working Group on Design-By-Analysis (BPV VIII)

B. F. Hantz, Chair	S. Krishnamurthy
T. W. Norton, Secretary	A. Mann
R. G. Brown	G. A. Miller
D. Dewees	C. Nadarajah
R. D. Dixon	P. Prueter
Z. Gu	M. D. Rana
C. F. Heberling II	T. G. Seipp
C. E. Hinnant	M. A. Shah
R. Jain	S. Terada

M. H. Jawad D. Arnett, Contributing Member

# Task Group on U-2(g) (BPV VIII)

rush drou	pon o z(g) (bi v viii)
G. Aurioles, Sr.	R. F. Reedy, Sr.
S. R. Babka	S. C. Roberts
R. J. Basile	M. A. Shah
D. K. Chandiramani	D. Srnic
R. Mahadeen	D. A. Swanson
U. R. Miller	J. P. Swezy, Jr.
T. W. Norton	R. Uebel
T. P. Pastor	K. K. Tam, Contributing Member
	aion <sup>3</sup> O
	Co
Subgroup on Heat Transfer Equipment (BPV VIII)	
G. Aurioles, Sr., Chair	U. R. Miller

# Subgroup on Fabrication and Examination (BPV VIII)

J. P. Swezy, Jr., Chair	B. F. Shelley
D. I. Morris, Vice Chair	P. L. Sturgill
E. A. Whittle, Vice Chair	E. Upitis
B. R. Morelock, Secretary	K. Oyamada, <i>Delegate</i>
N. Carter	W. J. Bees, Contributing Member
S. Flynn	L. F. Campbell, <i>Contributing</i>
S. Heater	Member
O. Mulet	
M. J. Pischke	W. S. Jacobs, Contributing Member
M. J. Rice	J. Lee, Contributing Member

# Subgroup on Heat Transfer Equipment (BPV VIII)

C Aurialas Cu Chain	U. R. Miller
G. Aurioles, Sr., Chair	U. R. Miller
S. R. Babka, Vice Chair	D. Srnic
P. Matkovics, Secretary	A. M. Voytko
D. Angstadt	R. P. Wiberg
M. Bahadori	I. G. Campbell, <i>Contributing</i>
J. H. Barbee	Member
O. A. Barsky	I. Garcia, Contributing Member
L. Bower	
A. Chaudouet	J. Mauritz, Contributing Member
· · · · · · · · · · · · · · · · · · ·	T. W. Norton, Contributing Member
M. D. Clark	, 0
S. Jeyakumar	F. Osweiller, Contributing Member
G. G. Karcher	J. Pasek, Contributing Member
D. L. Kurle	R. Tiwari, Contributing Member
R. Mahadeen	S. Yokell, Contributing Member
S. Mayeux	S. M. Caldwell, Honorary Member

# Subgroup on General Requirements (BPV VIII)

R. Uebel, Contributing Member

M. D. Lower, <i>Chair</i>	K. T. Lau
J. P. Glaspie, Vice Chair	T. P. Pastor
F. L. Richter, Secretary	S. C. Roberts
R. J. Basile	J. C. Sowinski
D. T. Davis	P. Speranza
D. B. DeMichael	D. B. Stewart
M. Faulkner	D. A. Swanson
F. Hamtak	R. Uebel
L. E. Hayden, Jr.	C. C. Neely, Contributing Member
	Ø.
	· ·

C. D. Rodery

# Task Group on Plate Heat Exchangers (BPV VIII)

P. Matkovics, Chair	R. Mahadeen
S. R. Babka	D. I. Morris
K. Devlin	M. J. Pischke
S. Flynn	C. M. Romero
J. F. Grubb	E. Soltow
F. Hamtak	D. Srnic

# Task Group on Subsea Applications (BPV VIII)

R. Cordes, Chair	F. Kirkemo
L. P. Antalffy	C. Lan
R. C. Biel	N. McKie
P. Bunch	S. K. Parimi
J. Ellens	M. Sarzynski
S. Harbert	•
X. Kaculi	Y. Wada
K. Karpanan	D. T. Peters, Contributing Member

Subgroup on High Pr	essure Vessels (BPV VIII)
D. T. Peters, Chair G. M. Mital, Vice Chair A. P. Maslowski, Staff Secretary L. P. Antalffy R. C. Biel P. N. Chaku R. Cordes R. D. Dixon L. Fridlund R. T. Hallman A. H. Honza J. A. Kapp J. Keltjens A. K. Khare N. McKie S. C. Mordre G. T. Nelson	E. A. Rodriguez E. D. Roll K. C. Simpson, Jr. J. R. Sims D. L. Stang F. W. Tatar S. Terada J. L. Traud R. Wink KJ. Young R. M. Hoshman, Contributing Member D. J. Burns, Honorary Member D. M. Fryer, Honorary Member G. J. Mraz, Honorary Member E. H. Perez, Honorary Member

# Task Group on UG-20(f) (BPV VIII)

S. Krishnamurthy, Chair B. R. Macejko T. Anderson J. Penso K. Bagnoli M. Prager R. P. Deubler M. D. Rana B. F. Hantz

# Subgroup on Materials (BPV VIII)

### J. Cameron, Chair D. W. Rahoi P. G. Wittenbach, Vice Chair R. C. Sutherlin K. Xu, Secretary E. Upitis

A. Di Rienzo G. S. Dixit, Contributing Member J. D. Fritz M. Gold, Contributing Member J. F. Grubb M. Katcher, Contributing Member M. Kowalczyk J. A. McMaster, Contributing

W. M. Lundy Member

E. G. Nisbett, Contributing Member J. Penso

## Italy International Working Group (BPV VIII)

.eli
I. Venier
G. Gobbi, Contributing Member G. Pontiggia, Chair M. Guglielmetti A. Veroni, Secretary B. G. Alborali P. Angelini R. Boatti A. Camanni P. Conti P. L. Dinelli

F. Finco

# Subgroup on Toughness (BPV II & BPV VIII)

### D. L. Kurle, Chair J. P. Swezy, Jr. K. Xu, Vice Chair S. Terada N. Carter E. Upitis W. S. Jacobs J. Vattappilly K. E. Orie K. Oyamada, Delegate M. D. Rana K. Mokhtarian, Contributing F. L. Richter Member K. Subramanian

D. A. Swanson C. C. Neely, Contributing Member

# Special Working Group on Bolted Flanged Joints (BPV VIII)

Working Group on Design (BPV VIII Div. 3)

K. C. Simpson

R. W. Mikitka, Chair J. R. Payne W. Brown G. B. Rawls, Jr. M. S. Shelton H. Chen W. J. Koves

## C. Becht V

E. D. Roll, Chair

J. R. Sims R. C. Biel D. L. Stang R. Cordes K. Subramanian R. D. Dixon S. Terada L. Fridlund J. L. Traud R. T. Hallman R. Wink K. Karpanan Y. Xu

J. Keltjens F. Kirkemo, Contributing Member N. McKie D. J. Burns, Honorary Member G. M. Mital D. M. Fryer, Honorary Member S. C. Mordre G. T. Nelson G. J. Mraz, Honorary Member D. T. Peters E. H. Perez, Honorary Member

## Subgroup on Graphite Pressure Equipment (BPV VIII)

A. Viet, Chair C. W. Cary G. C. Becherer E. Soltow F. L. Brown A. A. Stupica

# China International Working Group (BPV VIII)

X. Chen, Chair D. Luo Y. Luo B. Shou, Vice Chair Z. Fan, Secretary C. Miao X. Qian Y. Chen Z. Chen B. Wang J. Cui F. Xu R. Duan F. Xuan W. Guo K. Zhang B. Han Y. Zhang J. Hu S. Zhao Q. Hu J. Zheng H. Hui G. Zhu

## Working Group on Materials (BPV VIII Div. 3)

F. W. Tatar, Chair J. A. Kapp L. P. Antalffy A. K. Khare

P. N. Chaku

# Task Group on Impulsively Loaded Vessels (BPV VIII)

E. A. Rodriguez, Chair R. A. Leishear G. A. Antaki P. O. Leslie Germany International Working Group (BPV VIII) J. K. Asahina F. Ohlson D. Koelbl D. D. Barker C. Romero A. M. Clayton N. Rushton S. Krebs J. E. Didlake, Jr. J. H. Stofleth T. Ludwig T. A. Duffey Q. Dong, Contributing Member R. A. Meyers

B. L. Haroldsen H.-P. Schildberg, Contributing K. Hayashi Member

D. Hilding J. E. Shepherd, Contributing

K. W. King Member

R. Kitamura M. Yip, Contributing Member

P. Chavdarov, Chair A. Spangenberg, Vice Chair H. P. Schmitz, Secretary B. Daume A. Emrich H. Michael J. Fleischfresser P. Paluszkiewicz A. Gastberg H. Schroeder R. Helmholdt R. Kauer M. Sykora

# Subgroup on Interpretations (BPV VIII)

U. R. Miller, Chair	D. I. Morris
E. Lawson, Staff Secretary	D. T. Peters
G. Aurioles, Sr.	S. C. Roberts
R. J. Basile	C. D. Rodery
J. Cameron	D. B. Stewart
R. D. Dixon	P. L. Sturgill
M. Kowalczyk	D. A. Swanson
D. L. Kurle	J. P. Swezy, Jr.
M. D. Lower	J. Vattappilly
R. Mahadeen	P. G. Wittenbach

G. M. Mital T. P. Pastor, Contributing Member

# Subgroup on Plastic Fusing (BPV IX)

E. W. Woelfel <i>, Chair</i>	J. E. O'Sullivan
D. Burwell	E. G. Reichelt
M. Ghahremani	M. J. Rice
K. L. Hayes	S. Schuessler
R. M. Jessee	M. Troughton
. Johnston, Jr.	J. Wright

## COMMITTEE ON WELDING, BRAZING, AND FUSING (BPV IX)

D. A. Bowers, <i>Chair</i>	W. J. Sperko
M. J. Pischke, Vice Chair	M. J. Stanko
S. J. Rossi, Staff Secretary	P. L. Sturgill
M. Bernasek	J. P. Swezy, Jr.
M. A. Boring	P. L. Van Fosson
J. G. Feldstein	E. W. Woelfel
P. D. Flenner	A. Roza, Delegate
S. E. Gingrich	M. Consonni, Contributing Member
K. L. Hayes	S. A. Jones, Contributing Member
R. M. Jessee	A. S. Olivares, Contributing
J. S. Lee	Member
W. M. Lundy	S. Raghunathan, Contributing
T. Melfi	Member
W. F. Newell, Jr.	R. K. Brown, Jr., Honorary Member
D. K. Peetz	M. L. Carpenter, Honorary Member
E. G. Reichelt	B. R. Newmark, Honorary Member
M. J. Rice	S. D. Reynolds, Jr., Honorary
M. B. Sims	Member

# Subgroup on Welding Qualifications (RPV IX)

<b>5</b> . 5 .	
M. J. Rice, Chair	E. G. Reichelt
J. S. Lee, Vice Chair	M. B. Sims
M. Bernasek	W.J. Sperko
M. A. Boring	S. A. Sprague
D. A. Bowers	P. L. Sturgill
R. B. Corbit	J. P. Swezy, Jr.
P. D. Flenner	P. L. Van Fosson
L. S. Harbison	
K. L. Hayes	T. C. Wiesner
W. M. Lundy	A. D. Wilson
T. Melfi	D. Chandiramani, Contributing
W. F. Newell, Jr.	Member
B. R. Newton	M. Consonni, Contributing Member
S. Raghunathan	M. Degan, Contributing Member

# Subgroup on Brazing (BPV IX)

M. J. Pischke, Chair	A. F. Garbolevsky
E. W. Beckman	N. Mohr
L. F. Campbell	A. R. Nywening
M. L. Carpenter	J. P. Swezy Jr.

# Subgroup on General Requirements (BPV IX)

P. L. Sturgill, <i>Chair</i>	R. M. Jessee
E. W. Beckman	D. Mobley
J. P. Bell	D. K. Peetz
D. A. Bowers	J. Pillow
G. Chandler	H. B. Porter
P. R. Evans	J. P. Swezy, Jr.
S. Flynn	K. R. Willens
P. Gilston	E. W. Woelfel
F. Hamtak	E. Molina, Delegate
A. Howard	B. R. Newmark. Ho.

# B. R. Newmark, Honorary Member

# Italy International Working Group (BPV IX)

N. Maestri

A. Veroni, Secretary	M. Mandina
P. Angelini	M. Massobrio
R. Boatti	L. Moracchioli
P. L. Dinelli	G. Pontiggia
F. Ferrarese A. Ghidini	S. Verderame
E. Lazzari	A. Volpi
L. Lotti	G. Gobbi, Contributing Member

COMMITTEE ON FIBER-REINFORCED PLASTIC PRESSURE VESSELS

# (BPV X)

A. Camanni, Chair

D. Eisberg, Chair	D. H. Hodgkinson
B. F. Shelley, Vice Chair	L. E. Hunt
P. D. Stumpf, Staff Secretary	D. L. Keeler
A. L. Beckwith	B. M. Linnemann
D. Bentley	D. H. McCauley
F. L. Brown	N. L. Newhouse
J. L. Bustillos	D. J. Painter
B. R. Colley	,
T. W. Cowley	A. A. Pollock
I. L. Dinovo	G. Ramirez
M. R. Gorman	J. R. Richter
B. Hebb	D. O. Yancey, Jr.
M. J. Hendrix	P. H. Ziehl

# Subgroup on Materials (BPV IX)

<i>)</i>	
M. Bernasek <i>, Chair</i>	M. J. Pischke
T. Anderson	A. Roza
J. L. Arnold	C. E. Sainz
E. Cutlip	W. J. Sperko
S. S. Fiore	, ,
S. E. Gingrich	M. J. Stanko
L. S. Harbison	P. L. Sturgill
R. M. Jessee	J. Warren
T. Melfi	C. Zanfir

# COMMITTEE ON NUCLEAR INSERVICE INSPECTION (BPV XI)

G. C. Park, <i>Chair</i>	S. A. Norman
S. D. Kulat, Vice Chair	J. E. O'Sullivan
R. W. Swayne, Vice Chair	R. K. Rhyne
L. Powers, Staff Secretary	A. T. Roberts III
V. L. Armentrout	D. A. Scarth
J. F. Ball	F. J. Schaaf, Jr.
W. H. Bamford	J. C. Spanner, Jr.
S. B. Brown	D. J. Tilly
T. L. Chan	D. E. Waskey
R. C. Cipolla	J. G. Weicks
D. R. Cordes	H. D. Chung, Deleg
D. D. Davis	C. Ye, Delegate

D R. L. Dyle E. V. Farrell, Jr. M. I. Ferlisi P. D. Fisher E. B. Gerlach T. J. Griesbach J. Hakii

D. O. Henry W. C. Holston D. W. Lamond D. R. Lee G. A. Lofthus E. J. Maloney

G. Navratil

gate R. E. Gimple, Contributing Member

R. D. Kerr, Contributing Member B. R. Newton. Contributing Member R. A. West, Contributing Member R. A. Yonekawa, Contributing Member

M. L. Benson, Alternate J. T. Lindberg, Alternate R. O. McGill, Alternate C. J. Wirtz, Alternate

C. D. Cowfer, Honorary Member F. E. Gregor, Honorary Member O. F. Hedden, Honorary Member P. C. Riccardella, Honorary Member

# China International Working Group (BPV XI)

J. H. Liu <i>, Chair</i>	Y. Liu
Y. Nie, Vice Chair	W. N. Pei
C. Ye, Vice Chair	C. L. Peng
M. W. Zhou, Secretary	G. X. Tang
J. F. Cai	Q. Wang
D. X. Chen	Q. W. Wang
H. Chen	Z. S. Wang
H. D. Chen	L. Wei
Y. B. Guo	F. Xu
Y. Hou	Z. Y. Xu
D. M. Kang	Q. Yin
S. W. Li	K. Zhang
X. Y. Liang	X. L. Zhang
S. X. Lin	Y. Zhang
L. Q. Liu	Z. M. Zhong

# Germany International Working Group (BPV XI)

H.-R. Bath U Jendrich R. Doring H. Schau B. Erhard H.-J. Scholtka M. Hagenbruch X. Schuler B. Hoffmann E. Iacopetta J. Wendt

## Special Working Group on Editing and Review (BPV XI)

R. W. Swayne, Chair J. E. Staffiera C. E. Moyer D. J. Tilly C. J. Wirtz

## **Executive Committee (BPV XI)**

S. D. Kulat, Chair W. C. Holston G. C. Park, Vice Chair D. W. Lamond L. Powers, Staff Secretary J. T. Lindberg W. H. Bamford R. K. Rhyne J. C. Spanner Jr. R. L. Dyle M. J. Ferlisi R. W. Swayne E. B. Gerlach M. L. Benson, Alternate

# Task Group on Inspectability (BPV XI)

J. T. Lindberg, Chair D. Lieb M. J. Ferlisi, Secretary G. A. Lofthus W. H. Bamford D. E. Matthews A. Cardillo P. J. O'Regan D. R. Cordes J. Ossmann D. O. Henry R. Rishel E. Henry S. A. Sabo J. Honcharik P. Sullivan I. Howard R. Klein C. Thomas C. Latiolais J. Tucker

# Argentina International Working Group (BPV XI)

O. Martinez, Staff Secretary R. J. Lopez D. A. Cipolla M. Magliocchi A. Claus L. R. Miño D. Costa I. Monte D. P. Delfino M. D. Pereda D. N. Dell'Erba A. Politi A. Dominguez C. G. Real S. A. Echeverria F. M. Schroeter E. P. Fresquet G. J. Scian M. M. Gamizo M. J. Solari I. M. Guerreiro P. N. Torano M. F. Liendo O. A. Verastegui F. Llorente P. Yamamoto

# Task Group on ISI of Spent Nuclear Fuel Storage and Transportation Containment Systems (BPV XI)

K. Hunter, Chair R. M. Meyer A. Alleshwaram, Secretary B. L. Montgomery D. J. Ammerman M. Moran W. H. Borter T. Nuoffer J. Broussard M. Orihuela S. Brown R. Pace C. R. Bryan E. L. Pleins T. Carraher R. Sindelar D. Dunn H. Smith N. Fales J. C. Spanner, Jr. R. C. Folley C. J. Temus B. Gutherman G. White S. Horowitz X. I. Zhai M. W. Joseph P.-S. Lam, Alternate H. Jung J. Wise, Alternate M. Liu

# Subgroup on Evaluation Standards (SG-ES) (BPV XI)

W. H. Bamford, Chair	Y. S. Li
N. A. Palm, Secretary	R. O. McGill
H. D. Chung	H. S. Mehta
R. C. Cipolla	K. Miyazaki
R. L. Dyle	R. Pace
C. M. Faidy	J. C. Poehler
B. R. Ganta	S. Ranganath
T. J. Griesbach	D. A. Scarth
K. Hasegawa	T. V. Vo
K. Hojo	K. R. Wichman
D. N. Hopkins	S. X. Xu

K. Koyama M. L. Benson, Alternate
D. R. Lee T. Hardin, Alternate

# Task Group on Evaluation of Beyond Design Basis Events (SG-ES) (BPV XI)

R. Pace, Chair	S. A. Kleinsmith
K. E. Woods, Secretary	H. S. Mehta
G. A. Antaki	D. V. Sommerville
P. R. Donavin	T. V. Vo
R. G. Gilada	K. R. Wichman
T. J. Griesbach	
H. L. Gustin	G. M. Wilkowski
M. Hayashi	S. X. Xu

K. Hojo T. Weaver, Contributing Member

# Task Group on Evaluation Procedures for Degraded Buried Pipe (WG-PFE) (BPV XI)

R. O. McGill, <i>Chair</i> S. X. Xu, <i>Secretary</i> G. A. Antaki R. C. Cipolla K. Hasegawa	G. A. A. Miessi M. Moenssens D. P. Munson R. Pace P. J. Rush
K. Hasegawa K. M. Hoffman	P. J. Rush D. A. Scarth

# Working Group on Operating Plant Criteria (SGES) (BPV XI)

N. A. Palm, Chair	A. D. Odell
A. E. Freed, Secretary	R. Pace
V. Marthandam, Secretary	J. C. Poebler
K. R. Baker	S. Ranganath
W. H. Bamford	W. L. Server
M. Brumovsky	D. V. Sommerville
T. L. Dickson	C. A. Tomes
R. L. Dyle	
S. R. Gosselin	A. Udyawar
T. J. Griesbach	T. V. Vo
M. Hayashi	D. P. Weakland
S. A. Kleinsmith	K. E. Woods
H. Kobayashi	H. Q. Xu
H.C.Malas	T II 1:

H. S. Mehta T. Hardin, Alternate

# Working Group on Flaw Evaluation (SG-ES) (BPV XI)

R. C. Cipolla <i>, Chair</i>	D. R. Lee
S. X. Xu, Secretary	Y. S. Li
W. H. Bamford	M. Liu
M. L. Benson	H. S. Mehta
B. Bezensek	G. A. A. Miessi
M. Brumovsky	K. Miyazaki
H. D. Chung	R. K. Qashu
T. E. Demers	S. Ranganath
C. M. Faidy	P.D. Rush
B. R. Ganta	D. A. Scarth
R. G. Gilada	W. L. Server
H. L. Gustin	DJ. Shim
F. D. Hayes	A. Udyawar
P. H. Hoang	•
K. Hojo 🙀 🔾	T. V. Vo
D. N. Hopkins	B. Wasiluk
Y. Kim	K. R. Wichman
K. Koyama	G. M. Wilkowski

D. R. Lee
Y. S. Li
M. Liu
H. S. Mehta
G. A. A. Miessi
K. Miyazaki
R. K. Qashu
S. Ranganath
P. P. Rush
D. A. Scarth
W. L. Server
C. M. Faidy
D.-J. Shim
A. Udyawar
T. V. Vo
B. Wasiluk
K. R. Wichman
G. M. Wilkowski
D. L. Rudland, Alternate
G. M. Wilkowski
D. N. Hopkins

Working Group on Pipe Flaw Evaluation (SG-ES) (BPV XI) D. A. Scarth, Chair E. J. Houston G. M. Wilkowski, Secretary R. Janowiak S. Kalyanam W. H. Bamford K. Kashima M. L. Benson V. Lacroix M. Brumovsky Y. S. Li F. W. Brust R. O. McGill H. D. Chung H. S. Mehta R. C. Cipolla G. A. A. Miessi K. Miyazaki S. H. Pellet T. E. Demers H. Rathbun C. M. Faidy P. J. Rush D.-J. Shim S. R. Gosselin A. Udyawar C. E. Guzman-Leong T. V. Vo K. Hasegawa B. Wasiluk P. H. Hoang S. X. Xu

## Task Group on Crack Growth Reference Curves (BPV XI)

rask droup on crack drowin	Mererence et
D. A. Scarth, Chair	D. N. Hopkins
H. I. Gustin, Secretary	K. Kashima
W. H. Bamford	K. Koyama
M. L. Benson	D. R. Lee
F. W. Brust	H. S. Mehta
R. C. Cipolla	K. Miyazaki
R. L. Dyle	S. Ranganath
K. Hasegawa	T. V. Vo

# Subgroup on Nondestructive Examination (SG-NDE) (BPV XI)

A. Alleshwaram, Alternate

J. C. Spanner, Jr., Chair	J. T. Lindberg	
D. R. Cordes, Secretary	G. A. Lofthus	
T. L. Chan	G. R. Perkins	
S. E. Cumblidge	S. A. Sabo	
F. E. Dohmen	F. J. Schaaf, Jr.	
K. J. Hacker	R. V. Swain	
J. Harrison	C. J. Wirtz	
D. O. Hammy	C A Nova Altama	

D. O. Henry C. A. Nove, Alternate

# Working Group on Personnel Qualification and Surface Visual and Eddy Current Examination (SG-NDE) (BPV XI)

J. T. Lindberg, <i>Chair</i>	D. O. Henry
J. E. Aycock, Secretary	J. W. Houf
C. Brown, Secretary	C. Shinsky
S. E. Cumblidge	J. C. Spanner, Jr.
A. Diaz	J. T. Timm
N. Farenhaugh	C. L. Wirtz

# Working Group on Procedure Qualification and Volumetric

Examination (SG-NDE) (BPV XI)

G. A. Lofthus, Chair	F. E. Dohmen
J. Harrison, Secretary	K. J. Hacker
G. R. Perkins, Secretary	D. A. Kull
M. T. Anderson	C. A. Nove
M. Briley	D. Nowakowski
A. Bushmire	S. A. Sabo
D. R. Cordes	R. V. Swain
M. Dennis	S. J. Todd
S. R. Doctor	D. K. Zimmerman

# Subgroup on Repair/Replacement Activities (SG-RRA) (BPV XI)

E. B. Gerlach, Chair	J. E. O'Sullivan
E. V. Farrell, Jr., Secretary	S. Schuessler
J. F. Ball	R. R. Stevenson
S. B. Brown	R. W. Swayne
R. Clow	•
P. D. Fisher	D. J. Tilly
K. J. Karwoski	D. E. Waskey
S. L. McCracken	J. G. Weicks
B. R. Newton	P. Raynaud, Alterno

# Working Group on Welding and Special Repair Processes (SG-RRA)

	(BPV XI)
D. E. Waskey, Chair	M. Kris
D. J. Tilly, Secretary	S. L. McCracken
D. Barborak	D. B. Meredith
S. J. Findlan	B. R. Newton
P. D. Fisher	
M. L. Hall	J. E. O'Sullivan
K. J. Karwoski	D. Segletes
C. C. Kim	J. G. Weicks
$O_{I_A}$	

# Working Group on Nonmetals Repair/Replacement Activities (SG-RRA) (BPV XI)

J. E. O'Sullivan, Chair	T. M. Musto
S. Schuessler, Secretary	S. Patterson
J. Johnston, Jr.	A. Pridmore
M. Lashley	P. Raynaud
M. P. Marohl	F. J. Schaaf, Jr.

# Task Group on Repair by Carbon Fiber Composites (WGN-MRR) (BPV XI)

J. E. O'Sullivan <i>, Chair</i>	P. Raynaud
B. Davenport	C. W. Rowley
M. Golliet	V. Roy
L. S. Gordon	J. Sealey
M. P. Marohl	N. Stoeva
N. Meyer	M. F. Uddin
R. P. Ojdrovic	J. Wen
D. Peguero	T. Iimenez. Al

D. Peguero T. Jimenez, *Alternate*A. Pridmore G. M. Lupia, *Alternate* 

# Working Group on Design and Programs (SG-RRA) (BPV XI)

S. B. Brown, Chair	H. Malikowski 🧲
A. B. Meichler, Secretary	M. A. Pyne
O. Bhatty	P. Raynaud
R. Clow	~~~
R. R. Croft	R. R. Stevenson
E. V. Farrell, Jr.	R. W. Swayne
E. B. Gerlach	R Turner
	@17

# Subgroup on Water-Cooled Systems (SG-WCS) (BPV XI)

D. W. Lamond, Chair	K. W. Hall
G Navratil, Secretary	P. J. Hennessey
J. M. Agold	K. Hoffman
V. L. Armentrout	S. D. Kulat
J. M. Boughman	T. Nomura
S. B. Brown S. T. Chesworth	T. Nuoffer
D. D. Davis	G. C. Park
H, Q. Do	H. M. Stephens, Jr.
M. J. Ferlisi	M. J. Homiack, Alternate

# Task Group on High Strength Nickel Alloys Issues (SG-WCS) (BPV XI)

R. L. Dyle, <i>Chair</i>	H. Malikowski
B. L. Montgomery, Secretary	S. E. Marlette
W. H. Bamford	G. C. Park
P. R. Donavin	G. R. Poling
K. Hoffman	J. M. Shuping
K. Koyama	J. C. Spanner, Jr.
C. Lohse	D. P. Weakland

# Working Group on Containment (SG-WCS) (BPV XI)

H. M. Stephens, Jr., Chair	J. McIntyre
S. G. Brown, Secretary	J. A. Munshi
P. S. Ghosal	M. Sircar
H. T. Hill	S. Walden. Alte

R. D. Hough S. Walden, *Alternate*B. Lehman T. J. Herrity, *Alternate* 

# Working Group on Inspection of Systems and Components (SG-WCS) (BPV XI)

M. J. Ferlisi, <i>Chair</i>	S. D. Kulat
N. Granback, Secretary	A. Lee
J. M. Agold	G. J. Navratil
R. W. Blyde	T. Nomura
C. Cueto-Felgueroso	
H. Q. Do	J. C. Nygaard
K. W. Hall	R. Rishel
K. M. Hoffman	J. C. Younger

xxxiv

### Working Group on Pressure Testing (SG-WCS) (BPV XI)

J. M. Boughman, Chair	A. E. Keyser
S. A. Norman, Secretary	D. W. Lamond
T. Anselmi	J. K. McClanahan
YK. Chung	B. L. Montgomery
M. J. Homiack	C. Thomas

### **Task Group on Buried Components Inspection and Testing** (WG-PT) (BPV XI)

D. W. Lamond, Chair	B. Davenport
J. M. Boughman, Secretary	A. Hiser
M. Moenssens, Secretary	J. Ossmann
T A 1 !	

### T. Anselmi

### Working Group on Risk-Informed Activities (SG-WCS) (BPV XI)

M. A. Pyne, Chair	D. W. Lamond
S. T. Chesworth, Secretary	R. K. Mattu
J. M. Agold	A. McNeill III
C. Cueto-Felgueroso	G. J. Navratil
R. Haessler	P. J. O'Regan
J. Hakii	, 0
K. W. Hall	N. A. Palm
M. J. Homiack	D. Vetter
S. D. Kulat	J. C. Younger

### Working Group on General Requirements (BPV XI)

R. K. Rhyne, Chair	P. J. Hennessey
C. E. Moyer, Secretary	E. J. Maloney
J. F. Ball	R. K. Mattu
T. L. Chan	T. Nuoffer

### Special Working Group on Reliability and Integrity Management Program (RPV XI)

,	riogiani (brv xij
F. J. Schaaf, Jr., Chair	D.M. Jones
A. T. Roberts III, Secretary	A. L. Krinzman
N. Broom	D. R. Lee
S. R. Doctor	R. K. Miller
S. Downey	M. N. Mitchell
J. D. Fletcher	R. Morrill
J. T. Fong	
T. Graham	T. Roney
N. Granback	R. W. Swayne
I Grimm	S Takaya

### **COMMITTEE ON TRANSPORT TANKS (BPV XII)**

M. D. Rana <i>, Chair</i>	M. Pitts
N. J. Paulick <i>, Vice Chair</i>	T. A. Rogers
R. Lucas <i>, Staff Secretary</i>	S. Staniszewski
A. N. Antoniou	A. P. Varghese
P Chilukuri	I A Rvers Contr

P. Chilukuri	J. A. Byers, Contributing Member
W. L. Garfield	R. Meyers, Contributing Member
G. G. Karcher	M. R. Ward, Contributing Member

### Executive Committee (BPV XII)

N. J. Paulick, Chair	M. D. Rana
R. Lucas, Staff Secretary	S. Staniszewski
M. Pitts	A. P. Varghese

### Subgroup on Design and Materials (BPV XII)

gg.	
A. P. Varghese, Chair	T. A. Rogers
R. C. Sallash, Secretary	S. Staniszewski
D. K. Chandiramani	K. Xu
P. Chilukuri	4 m B 11 d 11 11
Y. Doron	A. T. Duggleby, Contributing
R. D. Hayworth	Member
A	T. J. Hitchcock, Contributing
G. G. Karcher	,
S. L. McWilliams	Member
N. J. Paulick	M. R. Ward, Contributing Member

### Subgroup on Fabrication, Inspection, and Continued Service (BPV XII)

M. Pitts, Chair	L. Selensky
P. Chilukuri, Secretary	S. Staniszewski
R. D. Hayworth	S. E. Benet, Contributing Member
K. Mansker	J. A. Byers, Contributing Member
G. McRae	A. S. Olivares, Contributing
O. Mulet	Member
T. A. Rogers	L. H. Strouse, Contributing Member
M. Rudek	S. V. Voorhees, Contributing
R. C. Sallash	Member

### Subgroup on General Requirements (BPV XII)

J. T. Fong	K. MOITH
T. Graham	T. Roney
N. Granback	R. W. Swayne
J. Grimm	S. Takaya
SME/ASME J	oint Task Group for System-Based Code (SWG-RIM) (BPV XI)
T. Asayama, Ch	air D. R. Lee
S. R. Doctor	H. Machida
K. Dozaki	M. Morishita
S. R. Gosselin	A. T. Roberts III
M. Hayashi	F. J. Schaaf, Jr.
D. M. Jones Y. Kamishima	S. Takaya
A. L. Krinzman	D. Watanabe

	Subgroup
S. Staniszev	vski, <i>Chair</i>
B. F. Pittel,	Secretary
A. N. Anton	iou
Y. Doron	
J. L. Freiler	
W. L. Garfie	ld
O. Mulet	
M. Pitts	
T. Rummel	
R. C. Sallash	1
L. Selensky	

P. Chilukuri, Contributing Member K. L. Gilmore, Contributing Member T. J. Hitchcock, Contributing Member G. McRae, Contributing Member S. L. McWilliams, Contributing Member T. A. Rogers, Contributing Member D. G. Shelton, Contributing Member L. H. Strouse, Contributing Member M. R. Ward, Contributing Member

J. Zheng, Contributing Member

### Subgroup on Nonmandatory Appendices (BPV XII)

### N. J. Paulick, Chair S. Staniszewski, Secretary

P. Chilukuri R. D. Hayworth

K. Mansker S. L. McWilliams N. J. Paulick

M. Pitts T. A. Rogers R. C. Sallash

J. P. Glaspie

D. G. Shelton

S. E. Benet, Contributing Member

D. D. Brusewitz, Contributing Member

T. J. Hitchcock, Contributing

Member

A. P. Varghese, Contributing

Member

M. R. Ward, Contributing Member

#### Subcommittee on Safety Valve Requirements (SC-SVR)

Z. Wang

D. B. DeMichael, Chair W. F. Hart C. E. O'Brien, Staff Secretary D. Miller J. F. Ball B. K. Nutter J. Burgess T. Patel S. Cammeresi M. Poehlmann J. A. Cox

R. D. Danzy J. A. West J. P. Glaspie

section 3 Division 1 NK S. F. Harrison S. R. Irvin, Sr., Alternate

### COMMITTEE ON OVERPRESSURE PROTECTION (BPV XIII)

D. B. DeMichael, Chair S. F. Harrison C. E. O'Brien, Staff Secretary W. F. Hart D. Miller J. F. Ball J. Burgess B. K. Nutter T. Patel S. Cammeresi J. A. Cox M. Poehlmann R. D. Danzy Z. Wang

### Subgroup on Design (SC-SVR)

D. Miller, Chair T. Patel C. E. Beair J. A. West

R. D. Danzy, Contributing Member B. Joergensen

B. J. Mollitor

### COMMITTEE ON BOILER AND PRESSURE VESSEL CONFORMITY ASSESSMENT (CBPVCA)

J. A. West

D. Cheetham, Contributing Member P. D. Edwards, Chair L. E. McDonald, Vice Chair T. P. Beirne, Alternate K. I. Baron, Staff Secretary J. B. Carr, Alternate M. Vazquez, Staff Secretary J. W. Dickson, Alternate J. P. Chicoine J. M. Downs, Alternate D. C. Cook B. J. Hackett, Alternate T. E. Hansen B. L. Krasiun, Alternate K. T. Lau D. W. Linaweaver, Alternate D. Miller P. F. Martin, Alternate B. R. Morelock I. Powell, Alternate J. D. O'Leary R. Rockwood, Alternate G. Scribner L. Skarin, Alternate B. C. Turczynski

R. D. Troutt, Alternate S. V. Voorbees, Alternate P. Williams, Alternate

A. J. Spencer, Honorary Member

### Subgroup on General Requirements (SC-SVR)

J. F. Ball, Chair J. P. Glaspie G. Brazier B. F. Pittel J. Burgess M. Poehlmann D.B. DeMichael D. E. Tuttle S. T. French J. White

### COMMITTEE ON NUCLEAR CERTIFICATION (CNC)

R. R. Stevenson, Chair J. DeKleine, Vice Chair E. Suarez, Staff Secretar G. Gobbi

S. M. Goodwin J. W. Highlands K. A. Huber J. C. Krane

D. E. Tuttle

E. A. Whittle

R. V. Wielgoszinski

R Hebel

M.A. Lockwood R. P. McIntyre L. M. Plante H. B. Prasse T. E. Quaka C. T. Smith

C. Turylo D. M. Vickery E. A. Whittle C. S. Withers

S. F. Harrison, Contributing Member

S. Andrews, Alternate D. Arrigo, Alternate J. Ball, Alternate P. J. Coco, Alternate P. D. Edwards, Alternate D. P. Gobbi, Alternate

K. M. Hottle. Alternate K. A. Kavanagh, Alternate P. Krane, Alternate D. Nenstiel, Alternate M. Paris. Alternate G. Szabatura, Alternate A. Torosyan, Alternate S. V. Voorhees, Alternate

S. Yang, Alternate

### Subgroup on Testing (SC-SVR)

W. F. Hart, Chair A. Donaldson T. P. Beirne G. D. Goodson J. E. Britt B. K. Nutter J. Buehrer C. Sharpe S. Cammeresi Z. Wang I. A. Cox A. Wilson J. W. Dickson S. R. Irvin, Sr., Alternate

### U.S. Technical Advisory Group ISO/TC 185 Safety Relief Valves

T. J. Bevilacqua, Chair D. Miller C. E. O'Brien, Staff Secretary B. K. Nutter I. F. Ball T. Patel G. Brazier J. A. West

D. B. DeMichael

### **ORGANIZATION OF SECTION III**

17)

### 1 GENERAL

Section III consists of Division 1, Division 2, Division 3, and Division 5. These Divisions are broken down into Subsections and are designated by capital letters preceded by the letter "N" for Division 1, by the letter "C" for Division 2, by the letter "W" for Division 3, and by the letter "H" for Division 5. Each Subsection is published separately, with the exception of those listed for Divisions 2, 3, and 5.

- Subsection NCA General Requirements for Division 1 and Division 2
- Appendices
- Division 1<sup>\*</sup>
  - Subsection NB Class 1 Components
  - Subsection NC Class 2 Components
  - Subsection ND Class 3 Components
  - Subsection NE Class MC Components
  - Subsection NF Supports
  - Subsection NG Core Support Structures
- Division 2 Code for Concrete Containments
  - Subsection CC Concrete Containments
- Division 3 Containment Systems for Transportation and Storage of Spent Nuclear Fuel and High-Level Radioactive Material
  - Subsection WA General Requirements for Division 3
  - Subsection WB Class TC Transportation Containments
  - Subsection WC Class SC Storage Containments
  - Subsection WD Class ISS Internal Support Structures
- Division 5 High Temperature Reactors
  - Subsection HA General Requirements
    - Subpart A Metallic Materials
    - Subpart B Graphite Materials
    - Subpart C Composite Materials
  - Subsection HB Class A Metallic Pressure Boundary Components
    - Subpart A Low Temperature Service
    - Subpart B Elevated Temperature Service
  - Subsection HC Class B Metallic Pressure Boundary Components
    - Subpart A Low Temperature Service
    - Subpart B Elevated Temperature Service
  - Subsection HF Class A and B Metallic Supports
    - Subpart A Low Temperature Service
  - Subsection HG Class A Metallic Core Support Structures
    - Subpart A Low Temperature Service
    - Subpart B Elevated Temperature Service
  - Subsection HH Class A Nonmetallic Core Support Structures
    - Subpart A Graphite Materials
    - Subpart B Composite Materials

### 2 SUBSECTIONS

Subsections are divided into Articles, subarticles, paragraphs, and, where necessary, subparagraphs and subsubparagraphs.

<sup>\*</sup> The 2015 Edition of Section III was the last edition in which Section III, Division 1, Subsection NH, Class 1 Components in Elevated Temperature Service, was published. The requirements located within Subsection NH were moved to Section III, Division 5, Subsection HB, Subpart B for the elevated temperature construction of Class A components.

### 3 ARTICLES

Articles are designated by the applicable letters indicated above for the Subsections followed by Arabic numbers, such as NB-1000. Where possible, Articles dealing with the same topics are given the same number in each Subsection, except NCA, in accordance with the following general scheme:

Article Number	Title
1000	Introduction or Scope
2000	Material
3000	Design
4000	Fabrication and Installation
5000	Examination
6000	Testing
7000	Overpressure Protection
8000	Nameplates, Stamping With Certification Mark and Reports

The numbering of Articles and the material contained in the Articles may not, however, be consecutive. Due to the fact that the complete outline may cover phases not applicable to a particular Subsection or Article, the rules have been prepared with some gaps in the numbering.

### 4 SUBARTICLES

Subarticles are numbered in units of 100, such as NB-1100.

### **5 SUBSUBARTICLES**

Subsubarticles are numbered in units of 10, such as NB-2130, and generally have no text. When a number such as NB-1110 is followed by text, it is considered a paragraph.

### 6 PARAGRAPHS

Paragraphs are numbered in units of 1, such as NB-2121

### **7 SUBPARAGRAPHS**

Subparagraphs, when they are *major* subdivisions of a paragraph, are designated by adding a decimal followed by one or more digits to the paragraph number, such as NB-1132.1. When they are *minor* subdivisions of a paragraph, subparagraphs may be designated by lowercase letters in parentheses, such as NB-2121(a).

### 8 SUBSUBPARAGRAPHS

Subsubparagraphs are designated by adding lowercase letters in parentheses to the *major* subparagraph numbers, such as NB-1132.1(a). When further subdivisions of *minor* subparagraphs are necessary, subsubparagraphs are designated by adding Arabic numerals in parentheses to the subparagraph designation, such as NB-2121(a)(1).

### 9 REFERENCES

References used within Section III generally fall into one of the following four categories:

- (a) References to Other Portions of Section III. When a reference is made to another Article, subarticle, or paragraph, all numbers subsidiary to that reference shall be included. For example, reference to Article NB-3000 includes all material in Article NB-3000; reference to NB-3100 includes all material in subarticle NB-3100; reference to NB-3110 includes all paragraphs, NB-3111 through NB-3113.
  - (b) References to Other Sections. Other Sections referred to in Section III are the following:
- (1) Section II, Materials. When a requirement for a material, or for the examination or testing of a material, is to be in accordance with a specification such as SA-105, SA-370, or SB-160, the reference is to material specifications in Section II. These references begin with the letter "S."

- (2) Section V, Nondestructive Examination. Section V references begin with the letter "T" and relate to the nondestructive examination of material or welds.
- (3) Section IX, Welding and Brazing Qualifications. Section IX references begin with the letter "Q" and relate to welding and brazing requirements.
- (4) Section XI, Rules for Inservice Inspection of Nuclear Power Plant Components. When a reference is made to inservice inspection, the rules of Section XI shall apply.
  - (c) Reference to Specifications and Standards Other Than Published in Code Sections
- (1) Specifications for examination methods and acceptance standards to be used in connection with them are published by the American Society for Testing and Materials (ASTM). At the time of publication of Section III, some such specifications were not included in Section II of this Code. A reference to ASTM E94 refers to the specification so designated by and published by ASTM, 100 Barr Harbor Drive, West Conshohocken, PA 19428.
- (2) Dimensional standards covering products such as valves, flanges, and fittings are sponsored and published by The American Society of Mechanical Engineers and approved by the American National Standards Institute. When a product is to conform to such a standard, for example ASME B16.5, the standard is approved by the American National Standards Institute. The applicable year of issue is that suffixed to its numerical designation in Table NCA-7100-1, for example ASME B16.5-2003. Standards published by The American Society of Mechanical Engineers are available from ASME (https://www.asme.org/).
- (3) Dimensional and other types of standards covering products such as valves, flanges, and fittings are also published by the Manufacturers Standardization Society of the Valve and Fittings Industry and are known as Standard Practices. When a product is required by these rules to conform to a Standard Practice, for example MSS SP-100, the Standard Practice referred to is published by the Manufacturers Standardization Society of the Valve and Fittings Industry, Inc. (MSS), 127 Park Street, NE, Vienna, VA 22180. The applicable year of issue of such a Standard Practice is that suffixed to its numerical designation in Table NCA-7100-1, for example MSS SP-89-2003.
- (4) Specifications for welding and brazing materials are published by the American Welding Society (AWS), 8669 Doral Boulevard, Suite 130, Doral, FL 33166. Specifications of this type are incorporated in Section II and are identified by the AWS designation with the prefix "SF," for example SFA-5.1.
- (5) Standards applicable to the design and construction of tanks and flanges are published by the American Petroleum Institute and have designations such as API-605. When documents so designated are referred to in Section III, for example API-605–1988, they are standards published by the American Petroleum Institute and are listed in Table NCA-7100-1.
- (d) References to Appendices. Section III uses two types of appendices that are designated as either Section III Appendices or Subsection Appendices. Either of these appendices is further designated as either Mandatory or Nonmandatory for use. Mandatory Appendices are referred to in the Section III rules and contain requirements that must be followed in construction. Nonmandatory Appendices provide additional information or guidance when using Section III.
- (1) Section III Appendices are contained in a separate book titled "Appendices." These appendices have the potential for multiple subsection applicability. Mandatory Appendices are designated by a Roman numeral followed, when appropriate, by Arabic numerals to indicate various articles, subarticles, and paragraphs of the appendix, such as II-1500 or XIII-1210. Nonmandatory Appendices are designated by a capital letter followed, when appropriate, by Arabic numerals to indicate various articles, subarticles, and paragraphs of the appendix, such as D-1200 or Y-1440.
- (2) Subsection Appendices are specifically applicable to just one subsection and are contained within that subsection. Subsection specific mandatory and nonmandatory appendices are numbered in the same manner as Section III Appendices, but with a subsection identifier (e.g., NF, NH, D2, etc.) preceding either the Roman numeral or the capital letter for a unique designation. For example, NF-II-1100 or NF-A-1200 would be part of a Subsection NF mandatory or nonmandatory appendix, respectively. For Subsection CC, D2-IV-1120 or D2-D-1330 would be part of a Subsection CC mandatory or nonmandatory appendix, respectively.
- (3) It is the intent of this Section that the information provided in both Mandatory and Nonmandatory Appendices may be used to meet the rules of any Division or Subsection. In case of conflict between Appendix rules and Division/Subsection rules, the requirements contained in the Division/Subsection shall govern. Additional guidance on Appendix usage is provided in the front matter of Section III Appendices.

<sup>\*\*</sup> The American National Standards Institute (ANSI) was formerly known as the American Standards Association. Standards approved by the Association were designated by the prefix "ASA" followed by the number of the standard and the year of publication. More recently, the American National Standards Institute was known as the United States of America Standards Institute. Standards were designated by the prefix "USAS" followed by the number of the standard and the year of publication. While the letters of the prefix have changed with the name of the organization, the numbers of the standards have remained unchanged.

### **SUMMARY OF CHANGES**

Errata to the BPV Code may be posted on the ASME Web site to provide corrections to incorrectly published items, or to correct typographical or grammatical errors in the BPV Code. Such Errata shall be used on the date posted.

Information regarding Special Notices and Errata is published by ASME at http://go.asme.org/BPVCerrata.

Changes given below are identified on the pages by a margin note, (17), placed next to the affected area.

The Record Numbers listed below are explained in more detail in "List of Changes in Record Number Order" following this Summary of Changes.

Page	Location	Change (Record Number)
X	List of Sections	Updated
xv	Submittal of Technical Inquiries to the Boiler and Pressure Vessel Standards Committees	Change (Record Number)  Updated  Revised in its entirety (13-2222)  Updated  (1) In "1 General," title of Section III, Division 3 revised (13-1594)
xviii	Personnel	Updated
xxxvii	Organization of Section III	<ul> <li>(1) In "1 General," title of Section III, Division 3 revised (13-1594)</li> <li>(2) In "1 General," for Section III, Division 3, Subsection WD added</li> <li>(3) In "1 General," entry for Subsection NH deleted and footnote editorially revised</li> <li>(4) In "9 References," subparas. (a) and (d)(1) revised (16-148)</li> </ul>
1	NF-1110	<ul> <li>(1) Former endnotes 1 and 2 incorporated into subparas. (c) and (e), respectively, and deleted from Endnotes (16-2026)</li> <li>(2) Subparagraph (e) revised (14-1489)</li> </ul>
13	NF-2311 NF-2420 Table NF-2432.14	Former endnote 4 added as first paragraph and deleted from Endnotes (16-2026)
21	NF-2420	Revised in its entirety (14-757, 16-2906)
23	Table NF-2432.1	Heading added to first column (16-1033)
24	Table NF-2432.13 NF-2580	<ul><li>(1) Former endnote 8 added as first paragraph to NF-2580 and deleted from Endnotes (16-2026)</li><li>(2) NF-2581.1 revised (14-1396)</li><li>(3) NF-2582 revised (14-1396)</li></ul>
27	NF-3115	Added (15-2527)
27	NF-3121.1	In subpara. (a), cross-reference revised (15-2527)
31	NF-3142	Revised in its entirety (15-2527)
31	NF-3143	Subparagraphs (a)(2) and (b) revised (15-2527)
30.	NF-3212.1	Former endnote 9 added as second paragraph and deleted from Endnotes (16-2026)
32	Table NF-3221.2-1	(1) Sixth column revised (15-2527) (2) Former Note (6) redesignated as (8) (15-2527) (3) Notes (6) and (7) added (15-2527)
33	NF-3221.4	Revised (15-2527)

Page	Location	Change (Record Number)
33	NF-3223.1	Subparagraph (d) added (15-2527)
34	Table NF-3225.2-1	(1) Entries in second row moved to end of first row by errata (15-2745)
		(2) Entry in sixth column revised (15-2527) (3) Note (2) revised (15-2527)
38	NF-3251.1	Former endnote 10 incorporated into first paragraph and deleted from Endnotes (16-2026)
38	NF-3252.1	Subparagraph (d) added (15-2527)
39	Table NF-3251.2-1	from Endnotes (16-2026)  Subparagraph (d) added (15-2527)  (1) Sixth column revised (15-2527)  (2) Note (7) added (15-2527)
42	NF-3282	(1) Paragraph revised (15-2527) (2) In NF-3282.1, nomenclature revised (15-2527) (3) In NF-3282.2 and NF-3282.3, Level D limits added (15-2527) (4) NF-3282.4(a)(1) revised (15-2527)
44	Table NF-3312.1(b)-1	(1) Sixth column revised (15-2527) (2) Notes (3) and (6) revised (15-2527) (3) Former Note (7) redesignated as (10) (15-2527) (4) Notes (7), (8), and (9) added (15-2527)
47	NF-3322.1	<ul> <li>(1) Title and first paragraph revised (15-2527)</li> <li>(2) Former endnote 11 incorporated into subparas. (a)(1) and (a)(2) and deleted from Endnotes (16-2026)</li> <li>(3) Former endnote 12 incorporated into subpara. (c)(1)(-c) and deleted from Endnotes (16-2026)</li> <li>(4) Subparagraph (c)(4) added (15-2527)</li> <li>(5) Former endnotes 13 and 14 incorporated into subpara. (d)(5)(-b) and deleted from Endnotes (16-2026)</li> <li>(6) Subparagraph (e)(1)(-d) added (15-2527)</li> <li>(7) In subpara. (e)(2), Level D added (15-2527)</li> <li>(8) Former endnote 16 incorporated into subpara. (f)(1) and deleted from Endnotes (16-2026)</li> <li>(9) Subparagraph (f)(5) added (15-2527)</li> </ul>
52	rien the	<ul> <li>(1) Former endnote 17 incorporated into subpara. (d)(2)(-b) and deleted from Endnotes (16-2026)</li> <li>(2) Former endnote 18 added to subpara. (d)(2)(-b)(-2) as in-text Note and deleted from Endnotes (16-2026)</li> <li>(3) Subparagraph (e)(3)(-c) revised (15-2527)</li> </ul>
60	NF-3322.6(e)(2)	Former endnote 19 added below eq. (45) as in-text Note and deleted from Endnotes (16-2026)
60 62 Click	NF-3322.8(c)(4)(-a)	Former endnote 20 added as in-text Note and deleted from Endnotes (16-2026)
68	NF-3324.6(b)(2)	Added (15-2527)
72	NF-3341.1	<ul><li>(1) Subparagraph (a) revised (15-2527)</li><li>(2) Former endnote 21 incorporated into subpara. (a) and deleted from Endnotes (16-2026)</li></ul>
78	NF-3370	Last sentence added (15-2527)

	Page	Location	Change (Record Number)
	78	NF-3382	<ul> <li>(1) Paragraph revised (15-2527)</li> <li>(2) In NF-3382.1, nomenclature revised (15-2527, 16-2831)</li> <li>(3) In NF-3382.2 and NF-3382.3, Level D limits added (15-2527, 16-2137, 16-2831)</li> <li>(4) NF-3382.4(a)(1) revised (15-2527)</li> </ul>
	95	Figure NF-4441-1	General Notes (c) and (d) editorially revised Subparagraph (b) revised (01-931) In second column, second entry revised (14-1364) Added (01-931) Subparagraphs (a) and (b)(1) revised (15-281)
	98	NF-4622.7	Subparagraph (b) revised (01-931)
	99	Table NF-4622.7(b)-1	In second column, second entry revised (14-1364)
	100	Table NF-4622.7(b)-2	Added (01-931)
	106	NF-5342	Subparagraphs (a) and (b)(1) revised (15-281)
	107	NF-5352	Subparagraphs (a) and (b)(1) revised (15-281)
	112	NF-I-2100	Former endnote 26 incorporated into paragraph and deleted from Endnotes (16-2027)
	113	Table NF-I-2100-1	<ul> <li>(1) Entries in Stress and Strength columns duplicated wherever multiple materials previously shared one set of four rows (16-2907)</li> <li>(2) Under Low Alloy Steels, A992 added (15-2304)</li> </ul>
	130	Table NF-I-2100-1M	<ul> <li>(1) Entries in Stress and Strength columns duplicated wherever multiple materials previously shared one set of four rows (16-2907)</li> <li>(2) Under Low Alloy Steels, A992 added (15-2304)</li> <li>(3) Under Martensitic Stainless Steels, for A276 420, entries for 175°C and 200°C revised (15-2304)</li> </ul>
	144	Table NF-I-2100-2	A9927A992M added (15-2304)
	159	NF-A-1311	Former endnote 27 incorporated into third paragraph and deleted from Endnotes (16-2027)
MORIN	OC. COM.	NF-A-1311  NF-A-1311  NF-A-1311	
	₹, ₹,		xlii
	1		Alli

xlii

### LIST OF CHANGES IN RECORD NUMBER ORDER

Record Number	Change
01-931	Code Case N-71 contains welding requirements and PWHT exemptions for select S-number/P-number/Group number materials. These exemption details were added to a new table for incorporation into NF-4600.
13-1594	Revised the Organization of Section III to address the incorporation of a new proposed Subsection WD that addresses internal support structures.
13-2222	Revised the front guidance on interpretations in its entirety.
14-757	Replaced lot definitions for welding consumables with SFA-5.01 definitions in NF-2420.
14-737	In Table NF-4622.7(b)-1, for P-Number 1 materials, where the nominal thickness is $\frac{3}{4}$ in. or
11 1301	less, the "Type of Weld" was changed to "All welds in material over $1^{1}\sqrt{1}$ in. (38 mm)."
14-1396	NF-2582 Visual Examination was revised to include ASTM F788 and ASTM F812 references.
14-1489	Revised NF-1110(e) to add padding.
15-281	Deleted "cracks or" from beginning of NF-5342(b)(1) and NF-5352(b)(1).
15-2304	Appendix NF-1 was revised to correct values for A276 Type 420 and to add A992.
15-2527	Updated and moved Level D design rules for supports to Subsection NF from Nonmandatory Appendix F.
15-2745	Revised the table entries to correctly reflect stress limit factors.
16-148	Revised numerous sections of text related to incorporation of revised Mandatory Appendix XIII.
16-1033	Added the heading "Welding Material" to column 1 of Table NF-2432.1-1.
16-2026	In general, endnotes that detailed technical content were added to the body of the Code.
16-2027	NF-I-2100 revised to incorporate endnote 26. NF-A-1311 revised to incorporate endnote 27. Endnotes 26 and 27 deleted.
16-2137	For NF-3382.2 and NF-3382.3, corrected approved record 15-2527 to reflect limits for Linear
4 6 0004	supports rather than Plate and Shell supports.
16-2831	Revised definition of $S_T$ in NF 3382 (corrects record 16-2527).
16-2906	Editorial revision to new language in NF-2420 as approved in record 14-757.
16-2907	Editorial revision to Tables NF-I-2100-1 and NF-I-2100-1M to expand the values in previously "bracketed" areas.
NORMOC. COM. Click to	"bracketed" areas.
	xliii

# There have been structural and stylistic changes to BPVC, starting with the 2011 Addenda, that should be noted to ad avigating the contents. The following is an overview of the changes: Ibparagraph Breakdowns/Nested Lists Hierarchy • First-level breakdowns are designed. • Second-level breakdowns are designed. CROSS-REFERENCING AND STYLISTIC CHANGES IN THE BOILER

JE ASME BRYC Section navigating the contents. The following is an overview of the changes:

### Subparagraph Breakdowns/Nested Lists Hierarchy

- Second-level breakdowns are designated as (1), (2), (3), etc., as in the past.
- Third-level breakdowns are now designated as (-a), (-b), (-c), etc.
- Fourth-level breakdowns are now designated as (-1), (-2), (-3), etc.
- Fifth-level breakdowns are now designated as (+a), (+b), (+c), etc.
- Sixth-level breakdowns are now designated as (+1), (+2), etc.

### **Footnotes**

With the exception of those included in the front matter (roman-numbered pages), all footnotes are treated as endnotes. The endnotes are referenced in numeric order and appear at the end of each BPVC section/subsection.

### Submittal of Technical Inquiries to the Boiler and Pressure Vessel Standards Committees

Submittal of Technical Inquiries to the Boiler and Pressure Vessel Standards Committees has been moved to the front matter. This information now appears in all Boiler Code Sections (except for Code Case books).

### **Cross-References**

It is our intention to establish cross-reference link functionality in the current edition and moving forward. To facilitate this, cross-reference style has changed Cross-references within a subsection or subarticle will not include the designator/identifier of that subsection/subarticle. Examples follow:

- (Sub-)Paragraph Cross-References. The cross-references to subparagraph breakdowns will follow the hierarchy of the designators under which the breakdown appears.
  - If subparagraph (-a) appears in X.1(c)(1) and is referenced in X.1(c)(1), it will be referenced as (-a).
  - If subparagraph (-a) appears in X.1(c)(1) but is referenced in X.1(c)(2), it will be referenced as (1)(-a).
  - If subparagraph (-a) appears in X.1(c)(1) but is referenced in X.1(e)(1), it will be referenced as (c)(1)(-a).
  - If subparagraph (a) appears in X.1(c)(1) but is referenced in X.2(c)(2), it will be referenced as X.1(c)(1)(-a).
- Equation Cross-References. The cross-references to equations will follow the same logic. For example, if eq. (1) appears in X.1(a)(1) but is referenced in X.1(b), it will be referenced as eq. (a)(1)(1). If eq. (1) appears in X.1(a)(1) but is referenced in a different subsection/subarticle/paragraph, it will be referenced as eq. X.1(a)(1)(1).

# ARTICLE NF-1000 INTRODUCTION

# NF-1100 SCOPE AND GENERAL REQUIREMENTS

# (17) NF-1110 ASPECTS OF CONSTRUCTION COVERED BY THESE RULES

- (a) Subsection NF contains rules for the material, design, fabrication, examination, installation, and preparation of certification documents (Certificate of Compliance and NS-1 Certificate of Conformance) for supports for components and piping which are intended to conform to the requirements for Class 1, 2, 3, and MC construction as set forth in Subsections NB, NC, ND, and NE, respectively, of this Section.
- (b) They do not cover deterioration that may occur in service as a result of corrosion, erosion, radiation effects, or metallurgical instability of the materials (NCA-1130).
- (c) Nuclear power plant supports (NCA-9200 provides the definitions of "support" and other terms) for which the rules are specified in this Subsection are those metal elements which transmit loads between components (NCA-1210), including piping systems, and intervening elements and the building structure. However, the term supports does not encompass a structural element the sole function of which is to carry dynamic loads caused by a postulated loss of pressure-retaining integrity.
- (d) The Owner shall be responsible for assuring the adequacy of the building structure and all intervening elements in the support load path in accordance with the requirements of NCA-3240 and NCA-3250. To the extent necessary, the support designer shall consider the structural interaction with intervening elements and the building structure.
- (e) Except for the requirements listed in (1) through (11), the requirements of Subsection NF do not apply to bearings, bushings, gaskets, hydraulic fluids, seals, shims, slide plates, retaining rings, wear shoes, springs, washers, wire rope, compression spring end plates, thread locking devices, cotter pins, sight glass assemblies, spring hanger travel and hydro stops, nameplates, nameplate attachment devices, padding between piping and supports, or for compression dynamic stops used as stops (stops do not include snubbers and dampers; see NF-3412.4 and NF-3412.5) for seismic and other dynamic loads that are designed primarily for compressive loading and are not connected to the support or pressure boundary.

- (1) The material of the exempt items shall be selected to tolerate the environmental conditions to which they will be exposed, such as temperature, fluids, humidity, and irradiation.
- (2) The exempt item shall be designed for the loading conditions and other requirements identified in the Design Specification.
- (3) Design Output Documents (NCA-3550) shall indicate items that are exempt.
- (4) Materials, fabrication, and installation of the exempt items shall comply with Design Output Documents.
- (5) Class 1 springs shall be inspected in accordance with NF-2520
- (6) Washers shall comply with the requirements of NF-2128(b) and NF-4700.
- (7) Wire rope shall comply with the requirements of NF-2530 and Article NF-3000.
- (8) Compression spring end plates shall comply with the requirements of Articles NF-3000, NF-4000, NF-5000, and NF-8000.
- (9) Compression dynamic stops shall comply with the requirements of Articles NF-3000, NF-4000, NF-5000, and NF-8000.
- (10) Thread locking devices shall comply with the requirements of NF-4725.1.
- (11) The means by which exempt items are attached to supports shall comply with the applicable requirements of this Subsection.

## NF-1120 RULES FOR SUPPORTS AND THEIR CLASSIFICATION

### NF-1121 Rules for Supports

The rules of Subsection NF provide requirements for new construction and include consideration of mechanical stresses and effects which result from the constraint of free-end displacements and anchor point motions defined in NF-3121.12 and NF-3121.13, but not thermal or peak stresses.

### NF-1122 Classification of Supports

Supports shall be constructed to the requirements of this Subsection that are applicable to the class of the component, including piping system, they are intended to support. Supports may be optionally classified as permitted in NCA-2134. When the components are optionally classified to a higher class as permitted in NCA-2134(d), the support need not be classified to the higher class.

# NF-1130 BOUNDARIES OF JURISDICTION NF-1131 Boundary Between Components and Supports

The jurisdictional boundary between components, including piping systems, and supports shall meet the requirements of NB-1132, NC-1132, ND-1132, or NE-1132 as applicable to the class of component.

# NF-1132 Boundary Between Supports and the Building Structure

- (a) Supports may bear on or may be welded, bolted, pinned, or clamped to the building structure. Typical examples of jurisdictional boundaries defined between supports and building structures are shown in Figure NF-1132-1.
- (b) The jurisdictional boundary between a support and the load carrying building structure is the surface of the building structure.
- (c) For the purpose of defining the jurisdictional boundary between a support and the building structure, structural members shown on the civil/structural drawings of the plant and considered in the building structural analysis may be designated building structure even though located in the support load path. However, structural members, except as defined in (d), detailed on support drawings which are installed and used for the primary purpose of supporting piping or components shall be designated supports and be constructed to the rules of this Subsection.
- (d) The Design Specification shall designate whether surface-mounted base plates intended to receive loads transmitted by supports are to be within the jurisdiction of Subsection NF or building structure. Fully or partially embedded steel elements intended to receive loads transmitted by supports; grout; concrete anchors; and holddown bolts, nuts, and washers shall be designated building structure.
- (e) If the means by which the support is connected to the building structure is a weld, the weld shall fall within the jurisdiction of this Subsection.
- (f) If the means by which the support is connected to the building structure is a bolted joint to a building structural steel member, the bolts shall fall within the jurisdiction of this Subsection.

# NF-1133 Boundary Between Supports and Intervening Elements

The jurisdictional boundary between supports and intervening elements in the support load path is the surface of the intervening elements. Supports may bear on or may be welded, bolted, pinned, or clamped to intervening elements. The means by which supports are connected to intervening elements shall fall within the jurisdiction of this Subsection.

# NF-1200 TYPES OF SUPPORTS AND ATTACHMENTS

### NF-1210 TYPES OF SUPPORTS

### NF-1211 General Requirements

In this Subsection all supports are categorized into three separate types based on the general design procedure used for analysis of the support (NF-3140). Requirements for materials, design, fabrication, and examination for each of the three types are provided in the following Articles of this Subsection. These support types are defined in NF-1212 through NF-1215.

### NF-1212 Plate- and Shell-Type Supports

A *Plate- and Shell-Type Support* is a support such as a skirt or saddle which is fabricated from plate and shell elements and is normally subjected to a biaxial stress field.

### NF-1213 Linear-Type Support

A *Linear-Type Support* is defined as acting under essentially a single component of direct stress. Such elements may also be subjected to shear stresses. Examples of such structural elements are tension and compression struts, beams and columns subjected to bending, trusses, frames, rings, arches, and cables. Energy-absorbing parts designed to dissipate energy by yielding and which are incorporated into a linear-type piping support shall be constructed in accordance with Mandatory Appendix NF-III.

### **NF-1214** Standard Supports

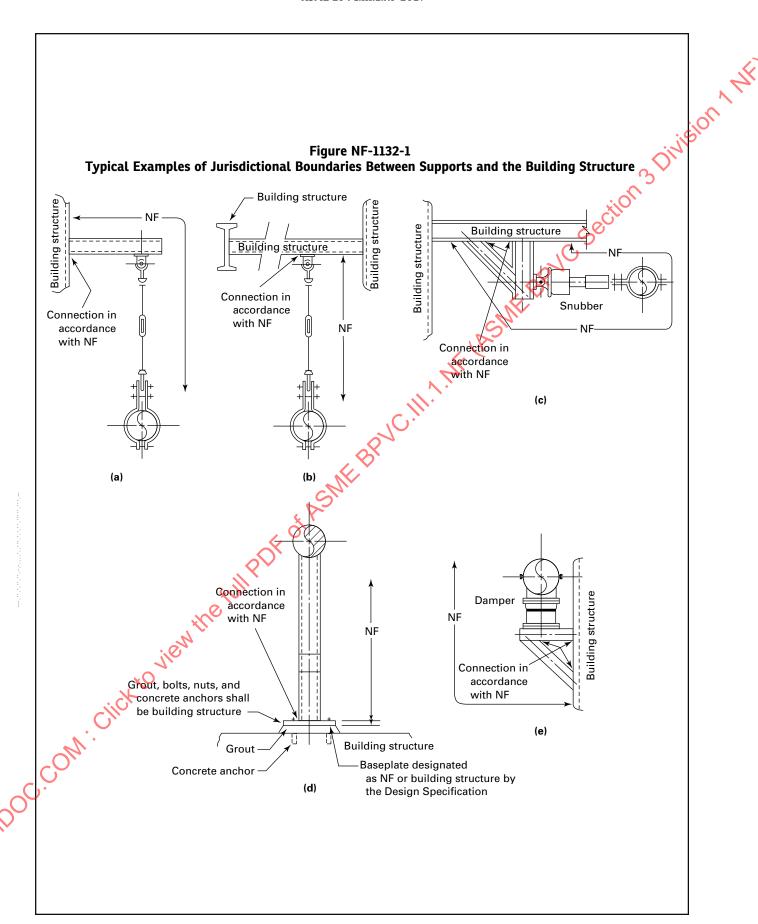
Typical Standard Supports are described in MSS SP-58, Pipe Hangers and Supports, Materials, Design, and Manufacture, which was developed and approved by the Manufacturers Standardization Society of the Valve and Fittings Industry. Typical catalog items are shown in Figure NF-1214-1. The capacities of standard supports may be determined using plate and shell analysis or linear analysis or load rating. Examples of standard supports are

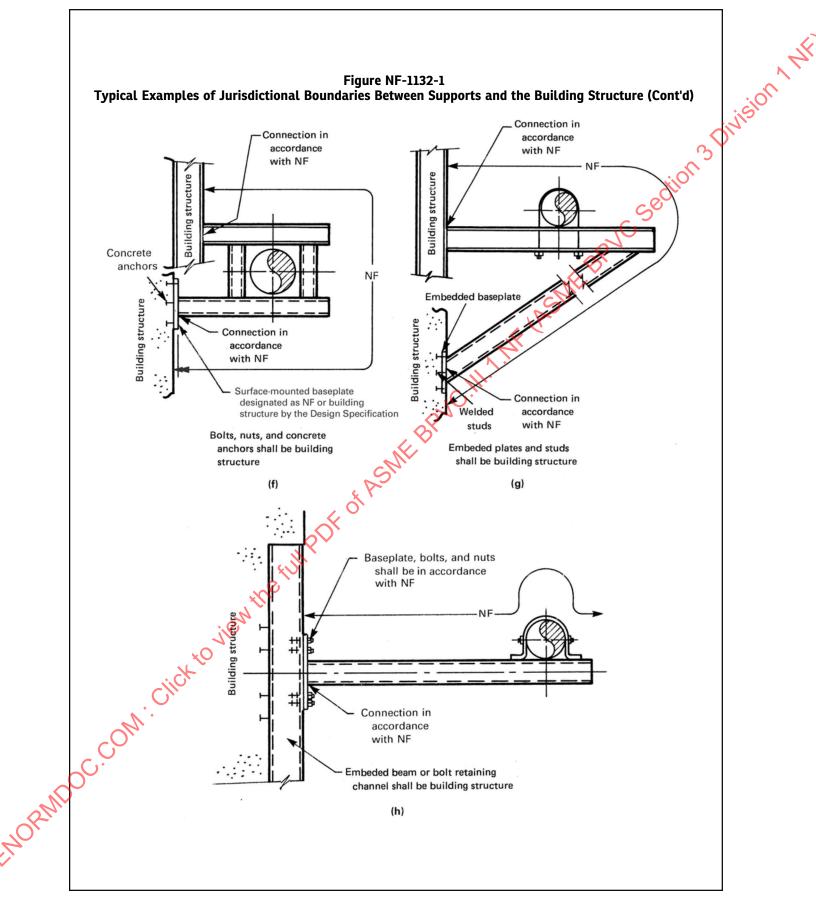
- (a) rigid supports consisting of anchors, guides, restraints, rolling or sliding supports, and rod-type hangers
  - (b) constant and variable type spring hangers
  - (c) snubbers
  - (d) sway braces and vibration dampeners
- (e) structural attachments such as ears, shoes, lugs, rings, clamps, slings, straps, and clevises
  - (f) dampers

### NF-1215 Primary and Secondary Members

Support members are also categorized as primary or secondary according to function. These member types are defined in the following paragraphs.

(a) Primary Members. Primary members of supports are defined as those members designed to carry loads under any postulated load condition.





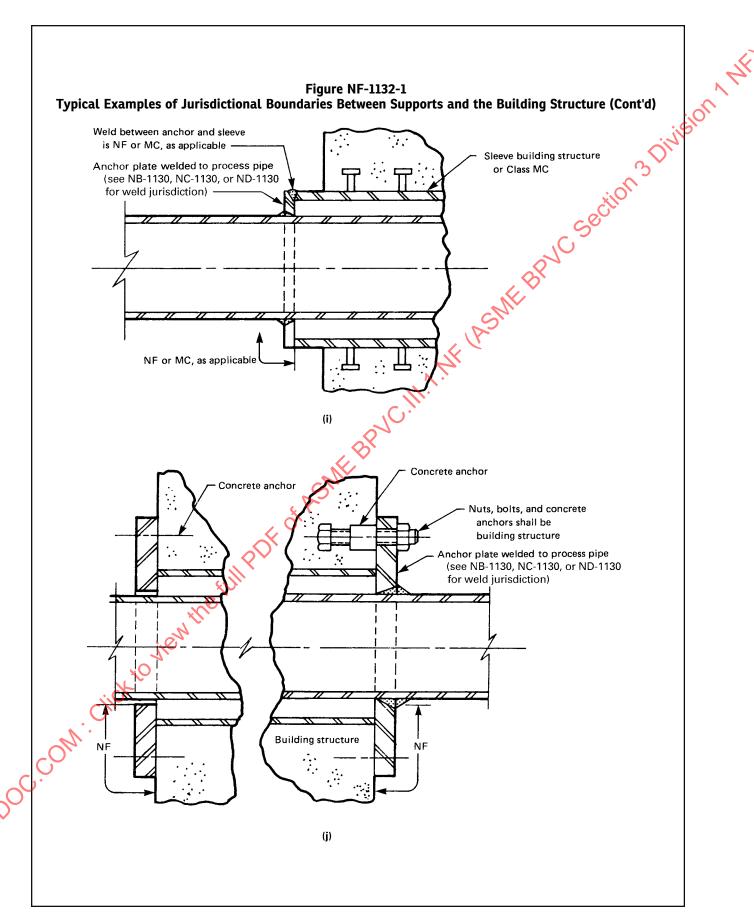
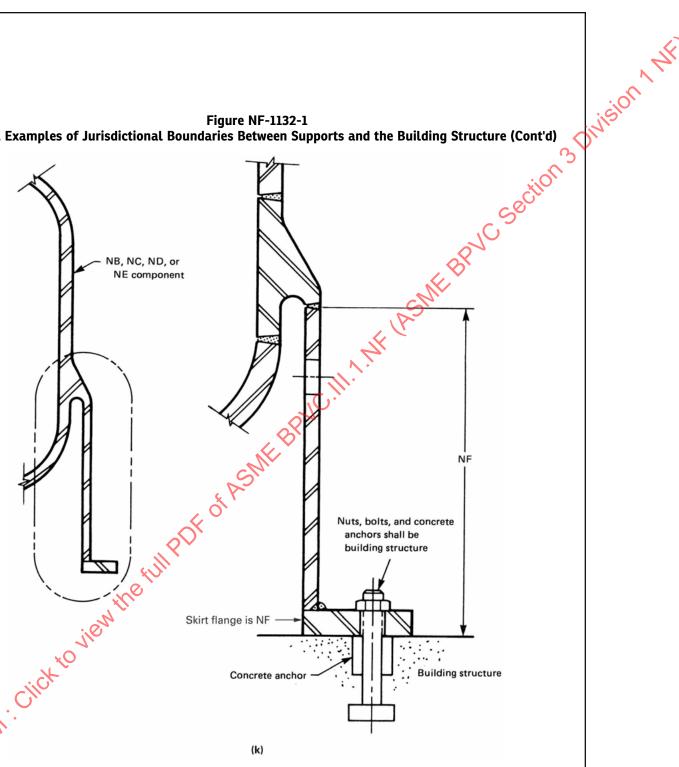
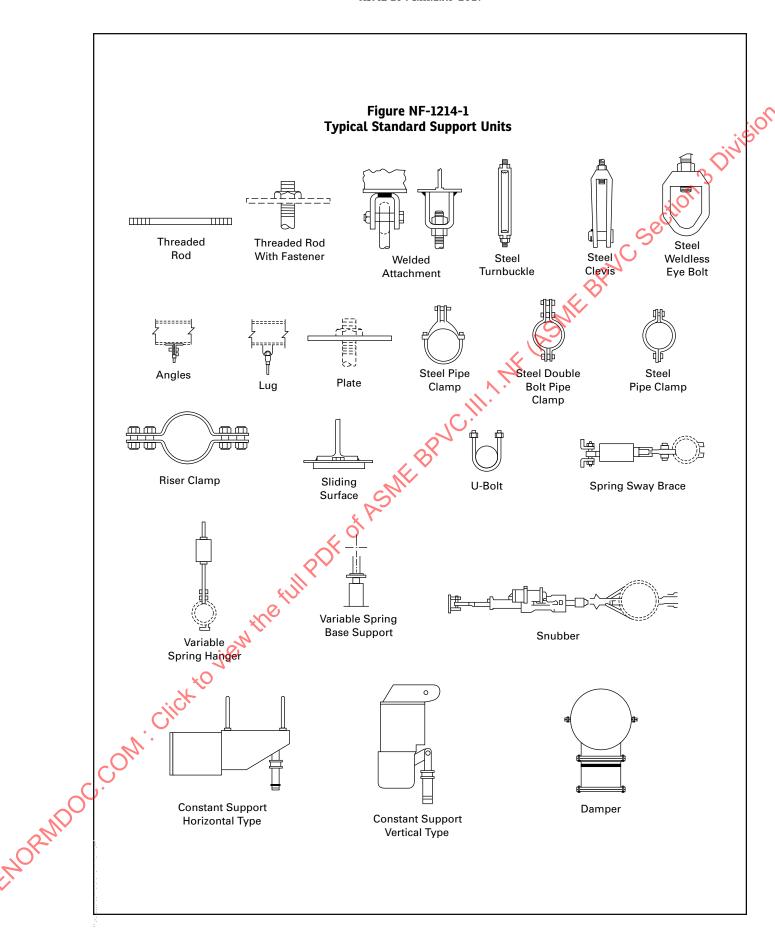


Figure NF-1132-1 Typical Examples of Jurisdictional Boundaries Between Supports and the Building Structure (Cont'd)



GENERAL NOTE: Sketches (a) through (k) are intended to show jurisdictional concepts and should not be considered as recommended configurations.



And and Cook to view the full politic of a safe bary C. III. May be a safe to the safe bary C. III. And the safe bary C. I

### ARTICLE NF-2000 MATERIAL

# NF-2100 GENERAL REQUIREMENTS FOR MATERIAL

### NF-2110 SCOPE OF PRINCIPAL TERMS EMPLOYED

- (a) The term *material* as used in this Subsection is defined in Article NCA-9000. The term *Material Organization* (Metallic) is defined in Article NCA-9000.
- (b) The requirements of this Article make reference to the term *thickness*. For the purpose intended, the following definitions of nominal thickness apply:
- (1) plate: the thickness is the dimension of the short transverse direction.
- (2) forgings: the thickness is the dimension defined as follows:
- (-a) hollow forgings: the nominal thickness is measured between the inside and outside surfaces (radial thickness).
- (-b) disk forgings (axial length less than the outside diameter): the nominal thickness is the axial length.
- (-c) flat ring forgings (axial length less than the radial thickness): for axial length ≤2 in. (50 mm), the axial length is the nominal thickness. For axial length >2 in. (50 mm), the radial thickness is the nominal thickness.
- (-d) rectangular solid forgings: the least rectangular dimension is the nominal thickness.
- (3) castings: thickness, t, is defined as the largest nominal thickness of the load carrying portion of the casting.

# NF-2120 SUPPORT MATERIAL NF-2121 Permitted Material Specifications

- (a) Except as provided in (b) or NF-2128, material for supports shall conform to the requirements of the specifications for material listed in the tables of Section II, Part D, including all applicable footnotes in the table, applicable to the Class of construction, as indicated in Table NF 2121(a)-1.
- (b) The requirements of Article NF-2000 do not apply to exempt items as indicated in NF-1110(e). Certificates of Compliance (NF-2130) are not required for exempt items.
- (c) Welding and brazing material used in manufacture of items shall comply with an SFA Specification in Section II, Part C, except as otherwise permitted in Section IX, and shall also comply with the applicable requirements of this

Article. The requirements of this Article do not apply to materials used as backing rings or backing strips in welded joints.

# NF-2122 Special Requirements Conflicting With Permitted Material Specifications

- (a) Special requirements stipulated in this Article shall apply in lieu of the requirements of the material specifications wherever the special requirements conflict with the material specification requirements [NCA-3856.3(d)]. Where the special requirements include an examination, test, or treatment which is also required by the material specification, the examination, test, or treatment need be performed only once. Required nondestructive examinations shall be performed as specified for each product form in NF-2500. Any examination, repair, test, or treatment required by the material specification or by this Article may be performed by the Material Organization or the Certificate Holder, as provided in NF-4121.1. Pipe or tube material used under the rules of this Subsection need not be hydrostatically or pneumatically tested, provided these materials are not used in a pressure-retaining
- (b) Copper and copper-based alloys shall not be used for structural members.
- (c) The stress rupture test of SA-453 and SA-638 for Grade 660 (UNS S66286) is not required for design temperatures of 800°F (427°C) and below.

### NF-2124 Size Ranges

Material outside the limits of size or thickness given in any specification in Section II may be used if the material is in compliance with the other requirements of the specification and no size limitation is given in the rules for construction. In those specifications in which chemical composition or mechanical properties are indicated to vary with size or thickness, any material outside the specification range shall be required to conform to the composition and mechanical properties shown for the nearest specified range [NCA-3856.3(d)].

### NF-2128 Bolting Material

(a) Material for bolts and studs shall conform to the requirements of one of the specifications listed in Section II, Part D, Subpart 1, Table 3 or Table 4. Materials listed in Section II, Part D, Subpart 1, Table 3 may be used for Class 2 and Class 3 supports and for Class 1 Standard Supports. Materials listed in Section II, Part D, Subpart 1, Table 4

Table NF-2121(a)-1	
Material Tables Required for Supports	ò

Type and Class	Section II, Part D, Subpart 1, Tables 2A and 2B, Design Stress Intensity Values, <i>S</i> <sub>m</sub>	Section II, Part D, Subpart 1, Tables 1A and 1B, Allowable Stress Values, S	Section II, Part D, Subpart 1, Table Y-1, Yield Strength Values, S <sub>y</sub>	Section II, Part D, Subpart 1, Table U, Tensile Strength Values, S <sub>u</sub>
Plate and shell				
Class 1	X		x	X
Class 2 and MC		X	Х	X
Class 3		X	X	x
Bolting (all classes)			X	х
Linear				Sec
Class 1			X	(x)
Class 2 and MC			X	X
Class 3		***	X	x
Bolting (all classes)			X	x
Standard support				
Class 1	x (plate and shell)		x (linear)	x (All Types)
Class 2		x (plate and shell)	x (linear)	x (All Types)
Class 3		x (plate and shell)	x (linear)	x (All Types)
Bolting (all classes)			x (linear)	x (All Types)
Applicable Class	Cl. 1	Cl. 2, Cl. 3, and MC	All Classes	All Classes

#### **GENERAL NOTES:**

- (a) Section II, Part D, Subpart 1, Table Y-2 provides permanent strain limiting factor for plate- and shell-type supports.
- (b) Section II, Part D, Subpart 2, Tables TE and TM provide values of the coefficient of thermal expansion and modulus of elasticity, respectively.

may be used for Class 1 supports. Material for nuts shall conform to SA-194, SA-563, or to the requirements of one of the specifications for nuts or bolting listed in Section II, Part D, Subpart 1, Tables 3 and 4. Material for nuts may also conform to the specification for suitable nuts identified in a permitted bolting specification. Table NF-2121(a)-1 provides the source for the yield and tensile strength values required for design.

(b) Unless required by NF-4720, the use of washers is optional. When used, they shall be made of wrought material with mechanical properties compatible with the nuts with which they are to be employed.

### NF-2130 CERTIFICATION OF MATERIAL

- (a) Material used in the construction of supports shall be certified Certified Material Test Reports in accordance with NCA-3862 shall be provided for material used for primary members for Class 1 Plate- and Shell-Type Supports, and Class 1 Linear-Type Supports. Material for other Classes of supports and all Classes of Standard Supports shall be provided with Certified Material Test Reports when impact testing is required (NF-2311). Copies of these certificates shall be provided with the support.
- (b) Certificates of Compliance with the material specification, grade, class, and heat-treated condition, as applicable, or Certified Material Test Reports shall be provided

by the Material Organization for material for all other supports, Standard Supports, and secondary members of all types and Classes of supports.

- (c) When Design Documents, such as the Design Report or Load Capacity Data Sheet, permit the use of alternative materials for which certificates of compliance are permitted, the alternative materials may be documented in the Design Documents or the documentation for the support in lieu of documenting the specific material used. The requirements of NF-2150 for identification of materials shall apply.
- (d) When material Certificates of Compliance are permitted, the support manufacturer shall transmit copies of the material Certificates of Compliance applicable to each support. Alternatively, he shall execute and furnish a single document certifying a listing of all material specifications involved in the shipment. When a single document is furnished, his quality program shall provide for his retention of the material Certificates of Compliance for each of the materials incorporated into the supports.

### NF-2140 WELDING MATERIAL

For the requirements governing the material to be used for welding, see NF-2400.

### NF-2150 MATERIAL IDENTIFICATION

The identification of material requiring Certified Material Test Reports shall meet the requirements of NCA-3856. Material furnished by a Material Organization with Certificates of Compliance shall be identified by a controlled system meeting the requirements of the applicable material specification, grade, and class. Identification of the material to the Material Organization's Certificate of Compliance is not required after the support manufacturer has verified that the material meets the requirements of this Section. Material for small items shall be controlled during the manufacture of the supports so that they are identifiable as acceptable material at all times. Welding material shall be controlled during the repair of material and the manufacture and installation of supports so that it is identifiable as acceptable material until the material is actually consumed in the process (NF-4122).

### NF-2160 DETERIORATION OF MATERIAL IN SERVICE

Consideration of deterioration of material caused by service is generally outside the scope of this Subsection. It is the responsibility of the Owner to select material suitable for the conditions stated in the Design Specifications (NCA-3250), with specific attention being given to the effect of Service Conditions upon the properties of the material.

# NF-2170 HEAT TREATMENT TO ENHANCE IMPACT PROPERTIES

Carbon steel, low alloy steels, and high alloy chromium (Series 4XX) steels may be heat treated by quenching and tempering to enhance their impact properties. Postweld heat treatment of the support at a temperature of not less than 1,100°F (595°C) may be considered to be the tempering phase of the heat treatment.

### NF-2180 PROCEDURES FOR HEAT TREATMENT OF MATERIAL

When heat treating temperature or time is required by the material specification and the rules of this Subsection, the heat treating shall be performed in temperature-surveyed and temperature-calibrated furnaces or the heat treating shall be controlled by measurement of material temperature thermocouples in contact with the material or attached to blocks in contact with the material or by calibrated pyrometric instruments. Heat treating shall be performed under furnace loading conditions such that the heat treatment is in accordance with the material specification and the rules of this Subsection.

# NF-2200 MATERIAL TEST COUPONS AND SPECIMENS FOR FERRITIC STEEL MATERIAL

### NF-2210 HEAT TREATMENT REQUIREMENTS

### NF-2211 Test Coupon Heat Treatment for Ferritic Material 1

If ferritic steel material is subjected to heat treatment during construction of a support, the material used for the impact test specimens shall be heat treated in the same manner as the support, except that test coupons and specimens for P-No. 1 Group Nos. 1 and 2 material with a nominal thickness of 2 in. (50 mm) or less are not required to be so heat treated. The support manufacturer shall provide the Material Organization with the temperature and heating and cooling rate to be used. In the case of postweld heat treatment, the total time at temperature or temperatures for the test material shall be at least 80% of the total time at temperature or temperatures during actual postweld heat treatment of the material, and the total time at temperature or temperatures for the test material, coupon, or specimen may be performed in a single cycle.

### NF-2212 Test Coupon Heat Treatment for Quenched and Tempered Material

NF-2212.1 Cooling Rates. When ferritic steel material is subjected to quenching from the austenitizing temperature, the test coupons representing those materials shall be cooled at a rate similar to and no faster than the main body of the material except in the case of certain forgings and castings (NF-2223.3 and NF-2226.4). This rule shall apply for coupons taken directly from the material as well as for separate test coupons representing the material, and one of the general procedures described in NF-2212.2 or one of the specific procedures described in NF-2220 shall be used for each product form.

**NF-2212.2 General Procedures.** One of the general procedures in (a), (b), and (c) may be applied to quenched and tempered material or test coupons representing the material, provided the specimens are taken relative to the surface of the product in accordance with NF-2220. Further specific details of the methods to be used shall be the obligation of the Material Organization and the Certificate Holder.

- (a) Any procedure may be used which can be demonstrated to produce a cooling rate in the test material that matches the cooling rate of the main body of the product within  $25^{\circ}F$  ( $14^{\circ}C$ ) and 20 sec at all temperatures after cooling begins.
- (b) If cooling rate data for the material and cooling rate control devices for the test specimens are available, the test specimens may be heat treated in the device to represent the material, provided that the provisions of (a) are met.

- (c) When any of the specific procedures described in NF-2220 are used, faster cooling rates at the edges may be compensated for by
- (1) taking the test specimens at least t from a quenched edge, where t equals the material thickness
- (2) attaching a steel pad at least t wide by a partial penetration weld (which completely seals the buffered surface) to the edge where specimens are to be removed or
- (3) using thermal barriers or insulation at the edge where specimens are to be removed

It shall be demonstrated (and this information shall be included in the Certified Material Test Report) that the cooling rates are equivalent to (a) or (b).

# NF-2220 PROCEDURE FOR OBTAINING TEST COUPONS AND SPECIMENS FOR QUENCHED AND TEMPERED MATERIAL

### NF-2221 General Requirements

The procedure for obtaining impact test specimens for quenched and tempered material is related to the product form. Coupon and specimen location shall be as required by the material specification, except as stated in the following paragraphs of this subarticle. References to dimensions signify nominal values.

### NF-2222 Plates

NF-2222.1 Number of Tension Test Coupons. The number of tension test coupons required shall be in accordance with the material specification and with SA-20, except that from carbon steel plates weighing 42,000 lb (19 000 kg) and over and allow steel plates weighing 40,000 lb (18 000 kg) and over, two tension test coupons shall be taken, one representing the top end of the plate and one representing the bottom end of the plate.

# **NF-2222.2 Orientation and Location of Coupons.** Coupons shall be taken so that specimens shall have their longitudinal axes at least $\frac{1}{4}t$ from a rolled surface and with the midlength of the specimen at least t from any heat-treated edge, where t is the nominal thickness of the material. The orientation of the coupons shall be as specified in the material specification.

**NF-2222.3 Requirements for Separate Test Coupons.** Where a separate test coupon is used to represent the support material, it shall be of sufficient size to ensure that the cooling rate of the region from which the test coupons are removed represents the cooling rate of the material at least  $\frac{1}{4}t$  deep and t from any edge of the product. Unless cooling rates applicable to the bulk pieces or product are simulated in accordance with NF-2212.2(b), the dimensions of the coupon shall not be less than  $3t \times 3t \times t$ , where t is the nominal material thickness.

### NF-2223 Forgings

**NF-223.1 Location of Coupons.** Coupons shall be taken so that specimens shall have their longitudinal axes at least  $\frac{1}{4}t$  from any surface and with the midlength of the specimens at least t from any second surface, where t is the maximum heat-treated thickness. A thermal buffer as described in NF-2212.2(c) may be used to achieve these conditions, unless cooling rates applicable to the bulk forgings are simulated as otherwise provided in NF-2212.2.

NF-2223.2 Very Thick and Complex Forgings. Test coupons for forgings that are both very thick and complex, such as trunnions, support forgings, flanges, and other complex forgings that are contour shaped or machined to essentially the finished product configuration prior to heat treatment, may be removed from prolongations or other stock provided on the product. The Certificate Holder shall specify the surfaces of the finished product subjected to high tensile stresses in service. The coupons shall be taken so that specimens shall have their longitudinal axes at a distance below the nearest heat-treated surface, equivalent to at least the greatest distance that the indicated high tensile stress surface will be from the nearest surface during heat treatment, and with the midlength of the specimens a minimum of twice this distance from a second heat-treated surface. In any case, the longitudinal axes of the specimens shall not be pearer than  $\frac{3}{4}$  in. (19 mm) to any heat-treated surface and the midlength of the specimens shall be at least  $1\frac{1}{2}$  in. (38 mm) from any heat-treated surface.

# NF-2223.3 Coupons From Separately Produced Test Forgings. Test coupons representing forgings from one heat and one heat treatment lot may be taken from a separately forged piece under the conditions given in (a) through (e).

- (a) The separate test forging shall be of the same heat of material and shall be subjected to substantially the same reduction and working as the production forging it represents.
- (b) The separate test forging shall be heat treated in the same furnace charge and under the same conditions as the production forging.
- (c) The separate test forging shall be of the same nominal thickness as the production forging.
- (d) Test coupons for simple forgings shall be taken so that specimens shall have their longitudinal axes at the region midway between midthickness and the surface and with the midlength of the specimens no nearer to any heat-treated edge than a distance equal to the forging thickness, except when the thickness–length ratio of the production forging does not permit, in which case a production forging shall be used as the test forging and the midlength of the specimens shall be at the midlength of the test forgings.
- (e) Test coupons for complex forgings shall be taken in accordance with NF-2223.2.

### NF-2224 Bars and Bolting Material

**NF-2224.1 Location of Coupons.** Coupons shall be taken so that specimens shall have their longitudinal axes at least  $\frac{1}{4}t$  from the outside or rolled surface and with the midlength of the specimens at least t from a heat-treated end, where t is either the bar diameter or thickness.

**NF-2224.2 Bolting Material.** For bolting material, the coupons shall be taken in conformance with the applicable material specification and with the midlength of the specimen at least one diameter or thickness from a heat-treated end. When the studs, nuts, or bolts are not of sufficient length, the midlength of the specimen shall be at the midlength of the studs, nuts, or bolts. The studs, nuts, or bolts selected to provide test coupon material shall be identical with respect to the quenched contour and size, except for length, which shall equal or exceed the length of the represented studs, nuts, or bolts.

### NF-2225 Tubular Products and Fittings

**NF-2225.1 Location of Coupons.** Coupons shall be taken so that specimens shall have their longitudinal axes at least  $\frac{1}{4}t$  from the inside or outside surface and with the midlength of the specimens at least t from a heat-treated end, where t is the nominal wall thickness of the tubular product.

NF-2225.2 Separately Produced Coupons Representing Fittings. Separately produced test coupons representing fittings may be used. When separately produced coupons are used, the requirements of NF-2223.3 shall be met.

### NF-2226 Castings

NF-2226.1 Castings With 2 in. (50 mm) Maximum Thickness and Less. For castings with a maximum thickness of 2 in. (50 mm) and less, the specimens shall be taken from either the standard separately cast coupons or the casting, in accordance with the material specification.

NF-2226.2 Castings With Thicknesses Exceeding 2 in. (50 mm) Maximum Thickness. For castings exceeding a thickness of 2 in. (50 mm), the coupons shall be taken from the casting (or an extension of it) so that specimens shall have their longitudinal axes at least  $\frac{1}{4}t$  of the maximum heat-treated thickness from any surface and with the midlength of the specimens at least t from any second surface. A thermal buffer may be used [NF-2212.2(c)(3)].

NF-2226.3 Separately Cast Test Coupons for Castings With Thicknesses Exceeding 2 in. (50 mm). In lieu of the requirements of NF-2226.2, separately cast test coupons may be used under the conditions of (a) through (c).

(a) The separate test coupon representing castings from one heat and one heat-treated lot shall be of the same heat of material and shall be subjected to substantially the same foundry practices as the production casting it represents.

(b) The separate test coupon shall be heat treated in the same furnace charge and under the same conditions as the production casting, unless cooling rates applicable to the bulk castings are simulated in accordance with NF-2212.2.

(c) The separate test coupon shall not be less than  $3t \times 3t \times t$ , where t equals the nominal thickness of the casting. Test specimens shall be taken with their longitudinal axes at the region midway between midthickness and the surface and with the midlength of the specimens no nearer any heat-treated edge than a distance equal to the casting thickness.

NF-2226.4 Castings Machined or Cast to Finished Configuration Before Heat Treatment. In lieu of the requirements of NF-2226.1, NF-2226.2, or NF-2226.3, test coupons may be removed from prolongations or other stock provided on the product. The coupons shall be taken so that specimens shall have their longitudinal axes at a distance below the nearest heat-treated surface equivalent at least to the greatest distance that the indicated high tensile stress surface will be from the nearest surface during heat treatment and with the midlength of the specimens a minimum of twice this distance from a second heat-treated surface. In any case, the longitudinal axes of the specimens shall be at least  $\frac{3}{4}$  in. (19 mm) from any heat-treated surface and the midlength of the specimens shall be at least  $1\frac{1}{2}$  in. (38 mm) from any second heat-treated surface. The Certificate Holder shall specify the surfaces of the finished product subjected to high tensile stresses in service.

### NF-2227 Rolled Shapes

For rolled shapes, the coupons shall be taken so that specimens shall have their longitudinal axes on a line representing the center of the thickest element of the shape and with the midlength of the specimen at least *t* from a heat-treated end.

# NF-2300 FRACTURE TOUGHNESS REQUIREMENTS FOR MATERIAL

### NF-2310 MATERIAL TO BE IMPACT TESTED

# NF-2311 Supports for Which Impact Testing of (17) Material Is Required

When impact testing is required, the methods of Section III Appendices, Nonmandatory Appendix G may be used as an alternative design procedure for assuring protection against nonductile failure.

- (a) Support materials shall be impact tested in accordance with the requirements listed below.
- (1) Attachments to the component or piping shall meet the requirements for impact testing stipulated in the applicable Subsection.
- (2) Class 1, 2, 3, and MC component supports shall meet the requirements of NF-2300.

- (3) For Class 1, 2, and 3 piping supports, Class 1, 2, and 3 Standard Supports, and all other types and Classes of supports, the Design Specification (NCA-3250) shall state whether or not impact testing is required for the material of which the support is constructed. When impact testing is required, the tests shall meet the requirements of NF-2300 for Class 1, 2, 3, or MC, respectively, and shall become a requirement of this Subsection.
- (b) The requirements for supports shall be as specified in NF-2300, except that the materials described in (1) through (13) are not to be impact tested as a requirement of this Subsection.
- (1) Material with a nominal section thickness of  $\frac{5}{8}$  in. (16 mm) and less.
- (2) Bolting, including studs, nuts, and bolts, with a nominal size of 1 in. (25 mm) and less.
- (3) Bars with a nominal cross-sectional area of 1 in.<sup>2</sup> (650 mm<sup>2</sup>) and less.
- (4) Material for fittings with all pipe connections of  $\frac{5}{8}$  in. (16 mm) nominal wall thickness and less.
- (5) Austenitic stainless steels, including precipitation-hardened austenitic Grade 660 (UNS S66286).
  - (6) Nonferrous materials.
- (7) Material for supports when the maximum stress does not exceed 6,000 psi (40 MPa) tension or is compressive.
- (8) Rolled structural shapes, when the thickness of a flange is  $\frac{5}{8}$  in. (16 mm) or less.
- (9) Materials for Class 1, 2, or MC supports, listed in Table NF-2311(b)-1, for thicknesses  $2^{1}/_{2}$  in. (64 mm) and less when the Lowest Service Temperature<sup>2</sup> is at least 30°F (15°C) above the tabulated temperature. This exemption from impact testing does not apply to either the weld metal (NF-2430) or the weld procedure qualification (NF-4335).
- (10) Materials for Class 3 supports, listed in Table NF-2311(b)-1, for thicknesses  $2^{1}/2$  in. (64 mm) and less, when the Lowest Service Temperature is equal to or greater than the tabulated temperature. This exemption from impact testing does not apply to either the weld metal (NF-2430) or the weld procedure qualification (NF-4335).
- (11) Materials for Class 2 and MC supports for which the Lowest Service Temperature exceeds 150°F (65°C).
- (12) Materials for Class 3 supports for which the Lowest Service Temperature exceeds 100°F (38°C).
- (13) Materials for Class 2, 3, and MC supports for which the Lowest Service Temperature (LST) is equal or above the Minimum Design Metal Temperature of Figure NF-2311(b)-1 for the materials listed and the applicable material thicknesses.
- (c) The Design Specification (NCA-3250) shall state the Lowest Service Temperature (LST) for the support and the designated impact test temperature, when required.

### Table NF-2311(b)-1 Exemptions From Impact Testing Under NF-2311(b)(9) and NF-2311(b)(10) for Material Thickness to $2\frac{1}{2}$ in. (64 mm), Inclusive

	Material Condition	T <sub>NDT</sub> , °F (°C) [Note (2)],
Material [Note (1)]	[Note (1)]	[Note (3)]
SA-537, Class 1	N	-30 (-35)
SA-516, Grade 70	Q & T	-10 (-25)
SA-516, Grade 70	N	0 (-18)
SA-508, Class 1	Q & T	+10 (-10)
SA-533, Grade B	Q & T	<del>4</del> 10 (-10)
SA-299 [Note (4)]	N	<b>→</b> +20 (-7)
SA-216, Grades WCB, WCC	Q & T \	+30 (0)
SA-36 (Plate)	HR	+40 (5)
SA-508, Class 2	Q & T	+40 (5)

### NOTES:

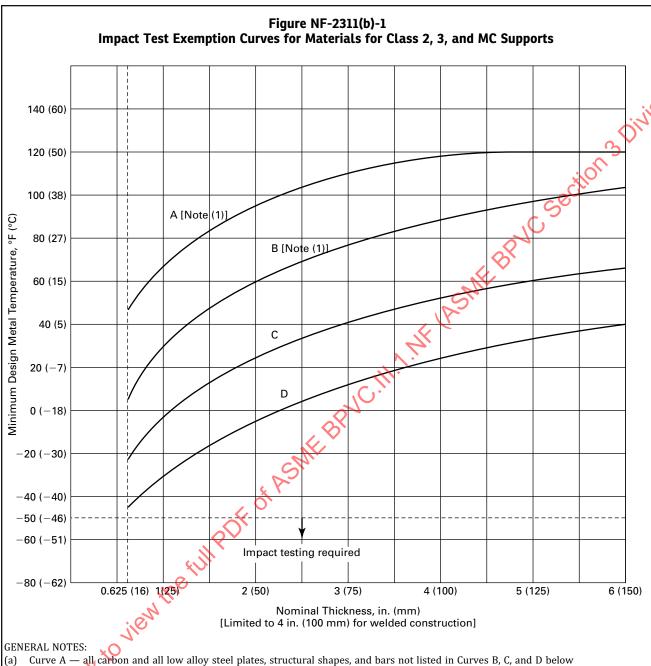
- (1) Material Condition letters refer to:
  - N = Normalize
  - Q & T = Quench and Temper
    - HR = Hot Rolled
- (2) These values for  $T_{\rm NDT}$  were established from data on heavy section steel [thickness greater than  $2\frac{1}{2}$  in. (64 mm)]. Values for sections less than  $2\frac{1}{2}$  in. (64 mm) thick are held constant until additional data are obtained.
- (3)  $T_{\rm NDT}$  = temperature at or above nil-ductility transition temperature (ASTM E208);  $T_{\rm NDT}$  is 10°F (5.6°C) below the temperature at which at least two specimens show no-break performance.
- (4) Materials made to a fine grain melting practice.

# NF-2320 IMPACT TEST PROCEDURES NF-2321 Charpy V-Notch Tests

The Charpy V-notch test ( $C_v$ ), when required, shall be performed in accordance with SA-370. Specimens shall be in accordance with SA-370, Figure 11, Type A. A test shall consist of a set of three full-size 10 mm  $\times$  10 mm specimens. The lateral expansion and absorbed energy, as applicable, and the test temperature, as well as the orientation and location of all tests performed to meet the requirements of NF-2330 shall be reported in the Certified Material Test Report.

### NF-2322 Location and Orientation of Test Specimens

Impact test specimens for quenched and tempered material shall be removed from the locations and orientation specified for tensile test specimens in each product form in NF-2220 except that for plates the orientation of the impact test specimens shall be longitudinal. For material in other heat-treated conditions, impact test specimens shall be removed from the locations and orientations specified for tensile test specimens in the material specification except that for plates the orientation of the impact specimens shall be longitudinal, and for structural shapes



- (b) Curve B
  - (1) 8A-285 Grades A and B
    - (a) SA-414 Grade A
    - (b) SA-442 Grade 55 > 1 in. (25 mm) if not to fine grain practice and normalized
    - (c) SA-442 Grade 60 if not to fine grain practice and normalized
    - (d) SA-515 Grades 55 and 60
    - (e) SA-516 Grades 65 and 70 if not normalized
    - (f) SA-612 if not normalized
    - (g) SA-662 Grade B if not normalized
    - (h) SA-724 if not normalized
  - (2) all materials of Curve A if produced to fine grain practice and normalized which are not listed for Curves C and D below;
  - (3) except for bolting [see (e) below], plates, structural shapes, and bars, all other product forms (such as pipe, fittings, and tubing) not listed for Curves C and D below
- (c) Curve C
  - (1) SA-182 Grades 21 and 22 if normalized and tempered
    - (a) SA-302 Grades C and D

# SME BRUC Section 3 Invision Amis Figure NF-2311(b)-1 Impact Test Exemption Curves for Materials for Class 2, 3, and MC Supports (Cont'd)

GENERAL NOTES (CONT'D):

- (b) SA-336 Grades F21 and F22 if normalized and tempered
- (c) SA-387 Grades 21 and 22 if normalized and tempered
- (d) SA-442 Grade  $55 \le 1$  in. (25 mm) if not to fine grain practice and normalized
- (e) SA-516 Grades 55 and 60 if not normalized
- (f) SA-533 Grades B and C
- (g) SA-662 Grade A;
- (2) all material of Curve B if produced to fine grain practice and normalized and not listed for Curve D below
- (d) Curve D
  - (a) SA-203
  - (b) SA-442 if to fine grain practice and normalized
  - (c) SA-508 Class 1
  - (d) SA-516 if normalized
  - (e) SA-524 Classes 1 and 2
  - (f) SA-537 Classes 1 and 2
  - (g) SA-612 if normalized
  - (h) SA-662 if normalized
- (i) SA-724 if normalized

(e) For bolting the following impact test exemption temperature shall apply:

Spec. No.	Grade	Impact Test Exemption Temperature, °F (°C)
SA-193	B5	-20 (-30)
	В7	-40 (-40)
	B7M	-50 (-45)
	B16	-20 (-30)
SA-307	В	-20 (-30)
SA-320	L7, L43	Impact tested
SA-325	1	-20 (-30)
SA-354	BB	-20 (-30)
	BC 😽	0 (-18)
	BD 🕻 💙	+20 (-7)
SA-449	A O'	-20 (-30)

NOTE:

(1) See NF-2311(b)(12) for Class 3 lowest service temperature limit.

the location and orientation of the impact test specimens shall be as specified in ASTMA673. Alternatively, the orientation may be in the direction of maximum stress regardless of heat treatment. For bolting, the C<sub>v</sub> impact test specimens shall be prepared with the longitudinal axis of the specimen located at least one-half radius or 1 in. (25 mm) below the surface plus the machining allowance per side, whichever is less. The fracture plane of the specimen shall be at least one diameter or thickness from the heat-treated end. When the studs, nuts, or bolts are not of sufficient length, the midlength of the specimen shall be at the midlength of the studs, nuts, or bolts. The studs, nuts, or bolts selected to provide test coupon material shall be identical with respect to the quenched contour and size, except for length, which shall equal or exceed the length of the represented studs, nuts, or bolts. For all material, the number of tests shall be in accordance with NF-2340.

### NF-2330 **TEST REQUIREMENTS AND ACCEPTANCE STANDARDS**

#### NF-2331 Material (Excluding Bolting)

When impact testing is required, material (other than bolting) shall be tested as required by (a) and (b).

- (a) Test three C<sub>v</sub> specimens at the designated temperature (NF-2311). All three specimens shall meet one of the acceptance standards applicable to the specific test method.
- (1) Charpy V-Notch Testing for Lateral Expansion Values. The test results of the three specimens, collectively and singly, shall meet the respective requirements of Table NF-2331(a)-1 for Class 1, of Table NF-2331(a)-2 for Class 2 and MC, and of Table NF-2331(a)-3 for Class 3 supports.

# Table NF-2331(a)-1 Required $C_{\nu}$ Lateral Expansion Values for Class 1 Support Material Other Than Bolting

Nominal Wall Thickness, in. (mm)	Lateral Expansion, mils (mm)
<sup>5</sup> / <sub>8</sub> (16) or less	No test required
Over $\frac{5}{8}$ to 1 (16 to 25), incl.	15 (0.38)
Over 1 (25)	25 (0.64)

- (2) Charpy V-Notch Testing for Absorbed Energy Values. The test results of the three specimens, collectively and singly, shall meet the respective requirements of Table NF-2331(a)-4 for Class 1, of Figure NF-2331(a)-1 for Classes 2 and MC, and of Figure NF-2331(a)-2 for Class 3 supports.
  - (b) Apply the procedures of (a) to
    - (1) base material<sup>3</sup>
- (2) the base material, the heat-affected zone, and weld metal from the weld procedure qualification tests of NF-4330
  - (3) the weld metal of NF-2431

### NF-2333 Bolting Material

When impact testing is required, for bolting material including studs, nuts, and bolts, test three  $C_v$  specimens at the designated temperature. All three specimens shall meet the requirements of Table NF-2333-1.

# NF-2340 NUMBER OF IMPACT TESTS REQUIRED NF-2341 Plates

One test shall be made from each plate as heat treated. Where plates are furnished in the unheat-treated condition and qualified by heat-treated test specimens, one test shall be made for each plate as-rolled. The term *as-rolled* refers to the plate rolled from a slab or directly from an ingot, not to its heat-treated condition.

### Table NF-2331(a)-2 Required C<sub>v</sub> Lateral Expansion Values for Class 2 and MC Support Material Other Than Bolting

Nominal Wall Thickness, in. (mm)	Lateral Expansion, mils (mm)
<sup>5</sup> / <sub>8</sub> (16) or less	No test required
<sup>5</sup> / <sub>8</sub> (16) or less Over <sup>5</sup> / <sub>8</sub> to 1 (16 to 25), incl.	15 (0.38)
Over 1 (25)	20 (0.50)

# Table NF-2331(a)-3 Required $C_{\nu}$ Lateral Expansion Values for Class 3 Support Material Other Than Bolting

Nominal Wall Thickness, in. (mm)	Lateral Expansion, mils (mm)
$\frac{5}{8}$ (16) or less Over $\frac{5}{8}$ (16)	No test required
Over <sup>5</sup> / <sub>8</sub> (16)	15 (0.38)

### NF-2342 Forgings and Castings

- (a) Where the weight of an individual forging or casting is less than 1,000 lb (450 kg), one test shall be made to represent each heat in each heat treatment lot.
- (b) When heat treatment is performed in a continuoustype furnace with suitable temperature controls and equipped with recording pyrometers so that complete heat treatment records are available, a heat treatment lot shall be considered as the lesser of a continuous run not exceeding 8 hr duration or a total weight, so treated, not exceeding 2,000 lb (900 kg).
- (c) One test shall be made for each forging or casting of 1,000 lb to 10,000 lb (450 kg to 4 500 kg) in weight.
- (d) As an alternative to (c), a separate test forging or casting may be used to represent forgings or castings of different sizes in one heat and heat treat lot, provided the test piece is a representation of the greatest thickness in the heat treat lot. In addition, test forgings shall have been subjected to substantially the same reduction and working as the forgings represented.
- (e) Forgings or castings larger than 10,000 lb (4 500 kg) shall have two tests per part for Charpy V-notch. The location of  $C_{\rm v}$  impact test specimens shall be selected so that an equal number of specimens is obtained from positions in the forging or casting 180 deg apart.
- (f) As an alternative to (e) for static castings, a separately cast test coupon (NF-2226.3) may be used; one test shall be made for Charpy V-notch.

### NF-2343 Bars and Rolled Shapes

- (a) For bars, one test shall be made for each diameter or size greater than 1 in. (25 mm) in each lot, where a lot is defined as one heat of material
  - (1) as-rolled in one continuous operation or
- (2) heat treated in one charge, or each 6,000 lb (2 700 kg) or less from one continuous operation
- (b) For rolled shapes, the frequency of testing shall be one test (a set of three specimens) for at least each 15 tons (14 000 kg) or each single length of 15 tons (14 000 kg) or more, of the same nominal shape size, excluding length, for each heat in the as-rolled condition. If the shapes are heat treated, one test shall be taken from

# Table NF-2331(a)-4 Required $C_{\nu}$ Energy Values for Class 1 Support Material Other Than Bolting

Customary Units						
	Energy, ft-lb for Base Materials of Specified Minimum Yield Strength, ksi					
	55 ksi	or Below	Over 55 ksi to 75 ksi, Incl.		Over 75 ksi to 105 ksi, Incl.	
Nominal Wall Thickness, in.	Average of 3	Lowest 1 of 3	Average of 3	Lowest 1 of 3	Average of 3	Lowest 1 of 3
<sup>5</sup> / <sub>8</sub> or less [Note (1)]						
Over $\frac{5}{8}$ to 1, incl. [Note (2)]	15	10	20	15	25	20
Over 1 [Note (2)]	25	20	30	25	35	30

or ones						
	Energy, J for Base Materials of Specified Minimum Yield Strength, MPa					
	380 MPa or Below		Over 380 MPa to 515 MPa, Incl.		Over 515 MPa to 725 MPa, Incl.	
Nominal Wall Thickness, mm	Average of 3	Lowest 1 of 3	Average of 3	Lowest 1 of 3	Average of 3	Lowest 1 of 3
16 or less [Note (1)]						
Over 16 to 25, incl. [Note (2)]	20	14	27	20	34	27
Over 25 [Note (2)]	34	27	41	34	47	41

SI Units

### NOTES:

- (1) No test required.
- (2) Where weld metal tests of NF-2400 are made to these requirements, the impact energy shall conform to the requirements of either of the base materials being joined.

each heat of each nominal shape size, excluding length, in each furnace lot. For shapes heat treated in a continuous furnace, a lot shall not exceed 15 tons (14 000 kg).

### NF-2344 Tubular Products and Fittings

On products that are seamless or welded without filler metal, one test shall be made from each lot. On products which are welded with filler metal, one additional test with the specimens taken from the weld area shall also be made on each lot. A lot shall be defined as stated in the applicable material specification, but in no case shall a lot consist of products from more than one heat of material and of more than one diameter, with the nominal thickness of any product included not exceeding that to be impact tested by more than \( \frac{1}{1} \) in. (6 mm); such a lot shall be in a single heat treatment load or in the same continuous run in a continuous furnace controlled within a 50°F (28°C) range and equipped with recording pyrometers.

### NF-2345 Bolting Material

One test shall be made for each lot of material where a lot is defined as one heat of material heat treated in one charge or as one continuous operation, not to exceed the following (in weight):

Diameter, in. (mm)	Weight, lb (kg)		
1 <sup>3</sup> / <sub>4</sub> (44) and less	1,500 (700)		
Over $1^{3}/_{4}$ to $2^{1}/_{2}$ (44 to 64)	3,000 (1 350)		
Over $2^{1}/_{2}$ to 5 (64 to 125)	6,000 (2 700)		
Over 5 (125)	10,000 (4 500)		
	, ,		

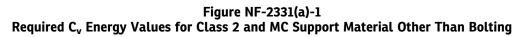
### NF-2350 RETESTS

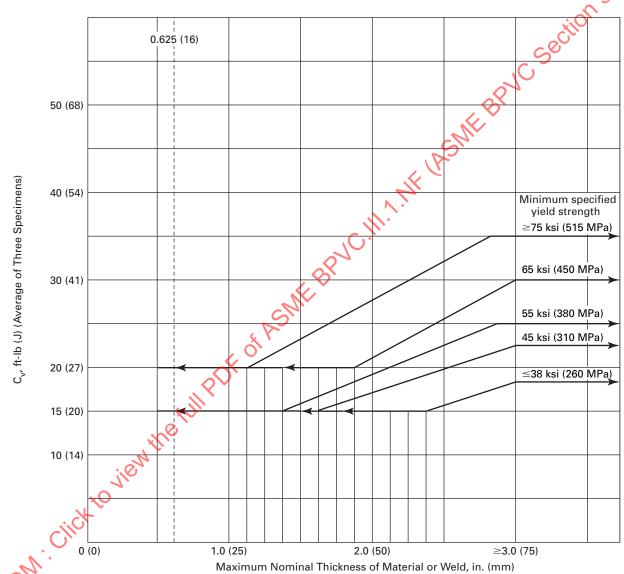
# NF-2351 Retests for Class 1, 2, 3, and MC Material Other Than Bolting

- (a) For Charpy V-notch tests required by NF-2331 to meet the acceptance standards of Tables NF-2331(a)-1, NF-2331(a)-2, and NF-2331(a)-3, respectively, one retest at the same temperature may be conducted, provided
- (1) the average value of the test results meets the minimum requirements,
- (2) not more than one specimen per test is below the minimum requirements, and
- (3) the specimens not meeting the minimum requirements are not lower than 5 mils (0.13 mm) below the specified requirements.

A retest consists of two additional specimens taken as near as practicable to the failed specimens. For acceptance of the retests, both specimens shall meet the minimum requirements.

- (b) For Charpy V-notch tests required by NF-2331 to meet the acceptance standards of Table NF-2331(a)-4, one retest at the same temperature may be conducted provided
- (1) the average value of the test results meets the "Average of 3" requirements specified.
- (2) not more than one specimen per test is below the "Lowest 1 of 3" requirements specified.
- (3) the specimen not meeting the requirements is not lower than 5 ft-lb (7 J) below the "Lowest 1 of 3" requirements specified.



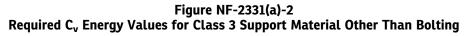


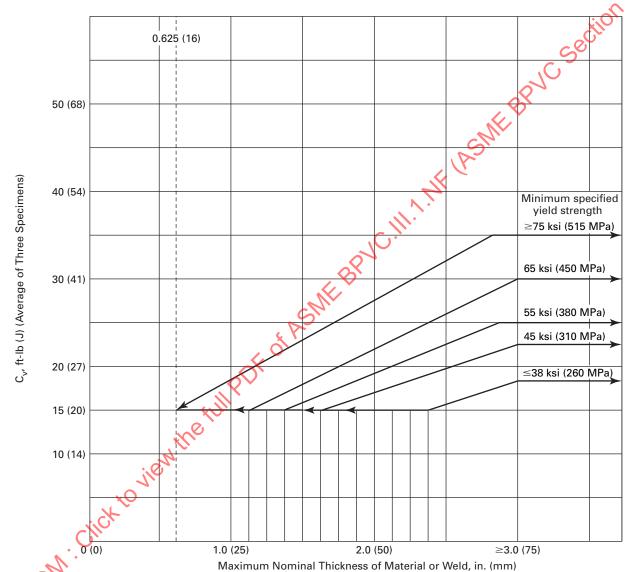
GENERAL NOTES:

(a) Interpolation between yield strengths shown is permitted.

(b) The lowest of three specimens tested shall not be less than 5 ft-lbf (7 J) below the average value required.

19





GENERAL NOTES:

(a) Interpolation between yield strengths shown is permitted.

The lowest of three specimens tested shall not be less than 5 ft-lbf (7 J) below the average value required.

20

### Table NF-2333-1 Required C<sub>v</sub> Values for Bolting Material

Nominal Diameter,	Lateral Expansion,		
in. (mm)	mils (mm)		
1 (25) or less	No test required		
Over 1 (25)	25 (0.64)		

(4) a retest consists of two additional specimens taken as near as practicable to the failed specimens. For acceptance of the retests, both specimens shall be equal to or greater than the "Average of 3" requirements specified.

### NF-2352 Retests for Bolting for Charpy V-Notch Tests

For Charpy V-notch tests of bolting required by NF-2333 to meet the acceptance standards of Table NF-2333-1, one retest at the same temperature may be conducted, provided

- (a) the average value of the test results meets the minimum requirements.
- (b) not more than one specimen per test is below the minimum requirements.
- (c) the specimens not meeting the minimum requirements are not lower than 5 mils (0.13 mm) below the specified requirements.
- (d) a retest consists of two additional specimens taken as near as practicable to the failed specimens. For acceptance of the retests, both specimens shall meet the minimum requirements.

# NF-2360 CALIBRATION OF INSTRUMENTS AND EQUIPMENT

Calibration of temperature instruments and  $C_v$  impact test machines used in impact testing shall be performed at the frequency specified in (a) or (b).

- (a) Temperature instruments used to control test temperature of specimens shall be calibrated and the results recorded to meet the requirements of NCA-3858.2 at least once in each 3 month interval.
- (b) C<sub>v</sub> impact test machines shall be calibrated and the results recorded to meet the requirements of NCA-3858.2. The calibrations shall be performed using the frequency and methods outlined in ASTM E23 and employing standard specimens obtained from the National Institute of Standards and Technology, or any supplier of subcontracted calibration services accredited in accordance with the requirements of NCA-3126 and NCA-3855.3(c).

# NF-2400 WELDING MATERIAL NF-2410 GENERAL REQUIREMENTS

- (a) All welding material used in the construction and repair of supports or material, except welding material used for cladding or hard surfacing, shall conform to the requirements of the welding material specification or to the requirements for other welding material as permitted in Section IX. In addition, welding material shall conform to the requirements stated in this subarticle and to the rules covering identification in NF-2150.
- (b) The Certificate Holder shall provide the organization performing the testing with the following information, as applicable:
  - (1) welding process
  - (2) SFA Specification and classification
  - (3) other identification if no SFA Specification applies
- (4) minimum tensile strength [NF-2431.1(e)] in either the as-welder or heat-treated condition, or both [NF-2431.1(c)]
- (5) Charpy V-notch test for material as-welded or heat treated, or both (NF-2331); the test temperature, and the lateral expansion or the absorbed energy, shall be provided
- (6) the preheat and interpass temperatures to be used during welding of the test coupon [NF-2431.1(c)]
- (7) postweld heat treatment time, temperature range, and maximum cooling rate, if the production weld will be heat treated [NF-2431.1(c)]
- (8) elements for which chemical analysis is required per the SFA Specification or WPS and NF-2432

**(17**)

(9) minimum delta ferrite (NF-2433)

### NF-2420 REQUIRED TESTS

The required tests shall be conducted for each lot of covered, flux-cored, or fabricated electrodes; for each heat of bare electrodes, rod, or wire for use with the OFW, GMAW, GTAW, PAW, and EGW (electrogas welding) processes (Section IX, QG-109); for each heat of consumable inserts; for each combination of heat of bare electrodes and lot of submerged arc flux; for each combination of lot of fabricated electrodes and lot of submerged arc flux; for each combination of heat of bare electrodes or lot of fabricated electrodes, and dry blend of supplementary powdered filler metal, and lot of submerged arc flux; or for each combination of heat of bare electrodes and lot of electroslag flux. The definitions in SFA-5.01 and the Lot Classes specified in (a) through (e) below shall apply.

- (a) each Lot Class C3 of covered electrodes.
- (b) each Lot Class T2 of tubular-cored electrodes and rods (flux cored or fabricated).
- (c) each Lot Class S2 of fully metallic solid welding consumables (bare electrode, rod, wire, consumable insert, or powdered filler metal).

(d) each Lot Class S2 of fully metallic solid welding electrodes or each Lot Class T2 of tubular-cored (fabricated) electrodes and each Lot Class F2 of submerged arc or electroslag welding flux.

(e) each Lot Class S2 of fully metallic solid welding electrodes or each Lot Class T2 of tubular-cored (fabricated) electrodes and each Lot Class F2 of submerged arc or electroslag welding flux and each Lot Class S2 of supplementary powdered filler metal. The chemical analysis range of the supplemental powdered filler metal shall be the same as that of the welding electrode, and the ratio of powder to electrode used to make the test coupon shall be the maximum permitted for production welding.

In all cases, when filler metal of controlled chemical composition (as opposed to heat control) is used, each container of welding consumable shall be coded for identification and shall be traceable to the production period, the shift, the manufacturing line, and the analysis of the steel rod or strip. Carbon, manganese, silicon, and other intentionally added elements shall be identified to ensure that the material conforms to the SFA or user's material specification. The use of controlled chemical composition is only permitted for carbon and low alloy steel consumables. Tests performed on welding material in the qualification of weld procedures will satisfy the testing requirements for the lot, heat, or combination of heat and batch of welding material used, provided the tests required by Article NF-4000 and this subarticle are made and the results conform to the requirements of this Article.

# NF-2430 WELD METAL TESTS NF-2431 Mechanical Properties Test

Tensile and impact tests shall be made, in accordance with this paragraph, of welding materials which are used to join P-Nos. 1, 3, 4, 5, 6, 7, 9, and 11 base materials in any combination, with the exceptions listed in (a) through (d).

- (a) austenitic stainless steel and nonferrous welding material used to join the listed P-Numbers
  - (b) consumable inserts (backing filler material)
- (c) welding material used for GTAW root deposits with a maximum of two layers
- (d) welding material to be used for the welding of base materials exempted from impact testing by NF-2311(b)(1) through NF-2311(b)(8) and NF-2311(b)(11) and NF-2311(b)(12) shall likewise be exempted from the impact testing required by this paragraph

**NF-2431.1 General Test Requirements.** The welding test coupon shall be made in accordance with (a) through (f) using each process with which the weld material will be used in production welding.

(a) Test coupons shall be of sufficient size and thickness that the test specimens required herein can be removed.

- (b) The weld metal to be tested for all processes except electroslag welding shall be deposited in such a manner as to eliminate substantially the influence of the base material on the results of the tests. Weld metal to be used with the electroslag process shall be deposited in such a manner as to conform to one of the applicable Welding Procedure Specifications (WPS) for production welding. The base material shall conform with the requirements of Section IX, QW-403.1 or QW-403.4, as applicable.
- (c) The welding of the test coupon shall be performed within the range of preheat and interpass temperatures which will be used in production welding. Coupons shall be tested in the as-welded condition or they shall be tested in the applicable postweld heat-treated condition when the production welds are to be postweld heattreated. The postweld heat treatment holding time shall be at least 80% of the maximum time to be applied to the weld metal in production application. The total time for postweld heat treatment of the test coupon may be applied in one heating cycle. The cooling rate from the postweld heat treatment temperature shall be of the same order as that applicable to the weld metal in the support. In addition, weld coupons for weld metal to be used with the electroslag process that are tested in the as-welded condition, or following a postweld heat treatment within the holding temperature ranges of Table NF-4622.1-1 or Table NF-4622.4(c)-1, shall have a thickness within the range of 0.5 to 1.1 times the thickness of the welds to be made in production. Electroslag weld coupons to be tested following a postweld heat treatment, which will include heating the coupon to a temperature above the "Holding Temperature Range" of Table NF-4622.1-1 for the type of material being tested, shall have a thickness within the range of 0.9 to 1.1 times the thickness of the welds to be made in production.
- (d) The tensile specimens, and the  $C_v$  impact specimens where required, shall be located and prepared in accordance with the requirements of SFA-5.1, or the applicable SFA specification. For impact specimen preparation and testing, the applicable parts of NF-2321 shall apply. The longitudinal axis of the specimen shall be at a minimum depth of  $^1/_4t$  from a surface, where t is the thickness of the test weld.
- (e) One all weld metal tensile specimen shall be tested and shall meet the specified minimum tensile strength requirements of the base material specification. When base materials of different specifications are to be welded, the tensile strength requirements shall conform to the specified minimum tensile strength requirements of either of the base material specifications.
- (f) Impact specimens of the weld metal shall be tested where impact tests are required for either of the base materials of the production weld. The weld metal shall conform to the requirements of NF-2331(a), applicable to the base material. Where different requirements exist for the two base materials, the weld metal may conform to either of the two requirements.

**NF-2431.2 Standard Test Requirements.** In lieu of the use of the General Test Requirements specified in NF-2431.1, tensile and impact tests may be made in accordance with this subparagraph where they are required for mild and low alloy steel covered electrodes; the material combinations to require weld material testing, as listed in NF-2431, shall apply for this Standard Test Requirements option. The limitations and testing under this Standard Test option shall be in accordance with (a) through (f).

- (a) Testing to the requirements of this subparagraph shall be limited to electrode classifications included in specifications SFA-5.1 or SFA-5.5.
- (b) The test assembly required by SFA-5.1 or SFA-5.5, as applicable, shall be used for test coupon preparation, except that it shall be increased in size to obtain the number of  $C_{\rm v}$  specimens required by NF-2331(a), where applicable.
- (c) The welding of the test coupon shall conform to the requirements of the SFA Specification for the classification of electrode being tested. Coupons shall be tested in the as-welded condition and also in the postweld heattreated condition. The postweld heat treatment temperatures shall be in accordance with Table NF-4622.1-1 for the applicable P-Number equivalent. The time at postweld heat treatment temperature shall be 8 hr (this qualifies postweld heat treatments of 10 hr or less). When the postweld heat treatment of the production weld exceeds 10 hr, or the PWHT temperature is other than that required above, the general test of NF-2431.1 shall be used.
- (d) The tensile and C<sub>v</sub> specimens shall be located and prepared in accordance with the requirements of SFA-5.1 or SFA-5.5, as applicable.
- (e) One all weld metal tensile specimen shall be tested and shall meet the specified minimum tensile strength requirement of the SFA Specification for the applicable electrode classification.
- (f) The requirements of NF 2431.1(f) shall be applicable to the impact testing of this option.

### NF-2432 Chemical Analysis Test

Chemical analysis of filler metal or weld deposits shall be made in accordance with NF-2420 and as required by NF-2432.1 and NF-2432.2.

**NF-2432.1 Test Method.** The chemical analysis test shall be performed in accordance with this Subparagraph and Table NF-2432.1-1, and the results shall conform to NF-2432.2.

(a) A-No. 8 welding material to be used with GTAW and PAW processes and any other welding material to be used with any GTAW, PAW, or GMAW process shall have chemical analysis performed either on the filler metal or on a weld deposit made with the filler metal in accordance with (c) or (d).

### Table NF-2432.1-1 Sampling of Welding Materials for Chemical Analysis

			All Other
Welding Material	GTAW/ PAW	GMAW	Processes
A-No. 8 filler metal	Filler metal or weld deposit	Weld deposit	Weld deposit
All other filler metal	Filler metal or weld deposit	Filler metal or weld deposit	Weld deposit
1			

(b) A-No. 8 welding material to be used with other than the GTAW and PAW processes and other welding material to be used with other than the GTAW, PAW, or GMAW process shall have chemical analysis performed on a weld deposit of the material or combination of materials being certified in accordance with (c) or (d). The removal of chemical analysis samples shall be from an undiluted weld deposit made in accordance with (c). As an alternative, the deposit shall be made in accordance with (d) for material that will be used for corrosion resistant overlay cladding. Where the Welding Procedure Specification or the welding material specification specifies percentage composition limits for analysis, it shall state that the speoffied limits apply for either the filler metal analysis or the undiluted weld deposit analysis or for in situ cladding deposit analysis in conformance with the above required certification testing.

(c) The preparation of samples for chemical analysis of undiluted weld deposits shall comply with the method given in the applicable SFA Specification. Where a weld deposit method is not provided by the SFA Specification, the sample shall be removed from a weld pad, groove, or other test weld<sup>4</sup> made using the welding process that will be followed when the welding material or combination of welding materials being certified is consumed. The weld for A-No. 8 material to be used with the GMAW or EGW process shall be made using the shielding gas composition specified in the Welding Procedure Specification that will be followed when the material is consumed. The test sample for ESW shall be removed from the weld metal of the mechanical properties test coupon. Where a chemical analysis is required for a welding material which does not have a mechanical properties test requirement, a chemical analysis test coupon shall be prepared as required by NF-2431.1(c), except that heat treatment of the coupon is not required and the weld coupon thickness requirements of NF-2431.1(c) do not apply.

(d) The alternate method provided in (b) for the preparation of samples for chemical analysis of welding material to be used for corrosion resistant overlay cladding shall require a test weld made in accordance with the essential variables of the Welding Procedure Specification that will be followed when the welding material is consumed. The test weld shall be made in conformance with

17

the requirements of Section IX, QW-214.1. The removal of chemical analysis samples shall conform with Section IX, QW-453 for the minimum thickness for which the Welding Procedure Specification is qualified.

### NF-2432.2 Requirements for Chemical Analysis.

The chemical elements to be determined, the composition requirements of the weld metal, and the recording of results of the chemical analysis shall be in accordance with (a) through (c).

- (a) Welding material of ferrous alloy A-No. 8 (Section IX, Table QW-442) shall be analyzed for the elements listed in Table NF-2432.2(a)-1 and for any other elements specified either in the welding material specification referenced by the Welding Procedure Specification or in the Welding Procedure Specification.
- (b) The chemical composition of the weld metal or filler metal shall conform to the welding material specification for elements having specified percentage composition limits. Where the Welding Procedure Specification contains a modification of the composition limits of SFA or other referenced welding material specifications, or provides limits for additional elements, these composition limits of the Welding Procedure Specification shall apply for acceptability.
- (c) The results of the chemical analysis shall be reported in accordance with NCA-3862.1. Elements listed in Table NF-2432.2(a)-1 but not specified in the welding material specification or Welding Procedure Specification shall be reported for information only.

### NF-2433 Delta Ferrite Determination

A determination of delta ferrite shall be performed on A-No. 8 weld material (Section IX, Table QW-442) backing filler metal (consumable inserts); bare electrode, rod, or wire filler metal; or weld metal, except that delta ferrite determinations are not required for SFA-5.4, Type 16-8-2, or A-No. 8 weld filler metal to be used for weld metal cladding.

**NF-2433.1 Method.** Delta ferrite determinations of welding material, including consumable insert material, shall be made using a magnetic measuring instrument and weld deposits made in accordance with (b). Alternatively, the delta ferrite determinations for welding materials may be performed by the use of chemical analysis of NF-2432 in conjunction with Figure NF-2433.1-1.

(a) Calibration of magnetic instruments shall conform to AWS A4.2.

### Table NF-2432.2(a)-1 Welding Material Chemical Analysis

Materials	Elements		
Cr-Ni stainless materials	C, Cr, Mo, Ni, Mn, Si, Cb		

- (b) The weld deposit for magnetic delta ferrite determination shall be made in accordance with NF-2432.1(c).
- (c) A minimum of six ferrite readings shall be taken on the surface of the weld deposit. The readings obtained shall be averaged to a single Ferrite Number (FN).

**NF-2433.2 Acceptance Standards.** The minimum acceptable delta ferrite shall be 5FN. The results of the delta ferrite determination shall be included in the Certified Material Test Report of NF-2130 or NF-4120.

## NF-2440 STORAGE AND HANDLING OF WELDING MATERIAL

Suitable storage and handling of electrodes, flux, and other welding materials shall be maintained. Precautions shall be taken to minimize absorption of moisture by fluxes and cored, fabricated, and coated electrodes.

# NF-2500 EXAMINATION AND REPAIR OF MATERIAL

## NF-2510 EXAMINATION AND REPAIR OF MATERIAL OTHER THAN BOLTING

Material for supports shall be examined in accordance with the material specification. Defects may be repaired as permitted by the material specification.

# NP-2520 REQUIRED EXAMINATION AND ACCEPTANCE STANDARDS FOR CLASS 1 SPRINGS

Spring coils for Class 1 variable, constant, and sway brace standard supports shall be examined after coiling by either the magnetic particle or liquid penetrant method. Linear indications shall be removed by grinding and blending. If the depth of the ground area exceeds 3% of the bar diameter or thickness, the spring coil shall be rejected.

# NF-2530 REQUIRED EXAMINATION AND ACCEPTANCE STANDARDS FOR COILED WIRE ROPE USED AS A SPRING

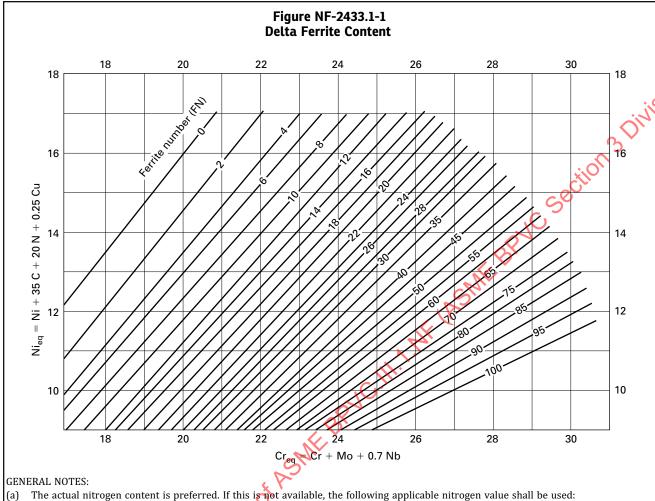
Coiled wire rope used as a spring shall be visually examined after coiling and shall be free from kinks, loose strands, loose wires, or broken protruding wires.

### NF-2580 EXAMINATION OF BOLTING MATERIAL (17) AND RODS

The word *rods* as used in this subarticle is intended to cover rods that are threaded. It also includes forged piston rods if they are threaded for a portion of their length. It does not apply to unthreaded bar stock.

### NF-2581 Required Examinations

**NF-2581.1 Class 1 Supports.** All bolting material and rods shall be visually examined in accordance with NF-2582. In addition, nominal sizes greater than 2 in.



- (1) GMAW welds, 0.08%, except that when self-shielding, flux-cored electrodes are used, 0.12%
- (2) welds made using other processes, 0.06%
- (b) This diagram is identical to the WRC-1992 Diagram, except that the solidification mode lines have been removed for ease of use.

(50 mm) shall be examined by either the magnetic particle or liquid penetrant method in accordance with NF-2583. In addition, nominal sizes greater than 4 in. (100 mm) shall be ultrasonically examined in accordance with NF-2584.

**NF-2581.2 Class 2, 3, and MC Supports.** Bolts, studs, and nuts shall be examined in accordance with the requirements of the material specification and NF-2582.

### NF-2582 Visual Examination

The final surfaces of threads, shanks, and heads shall be visually examined for workmanship, finish, and appearance in accordance with the requirements of ASTM F788 for bolting material and ASTM F812 for nuts. The visual examination personnel shall be trained and qualified in accordance with the Material Organization's Quality System Program or the Certificate Holder's Quality Assurance Program. These examinations are not required to be

performed either in accordance with procedures qualified to NF-5100 or by personnel qualified in accordance with NF-5500.

### NF-2583 Magnetic Particle or Liquid Penetrant Examination

All bolting material and rods greater than 2 in. (50 mm) nominal size for Class 1 supports shall be examined by a magnetic particle method or a liquid penetrant method. Such examination shall be performed on the finished bolting material after threading or on the material stock at approximately the finished diameter before threading and after heading (if involved). Linear nonaxial indications are unacceptable. Linear axial indications greater than 1 in. (25 mm) in length are unacceptable.

### NF-2584 Ultrasonic Examination for Sizes Greater Than 4 in. (100 mm)

All bolting material and rods greater than 4 in. (100 mm) nominal size for Class 1 supports shall be ultrasonically examined over the entire cylindrical surface prior to threading in accordance with the requirements of the following subparagraphs.

**NF-2584.1 Ultrasonic Method.** Examination shall be carried out by the straight beam, radial scan method.

**NF-2584.2 Examination Procedure.** Examination shall be performed at a nominal frequency of 2.25 MHz with a search unit not to exceed 1 in.<sup>2</sup> (650 mm<sup>2</sup>) area.

**NF-2584.3 Calibration of Equipment.** Calibration sensitivity shall be established by adjustment of the instrument so that the first back reflection is 75% to 90% of full screen height.

**NF-2584.4 Acceptance Standards.** Any discontinuity that causes an indication in excess of 20% of the height of the first back reflection or any discontinuity which prevents the production of a first back reflection of 50% of the calibration amplitude is not acceptable.

### NF-2586 Repair by Welding

Weld repairs of bolting material and rods are not permitted.

# NF-2600 MATERIAL MANUFACTURER'S QUALITY SYSTEM PROGRAMS

### NF-2610 DOCUMENTATION AND MAINTENANCE OF QUALITY SYSTEM PROGRAMS

(a) Except as provided in (b), Material Organizations shall have a Quality System Program that meets the requirements of NCA-3800.

- (b) The requirements of NCA-3862 shall be met as required by NF-2130. The other requirements of NCA-3800 need not be used by Material Organizations for small products, as defined in (c), for brazing material, and for material that is allowed by this Subsection to be furnished with a Certificate of Compliance. For these products, the Certificate Holder's Quality Assurance Program (Article NCA-4000) shall include measures to provide assurance that the material is furnished in accordance with the material specification and the requirements of this Subsection.
- (c) For the purpose of this paragraph, small products are defined as given in (1) through (3)
- (1) pipe, tube, pipe fittings, and flanges of 2 in. (50 mm) nominal size and less
- (2) bolting material, including studs, nuts, and bolts of 2 in. (50 mm) nominal diameter and less
- (3) structural material with a nominal crosssectional area of 2 in.<sup>2</sup> (1 300 mm<sup>2</sup>) and less
- (d) When impact testing is required in accordance with NF-2300, the material not exempted by NF-2311 shall be furnished with Certified Material Test Reports in accordance with NCA-3862.
- (e) Structural materials, which are permitted by this Section to be furnished with a Certificate of Compliance, may be repaired by welding using welders, documentation, and examination requirements specified in SA-6 and need not meet the requirements of NCA-3857.3.
- (f) Materials manufactured to a Material Specification prohibiting weld repair without user approval do not require documentation to indicate that weld repairs have not been performed.

# ARTICLE NF-3000 DESIGN

# NF-3100 GENERAL DESIGN REQUIREMENTS NF-3110 LOADING CRITERIA NF-3111 Loading Conditions

The loadings that shall be taken into account in designing a support include, but are not limited to, those in the following:

- (a) weight of the piping or component and normal contents under plant and system operating and test conditions, including loads due to static and dynamic head and fluid flow effects
  - (b) weight of the support
- (c) superimposed static and thermal loads and reactions induced by the supported system components
- (d) dynamic loads, including loads caused by earthquake and vibration
  - (e) effects from piping thermal expansion
  - (f) anchor and support movement effects
  - (g) environmental loads such as wind and snow loads/
- (h) expansion or contraction of a component produced by internal or external pressure

Guidelines for classification of these loadings into primary, secondary, or peak are listed in NF 3120 and NF-3220.

# NF-3112 Design Loadings

The Design Loadings shall be established in accordance with NCA-2142.1 and NF-3112.1 through NF-3112.3.

NF-3112.1 Design Temperature. The specified Design Temperature shall be established in accordance with NCA-2142.1(b). The metal temperature shall be determined by computation using accepted heat transfer procedures or by measurement from equipment in service under plant and system equivalent operating conditions. In lieu of heat transfer analysis or measurements, the component or piping Design Temperature may be used. In no case shall the temperature at the surface of the metal exceed the maximum temperature listed in Section II, Part D, Subpart 1, Tables 1A, 1B, 2A, 2B, 3, 4, and Y-1, or exceed the maximum temperature limitations specified elsewhere in this Subsection.

**NF-3112.2 Design Mechanical Loads.** The specified Design Mechanical Loads shall be established in accordance with NCA-2142.1(c), and shall include all loads from the component or piping acting on the support.

NF-3112.3 Design Stress Intensity and Allowable Stress Values. When the procedures of design by analysis (NF-3220) are employed, the applicable design stress intensity values  $S_m$  listed in Section II, Part D, Subpart 1, Table 2A, 2B, and 4 shall be used. When the procedures of linear elastic analysis (NF-3320) are employed, the allowable stress values shall be the applicable yield strength values  $S_y$  listed in Section II, Part D, Subpart 1, Table Y-1 as modified by the design factors given in NF-3322. The material shall not be used at metal and design temperatures that exceed the temperature limit in the applicability column for which stress or stress intensity values are listed. The values in the tables may be interpolated for intermediate temperatures.

# NF-3113 Service Conditions

Each Service Condition to which the piping or component may be subjected shall be categorized in accordance with NCA-2142.2, and Service Limits [NCA-2142.4(b)] shall be designated in the Design Specification in such detail as will provide a complete basis for design in accordance with this Article.

#### NF-3114 Test Conditions

The component and piping support loadings resulting from test conditions shall be identified (NCA-2142).

#### NF-3115 Design Interface

**(17**)

The design interface between the support and the component shall be established in the Design Specifications (NCA-3254). In order to establish the interface loads, a system analysis is performed (Section III Appendices, Mandatory Appendix XXVII, XXVII-1410). In some cases, the design procedures or service limits permitted for supports in this Article could result in support behavior different from that utilized in the system analysis. If necessary, the support Design Specification shall restrict the design procedure or establish more restrictive limits to reflect those utilized in the system analysis.

# NF-3120 DESIGN CONSIDERATIONS NF-3121 Terms Relating to Design by Analysis

# NF-3121.1 General Considerations.

**(17**)

(a) Terms that are common to the design by stress analysis of Plate- and Shell-Type, Linear-Type, and Standard Supports are defined in NF-3121.2 through NF-3121.17.

- (b) Terms unique to the design by stress analysis of Plate- and Shell-Type Supports are defined in NF-3212.
- (c) Terms unique to the design by stress analysis of Linear-Type Supports are defined in NF-3313.
- **NF-3121.2 Primary Stress.** Primary stress is any normal stress or shear stress developed by an imposed loading that is necessary to satisfy the laws of equilibrium of external and internal forces and moments. The basic characteristic of a primary stress is that it is not self-limiting. Primary stresses that considerably exceed the yield strength will result in failure or, at least, in gross distortion. Primary membrane stress is divided into general and local categories. A general primary membrane stress is one that is so distributed in the support that no redistribution of load occurs in the support as a result of yielding. Examples of primary stress are
- (a) general membrane stress in a circular cylindrical shell or a spherical shell due to internal pressure or to distributed loads
- (b) bending stress in a cantilever beam due to a normal end load

For evaluation purposes, stresses induced in the support by restraint of free end displacement [NF-3111(e)] and anchor motion [NF-3111(f)] of piping are considered primary stresses.

**NF-3121.3 Secondary Stress.** Secondary stress is a normal stress or a shear stress developed by the constraint of adjacent material or by self-constraint of the structure. The basic characteristic of a secondary stress is that it is self-limiting. Local yielding and minor distortions can satisfy the conditions that cause the stress to occur, and failure from one application of the stress is not to be expected.

Examples of secondary stress are

- (a) general thermal stress [NF-3121.11]
- (b) bending stress at a gross structural discontinuity

**NF-3121.4 Peak Stress.** Peak stress is that increment of stress that is additive to the primary plus secondary stresses by reason of local discontinuities or local thermal stress (NF-3121.11), including the effects, if any, of stress concentrations. The basic characteristic of a peak stress is that it does not cause any noticeable distortion and is objectionable only as a possible source of a fatigue crack or a brittle fracture. A stress that is not highly localized falls into this category if it is of a type that cannot cause noticeable distortion. Evaluation of peak stresses in the support is not required by this Subsection.

**NF-3121.5 Normal Stress.** Normal stress is the component of stress normal to the plane of reference. This is also referred to as direct stress. Usually the distribution of normal stress is not uniform through the thickness of a part, so this stress is considered to have two components, one uniformly distributed and equal to the average stress across the thickness under consideration, and the other varying from this average value across the thickness.

**NF-3121.6 Shear Stress.** Shear stress is the component of stress tangent to the plane of reference.

**NF-3121.7 Membrane Stress.** Membrane stress is the component of normal stress that is uniformly distributed and equal to the average stress across the thickness of the section under consideration.

**NF-3121.8 Bending Stress.** Bending stress is the component of normal stress that varies across the thickness. The variation may or may not be linear.

**NF-3121.9 Total Stress.** Total stress is the sum of the primary and secondary stress contributions. Recognition of each of the individual contributions is essential to establishment of appropriate stress limitations.

**NF-3121.10 Critical Buckling.** Critical buckling occurs when a support is loaded to a state at which an infinitesimal additional load of disturbance causes the support to change from an equilibrium condition to one of instability.

**NF-3121.11 Thermal Stress.** Thermal stress is a self-balancing stress produced by a nonuniform distribution of temperature or by differing thermal coefficients of expansion. Thermal stress is developed in a solid body whenever a volume of material is prevented from assuming the size and shape that it normally would under a change in temperature. Evaluation of thermal stresses in the support is not required by this Subsection.

NF-3121.12 Free End Displacement. Free end displacement consists of the relative motions that would occur between an attachment and connected structure or equipment if the two members were separated and permitted to move.

**NF-3121.13 Anchor Point Motion Stress.** Anchor point motion stresses are those stresses resulting from the differential motion of support points. An example is differential building settlement.

NF-3121.14 Gross Structural Discontinuity. Gross structural discontinuity is a geometric or material discontinuity that affects the stress or strain distribution through the entire thickness of the member. Gross discontinuity-type stresses are those portions of the actual stress distributions that produce net bending and membrane force resultants when integrated through the thickness. Examples of a gross structural discontinuity are head-to-shell junctions, flange-to-shell junctions, nozzles, and junctions between parts of different diameters or thicknesses.

**NF-3121.15 Limit Analysis** — **Collapse Load.** The methods of limit analysis are used to compute the maximum load or combination of loads a structure made of ideally plastic (nonstrain-hardening) material can carry. The deformations of an ideally plastic structure increase without bound at this load, which is termed the collapse load.

**NF-3121.16 Collapse Load** — **Lower Bound.** If, for a given load, any system of stresses can be found which everywhere satisfies equilibrium and nowhere exceeds the material yield strength, using shape factors associated with the cross section under consideration, the load is at or below the collapse load. This is the lower bound theorem of limit analysis which permits calculations of a lower bound to the collapse load.

NF-3121.17 Local Structural Discontinuity. Local structural discontinuity is a geometric or material discontinuity that affects the stress or strain distribution through a fractional part of the thickness. The stress distribution associated with a local discontinuity causes only very localized deformation or strain and has no significant effect on the shell-type discontinuity deformations. Examples are small fillet radii, small attachments, and partial penetration welds.

# NF-3122 Piping or Component Deformation

Deformation limits for the supported piping or component shall be stipulated in the Design Specifications, if required.

# NF-3123 Functional Requirements

When a support is to be designed to perform a specific function during any Service Condition, the functional requirements shall be designated in the Design Specifications.

# NF-3124 Rolling and Sliding Supports

- (a) Rolling and sliding supports shall permit free movement of the piping or component, or the piping or component shall be designed to include the imposed load and frictional resistance of these types of supports; their dimensions shall provide for the designed movement of the supported piping or component.
- (b) Sliding supports (or shoes) and brackets shall be designed to accommodate the forces caused by friction in addition to the loads imposed by bearing. The dimensions of the support shall provide for the design movement of the supported piping or component. Material and lubricants used in sliding supports shall be suitable for the environment of the metal at the point of sliding contact and shall be specified in the Design Documents.

## NF-3125 Sway Braces and Vibration Dampeners

Sway braces and vibration dampeners may be used to limit the effects of vibration and may be of the rigid strut (linear type) tension–compression type. If employed in the design or added as a result of observation under start-up or initial operating conditions, the effect of sway braces shall be included in the stress analysis of the piping or component for all of the specified conditions.

# NF-3127 Provisions for Movement of Supported Piping or Component

Consideration shall be given to the relative motion of the supported piping or component and the support. When clearances or travel ranges or both are required to accommodate piping or component movements, sufficient design margins shall be introduced to allow for vare iations due to fabrication and installation. Care shall be taken to ensure that design clearances and travel ranges are based on the maximum range that might occur between two operating conditions and not necessarily on the maximum cold to hot range. All parts of the support shall be fabricated and assembled so that they will not be disengaged by movement of the supported piping or component. Supports needed only to provide stability during Level C or Level D Service Conditions shall be designed and installed so as not to overstress the piping or component during Level A or Level B Service Conditions.

# NF-3128 Snubbers and Dampers

The end connection of the snubber shall be designed to accommodate vertical and horizontal movement of the piping or component, as required. Dampers shall be properly oriented and installed in accordance with manufacturer's requirements.

# NF-3130 GENERAL DESIGN REQUIREMENTS NF-3131 Requirements for Acceptability

The requirements for acceptability of support design are given in (a) through (e).

- (a) The design shall be such that the stress intensity values or the stress values (whichever are applicable) will not exceed the limits given in this subarticle. Table NF-3131(a)-1 indicates the rules and limits to be used for the various classes and types of design procedures. The applicable tables of allowable stress values for materials to be used with a specific design procedure are referenced in Table NF-2121(a)-1.
- (b) The design procedure shall be one of those given in Table NF-3131(a)-1 as being applicable to supports.
- (c) The design details shall conform to the rules of this subarticle or to those referenced herein.
- (d) For configurations where compressive stresses occur, the critical buckling stress shall be taken into account in addition to the requirements of (a), (b), and (c).
- (e) For Class 1 supports, protection against nonductile fracture shall be provided. An acceptable procedure for nonductile failure prevention is given in Section III Appendices, Nonmandatory Appendix G.

# NF-3132 Dimensional Standards

Dimensions of supports may comply with the standards and specifications listed in Table NCA-7100-1 but are not mandatory. Compliance with these standards does not replace or eliminate the requirements for stress analysis or load rating.

Table NF-3131(a)-1
Reference Paragraphs for Procedures for Design of Component Supports, Piping Supports, and
Standard Supports

	Plate and Shell				Linear					
Type and Class of Support	Design by Analysis	Bolting	Weld Joint	Experimen- tal Analysis	Load Rating	Design by Analysis	Bolting	Weld Joints	Experimen- tal Analysis	Load Rating
Component										
Class 1	NF-3220, NF-3522	NF-3225, NF-3525	NF-3226, NF-3526	NF-3270	NF-3280	NF-3320, NF-3523	NF-3324, NF-3525	NF-3324, NF-3526	NF-3370	NF-3380
Class 2 and MC [Note (1)]	NF-3250, NF-3552	NF-3255, NF-3555	NF-3256, NF-3556	NF-3270	NF-3280	NF-3350, NF-3553	NF-3324, NF-3555	NF-3324, NF-3556	NF-3370	NF-3380
Class 3	NF-3260, NF-3552	NF-3265, NF-3555	NF-3266, NF-3556	NF-3270	NF-3280	NF-3360, NF-3553	NF-3324, NF-3555	NF-3324, NF-3556	NF-3370	NF-3380
Piping									70	
Class 1	NF-3220, NF-3622	NF-3225, NF-3625	NF-3226, NF-3626	NF-3270	NF-3280	NF-3320, NF-3623	NF-3324, NF-3625	NF-3324 NF-3626	NF-3370	NF-3380
Class 2	NF-3250, NF-3652	NF-3255, NF-3655	NF-3256, NF-3656	NF-3270	NF-3280	NF-3350, NF-3653	NF-3324, NF-3655	NF-3324, NF-3656	NF-3370	NF-3380
Class 3	NF-3260, NF-3652	NF-3265, NF-3655	NF-3266, NF-3656	NF-3270	NF-3280	NF-3360, NF-3653	NF-3324, NF-3655	NF-3324, NF-3656	NF-3370	NF-3380
Standard							4,			
Class 1 [Note (1)]	NF-3220, NF-3422	NF-3225, NF-3425	NF-3226, NF-3426	NF-3270	NF-3280	NF-3320, NF-3423	NF-3324, NF-3425	NF-3324, NF-3426	NF-3370	NF-3380
Class 2 [Note (1)]	NF-3250, NF-3452	NF-3255, NF-3455	NF-3256, NF-3456	NF-3270	NF-3280	NF-3350	NF-3453	NF-3324, NF-3455	NF-3324, NF-3456	NF-3370, NF-3380
Class 3	NF-3260, NF-3452	NF-3265, NF-3455	NF-3266, NF-3456	NF-3270	NF-3280	NF-3360, NF-3453	NF-3324, NF-3455	NF-3324, NF-3456	NF-3370	NF-3380

#### NOTE:

(1) Supports for Class 2 vessels designed to NC-3200 shall be designed in accordance with Class 1 requirements.

# NF-3133 Stress Analysis

A detailed stress analysis or Design Report, as required by NCA-3550 for all supports, shall be prepared in sufficient detail to show that each of the stress limits of NF-3200 or NF-3300 is satisfied when the support is subjected to the loadings of NF-3110.

# NF-3134 Support Tolerances

- (a) Tolerances shall be specified by the designer in the design output documents. Fabrication tolerances and local installation tolerances as contained in Nonmandatory Appendix NF-D are only mandatory when invoked by the designer.
- (b) When specifying the support tolerances, the support designer shall consider the piping support location/orientation tolerances specified by the piping designer (Section III Appendices, Nonmandatory Appendix T, T-1230).

# NF-3140 GENERAL DESIGN PROCEDURES NF-3141 Types of Procedures

- (a) The design procedure which may be used is dependent on the type of support being designed and the Class of construction involved. Three design procedures are recognized, namely
  - (1) design by analysis
    - (-a) maximum shear stress theory
    - (-b) maximum stress theory
- (2) experimental stress analysis (Section III Appendices, Mandatory Appendix II)
  - (3) load rating
- (b) Unless either the experimental stress analysis procedure or the load rating procedure is used, the requirements of the following paragraphs apply.

# (17) NF-3142 Plate- and Shell-Type Supports — Analysis Procedure

## NF-3142.1 Elastic Analysis.

- (a) Elastic analysis used in the design of Plate- and Shell-Type Supports of Class 1 construction shall be based on maximum shear stress theory in accordance with the rules of NF-3200.
- (b) Elastic analysis used in the design of Plate- and Shell-Type Supports of Class 2, 3, and MC construction shall be based on maximum stress theory. Supports for Class 2 vessels designed to NC-3200 shall be designed in accordance with Class 1 requirements.

# NF-3142.2 Limit Analysis.

- (a) Limit analysis used in the design of Plate- and Shell-Type Supports of Class 1 construction shall be in accordance with the procedures of NF-3221.4.
- (b) Limit analysis used in the design of Plate- and Shell-Type Supports of Class 2, 3, and MC construction shall be in accordance with the procedures of NF-3251.4.

# (17) NF-3143 Linear-Type Supports — Analysis Procedure

- (a) The analysis procedure shall comply with (1) or (2).
- (1) Elastic analysis based on maximum stress theory in accordance with the rules of NF-3300 shall be used for the design of Linear-Type Supports for Class 1, 2, 3, and MC construction.
- (2) Limit analysis in accordance with the procedures of NF-3340 shall be used in the design of Linear-Type Supports for Class 1, 2, 3, and MC construction.
- (b) High cycle fatigue analysis in accordance with procedures of NF-3330 shall be used in the design of Linear-Type Supports for Class 1 construction when members and their connections are subject to high cycle fatigue as defined in NF-3331.

# NF-3144 Standard Supports — Analysis Procedure

Standard Supports designed by analysis shall be designed to either the requirements of NF-3142 or NF-3143, according to whether they are Plate- and Shell-Type or Linear Type Standard Supports.

# NF-3200 DESIGN RULES FOR PLATE- AND SHELL-TYPE SUPPORTS

# NF-3210 GENERAL REQUIREMENTS

# NF-3211 Basis for Determining Stresses in Design by Analysis

The theory of failure used in the rules of this Subsection for combining stresses for the design of Class 1 Plate- and Shell-Type Supports is the maximum shear stress theory; for Class 2, 3, and MC Plate- and Shell-Type Supports, it is the maximum stress theory.

#### NF-3212 Definitions

Terms used in the design of Plate- and Shell-Type Supports by stress analysis are defined in NF-3121 and in NF-3212.1 below.

**NF-3212.1 Stress Intensity.** *Stress intensity* is defined as twice the maximum shear stress which is the difference between the algebraically largest principal stress and the algebraically smallest principal stress at a given point. Tensile stresses are considered positive and compressive stresses are considered negative.

This definition of *stress intensity* is not related to the definition of stress intensity applied in the field of fracture mechanics.

# NF-3220 DESIGN BY ANALYSIS FOR CLASS 1 NF-3221 Stress Limits

Stress limits for elements of Class 1 supports are given in this paragraph. Stress limits for bolts and welds are given in NF-3225 and NF-3226. General requirements concerning stress determinations, definitions, derivations of stress intensities, and classification of stresses are given in NF-3210.

Plate and Shell-Type Supports may be designed by either elastic or limit analysis, stress intensity limits for which are given in NF-3221.1 through NF-3221.4.

- **NF-3221.1 Design Limits.** The stress intensity limits which must be satisfied for the Design Loadings stated in the Design Specification are the two limits of this paragraph and the Special Stress Limits of NF-3223. The design stress intensity values  $S_m$  are given in NF-3224.
- (a) General primary membrane stress intensity  $P_m$  is derived from the average value across the thickness of a section of the general primary stresses produced by specified Design Mechanical Loads, but excluding all secondary stresses. Averaging is to be applied to the stress components prior to determination of the stress intensity values. The allowable value of this stress intensity is  $S_m$  at the Design Temperature.
- (b) Primary membrane plus primary bending stress intensity  $P_m + P_b$  is derived from the highest value across the thickness of a section of the general membrane stresses plus primary bending stresses produced by the specified Design Mechanical Loads, but excluding all secondary stresses. The allowable value of this stress intensity is  $1.5S_m$ .

# **NF-3221.2 Service Level A Through D Service Limits.** The rules and stress limits which must be satisfied in an elastic analysis for any Level A through D Service Loadings stated in the Design Specification are those given in NF-3221.1 and NF-3223 multiplied by the appropriate stress intensity limit factor from Table NF-3221.2-1 for the particular loading and stress categories.

**NF-3221.3 Test Limits.** The rules and stress limits which must be satisfied for any Test Loadings stated in the Design Specification are those given in NF-3221.1

**(17)** 

# Table NF-3221.2-1 Elastic Analysis Stress Categories and Stress Limit Factors for Class 1 Plate- and Shell-Type Supports Designed by Analysis

	Stress Limit Factors for Loading Categories [Note (1)]						
Stress Category	Design	Service Level A	Service Level B [Note (2)]	Service Level C [Note (3)]	Service Level D [Note (3)]	Test Loadings	
Primary stresses [Note (4)], [Note (5)]	$K_m = 1.0$	$K_m = 1.0$	$K_m = 1.33$	$K_m = 1.5$	$K_m$ = greater of 1.5 or $1.2S_y/S_m$	K <sub>m</sub> = 1.33	
	$K_{v} = 1.0$	$K_{v} = 1.0$	$K_{v} = 1.33$	$K_{v} = 1.5$	$K_v$ = greater of 1.5 or $1.2S_y/S_m$ [Note (6)]	<i>K<sub>ν</sub></i> = 1.33	
	$K_{bk} = 1.0$	$K_{bk} = 1.0$	$K_{bk} = 1.33$	$K_{bk} = 1.5$	$K_{bk}$ = greater of 1.5 or $1.2S_y/S_m$	$K_{0k} = 1.33$	
	but stress ≤ ½ of critical buckling stress	•	•	•	but stress ≤ ½ of critical buckling stress Note	•	
Primary plus secondary stresses [Note (5)], [Note (8)]	Evaluation is require this evaluation.	red for critical buckl	ing for all loading ca	tegories. The require	ements of this subar	ticle shall be met for	
Peak stresses	Evaluation not req	uired.		111.			

#### GENERAL NOTE:

- $K_{bk}$  = stress limit factor applicable to the Design allowable membrane stress intensity or membrane plus bending stress intensity (compression only)
- K<sub>m</sub> = stress limit factor applicable to the Design allowable membrane stress intensity or membrane plus bending stress intensity (see NF-3221.1 and NF-3221.2)
- $K_v$  = stress limit factor applicable to the Design allowable shear stress (see NF-3223.2)

#### NOTES:

- (1) Control of deformation is not assured by these stress limit factors. When required by Design Specification, deformation control must be considered separately.
- (2)  $K_m$ ,  $K_v$ , and  $K_{bk} = 1.0$  for design of snubbers and dampers.
- (3) Stress shall not exceed  $0.7S_u$ .
- (4) For Service Levels A, B, C, and D, stresses induced on the supports by restraint of free-end displacement and anchor motions of piping shall be considered as primary stresses.
- (5) Thermal stresses within the support as defined by NF-3121.11 need not be evaluated.
- (6) Shear stress shall not exceed  $0.42S_u$ .
- (7) Two-thirds of the critical buckling stress may be used, provided the critical buckling stress is determined by one of the following methods:
  - (a) comprehensive analysis that considers effects such as geometric imperfections, deformations due to existing loading conditions, nonlinearities, large deformations, residual stresses, and inertial forces
  - (b) tests of physical models under conditions of restraint and loading the same as those to which the configuration is expected to be subjected
- (8) Service Levels A and B, primary plus secondary stresses shall be limited to a range of 2S<sub>y</sub> or S<sub>u</sub> at temperature, whichever is less for component supports only.

and NF-3223 multiplied by the appropriate stress intensity limit factor in Table NF-3221.2-1 under Test Loadings.

NF-3221.4 **Limit Analysis.** The limits on primary **(17)** membrane plus primary bending stress intensity [NF-3221.1(b)] need not be satisfied at a specific location if it can be shown by means of limit analysis or by tests that the specified loadings do not exceed two-thirds of the lower bound collapse load for Design Loadings and Service Level A and B Loadings, and do not exceed 0.8 times the lower bound collapse load for Service Level C Loadings. Service Level D Loadings shall not exceed 0.9 times the lower bound collapse load. For Service Level D, the lower bound collapse load may be determined using a yield stress in the analysis that is the greater of  $1.2S_v$  and  $1.5S_m$ , but not larger than  $0.7S_u$ . For materials in Section II, Part D, Subpart 1 for which allowable stresses, or stress intensities, may reach 90% of the yield strength  $S_{\nu}$  at temperature, the specified loading shall not exceed the product of the applicable permanent strain limiting factor of Section II, Part D, Subpart 1, Table Y-2 times the lower bound collapse load.

## NF-3222 Derivation of Stress Intensities

One requirement for the acceptability of a design (NF-3130) is that the calculated stress intensities shall not exceed specified allowable limits. These limits differ depending on the stress category (primary, secondary, etc.) from which the stress intensity is derived. This paragraph describes the procedure for the calculation of the stress intensities which are subject to the specified limits. The steps in the procedure are stipulated in (a) through (e).

- (a) At the point on the support which is being investigated, choose an orthogonal set of coordinates, such as tangential, longitudinal, and radial, and designate them by the subscripts t, l, and r. The stress components in these directions are then designated  $\sigma_t$ ,  $\sigma_l$ , and  $\sigma_r$  for direct stresses and  $T_{lt}$ ,  $T_{rr}$ , and  $T_{rt}$  for shear stresses.
- (b) Calculate the stress components for each type of loading to which the item will be subjected and assign each set of stress values to one or a group of the following categories

(1) general primary membrane stress  $P_m$  (NF-3121.7)

- (2) primary bending stress  $P_b$  (NF-3121.8)
- (3) secondary stress Q (NF-3121.3)
- (c) For each category, calculate the algebraic sum of the values of  $\sigma_t$  that result from the different types of loadings, and similarly for the other five stress components. Certain combinations of the categories must also be considered.
- (d) Translate the stress components for the t, l, and r directions into principal stresses  $\sigma_1$ ,  $\sigma_2$ , and  $\sigma_3$ .

(e) Calculate the stress differences  $S_{12}$ ,  $S_{23}$ , and  $S_{31}$  from the following equations:

$$S_{12} = \sigma_1 - \sigma_2$$

$$S_{23} = \sigma_2 - \sigma_3$$

$$S_{31} = \sigma_3 - \sigma_1$$

The stress intensity S is the largest absolute value of  $S_{12}$ ,  $S_{23}$ , and  $S_{31}$ .

NOTE: Membrane stress intensity is derived from the stress components averaged across the thickness of the section. The averaging shall be performed at the component stress level in (b) or (c).

# NF-3223 Special Stress Limits

The following deviations from the basic stress limits are provided to cover special Service Loadings or configurations. Some of these deviations are more restrictive and some are less restrictive than the basic stress limits. In cases of conflict between these requirements and the basic stress limits, the rules of this paragraph take precedence for the particular situations to which they apply.

## NF-3223.1 Bearing Loads.

(a) The average bearing stress for resistance to crushing under the maximum load, experienced as a result of Design Loadings, Test Loadings, or any Service Loadings, except those for which Level D Limits are designated, shall be limited to  $S_y$  at temperature, except that when the distance to a free edge is larger than the distance over which the bearing load is applied, a stress of  $1.5S_y$  at temperature is permitted.

**(17)** 

- (b) When bearing loads are applied near free edges, such as at a protruding ledge, the possibility of a shear failure shall be considered. The average shear stress shall be limited to  $0.6S_m$  in the case of primary stress (NF-3121.2) and  $0.5S_y$  in the case of primary stress plus secondary stress (NF-3121.9).
- (c) When considering bearing stresses in pins and similar members, the  $S_y$  at temperature is applicable, except that a value of  $1.5S_y$  may be used if no credit is given to the bearing area within one pin diameter from a plate edge.
- (d) Except for pinned and bolted joints, bearing stresses need not be evaluated for loads for which Level D Service Limits apply.

#### NF-3223.2 Pure Shear.

(a) The average primary shear stress across a section loaded in pure shear, experienced as a result of Design Loadings, Test Loadings, or any Service Loadings, except those for which Level D Limits are designated, shall be limited to  $0.6S_m$ .

(b) The maximum primary shear, experienced as a result of Design Loadings, Test Loadings, or any Service Loadings except those for which Level D Limits are designated, exclusive of stress concentration at the periphery of a solid circular section in torsion, shall be limited to  $0.8S_m$ . Primary plus secondary shear stresses shall be converted to stress intensities (equal to two times pure shear stress) and as such shall not exceed the basic stress limits of Table NF-3221.2-1.

**NF-3223.3 Triaxial Stresses.** The algebraic sum of the three primary principal stresses  $(\sigma_1 + \sigma_2 + \sigma_3)$  shall not exceed four times the tabulated value of  $S_m$ .

NF-3223.4 Applications of Elastic Analysis for Stresses Beyond the Yield Strength. Certain of the allowable stresses permitted in the design criteria are such that the maximum stress calculated on an elastic basis may exceed the yield strength of the material.

# NF-3224 Design Stress Values

The design stress intensity values  $S_m$  are given in Section II, Part D, Subpart 1, Tables 2A, 2B, and 4 for support material. Values for intermediate temperatures may be found by interpolation. Values of yield strength and ultimate tensile strength are given in Section II, Part D, Subpart 1, Tables Y-1 and U, respectively. Values of the coefficient of thermal expansion and of the modulus of elasticity are given in Section II, Part D, Subpart 2, Tables TE and TM.

## NF-3225 Design of Bolting

**NF-3225.1 Design Limits.** The rules and stress limits that must be satisfied for any Design Loading stated in the Design Specification are those given in NF-3324.6.

NF-3225.2 Service Limits, Level A Through D. The rules and stress limits which must be satisfied for any Level A through D Service Loading stated in the Design Specification are those given in NF-3324.6 multiplied by the appropriate stress limit factor for the particular service loading level and stress category specified in Table NF-3225.2-1. This product shall not exceed the yield strength of the material at temperature.

**NF-3225.3 Test Limits.** The rules and stress limits that must be satisfied for any Test Loadings stated in the Design Specification are those given in NF-3324.6 multiplied by the appropriate stress limit factor for the particular Service Loading Level and stress category specified in Table NF-3225.2-1.

**NF-3225.4 Friction-Type Joints.** Multiplying factors normally used to increase the allowable limits for Service Level B, C, and D are not applicable to friction-type joints [see NF-3324.6(a)(4)].

# NF-3226 Design of Welded Joints

NF-3226.1 Permissible Types of Wetded Joints in Plate- and Shell-Type Supports. All welded joints in Plate- and Shell-Type Supports shall be continuous and shall be of one of the types listed in (a) through (c). Typical examples of those permitted types are shown in Figure NF-3226.1-1. Fillet and partial penetration welds shall meet the requirements of NF-3324.5(d), except that NF-3324.5(d)(7) does not apply.

(a) Butt Joints. Butt joints shall be one of the following: (1) full penetration, single or double welded [Figure NF-3226.1-1, sketches (a-1) and (a-2)].

(2) partial penetration, double welded [Figure NF-3226.1-1, sketch (b)].

(3) the applicable welds may be either square groove, V groove, bevel groove, J groove, U groove, flare V groove, or flare bevel groove [see Figure NF-3226.1(a)-1].

(4) when angle joints are used for connecting a transition in diameter to a cylinder, the angle  $\alpha$  of Figure NF-3226.1-1, sketch (f) shall not exceed 30 deg.

(5) A tapered transition having a length not less than three times the offset between the adjacent surfaces of abutting sections, as shown in Figure NF-3226.1(a)-2, shall be provided at joints between sections that differ in thickness by more than one-fourth of the thickness of the thinner section or by more than  $\frac{1}{8}$  in. (3 mm), whichever is less. The transition may be formed by any process that will provide a uniform taper. The weld may be partly or entirely in the tapered section, or adjacent to it. This

# Table NF-3225.2-1 Stress Limit Factors for Class 1, 2, 3, and MC Bolt Design by Analysis

		Stress Limit Factors	for Loading Level	s Stated in Design	Specification [Note (1)	]
Stress Category	Design	Service Level A	Service Level B	Service Level C	Service Level D	Test Loading
Tension and shear	$K_{ho} = 1.0$	$K_{ho} = 1.0$	$K_{ho} = 1.15$	$K_{ho} = 1.25$	$K_{bo} = 1.5 [Note (2)]$	$K_{ho} = 1.25$

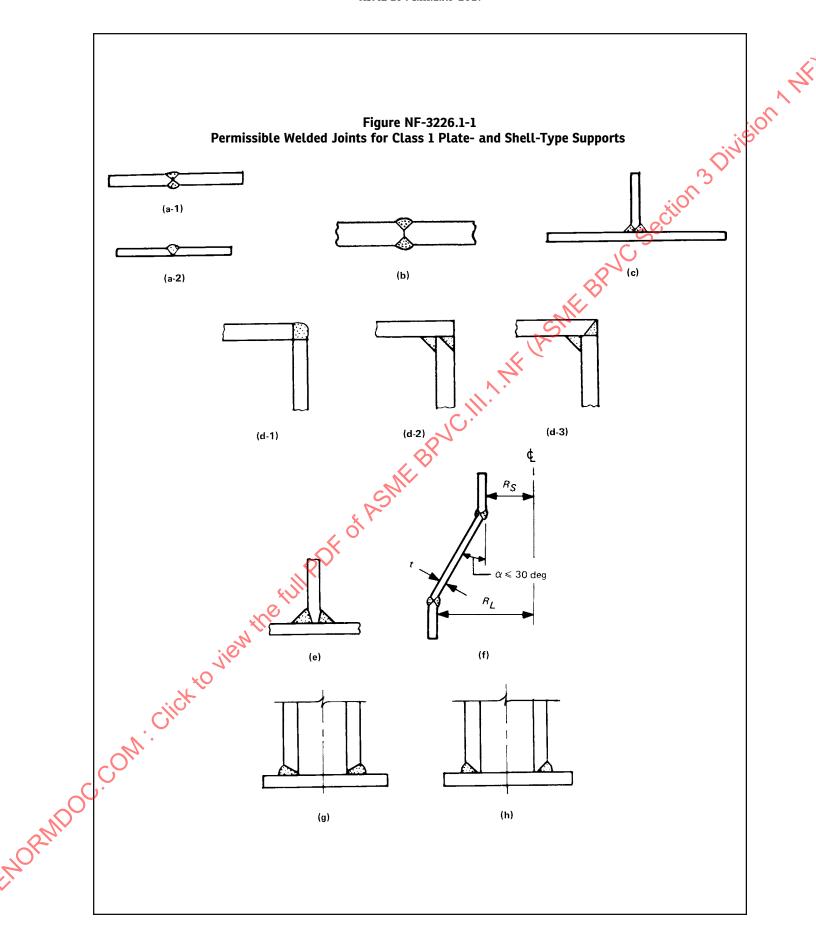
GENERAL NOTE:

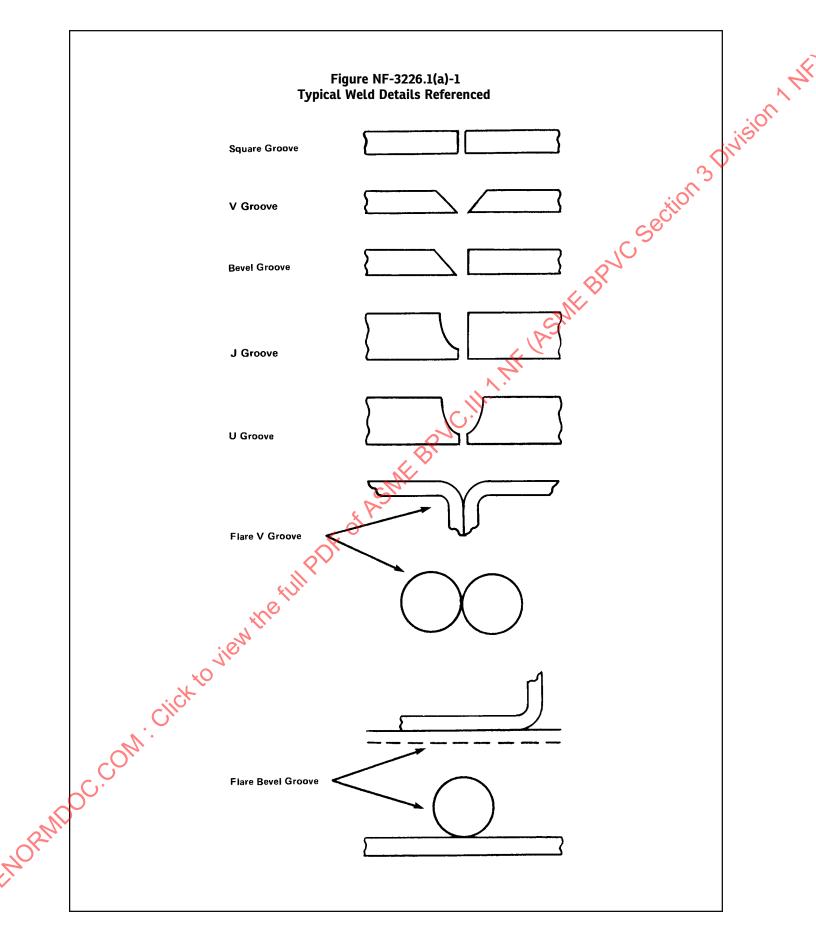
 $K_{bo}\,$  = stress limit factor applicable to the Design allowable tensile and shear stresses

#### NOTES

- (1) Not to be used for friction type connections.
- (2) Average tensile stress shall not exceed  $0.7S_u$  and  $S_y$ , and average bolt shear stress shall not exceed  $0.42S_u$  and  $0.6S_y$ .

**(17**)



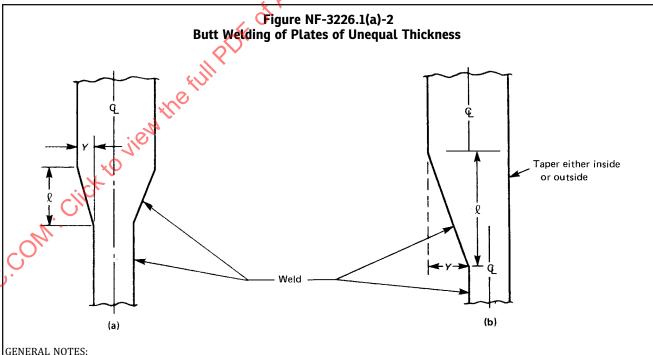


paragraph also applies when there is a reduction in thickness within a spherical shell, or cylindrical course or

- (6) When the use of backing rings will result in undesirable conditions such as severe stress or corrosion, the requirements of NF-4240 shall be met.
- (b) Corner Joints. Corner joints shall be one of the following:
- (1) full penetration, as shown in Figure NF-3226.1-1, sketch (d-1)
- (2) partial penetration with a fillet weld as shown in Figure NF-3226.1-1, sketches (d-2) and (d-3)
- (3) the applicable welds are fillet, square groove, V groove, bevel groove, U groove, J groove, flare V groove and flare bevel groove
- (c) Tee Joints. Tee joints shall be one of the following:
- (1) full penetration, Figure NF-3226.1-1, sketches (c) and (g)
- (2) partial penetration, double welded with or without additional fillet welds, Figure NF-3226.1-1, sketch (e)
- (3) partial penetration, single welded between the end surface of a closed tubular section or a closed formed section, Figure NF-3226.1-1, sketch (h)
- (4) the applicable welds may be square groove, bevel groove, I groove, flare V groove or flare bevel groove

# NF-3226.2 Stress Intensities and Stress Limits for Welded Joints in Plate- and Shell-Type Supports.

- (a) Design Limits. The stress intensity and allowable stress limits which must be satisfied for welds for the Design Loadings stated in the Design Specification shall be the following:
- (1) Full Penetration Groove Welds. The stress intensity limits for full penetration groove welds shall not ex ceed the applicable stress intensity value for the base metal being joined, as specified in NF-3221.1 and Table NF-3324.5(a)-1. See NF-3111.
  - (2) Partial Penetration Groove Welds
- (-a) Compression Normal to Effective Throat or Shear on Effective Throat. The stress intensity and stress limits shall be the same as those for the base metal as required in NF-3221.1.
- (-b) Tension Normal to the Axis on the Effective Throat. The stress limits shall be as specified in Table NF-3324.5(a)-1.
- (3) Fillet Welds The allowable stress limits for fillet welds shall be as specified in Table NF-3324.5(a)-1.
- (b) Service Limits, Levels A. B. C. and D. and Test, The rules and stress limits that must be satisfied for welds for any Level A, B, C, and D Service and Test Loadings stated in the Design Specification are those given in (a) multiplied by the appropriate base material stress limit factor given in Table NF-3251.2-1.
- (c) The effective sizes of welds shall be as given in NF-3324.5(d) and NF-3324.5(f).



GENERAL NOTES:

- $\ell \geq 3y$ , where  $\ell$  is required length of taper and y is the offset between the adjacent surfaces of abutting sections.
- Length of required taper  $\ell$  may include the width of the weld.
- In all cases  $\ell$  shall not be less than 3y.

NF-3226.3 Consideration of Lamellar Tearing.

Welded joint configurations causing significant through-thickness tensile stress [as defined in NF-1215(b)] during fabrication and/or service on rolled product forms should be avoided. However, if this type of construction is used, the designer should consider one or several of the following factors that may reduce the susceptibility of the joint to experience lamellar tearing and provide documentation, including fabrication requirements, in the Design Output Documents:

- (a) Reduce volume of weld metal to the extent practical.
- (b) Select materials that are resistant to lamellar tearing.
- (c) Invoke any of the special fabrication requirements of NF-4441.

#### NF-3250 **DESIGN BY ANALYSIS FOR CLASS 2** AND MC

Plate- and Shell-Type Supports may be designed by either elastic or limit analysis, limits for which are given in the following subparagraphs. Limits for bolts and welds are given in NF-3255 and NF-3256. For general requirements and definitions, see NF-3100 and NF-3210.

## NF-3251 Stress Limits

**NF-3251.1 Design Loadings.** The stress (maximum **(17)** normal stress, also referred to as principal stress) limits are satisfied for the Design Loadings (NCA-2142.1) stated in the Design Specifications if the requirements of eqs. [15] and (2) are met.

$$\sigma_m \leq 1.0S \tag{1}$$

$$\sigma_m + \sigma_b \leq 1.5S \tag{2}$$

$$\sigma_m + \sigma_b \le 1.5S \tag{2}$$

where

S = allowable stress value, ksi (MPa), from the applicable table of Section Part D, Subpart 1, as referenced in Table NF 2121(a)-1

 $\sigma_b$  = bending stress, ksi (MPa) (NF-3121.8)

 $\sigma_m$  = membrane stress, ksi (MPa) (NF-3121.7)

NF-3251.2 Service Loadings — Service Level A

**Through D** The rules and stress limits that must be satisfied in an elastic analysis for any Level A through D Service Loading stated in the Design Specification are those given for Design Loadings in NF-3251.1 multiplied by the appropriate stress limit factor specified in Table NF-3251.2-1.

**NF-3251.3 Test Loadings.** The rules and stress limits that must be satisfied for any Test Loadings stated in the Design Specification are those given for Design Loadings in NF-3251.1 multiplied by the appropriate stress limit factor specified in Table NF-3251.2-1.

NF-3251.4 Limit Analysis. The limits for Class 1 Plate- and Shell-Type Supports designed by limit analysis apply.

#### NF-3252 **Special Stress Limits**

The following deviations from the basic stress limits are provided to cover special Service Loadings or configurations. Some of these deviations are more restrictive and some are less restrictive than the basic stress limits. In cases of conflict between these requirements and the basic stress limits, the rules of this paragraph take precedence for the particular situations to which they apply.

## NF-3252.1 Bearing Loads.

(a) The average bearing stress for resistance to crushing under the maximum load, experienced as a result of Design Loadings, Test Loadings, or any Service Loadings, shall be limited to  $S_{\nu}$  at temperature, except that when the distance to a free edge is larger than the distance over which the bearing load is applied, a stress of  $1.5S_v$  at temperature is permitted.

**(17)** 

- (b) When bearing loads are applied near free edges, such as at a protruding ledge, the possibility of a shear failure shall be considered. In the case of load controlled stress only the average shear stress shall be limited to 0.6S in the case of primary stress (NF-3121.2).
- (g) When considering bearing stresses in pins and similar members, the  $S_v$  at temperature value is applicable, except that a value of  $1.5S_v$  may be used if no credit is given to the bearing area within one pin diameter from a plate edge.
- (d) Except for pinned and bolted joints, bearing stresses need not be evaluated for loads for which Level D Service Limits apply.

#### NF-3252.2 Pure Shear.

- (a) The average primary shear stress across a section loaded in pure shear, experienced as a result of Design Loadings, Test Loadings, or any Service Loadings, shall be limited to 0.6*S*.
- (b) The maximum primary shear, experienced as a result of Design Loadings, Test Loadings, or any Service Loadings exclusive of stress concentration at the periphery of a solid circular section in torsion, shall be limited to 0.8S.

#### NF-3255 **Design of Bolting**

The provisions of NF-3225 apply.

#### NF-3256 **Design of Welded Joints**

NF-3256.1 Permissible Types of Welded Joints in Plate- and Shell-Type Supports. All welded joints in Plate- and Shell-Type Supports shall be continuous and shall be of one of the types listed in (a) through (d). Typical examples of those permitted types are shown in Figure NF-3256.1-1. Fillet and partial penetration welds shall meet the requirements of NF-3324.5(d), except that

	Stress Limit Factors for Loading Categories [Note (1)]						
Stress Category	Design	Service Level A	Service Level B [Note (2)]	Service Level C [Note (3)]	Service Level D [Note (3)]	Test Loadings	
Primary stresses [Note (4)],	$K_m = 1.0$	$K_m = 1.0$	$K_m = 1.33$	$K_m = 1.5$	$K_m$ = greater of 1.5 or 1.2 $S_y/S$	$K_m = 1.33$	
[Note (5)]	$K_{\nu} = 1.0$	$K_{\nu} = 1.0$	$K_{\nu} = 1.33 \text{ [Note (6)]}$	$K_{\nu} = 1.5 \text{ [Note (6)]}$	$K_v$ = greater of 1.5 or $1.2S_y/S$ [Note (6)]	$K_{\nu} = 1.33$ [Note (6)]	
	$K_{bk}=1.0$	$K_{bk}=1.0$	$K_{bk}=1.33$	$K_{bk}=1.5$	$K_{bk}$ = greater of 1.5 or 1.2 $S_v/S_v$	$K_{bk} = 1.33$	
	but stress ≤ ½ of critical buckling stress	but stress ≤ ½ of critical buckling stress	, <del>-</del>	but stress ≤ ½ of critical buckling stress		but stress $\leq \frac{1}{2}$ of	

#### GENERAL NOTE:

 $K_{bk}$  = stress limit factor applicable to the Design allowable membrane stress or membrane plus bending stress (compression only)

 $K_m$  = stress limit factor applicable to the Design allowable membrane stress or membrane plus bending stress (see NF-3251.1 and NF-3261)

 $K_v$  = stress limit factor applicable to the Design allowable shear stress (see NF-32522 and NF-3261)

#### NOTES

- (1) Control of deformation is not assured by these stress limit factors. When required by Design Specification, deformation control must be considered separately.
- (2)  $K_m$  and  $K_{bk}$  = 1.0 for design of snubbers and dampers.
- (3) Stress shall not exceed  $0.7S_u$ .
- (4) For Service Levels A, B, C, and D, stresses induced on the supports by restraint of free-end displacement and anchor motions of piping shall be considered as primary stresses.
- (5) Thermal stresses within the support as defined by NF-3121.11 need not be evaluated.
- (6) Shear stress shall not exceed  $0.42S_u$ .
- (7) Two-thirds of the critical buckling stress may be used, provided the critical buckling stress is determined by one of the following methods:
  - (a) comprehensive analysis that considers effects such as geometric imperfections, deformations due to existing loading conditions, nonlinearities, large deformations, residual stresses, and inertial forces
  - (b) tests of physical models under conditions of restraint and loading the same as those to which the configuration is expected to be subjected

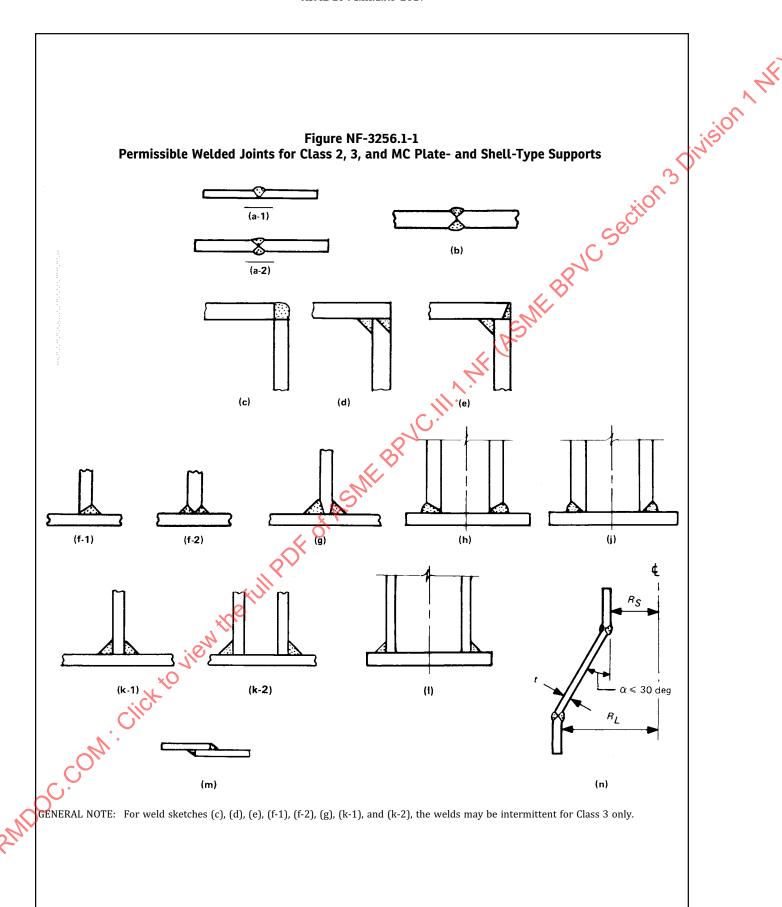
NF-3324.5(d)(7) does not apply. Plug and slot welds shall meet the requirements of NF-3324.5(e). Lap joints shall meet the requirements of NF-3324.3.

- (a) Butt Joints. Butt joints shall be one of the following:
- (1) full penetration, single and double welded, Figure NF-3256.1-1, sketches (a-1) and (a-2).
- (2) partial penetration, double welded, Figure NF-3256,14, sketch (b).
- (3) when angle joints are used for connecting a transition in diameter to a cylinder, the angle  $\alpha$  of Figure NF<sub>3</sub>3256.1-1, sketch (n) shall not exceed 30 deg.
- (4) A tapered transition having a length not less than three times the offset between the adjacent surfaces of abutting sections, as shown in Figure NF-3226.1(a)-2, shall be provided at joints between sections that differ in thickness by more than one-fourth of the thickness of the thinner section or by more than  $\frac{1}{8}$  in. (3 mm),

whichever is less. The transition may be formed by any process that will provide a uniform taper. The weld may be partly or entirely in the tapered section or adjacent to it.

- (5) When the use of backing rings will result in undesirable conditions such as severe stress or corrosion, the requirements of NF-4240 shall be met.
- (b) Corner Joints. Corner joints shall be one of the following:
- (1) full penetration, as shown in Figure NF-3256.1-1, sketch (c)
- (2) partial penetration with a fillet weld as shown in Figure NF-3256.1-1, sketches (d) and (e)
  - (c) Tee Joints. Tee joints shall be one of the following:
- (1) full penetration, single or double welded, Figure NF-3256.1-1, sketches (f-1), (f-2), and (h)
- (2) partial penetration, with or without additional fillet welds, Figure NF-3256.1-1, sketch (g)

**17**)



40

- (3) partial penetration, single welded between a plate and the end surface of a closed tubular section or a closed formed section, Figure NF-3256.1-1, sketch (j)
- (4) fillet, double welded, Figure NF-3256.1-1, sketch (k-1); single welded when double members are used, Figure NF-3256.1-1, sketch (k-2)
- (5) fillet, single welded between a flat surface and the end surface of a closed tubular section or a closed formed section, Figure NF-3256.1-1, sketch (l)
- (d) Lap Joints. Lap joints shall be fillet, double welded, Figure NF-3256.1-1, sketch (m).
- (e) The applicable welds for the joints permitted in (a) through (d) are
- (1) square groove, bevel groove, J groove, flare bevel groove, U groove and V groove
- (2) plug and slot welds are permitted in (c) and (d) only
- (3) fillet welds are permitted in (c)(4), (c)(5), and (d) only

#### NF-3256.2 Allowable Stress Limits.

- (a) Design Limits. The allowable stress limits which must be satisfied for the Design Loadings stated in the Design Specification shall be the following:
- (1) Full Penetration Groove Welds. The allowable stress limits for full penetration groove welds shall not exceed the applicable allowable stress value for the base metal being jointed, as specified in NF-3251.1 and Table NF-3324.5(a)-1.
  - (2) Partial Penetration Groove Welds
- (-a) Compression Normal to Effective Throat or Shear on Effective Throat. The allowable stress limits shall be as specified in Table NF-3324.5(a)-1.
- (-b) Tension Normal to the Axis on the Effective Throat. The stress limits shall be as specified in Table NF-3324.5(a)-1.
- (3) Fillet Welds. The allowable stress limit for fillet welds shall be as specified in Table NF-3324.5(a)-1.
- (b) Service Limits, Level A Through D, and Test. The rules and stress limits which must be satisfied for welds for any Level A through D Service and Test Loading stated in the Design Specification are those given in (a) multiplied by the appropriate base material stress limit factor given in Table NF-3251.2-1.
- **NF-3256.3 Effective Size.** The effective sizes of welds shall be as given in NF-3324.5(d) through NF-3324.5(f).
- Welded joint configurations causing significant through-thickness tensile stress [as defined in NF-1215(b)] during fabrication and/or service on rolled product forms should be avoided. However, if this type of construction is used, the designer should consider one or several of the following factors that may reduce the susceptibility of the joint to experience lamellar tearing and provide documentation, including fabrication requirements, in the Design Output Documents:
  - (a) Reduce volume of weld metal to the extent practical.

- (b) Select materials that are resistant to lamellar tearing.
- (c) Invoke any of the special fabrication requirements of NF-4441.

# NF-3260 DESIGN BY ANALYSIS FOR CLASS 3 NF-3261 Stress Limits

The design of Class 3 supports shall be in accordance with the requirements of NF-3250 using one of the design procedures indicated in Table NF-3131(a)-1 for Class 3 construction.

# NF-3265 Design of Bolting

The provisions of NF-3225 apply.

# NF-3266 Design of Welded Joints

The types of welded joints shall be as stipulated in NF-3256 for Class 2 and MC supports, except that for groove welded T-joints, groove welded corner joints, and fillet welded T-joints, as listed in NF-3256.1(a)(2) and NF-3256.1(a)(3), the welds may be intermittent instead of continuous. Intermittent fillet welds shall meet the requirements of NF-3324.5(d)(7). The allowable stress limits shall be as stipulated in NF-3256.2.

# NF-3270 EXPERIMENTAL STRESS ANALYSIS

Supports may be designed by experimental stress analysis in accordance with Section III Appendices, Mandatory Appendix II.

# NF-3280 DESIGN BY LOAD RATING NF-3281 Procedure for Load Rating

The procedure for load rating shall consist of imposing a total load on one or more duplicate full-size samples of a support equal to or less than the load under which the support fails to perform its required function. Full-size samples composed of various parts may have each part or a number of parts load rated, provided that all parts in the load path are either load rated or otherwise qualified per NF-3200, or by experimental stress analysis. When parts are connected by bolting or welding, the connection shall be either load rated or qualified per NF-3225 or NF-3226. Should more than one part be load rated in a single load test, then the load rating equations of NF-3280 shall be evaluated for each part using the part's  $S_{v(act)}$  and  $S_{u(act)}$  values. The part having the lowest load rating shall establish the load rating for the combination of parts. A single test sample is permitted but, in that case, the load ratings shall be reduced by 10%. Otherwise, tests shall be run on a statistically significant number of samples. The permissible types of welded joints shall be as permitted for the specific class of construction in NF-3226.1, NF-3256.1, and NF-3266. The full-size sample shall be fabricated for testing using welds not exceeding weld sizes stipulated in the Design Drawings. Bolted joints in the test sample shall be made up using the lowest strength bolt material and minimum edge distance allowed by the specification.

#### Load Ratings in Relation to Design NF-3282 Service and Test Loadings

The load rating for Design Loadings shall be determined in accordance with the requirements for Service Level A limits. The load ratings for Service Loadings for which Level A through Level D Limits have been designated shall be determined by means of the equations in the following subparagraphs. The load rating for Test Loadings shall be determined in accordance with the requirements for Service Level B limits.

NF-3282.1 Nomenclature. The symbols used in this paragraph are defined as follows:

Kl = load rating coefficient for support in compression

S = allowable stress value at the Design Temperature (NF-3112.1) from the applicable tables of Section II, Part D, Subpart 1, ksi (MPa)

 $S_T$  = specified minimum tensile strength of the material used in the support as given in the applicable tables of Section II, Part D, Subpart 1, ksi (MPa)

 $S_u = \text{see NF-3313.1}$ 

 $S_{u(act)}$  = actual tensile strength of the material used in the part or support which had reached ultimate capacity during the test, ksi (MPa) .

 $S_V$  = specified minimum yield strength of the material used in the support as given in the applicable tables of Section II, Part D, Subpart 1, ksi (MPa)

 $S_{v} = \text{see NF-3313.1}$ 

 $S_{y(act)}$  = actual yield strength of the material used in the part or support which yielded during the test, ksi (MPa)

 $TL_u$  = support test load at which a substantial increase in load displacement results in zero or negative increase in actual support load

 $TL_v$  = support test load at or below yield

NF-3282.2 Plate- and Shell-Type Supports. The load ratings for Plate- and Shell-Type Supports for the Service Loadings shall be determined by the following equations: Design and Level A Limits (lower of the two values)

load rating<sub>(yield)</sub> = 
$$TL_y \times 1.0 \frac{S}{S_{y(act)}}$$
 (1)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 1.0 \frac{S}{S_{u(act)}}$$
 (2)

Test and Level B Limits (lower of the two values)

load rating<sub>(yield)</sub> = 
$$TL_y \times 1.33 \frac{S}{S_{y(act)}}$$
 (3)

load rating<sub>(ult.)</sub> =  $TL_u \times 1.33 \frac{S}{S_{u(act)}}$  (4)

Limits (lower of the two values)

$$load \ rating_{\{ult.\}} = TL_u \times 1.33 \frac{S}{S_{u\{act\}}}$$

Level C Limits (lower of the two values)

load rating(yield) = 
$$TL_y \times 1.5 \frac{S}{S_{\text{con}}}$$
 (5)

load rating<sub>(ult.)</sub> = 
$$TL_u > 1.5 \frac{S}{S_u(act)}$$
 (6)

Level D Limits (lowest of the three values)

load rating(yield) = 
$$TL_y \times 2.0 \frac{S}{S_y(act)}$$
 (7)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 0.7 \frac{S_u}{S_{u(act)}}$$
 (8)

load rating(ult.) = 
$$TL_u \times 0.7$$
 (9)

NF-3282.3 Supports Loaded in Compression. The load rating for supports loaded in compression shall be determined for Service Levels A through D by the methods of NF-3282.2. In addition, for cases where buckling governs,  $TL_u$  shall be corrected for the ratio of Young's Modulus at the Design Temperature to Young's Modulus at the test temperature. The load rating shall be determined by the following equations:

Design and Level A Limits

load rating = 
$$0.50 \ Kl \times TL_u$$
 (10)

Test and Level B Limits

load rating = 
$$0.50 \, Kl \times TL_{\mu}$$
 (11)

Level C Limits

load rating = 
$$0.50 \ Kl \times TL_u$$
 (12)

Kl = 1.0 for elastic buckling

$$KI = \frac{S_y}{S_y(act)}$$
 for inelastic buckling

Level D Limits

load reading = 
$$0.50 \ Kl \times TL_u$$
 (13)

NF-3282.4 Alternative Load Rating Method Using  $TL_y$  or  $TL_u$  Test Results Only. Either  $TL_y$  or  $TL_u$  test results may be used to establish a support load rating, provided the load rating as determined by the method of NF-3282.2 or NF-3282.3 is modified as follows:

- (a) For load rating based on  $TL_y$  test results only
- (1) for supports constructed of materials having definitely determinable yield points and specified minimum  $S_Y/S_T$  ratios not greater than 0.625, the load rating shall be multiplied by 0.83. If  $S_{y(act)}$  of the material is not known, 0.67 shall be used in place of multiplier 0.83
- (2) for supports constructed of carbon steel with specified minimum tensile strengths of not over 70,000 psi (485 MPa),  $S_{y(act)}$  shall be increased by 5,000 psi (34 MPa) prior to determining the load rating
- (3) for all other materials the determined load rating shall be multiplied by 0.67

For supports loaded in compression, a  $TL_u$  test shall be performed with the compressive load rating determined in accordance with NF-3282.3.

(b) For load rating based on  $TL_u$  test results, the determined load rating shall be multiplied by 0.50.

# NF-3300 DESIGN RULES FOR LINEAR-TYPE SUPPORTS

# **NF-3310 GENERAL REQUIREMENTS**

This subarticle provides rules for the design of Linear-Type Supports by either linear elastic analysis (NF-3320) or plastic (limit) analysis (NF-3340) Linear elastic analytical procedures are also provided (NF-3330) for the design of members and connections which will be subjected to high cycle fatigue conditions in service.

# NF-3311 Design Considerations

NF-3311.1 Linear Elastic Analysis. The rules for linear elastic analysis are based on the yield strength values at temperature of the materials used in constructing linear-type supports that are set forth in Section II, Part D, Subpart 1, Table Y-1. The allowable stresses are determined in NF-3320 by applying factors of safety dependent on the structural member involved to these yield strength values.

**NF-3311.2 High Cycle Fatigue Analysis.** The rules for designing Linear-Type Supports given in NF-3330 are essentially the same as those given in NF-3320 for linear elastic analysis, except that the maximum range of stress, namely, the difference between the minimum and maximum value of the stress throughout each cycle and the frequency with which the support will be

subjected to this range of stress, shall be taken into consideration when so stipulated by the Design Specification (NCA-3250).

**NF-3311.3 Limit Analysis.** The rules for limit (plastic) analysis given in NF-3340 may be used as an alternative to the linear elastic analysis method. These rules permit proportioning Linear-Type Supports on the basis of limit design by determining their lower bound collapse loads. The requirements set forth in NF-3320 governing allowable stresses to be used in designing Linear-Type Supports are waived when the limit design procedure is employed, but all other pertinent provisions of NF-3320 shall apply.

**NF-3311.4 Basis for Determining Stress in Design by Analysis.** The theory of failure used in the rules for the design of Linear-Type Supports is the maximum stress theory. In the maximum stress theory, the controlling stress is the maximum principal stress.

**NF-3311.5 Terms Relating to Design by Analysis.** Terms used in the design of Linear-Type Supports by stress analysis are defined in NF-3313.

## NF-3312 Analysis Methods

Linear-Type Supports may be designed by either elastic or limit analysis, limits for which are given in NF-3312.1.

**NF-3312.1 Elastic Analysis.** In elastic analysis it is assumed that all component and support stiffnesses remain constant.

- (a) Design Limits. The rules and stress limits that must be satisfied in an elastic analysis for any Design Loading shall be stated in the Design Specification.
- (b) Service Limits, Level A Through D. The rules and stress limits which must be satisfied in an elastic analysis for any Level A through D Service Loading stated in the Design Specification are those given in NF-3321 multiplied by the appropriate stress limit factor from Table NF-3312.1(b)-1.
- (c) Test Limits. The rules and stress limits that must be satisfied for any Test Loadings stated in the Design Specification are those given in NF-3321 multiplied by the appropriate stress limit factor in Table NF-3312.1(b)-1 under Test Loadings.

# NF-3313 Nomenclature and Numbering of Equations

**NF-3313.1 Nomenclature.** Except where symbols are used in the text of the paragraphs that follow to represent the value of complex algebraic expressions, the nomenclature adopted in NF-3300 is defined as follows:

- A = gross area of an axially loaded compression member, in.<sup>2</sup> (mm<sup>2</sup>)
- a = clear distance between transverse stiffeners; dimension parallel to the direction of stress, Table NF-3332.3-1, in. (mm)

**(17)** 

# Table NF-3312.1(b)-1 Elastic Analysis Stress Categories and Stress Limit Factors for Class 1, 2, 3, and MC Linear-Type **Supports Designed by Analysis**

		Charac	- 1 ii+ F+ f		[N-4- (4)]	
Stress Category	Design	Service Level A	Service Level B [Note (2)], [Note (3)]	Service Level C [Note (3)]	Service Level D [Note (3)]	Test Loadings
Primary stresses [Note (4)], [Note (5)]	$K_s = 1.0$	$K_s = 1.0$	$K_s = 1.33$	$K_s = 1.5$	$K_s$ = lesser of 2.0 or 1.167 $S_u/S_y$ [Note (7)]	$K_s = 1.33$
	$K_v = 1.0$	$K_{\nu} = 1.0$	$K_{\nu} = 1.33$ [Note (6)]	$K_{\nu} = 1.5 \text{ [Note (6)]}$	$K_{\nu}$ = 2.0 [Note (6)], [Note (8)]	$K_{\nu} = 1.33$ [Note
	$K_{bk}$ = 1.0	$K_{bk}=1.0$	$K_{bk} = 1.33$	$K_{bk} = 1.5$	$K_{bk} = 1.5$ [Note (9)]	$K_{bk} = 1.33$
			but stress $\leq \frac{2}{3}$ of critical buckling stress	but stress ≤ <sup>2</sup> / <sub>3</sub> of critical buckling stress	but stress ≤ <sup>2</sup> / <sub>3</sub> of critical buckling stress	, ,
Primary plus secondary stresses [Note (5)], [Note (10)]	Evaluation i for this e	s required for critical valuation.	buckling for all loadi	ng categories. The re	urements of this su	ıbarticle shall be met
Peak stresses	Evaluation	not required.		, N.		

#### GENERAL NOTE:

 $K_{bk}$  = stress limit factor applicable to the Design allowable compressive axid and bending stresses to determine buckling limit

 $K_s$  = stress limit factor applicable to the Design allowable tensile and bending stresses (see NF-3321.1, NF-3322.1, NF-3350, and NF-3360)

 $K_v$  = stress limit factor applicable to the Design allowable shear stresses (see NF-3321.1, NF-3322.1, NF-3350, and NF-3360)

#### NOTES:

- (1) Control of deformation is not assured by these stress limit actors. When required by Design Specification, deformation control must be considered separately.
- (2)  $K_s$ ,  $K_v$ , and  $K_{bk} = 1.0$  for design of snubbers and dampers.
- (3) Stress shall not exceed  $0.7S_u$  and  $1.2S_v$ .
- (4) For Service Levels A, B, C, and D, stresses induced on the supports by restraint of free-end displacement and anchor motions of piping shall be considered as primary stresses.
- Thermal stresses within the support as defined by NF-3121.11 need not be evaluated.
- Shear stress shall not exceed  $0.42S_u$  and  $0.72S_v$ . (6)
- If  $S_u \le 1.2S_v$ ,  $K_s = 1.4$ .
- If  $S_u \le 1.2S_y$ ,  $K_s = 1.4$ . For fillet welds,  $K_v = 1.7$  and shear stress shall not exceed  $0.5S_u$ . (8)
- Except as required by NF-3322.1(c)(4). (9)
- (10) Service Levels A and B, primary plus secondary stresses, shall be limited to a range of  $2S_v$  or  $S_u$  at temperature, whichever is less for component supports only.

- $A_b$  = nominal body area of a fastener, in.<sup>2</sup> (mm<sup>2</sup>); area of an upset rod based upon the major diameter of its threads, i.e., the diameter of a coaxial cylinder that would bound the crests of the upset threads, in.<sup>2</sup> (mm<sup>2</sup>)
- $A_e$  = effective net area of an axially loaded tension member, in.<sup>2</sup> (mm<sup>2</sup>)
- $A_f$  = area of compression flange, in.<sup>2</sup> (mm<sup>2</sup>)
- $A_n$  = net area of an axially loaded tension member, in.<sup>2</sup> (mm<sup>2</sup>)
- $A_{st}$  = cross-sectional area of stiffener or pair of stiffeners, in.<sup>2</sup> (mm<sup>2</sup>)
- $A_w$  = area of girder web, in.<sup>2</sup> (mm<sup>2</sup>)
- a' = distance required at ends of welded partial length cover plate to develop stress, in. (mm)
- b = actual width of stiffened and unstiffened compression elements, in. (mm)
  - = dimensions normal to the direction of stress, Table NF-3332.3-1, in. (mm)
- $b_e$  = effective width of stiffened compression element, in. (mm)
- $b_f$  = flange width of rolled beam or plate girder, in. (mm)
- C = ratio of bolt tensile strength to tensile strength of connected part
- C<sub>1</sub> = increment used in computing minimum spacing of oversized and slotted holes
- C<sub>2</sub> = increment used in computing minimum edge distance for oversized and slotted holes
- C<sub>b</sub> = bending coefficient dependent upon moment gradient
- C<sub>c</sub> = slenderness ratio separating elastic and inelastic buckling
- $C_m$  = coefficient applied to bending term in interaction equation and dependent upon column curvature caused by applied moments
- $C_t$  = reduction coefficient in computing effective net area of an axially loaded tension member
- $C_{\nu}$  = ratio of critical web stress, according to the linear buckling theory, to the shear yield stress of web material
- $C'_{c}$  = effective column slenderness ratio
  - D = factor dependent upon type of transverse stiffeners
    - = outside diameter of tubular members, in. (mm)
  - d = depth of beam or girder, in. (mm)
    - diameter of roller or rocker bearing, in. (mm)
    - nominal diameter of fastener, in. (mm)
- $d_c$  = column web depth clear of fillets, in. (mm)
  - E = modulus of elasticity of steel at temperature, ksi (MPa)
  - e = horizontal displacement, in the direction of the span, between top and bottom of simply supported beam at its ends, in. (mm)
  - f = axial compression stress on member based on effective area, ksi (MPa)

- $F_a$  = axial compressive stress permitted in the absence of bending moment, ksi (MPa)
- $f_a$  = computed axial stress, ksi (MPa)
- $F_{as}$  = axial compressive stress permitted in the absence of bending moment, for bracing and other secondary members, ksi (MPa)
- $F_b$  = bending stress permitted in the absence of axial force, ksi (MPa)
- $f_b$  = computed bending stress, ksi (MPa)
- $F_p$  = allowable bearing stress on the projected area of bolts or pins, or on contact area of milled surfaces and ends of rollers, rockers, or fitted bearing stiffeners at temperature, ksi (MPa)
- $f_p$  = computed bearing stress, ksi (MPa)
- $F_{sr}$  = stress range, ksi (MPa)
- $F_t$  = allowable tensile stress, ksi (MPa)
- $f_t$  = computed tensile stress, ksi (MPa)
- $F_{tb}$  = allowable tensile stress at temperature, ksi (MPa)
- $F_v$  = allowable shear stress, ksi (MPa)
- $f_v$  = computed shear stress, ksi (MPa)
- $F_{vb}$  = allowable shear stress at temperature, ksi (MPa)
- $f_{vs}$  = shear between girder web and transverse stiffeners, kips/linear in. (N/mm) of single stiffener or pair of stiffeners
- $F_{yc}$  = column yield stress, ksi (MPa)
- $F_{y}$  stiffener yield stress, ksi (MPa)
- = allowable bending stress in compression flange of plate girders as reduced for hybrid girders or because of large web depth-thickness ratio, ksi (MPa)
- $f'_c$  = specified compression strength of concrete, ksi (MPa)
- $F'_{e}$  = Euler stress divided by factor of safety, ksi (MPa)
  - g = transverse center to center spacing (gage) between successive holes in line of stress, in. (mm)
  - h = clear distance between flanges of a beam or girder at the section under investigation, in. (mm)
  - K =effective length factor
  - k = coefficient relating linear buckling strength of a plate to its dimensions and condition of edge support
    - = also distance between outer face of column flange and web toe of its fillet if column is a rolled shape, or equivalent distance if column is a welded shape, in. (mm)
- $k_s$  = slip coefficient for the particular surface condition taken from Table NF-3324.6(a)(4)-1
- L = span length
  - = distance from center of bolt hole to edge of connected part, in. (mm)
- l = for beams, distance between cross section braced against twist or lateral displacement of the compression flange, in. (mm)
  - for columns, actual unbraced length of member or unsupported length of lacing bar, in. (mm)
- $l_b$  = actual unbraced length in plane of bending, in. (mm)

- $l_{cr}=$  critical unbraced length adjacent to plastic hinge, in. (mm)
- M = moment, in-kip (N·mm) (NF-3320); factoredbending moment, in-kip (N·mm) (NF-3340)
- m = number of shear planes per bolt
- $M_1$  = smaller moment at end of unbraced length of beam column, in-kip (N·mm)
- $M_2$  = larger moment at end of unbraced length of beam column, in-kip (N·mm)
- $M_m$  = critical moment that can be resisted by a plastically designed member in the absence of axial load, in-kip (N·mm)
- $M_p$  = plastic moment, in-kip (N·mm)
- M' = lesser of the factored moments at the ends of unbraced segment, in-kip (N·mm)
- N =length of bearing of applied load, in. (mm)
- n = number of bolts in the joint
- P = force transmitted by a fastener, kips (N) (NF-3320); factored axial load, kips (N) (NF-3340); axial load, kips (N) (NF-3340)
- $P_{bf}$  = factored beam flange or connection plate force in a restrained connection
  - = the computed force delivered by the flange or moment connection plate, multiplied by five-thirds for Level A loads and multiplied by four-thirds for Level B, C, and D loads, kips (N)
- $P_{cr}$  = maximum strength of axially loaded compression member or beam, kips (N)
- $P_e$  = Euler buckling load, kips (N)
- $P_s$  = maximum slip resistance of the joint, lb (N)
- $P_y$  = axial plastic load, equal to member area times specified minimum yield stress, kips (N)
- Q<sub>a</sub> = ratio of effective area of an axially loaded member to its total area
- $Q_s$  = axial stress reduction factor
- R = reaction or concentrated transverse load applied to beam or girder, kips (N); radius, in. (mm)
- r = governing radius of gyration in. (mm)
- $r_b$  = radius of gyration about axis of concurrent bending, in. (mm)
- $r_c$  = radius of gyration of a section comprising the compression flange plus one-third of the compression web area, taken about an axis in the plane of the web, in. (mm)
- $r_y$  = radius of gyration of a member about its weak axis, in. (mm)
- s = longitudinal center-to-center spacing (pitch) of any two consecutive holes, in. (mm)
- $S_u$  = tabulated values of ultimate tensile strength of material at temperature (Section II, Part D, Subpart 1, Table U), ksi (MPa)
- $S_y$  = yield strength at temperature (Section II, Part D, Subpart 1, Table Y-1), ksi (MPa)
- t = girder, beam, or column web thickness, in. (mm); thickness of a connected part, in. (mm)
  - = wall thickness of a tubular member, in. (mm)

- $T_b$  = specific pretension of a high-strength bolt, kips (N)
- $t_b$  = thickness of beam flange or moment connection plate at rigid beam to column connection, in. (mm)
- $t_f$  = flange thickness, in. (mm)
- $T_i$  = initial clamping force per bolt, lb (N)
- $t_1$  = thickness of thinner part joined by partial penetration groove weld, in. (mm)
- V<sub>u</sub> = shear produced by required factored loading, with mate load, kips (N)
- x = subscript relating to bending about the strong principal axis
- Y = ratio of yield stress of web steel to yield stress of stiffener steel
- y = subscript relating to bending about the weak principal axis
- $Z_x$  = plastic section modulus for axis of bending, in.<sup>3</sup> (mm<sup>3</sup>)
- Δ = displacement of the neutral axis of a loaded member from its position when the member is not loaded, in. (mm)
- $\alpha$  = ratio of hybrid girder web yield stress to flange yield stress

**NF-3313.2 Numbering of Equations.** A separate series of equation numbers is used for each paragraph and each series, beginning with NF-3322.1(a)(1), eq. (1).

# NF-3320 DESIGN BY LINEAR ELASTIC ANALYSIS FOR CLASS 1

## NF-3321 General Design Requirements

## NF-3321.1 Design and Service Conditions.

- (a) Except as otherwise provided in this subarticle, all items of the structure shall be so proportioned that the stress, ksi (MPa), for Design and Level A Service Limits, shall not exceed the values of NF-3322. For Level B, C, and D Service Limits, allowable stresses may be increased by the factors shown in Table NF-3312.1(b)-1. Except for pinned and bolted joints, bearing stresses need not be evaluated for loads for which Level D Service Limits are specified.
- (b) To avoid column buckling in compression members, local instability associated with compression flange buckling in flexural members, and web buckling in plate girders, the allowable stress shall be limited to two-thirds of the critical buckling stress.

**NF-3321.2 Design for High Cycle Fatigue Conditions.** Design for high cycle fatigue conditions shall be in accordance with the requirements of NF-3330, using the allowable stresses of NF-3322 as modified by NF-3332.4.

#### **Design Requirements for Structural** NF-3322 **Steel Members**

- **NF-3322.1 Allowable Stresses.** Except where noted, **(17)** the following are the allowable stresses for Design and Level A Service Conditions. See NF-3312.1 for other service condition allowable stresses.
  - (a) Stress in Tension. The allowable stress in tension shall be as given in (1) through (3)
  - (1) Except for pin-connected members and threaded members used in bolting applications,  $F_t$  shall be

$$F_t = 0.60S_y \tag{1}$$

but not more than  $0.5S_u$  on the effective net area (for determination of effective net area, see NF-3322.8).

(2) For pin-connected members, using the net area (for determination of effective net area, see NF-3322.8)

$$F_t = 0.45S_y \tag{2}$$

but not more than  $0.375S_{ii}$ .

- (3) For threaded parts in tension used in bolting applications, see NF-3324.6(a)(1).
  - (b) Stress in Shear
- (1) Except as provided in (2) and NF-3324.6(a)(2), on the cross-sectional area effective in resisting shear

$$F_{V} = 0.40S_{V} \tag{3a}$$

but not more than  $0.33S_u$ .

The effective area in resisting shear of rolled and fabricated shapes may be taken as the overall depth times the web thickness.

(2) At the beam end connections where the top flange is coped, and in similar situations where failure might occur by shear along a plane through the fasteners, or by a combination of shear along a plane through the fasteners plus tension along aperpendicular plane, on the effective area in resisting tearing failure shall be

$$E = 0.30S_u \tag{3b}$$

The effective area is the minimum net failure surface, bounded by the bolt holes.

- (c) Stress in Compression. The allowable stress in compression shall be as required by (1) through (3). The allowable compressive stress for columns, except those fabricated from austenitic stainless steel, shall be as required by (1). The allowable compressive stress for columns fabricated from austenitic stainless steel shall be as required by (2). The allowable compressive stress for member elements other than columns constructed of any material, including austenitic stainless steel, shall be as required by (3).
- (1) Gross Sections of Columns, Except Those Fabricated of Austenitic Stainless Steel

(-a) Where Kl/r Is Less Than  $C_c$ . On the gross sec-Fa =  $\frac{\left[1 - (Kl/r)^2/2C_c^2\right]s_y}{5/3 + \left[3(Kl/r)/8C_c\right] - \left[(Kl/r)^3/8C_c^3\right]}$ Where Kl/r is Graphing is less than  $C_c$ , the almost and  $C_c$  is  $C_c = \sqrt{2\pi^2 E/s_y}$ . tion of axially loaded compression members whose cross sections meet the provisions of NF-3322.2(d), when Kl/r(the largest effective slenderness ratio of any unbraced segment as defined in NF-3322.2) is less than  $C_c$ , the allowable stress in compression shall be

$$F_{a} = \frac{\left[1 - \left(\frac{Kl}{r}\right)^{2}/2C_{c}^{2}\right]S_{y}}{5/3 + \left[3\left(\frac{Kl}{r}\right)/8C_{c}\right] - \left[\left(\frac{Kl}{r}\right)^{3}/8C_{c}^{3}\right]}$$
(4)

where

$$C_c = \sqrt{2\pi^2 E / Sy}$$

section of axially loaded compression members when Kl/r exceeds  $C_c$ , the allowable stress in compression

$$F_a = \frac{12\pi^2 E}{23 \left( \frac{Kl}{r} \right)^2}$$
 (5)

(c) Where the Slenderness Ratio Exceeds 120. On the gross section of axially loaded bracing and secondary members, when l/r (for this case, K is taken as unity) exceeds 120 the allowable stress in compression shall be

$$F_{as} = \frac{F_a [\text{eq. (4) or (5)}]}{1.6 - (\frac{l}{200r})}$$
 (6)

- (2) Gross Sections of Columns Fabricated From Austenitic Stainless Steel. On the gross section of axially loaded compression members whose cross sections meet the provisions of NF-3322.2(d) the allowable stress in compression shall be as required by (-a) and (-b).
  - (-a) Where Kl/r Is Less Than or Equal to 120.

$$F_a = S_y \left( 0.47 - \frac{Kl/r}{444} \right)$$
 (6a)

(-b) Where Kl/r Is Greater Than 120. The following compressive stress limit applies to bracing and secondary members, as well as main members:

$$F_a = S_y \left( 0.40 - \frac{Kl/r}{600} \right)$$
 (6b)

(3) Member Elements Other Than Columns

(-a) Plate Girder Stiffeners. On the gross area of plate girder stiffeners, the allowable stress in compression shall be

$$F_a = 0.60S_v \tag{7}$$

but not more than  $0.50S_{\mu}$ .

(-b) Webs of Rolled Shapes. On the web of rolled shapes at the toe of the fillet, the allowable stress in compression shall be

$$F_a = 0.75S_v \tag{8}$$

but not more than  $0.63S_u$ .

For the use of stiffeners to avoid web crippling, see NF-3322.6(a).

(4) Level D. The maximum Level D allowable load for ferritic steels shall be determined in accordance with the following, provided that the initial out-of-straightness does not exceed  $\frac{1}{1000}$  of the unsupported length. Effects of deformations due to existing loads shall also be considered.

(-a) Except as noted in (-b), the following rules shall be applied:

For  $0 \le \lambda \le 1$ 

$$\frac{P}{P_y} = \frac{1 - \lambda^2/4}{1.11 + 0.50\lambda + 0.17\lambda^2 - 0.28\lambda^3}$$

For  $1 \le \lambda \le \sqrt{2}$ 

$$\frac{P}{P_y} = \frac{2}{3} \left( 1 - \lambda^2 / 4 \right)$$

For  $\lambda > \sqrt{2}$ 

$$\frac{P}{P_y} = \frac{2}{3\lambda^2}$$
tion
city
actor
ble load

where

 $A_q$  = area of gross section

E = modulus of elasticity

K = effective length factor

L = unbraced length

P = maximum allowable load

 $P_y = S_y A_g$  r = radius of gyration

$$\lambda = \left(\frac{KL}{r}\right) \frac{1}{\pi} \sqrt{\frac{S_y}{E}}$$

(-b) For nonstress-relieved heavy structural shapes [web or flange thickness greater than 1 in. (25 mm)] or for nonstress-relieved built-up members using universal mill plate, the following rules shall be applied:

For 
$$0 < \lambda < 1$$

$$\frac{P}{P_V} = \frac{1 - \lambda^2/4}{1.11 + 0.75\lambda + 0.83\lambda^2 - 0.81\lambda^3}$$

For  $1 < \lambda < \sqrt{2}$ 

$$\frac{P}{P_y} = \frac{1 - \lambda^2/4}{1.88}$$

For 
$$\lambda > \sqrt{2}$$

$$\frac{P}{P_y} = \frac{1}{1.88\lambda^2}$$

(d) Stress in Bending. The allowable bending stress resulting from tension and compression in structural members shall be as required by (1) through (6).

# (1) Compact Sections

(-a) Tension and compression on extreme fibers of compact hot rolled or built-up members (except hybrid beams and members made from 65 ksi (450 MPa) or greater  $S_v$  material) symmetrical about and loaded in the plane of their minor axes and meeting the requirements of Subsection NF shall result in a maximum bending stress

$$F_b = 0.66S_y \tag{9}$$

but not more than 0.55S

In order to qualify, a member shall meet the requirements of (-1) through (-7).

(-1) The flanges shall be continuously connected to the web or webs.

(-2) The width-thickness ratio of unstiffened projecting elements of the compression flange [as defined in W-3322.2(d)] shall not exceed  $65 / \sqrt{S_y}$  (for SI units, use  $170 / \sqrt{S_v}$ ).

(-3) The width-thickness ratio of stiffened elements of the compression flange [as defined in NF-3322.2(d)] shall not exceed  $190 / \sqrt{S_y}$  (for SI units, use 500 /  $\sqrt{S_v}$ ).

(-4) The depth-thickness ratio of the web or webs shall not exceed the value given by eq. (10) or (11), as applicable

(U.S. Customary Units)

$$d/t = \left[640/\sqrt{S_y}\right] \left[1 - 3.74 \left(f_a/S_y\right)\right]$$
 (10)

(SI Units)

$$d/t = \left(1.680 / \sqrt{S_y}\right) \left[1 - 3.74 \left(f_a / S_y\right)\right]$$
 (10)

when  $f_a/S_y \leq 0.16$ ;

(U.S. Customary Units)

$$d/t = 257/\sqrt{S_v} \tag{11}$$

(SI Units)

$$d/t = 673 / \sqrt{S_v} \tag{11}$$

when  $f_a/S_v > 0.16$ .

(-5) The laterally unsupported length of the compression flange of members other than box-shaped members shall not exceed the value  $76b_f/\sqrt{S_V}$  (for SI units, use  $200b_f / \sqrt{S_y}$ ) nor  $20,000/[(d/A_f)S_y]$  {for SI units, use 138 000/ $[(d/A_f) S_v]$ }.

(-6) The laterally unsupported length of the compression flange of a box-shaped member of rectangular cross section, whose depth is not more than six times the width and whose flange thickness is not more than two times the web thickness, shall not exceed the value

(U.S. Customary Units)

$$[1,950 + 1,200 (M_1/M_2)](b/S_y)$$
 (12)

(SI Units)

$$[13 400 + 8 300 (M_1/M_2)](b/S_y)$$
 (12)

except that it need not be less than 1,200  $(b/S_v)$  [for SI units, use 8 300  $(b/S_v)$ ].

(-7) The diameter-thickness ratio of hollow circular sections shall not exceed  $3,300/S_y$  (for SI units, use 23  $000/S_v$ ).

(-b) Beams and girders (except for hybrid girders and members made from 65 ksi (450 MPa) or greater  $S_v$ material) which meet the requirements of (-a)(-1)/through (-a)(-7) and are continuous over supports or are rigidly framed to columns by means of high-strength bolts or welds may be proportioned for nine-tenths of the negative moments produced by gravity loading which are maximum at points of support, provided that for such members the maximum positive moment shall be increased by one-tenth of the average negative moments. This reduction shall not apply to moments produced by loading on cantilevers. If the negative moment is resisted by a column rigidly framed to the beam or girder, the onetenth reduction may be used in proportioning the column for the combined axial and bending loading, provided that the stress  $f_a$  due to any concurrent axial load on the member does not exceed  $0.15F_a$ .

(2) Members With High Flange Width-Thickness Ratio. Members that meet the requirements of (1), except that  $b_f/2t_f$  exceeds  $65/\sqrt{S_y}$  (For SI units, use  $\sqrt{S_y}$ ) but is less than  $95/\sqrt{S_y}$  (For SI units, use  $250 / \sqrt{S_y}$ ), may be designed on the basis of an allowable bending stress

(U.S. Customary Units)

$$F_b = S_y \left[ 0.79 - 0.002 \left( b_f / 2t_f \right) \sqrt{S_y} \right]$$
 (13)

(SI Units)

$$F_b = S_y \left[ 0.79 - 0.0008 \left( \frac{b_f}{2t_f} \right) \sqrt{S_y} \right]$$
 (13)

(3) Doubly Symmetrical Members With Bending About the Minor Axis. Tension and compression on extreme fibers of doubly symmetrical I- and H-shaped members [except for hybrid girders and members made from 65 ksi (450 MPa) or greater  $S_{\nu}$  material] which are bent about their minor axis, meeting the requirements of (1)(-a)and (1)(-b); solid round and square bars; and solid rectangular sections bent about their weaker axis shall not exceed a bending stress of

$$F_b = 0.75S_b$$
 (14)

but not more than  $0.63S_u$ .

Doubly symmetrical L and H-shaped members bent about their minor axis lexcept for hybrid girders and members made from 65 ksi (450 MPa) or greater  $S_y$  material], meeting the requirements of (1)(-a), except where  $b_f/2t_f$  exceeds 65 /  $\sqrt{S_V}$  (for SI units, use 170 /  $\sqrt{S_V}$ ) but is less than  $95/\sqrt{S_y}$  (for SI units, use  $250/\sqrt{S_y}$ ), may be designed on the basis of an allowable bending stress

(U.S. Customary Units)

$$F_b = S_y \left[ 1.075 - 0.005 \left( b_f / 2t_f \right) \sqrt{S_y} \right]$$
 (15a)

(SI Units)

$$F_b = S_y \left[ 1.075 - 0.002 \left( \frac{b_f}{2t_f} \right) \sqrt{S_y} \right]$$
 (15a)

Rectangular tubular sections meeting the requirements of (1)(-a)(-1), (1)(-a)(-3), and (1)(-a)(-4), and bent about their minor axis, may be designed on the basis of an allowable bending stress

$$F_b = 0.66S_v \tag{15b}$$

but not more than  $0.55S_u$ .

(4) Box-Type Flexural Members. Tension and compression on extreme fibers of box-type flexural members whose compression flange or web width-thickness ratio does not meet the requirements of (1), but does conform to the requirements of NF-3322.2(d), shall not exceed a bending stress of

$$F_b = 0.60S_v \tag{16}$$

but not more than  $0.50S_{ij}$ .

Lateral torsional buckling need not be investigated for a box section whose depth is less than six times its width. Lateral support requirements for box sections of larger depth-width ratios must be determined by special analysis.

## (5) Miscellaneous Members

(-a) Tension on extreme fibers of flexural members not covered in (1) through (4), shall not exceed a bending stress as determined by using eq. (4)(16).

(-b) Compression on extreme fibers of flexural members included in (-a), meeting the requirements of NF-3322.2(d)(1)(-b), and having an axis of symmetry in and loaded in the plane of their web and compression on extreme fibers of channels [only eq. (19) is applicable to channels] bent about their major axis, shall have the larger value of bending stress computed by eqs. (17) through (19), as applicable, but not more than  $0.60S_y$  (see NF-3322.3 for further limitations in plate girder flange stress).

When

(U.S. Customary Units)

$$\left[\left(102\times 10^{3}C_{b}\right)/S_{y}\right]^{1/2}\leq l/r_{c}\leq \left[\left(510\times 10^{3}C_{b}\right)/S_{y}\right]^{1/2}$$

(SI Units)

$$\left[ \left( 703.3 \times 10^{3} C_{b} \right) / S_{y} \right]^{1/2} \le l / r_{c} \le \left[ \left( 3516 \times 10^{3} C_{b} \right) / S_{y} \right]^{1/2}$$

then

(U.S. Customary Units)

$$F_b = \left\{ \frac{2}{3} - \left[ S_y (1/r_c)^2 / \left( 1,530 \times 10^3 C_b \right) \right] \right\} S_y$$
 (17)

(SI Units)

$$F_b = \left\{ \frac{2}{3} - \left[ S_y (l/r_c)^2 / \left( 10550 \times 10^3 G_b \right) \right] \right\}_y$$
 (17)

When

(U.S. Customary Units)

$$1/r_c \geq \sqrt{\left(510 \times 10^3 C_b\right)/S_y}$$

(SI Units)

$$1/r_c \ge \sqrt{\left(3.516 \times 10^3 C_b\right)/S}$$

then

(U.S.)Customary Units)

$$F_b = \left(170 \times 10^3 C_b\right) / \left(l / r_c\right)^2 \tag{18}$$

(SI Units)

$$F_b = (1 \ 172 \times 10^3 C_b) / (l / r_c)^2$$
 (18)

When the compression flange is solid and approximately rectangular in cross section and its area is not less than that of the tension flange

(U.S. Customary Units)

$$F_b = \left(12 \times 10^3 C_b\right) / \left(ld / A_f\right) \tag{19}$$

(SI Units)

$$F_b = (82740C_b)/(ld/A_f)$$
 (19)

where

 $C_b = 1.75 + 1.05(M_1/M_2) + 0.3(M_1/M_2)^2 \le 2.3$ , where  $M_1$  is the smaller and  $M_2$  is the larger bending moment at the ends of the unbraced length taken about the strong axis of the member, and where  $M_1/M_2$  (the ratio of end moments) is positive when  $M_1$  and  $M_2$  have the same sign (reverse curvature bending) and is negative when they are of opposite signs (single curvature bending). When the bending moment at any point within an unbraced length is larger than that at both ends of this length, the value of  $C_b$  shall be taken as 1. When computing  $F_{bx}$  and  $F_{by}$  to be used in eq. (e)(1)(20),  $C_b$  may be computed by the equation given above for frames subject to joint translation.  $C_b$  may conservatively be taken as 1 for cantilever heams  $^5$ 

For cantilevers braced against twist only at the support,  $\boldsymbol{l}$  may be conservatively taken as the actual length.

- (-c) For hybrid plate girders,  $S_y$  for eqs. (-b)(17) and (-b)(18) is the yield stress of the compression flange. Equation (-b)(19) shall not apply to hybrid girders.
- (6) Miscellaneous Members Braced Laterally. Compression in extreme fibers of flexural members meeting the requirements of NF-3322.2(d)(1)(-b), but not included in (5)(-b), shall result in a maximum bending stress as determined by using eq. (4)(16), provided that sections bent about their major axis are braced laterally in the region of compression stress at intervals not exceeding  $76b_f/\sqrt{S_y}$  (for SI units, use  $200b_f/\sqrt{S_y}$ ).

## (e) Combined Stresses

(1) Axial Compression and Bending. For members subjected to both axial compression and bending, including bending moment resulting in secondary stress, stresses shall be proportioned to satisfy the requirements of eqs. (20) through (22)

$$\frac{f_a}{F_a} + \frac{C_{mx}f_{bx}}{\left(1 - f_a/F'_{ex}\right)F_{bx}} + \frac{C_{my}f_{by}}{\left(1 - f_a/F'_{ey}\right)F_{by}} \le 1.0$$
 (20)

$$\frac{f_a}{0.60S_v} + \frac{f_{bx}}{F_{bx}} + \frac{f_{by}}{F_{by}} \le 1.0$$
 (21)

After evaluating primary stresses, the right-hand side of eq. (22) may be replaced by 1.5 when both primary and secondary bending stresses are considered. When  $f_a/F_a \le 0.15$ , eq. (22) may be used in lieu of eqs. (20) and (21)

$$\frac{f_a}{F_a} + \frac{f_{bx}}{F_{bx}} + \frac{f_{by}}{F_{by}} \le 1.0$$
 (22)

In eqs. (20) through (22), the subscripts x and y indicate the axis of bending about which a particular stress or design property applies, and

$$F'_e = \frac{12\pi^2 E}{23(Kl_b/r_b)^2}$$

for all members except those fabricated of austenitic stainless steel, or

$$F'_e = \frac{\pi^2 E}{2.15 (K l_b / r_b)^2}$$

for members fabricated of austenitic stainless steel (in the expression for  $F'_e$ , as in the case of  $F_a$ ,  $F_b$ , and  $0.60S_v$ ,  $F'_e$ may be multiplied by the appropriate stress limit factor from Table NF-3312.1(b)-1 for the particular stress category), where  $C_m$  is a coefficient whose value shall be as given in (-a) through (-c)

- (-a) For compression members in frames subject to joint translation (sidesway),  $C_m = 0.85$ .
- (-b) For rotationally restrained compression members in frames braced against joint translation and not subject to transverse loading between their supports in the plane of bending

$$C_m = 0.6 - 0.4(M_1/M_2)$$

where  $M_1/M_2$  = the ratio of the smaller to larger moments at the ends of that portion of the member up braced in the plane of bands eration. when it is bent in single curvature.

- (-c) For compression members in frames braced against joint translation in the plane of loading and sub-(-1) for members whose ends are restrained otation in the plane of bending  $C_m = 0.97$ jected to transverse loading between their supports, the value of  $C_m$  may be determined by rational analysis; however, in lieu of such analysis, the following values may be used:
- against rotation in the plane of bending

$$C_m = 0.85$$

(-2) for members whose ends are unrestrained against rotation in the plane of bending

$$C_m = 1.0$$

(-d) For Level D, the allowable stresses in eqs. (20), (21), and (22) shall be defined as follows:

(-1)  $F_a = P/A_g$  where P shall be determined in accordance with (c)

(-2) The value of  $F_e$  shall be taken as

$$F_e' = \frac{\pi^2 E}{1.30 (K l_b / r_b)^2}$$

(-3)  $F_b$  shall be determined using (2)(-b) or (2)(💜 as appropriate.

(2) Axial Tension and Bending. Members subject to both axial tension and bending stresses shall be proportioned at all points along their length to satisfy the requirements of eq. (1)(21), where  $f_b$  is the computed bending tensile stress. However, the computed bending compressive stress, taken alone, shall not exceed the applicable value according to (d), "Stress in Bending."

For Level D

- (-a) the  $0.60S_y$  in eq. (1)(21) may be replaced with the smaller of  $1.2S_y$  or  $0.7S_u$
- (-b) for members qualifying as compact sections under criteria of (d)(1), the maximum bending stress shall be given by

$$F_b = f \times S_v$$

where

f = plastic shape factor for the cross section

- (-c) if members do not meet the compact section requirements, they shall be designed using one of the two methods below to determine  $F_b$  for use in the preceding equation.
- (-1) Allowable values for  $F_b$  given in (d)(2) may be increased by a factor of 1.11 (to maintain a nominal factor of safety of 1.5 against instability).
- (-2) Rigorous analysis of member stability may be used to determine critical bending stress. A factor of safety of 1.5 shall be used in determining allowable design bending stress.

(3) Shear and Tension. Bolts subject to combined shear and tension shall be proportioned in accordance with the requirements of NF-3324.

## (f) Bearing on Contact Area

(1) Milled Surfaces. On contact area of milled surfaces and ends of fitted bearing stiffeners on projected area of pins in reamed, drilled, or bored holes, the maximum bearing stress shall be

$$F_p = 0.90S_y \tag{23}$$

NOTE: When parts in contact have different yield stresses,  $S_y$  shall be the smaller value.

(2) Rollers and Rockers. The maximum bearing stress, ksi (MPa), shall be

(U.S. Customary Units)

$$F_p = \left(\frac{S_y - 13}{20}\right) 0.66d \tag{24a}$$

(SI Units)

$$F_p = \left(\frac{S_y - 90}{20}\right) 0.66d \tag{24a}$$

(3) Bolts. The maximum bearing load on projected area of bolts in bearing connections shall be

$$F_p = 1.5S_u \tag{24b}$$

(4) Concrete Bearing. In the absence of any other Code rules governing bearing stresses on concrete, the maximum bearing stress values shall be as given in (-a) and (-b)

(-a) on the full area of a concrete support

$$F_p = 0.35f'_c \tag{25}$$

(-b) on less than the full area of a concrete support

$$F_p = 0.35 f'_c \sqrt{A_2 / A_1} \le 0.7 f'_c$$
 (26)

where

 $A_1$  = bearing area, in.<sup>2</sup> (mm<sup>2</sup>)

A = full cross-sectional area of concrete support, in.<sup>2</sup> (mm<sup>2</sup>)

- (5) Pinned or Bolted Joints. For Level C and Level D Service Loadings, the bearing stress for pinned joints or bolted joints shall not exceed  $2.1S_u$ .
- (g) Single Angle Members. Single angle members shall be designed in accordance with Mandatory Appendix NF-II.

# NF-3322.2 Stability and Slenderness and Width- (17) Thickness Ratios.

#### (a) General Requirements

- (1) General stability shall be provided for the structure as a whole and for each compression element. Design consideration shall be given to significant load effects resulting from the deflected shape of the structure or of individual elements of the lateral load resisting system, including the effects on beams, columns, bracing, connections, and shear walls.
- (2) In determining the slenderness ratio of an axially loaded compression member, except as provided in NF-3322.1(c)(3), the length shall be taken as its effective length Kl, and r shall be taken as the corresponding radius of gyration.

# (b) Lateral Stability

- (1) Braced Frames. In frames [where lateral stability is provided by adequate attachment to diagonal bracing (to shear walls), to an adjacent structure having adequate lateral stability, or to floor slabs (or roof decks) secured horizontally by walls or bracing systems parallel to the plane of the frame] and in trusses, the effective length factor *K* for the compression members shall be taken as 1.0, unless analysis shows that a smaller value may be used.
- (2) Unbraced Frames. In frames where lateral stability is dependent upon the bending stiffness of rigidly connected beams and columns, the effective length *Kl* of compression members shall be determined by a rational method and shall not be less than the actual unbraced length.

# (c) Maximum Slenderness Ratios

- (1) The slenderness ratio Kl/r of compression members shall not exceed 200.
- (2) The slenderness ratio l/r of tension members, other than rods, preferably should not exceed 240 for main members, and 300 for lateral bracing members and other secondary members.

# (d) Width Ratios

# (1) Unstiffened Elements Under Compression

(-a) Effective Width. Unstiffened (projecting) compression elements are those having one free edge parallel to the direction of compression stress. The width of unstiffened plates shall be taken from the free edge to the first row of fasteners or welds; the width of legs of angles, channel and zee flanges, and stems of tees shall be taken as the full nominal dimension; the width of flanges of I-and H-shaped members and tees shall be taken as one-half of the full nominal width. The thickness of a sloping flange shall be measured at a section halfway between a free edge and the corresponding face of the web.

- (-b) Determination of Effective Width. Unstiffened elements subject to axial compression or compression due to bending shall be considered as fully effective when the width-thickness ratio is not greater than the following:
- (-1) for single angle struts and double angle struts with separators,  $76/\sqrt{S_y}$  (for SI units, use  $200 / \sqrt{S_v}$ ).
- (-2) for struts comprising double angles in contact; angles or plates projecting from girders, columns, or other compression members; compression flanges of beams; and stiffeners on plate girders,  $95 / \sqrt{S_y}$  (for SI units, use  $250 / \sqrt{S_v}$ ).
- (-3) for stems of tees,  $127 / \sqrt{S_y}$  (for SI units, use 333 /  $\sqrt{S_y}$ ).

When the actual width-thickness ratio exceeds these values, the design stress shall be governed by the provisions of (e).

(2) Stiffened Elements Under Compression

- (-a) Effective Width. Stiffened compression elements are those having lateral support along both edges which are parallel to the direction of the compression stress. The width of such elements shall be taken as follows:
- (-1) For webs of rolled, built-up, or formed sections, h is the clear distance between flanges.
- (-2) For webs of rolled, built-up, or formed sections, *d* is the full nominal depth.
- (-3) For flange or diaphragm plates in built-up sections, the width b is the distance between adjacent lines of fasteners or lines of welds.
- (-4) For flanges of rectangular hollow structural sections, the width b is the clear distance between webs less the inside corner radius on each side. If the corner radius is not known, the flat width may be taken as the total section width minus three times the thickness.
- (-b) Determination of Effective Width. Stiffened elements subject to axial compression or to uniform compression due to bending, as in the case of the flange of a flexural member [webs of flexural members are covered by the provisions of NF-3322.6(a) and NF-3322.6(e)(2) and are not subject to the provisions of this paragraph], shall be considered as fully effective when the width-thickness ratio is not greater than the following:
- (-1) for flanges of square and rectangular box sections of uniform thickness, 238 /  $\sqrt{S_y}$  (for SI units, use 624 /  $\sqrt{S_v}$ ).

(-2) for unsupported width of cover plates perforated with a succession of access holes,  $317 / \sqrt{S_y}$  (for

plate at widest hole as basis for computation stresses.

(-3) for all other uniformly compressed stiffened elements,  $253 / \sqrt{S_y}$  (for SI units, use  $663 / \sqrt{S_y}$ ).

Except in the case of perforated cover plates the actual width–thickness ratio exceed lesign stress shall be governed.

- (-c) Circular Tubular Elements Circular tubular elements subject to axial compression shall be considered as fully effective when the ratio of the outside diameter to the wall thickness is not greater than  $3,300/S_v$  $(22700/S_v)$ . For diameter to thickness ratios greater than  $3{,}300/S_v$  (22  $700/S_v$ ) but less than  $13{,}000/S_v$  $(89\,600/S_v)$ , the calculated stress shall be governed by the provisions of (e).
  - (e) Slender Compression Elements
- (1) General Requirements. Axially loaded members and flexural members, containing elements subject to compression and having a width-thickness ratio in excess of the applicable limit given in (d)(1) and (d)(2), shall be proportioned to meet the requirements of the following Subparagraphs:
  - (2) Unstiffened Compression Elements. Except as provided herein, stress on unstiffened compression elements, the width-thickness ratio of which exceeds the applicable limit given in (d)(1), shall be subject to a reduction of factor  $Q_s$ . The value of  $Q_s$  shall be determined by eqs. (-a)(27) to (-c)(32), as applicable, where b is the width of the unstiffened element as defined in (d)(1). When such elements comprise the compression flange of a flexural member, the maximum allowable bending stress shall not exceed  $0.6S_vQ_s$ , nor the applicable value as provided in (5) or (6). The allowable stress of axially loaded compression members shall be modified by the appropriate reduction factor  $Q_s$  as provided in (5).
    - (-a) For single angles, when

(U.S. Customary Units)

$$76/\sqrt{S_y} < b/t < 155/\sqrt{S_y}$$

(SI Units)

$$200/\sqrt{S_y} ~<~ b/t ~<~ 400/\sqrt{S_y}$$

then

(U.S. Customary Units)

$$Q_{S} = 1.340 - 0.00447(b/t)\sqrt{S_{y}}$$
 (27)

(SI Units)

$$Q_s = 1.340 - 0.00171(b/t)\sqrt{S_y}$$
 (27)

when

(U.S. Customary Units)

$$b/t \geq 155/\sqrt{S_y}$$

(SI Units)

$$b/t \geq 400/\sqrt{S_v}$$

then

(U.S. Customary Units)

$$Q_s = 15,500 / \left[ S_y (b/t)^2 \right]$$
 (28)

(SI Units)

$$Q_s = 106 869 / \left[ S_y (b/t)^2 \right]$$
 (28)

(-b) For angles or plates projecting from columns or other compression members and for projecting elements of compression flanges of girders, when

(U.S. Customary Units)

$$95/\sqrt{S_y/k_c} < b/t < 195/\sqrt{S_y/k_c}$$

(SI Units)

$$250/\sqrt{S_V/k_c} < b/t < 510/\sqrt{S_V/k_c}$$

then

(U.S. Customary Units)

$$Q_S = 1.293 - 0.00309(b/t)\sqrt{S_y/k_c}$$
 (29)

(SI Units)

$$Q_s = 1.293 - 0.00118(b/t)\sqrt{S_y/k_c}$$
 (29)

when

(U.S. Customary Units)

$$b/t > 1957 \sqrt{S_y/k_c}$$

(SI Units)

$$b/t > 510/\sqrt{S_y/k_c}$$

then

(U.S. Customary Units)

$$Q_s = 26,200 k_c / \left[ S_y (b/t)^2 \right]$$
 (30)

(SI Units)

$$Q_S = 180\ 000\ k_C / \left[ S_y(b/t)^2 \right]$$

 $k_c = \frac{4.05}{(h/t)^{0.46}}$  if  $\frac{h}{t} > 70$ , otherwise  $k_c = 1.0$ .

(-c) For stems of tees, when

(U.S. Customary Units)

$$127/\sqrt{S_y} < b / t < 176/\sqrt{S_y}$$

(SI Units)

$$333 \times \sqrt{S_y} < b/t < 461/\sqrt{S_y}$$

then

(U.S. Customary Units)

$$Q_S = 1.908 - 0.00715 (b/t) \sqrt{S_y}$$
 (31)

(SI Units)

$$Q_S = 1.908 - 0.00273 (b/t) \sqrt{S_V}$$
 (31)

when

(U.S. Customary Units)

$$b/t \ge 176/\sqrt{S_y}$$

(SI Units)

$$b/t \geq 461/\sqrt{S_y}$$

then

(U.S. Customary Units)

$$Q_S = 20,000 / \left[ S_y (b/t)^2 \right]$$
 (32)

(SI Units)

$$Q_{s} = 138\ 000 / \left[ S_{y}(b/t)^{2} \right]$$
 (32)

(-d) However, unstiffened elements of channels and tees whose proportions exceed the limits of (d)(1) shall conform to the limits given in Table NF-3322.2(e)(2)-1.

Table NF-3322.2(e)(2)-1						
<b>Limiting Proportions for Channels and Tees</b>						

Shape	Ratio of Flange Width to Profile Depth	Ratio of Flange Thickness to Web or Stem Thickness
Built-up or rolled	≤0.25	≤3.0
channels	≤0.50	≤2.0
Built-up tees	≥0.50	≥1.25
Rolled tees	≥0.50	≥1.10

# (3) Stiffened Compression Elements

(-a) When the width–thickness ratio of uniformly compressed stiffened elements, except for perforated cover plates, exceeds the applicable limit given in (d)(2), a reduced effective width  $b_e$  shall be used in computing the flexural design properties of the section containing the element and the permissible axial stress, except that the ratio  $b_e/t$  need not be taken as less than the applicable value permitted in (d)(2).

(-1) For the flanges of square and rectangular sections of uniform thickness

(U.S. Customary Units)

$$b_e = \frac{253t}{\sqrt{f}} \left[ 1 - \left( \frac{50.3}{(b/t)\sqrt{f}} \right) \right] \le b$$
 (33)

(SI Units)

$$b_e = \frac{663t}{\sqrt{f}} \left[ 1 - \left( \frac{132}{\left( b/t \right) \sqrt{f}} \right) \right] b \tag{33}$$

(-2) For other uniformly compressed elements:

(U.S. Customary Units)

$$b_e = \frac{253t}{\sqrt{5}} \left[ 1 - \left( \frac{44.3}{\left( b/t \right) \sqrt{f}} \right) \right] \le b \tag{34}$$

(SI IInits)

$$b_e = \frac{663t}{\sqrt{f}} \left[ 1 - \left( \frac{116}{\left( b/t \right) \sqrt{f}} \right) \right] \le b \tag{34}$$

where

b = the actual width, in. (mm), of a stiffened compression element as defined in (d)(2)

f = computed compressive stress, ksi (MPa), in the stiffened elements based on the design properties as specified in (4). If unstiffened elements are included in the total cross section for the stiffened

element, f must be such that the maximum compressive stress in the unstiffened element does not exceed  $F_aQ_s$  or  $F_bQ_s$ , as applicable.

(-b) The allowable stress for axially loaded circular tubular members not meeting the requirements of NF-3322.1(e), but having a diameter-to-thickness ratio of less than  $13,000/S_y$  (for SI units, use 89  $000/S_y$ ), shall not exceed either the smaller value determined by NF-3322.1(e) or

(U.S. Customary Units)

$$F_a = \left[\frac{662}{D/t} + 0.4S_0\right]$$

(SI Units)

$$F_a = \begin{bmatrix} 4560 \\ D/t \end{bmatrix} + 0.4S_y$$

(-c) When the allowable stresses are increased for Level C or Level D Service Limits, the effective width  $b_e$  shall be determined on the basis of 0.75 times the computed compressive stress, f, caused by Level C or Level D Service Loadings.

(4) Design Properties. Properties of sections shall be determined in accordance with conventional methods, using the full cross section of the member, except as required in (-a) and (-b).

(-a) In computing the moment of inertia and section modulus of flexural members with respect to the axis of bending under consideration, the effective width of stiffened compression elements parallel to the axis of bending and having a width-thickness ratio in excess of the applicable limit given in (d)(2), rather than the actual width, shall be used, and the axis of bending shall be located accordingly, except that for sections otherwise symmetrical, the properties may conservatively and more easily be computed using a corresponding effective area on the tension side of the neutral axis as well. That portion of the area that is neglected in arriving at the effective area shall be located at and symmetrical about the centerline of the stiffened element to which it applies.

(-b) The stress  $f_a$  due to axial loading, and the radius of gyration r, shall be computed on the basis of actual cross-sectional area. However, the allowable axial stress  $F_a$ , as provided in (5) shall be subject to the form factor

$$Q_a = \frac{\text{effective area}}{\text{actual area}}$$

where the effective area is equal to the actual area less  $\sum (b - b_e) t$ .

(5) Axially Loaded Compression Members. The allowable stress for axially loaded compression members containing unstiffened or stiffened elements shall not exceed

$$F_{a} = \frac{Q_{s}Q_{a}\left[1 - \frac{(KI/r)^{2}}{2(C'_{c})^{2}}\right]S_{y}}{\frac{5}{3} + \frac{3(KI/r)}{8C'_{c}} - \frac{(KI/r)^{3}}{8(C'_{c})^{3}}}$$
(35)

when the largest effective slenderness ratio of any unbraced segment is less than  $C'_c$  or the value given by eq. NF-3322.1(c)(1)(-b)(5) when Kl/r exceeds  $C'_{c}$ , where

$$C'_c = \sqrt{\frac{2\pi^2 E}{QS_y}}$$

and

$$Q = Q_s Q_o$$

(-a) Cross sections composed entirely of unstiffened elements

$$Q = Q_s$$
 (i.e.,  $Q_q = 1.0$ )

(-b) Cross sections composed entirely of stiffened elements

$$Q = Q_a \text{ (i.e., } Q_S = 1.0\text{)}$$

(-c) Cross sections composed of both stiffened and unstiffened elements

$$Q = Q_S Q_Q$$

(6) Combined Axial and Flexural Stress. In applying the provisions of NF-3322.1(e) to members subject to combined axial and flexural stress and containing stiffened elements whose width-thickness ratio exceeds the applicable limit given in (d)(2), the stresses  $F_a$ ,  $f_{bx}$ , and  $f_{by}$  shall be calculated on the basis of the section properties as provided in (4) and (5), as applicable. The allowable bending stress  $F_h$  for members containing unstiffened elements whose width-thickness ratio exceeds the applicable limit given in (d)(1) shall be the smaller value  $0.6S_vQ_s$  or that provided in NF-3322.1(d)(5).

# NF-3322.3 Plate Girders, Rolled Beams, and Built-Up Members.

(a) Girders and Rolled Beams. Rolled or welded shapes, plate girders, and cover-plated beams shall, in general, be proportioned by the moment of inertia of the gross section. No deduction shall be made for shop or field bolt or rivet holes in either flange provided that

$$0.5S_u A_{fn} \geq 0.6S_v A_{fq} \tag{35a}$$

where  $A_{fg}$  is the gross flange area and  $A_{fn}$  is the net flange area, calculated in accordance with the provisions of NF-3322.8.

If

$$0.5S_u A_{fn} < 0.6S_v A_{fq}$$
 (35b)

the member flexural properties shall be based on an effective tension flange area  $A_{fe}$ 

$$.5S_u A_{fn} < 0.6S_y A_{fg}$$
 (35b)  
properties shall be based on an effectea  $A_{fe}$ 

$$A_{fe} = \frac{5}{6} \frac{S_u}{S_y} A_{fn}$$
 (35c)

- (b) Hybrid Girders. Hybrid girders may be proportioned by the moment of inertia of their gross section, provided that they are not required to resist an axial force greater than  $0.15S_v$  times the area of the gross section, where  $S_v$ is the yield stress of the flange material. To qualify as hybrid girders, the flanges at any given section shall have the same cross-sectional area and be made of the same grade of steel.
- (c) Open Box-Type Beams and Grillages. When two or more rolled beams or channels are used side by side to form a flexural member, they shall be connected together at intervals of not more than 5 ft (1.5 m). Through-bolts and separators may be used, provided that in beams having a depth of 12 in. (300 mm) or more, no fewer than two bolts shall be used at each separator location. When concentrated loads are carried from one beam to the other or distributed between the beams, diaphragms having sufficient stiffness to distribute the load shall be bolted or welded between the beams.

## NF-3322.4 Compression Members.

- (a) Requirements for Fasteners
- (1) Transverse Spacing of Fasteners. All parts of builtup compression members and the transverse spacing of their lines of fasteners shall meet the requirements of NF-3322.2.
- (2) End Fasteners. At the ends of built-up compression members bearing on base plates or milled surfaces, all components in contact with one another shall be connected by bolts, spaced longitudinally not more than four diameters apart for a distance equal to  $1^{1}/_{2}$  times the maximum width of the member, or by continuous welds having a length not less than the maximum width of the member.
- (3) Intermediate Fasteners. The longitudinal spacing for intermediate bolts or intermittent welds in built-up members shall be adequate to provide for the transfer of calculated stress. The maximum longitudinal spacing of bolts or intermittent welds connecting two rolled shapes in contact with one another shall not exceed 24 in. (600 mm). In addition, for painted members and unpainted members not subject to corrosion where the outside component consists of a plate, the maximum longitudinal spacing shall not exceed:
- (-a)  $127 / \sqrt{S_v}$  (for SI units, use  $333 / \sqrt{S_v}$ ) times the thickness of the outside plate nor 12 in. (300 mm) when fasteners are not staggered along adjacent gage lines:

(-b)  $190 / \sqrt{S_y}$  (for SI units, use  $500 / \sqrt{S_y}$ ) times the thickness of the outside plate nor 18 in. (450 mm) when fasteners are staggered along adjacent gage lines.

(b) Local Slenderness Ratio. Compression members composed of two or more rolled shapes separated by intermittent fillers shall be connected at these fillers at intervals such that the slenderness ratio Kl/r of either shape, between the fasteners, does not exceed  $\frac{3}{4}$  times the governing slenderness ratio of the built-up member. The least radius of gyration r shall be used in computing the slenderness ratio of each component part. At least two intermediate connectors shall be used along the length of the built-up member.

(c) Tie Plates. Open sides of compression members built up from plates or shapes shall be provided with lacing having tie plates at each end and at intermediate points if the lacing is interrupted. Tie plates shall be as near to the ends as practicable. In main members carrying calculated stress, the end tie plates shall have a length of not less than the distance between the lines of bolts or welds connecting them to the elements of the member. Intermediate tie plates shall have a length not less than onehalf of this distance. The thickness of tie plates shall be not less than one-fiftieth of the distance between the lines of bolts or welds connecting them to the segments of the members. In bolted construction, the spacing in the direction of stress in tie plates shall be not more than six diameters and the tie plates shall be connected to each segment by at least three fasteners. In welded construction, the welding on each line connecting a tie plate shall aggregate not less than one-third of the length of the plate.

(d) Lacing. Lacing, including flat bars, angles, channels, or other shapes employed as lacing, shall be so spaced that the ratio l/r of the flange included between their connections shall not exceed  $\frac{3}{4}$  times the governing ratio for the member as a whole. Lacing shall be proportioned to resist a shearing stress normal to the axis of the member equal to 2% of the total compressive stress in the member. The ratio l/r for lacing bars arranged in single systems shall not exceed 140. For double lacing this ratio shall not exceed 200. Double lacing bars shall be joined at their intersections. For lacing bars in compression, the unsupported length of the lacing bar shall be taken as the distance between fasteners or welds connecting it to the components of the built-up member for single lacing, and 70% of that distance for double lacing. The inclination of lacing bars to the axis of the member shall preferably be not less than 60 deg for single lacing and 45 deg for double lacing. When the distance between the lines of fasteners or welds in the flanges is more than 15 in. (381 mm), the lacing preferably shall be double or be made of angles.

(e) Perforated Cover Plates. The function of tie plates and lacing may be performed by continuous cover plates perforated with a succession of access holes. The

unsupported width of such plates at access holes, as defined in NF-3322.2(d)(2), is assumed to be available to resist axial stress, provided that

- (1) the width-thickness ratio conforms to the limitations of NF-3322.2(d)(2)
- (2) the ratio of length in direction of stress to width of hole shall not exceed 2
- (3) the clear distance between holes in the direction of stress shall be not less than the transverse distance between nearest lines of connection bolts or welds
- (4) the periphery of the holes at all points connecting shall have a minimum radius of  $1^{1}/_{2}$  in. (38 mm)

# NF-3322.5 Built-Up Tension Members.

- (a) Longitudinal Spacing of Connectors. The longitudinal spacing of connectors between elements in continuous contact consisting of a plate and a shape or two plates shall not exceed
- (1) 24 times the thickness of the thinner plate, or 12 in. (300 mm) for painted members or unpainted members not subject to corrosion
- (2) 14 times the thickness of the thinner plate, or 7 in. (175 mm) for unpainted members of weathering steel subject to atmospheric corrosion

In a tension member the longitudinal spacing of fasteners and intermittent welds connecting two or more shapes in contact shall not exceed 24 in. (600 mm). Tension members composed of two or more shapes or plates separated by intermittent fillers shall be connected to one another at these fillers at intervals such that the slenderness ratio of either component between the fasteners does not exceed 300.

(b) Perforated Cover and Tie Plates. Either perforated cover plates or tie plates without lacing may be used on the open sides of built-up tension members. Tie plates shall have a length not less than two-thirds the distance between the lines of bolts or welds connecting them to the elements of the member. The thickness of such tie plates shall not be less than one-fiftieth of the distance between these lines. The longitudinal spacing of bolts or intermittent welds at tie plates shall not exceed 6 in. (150 mm). The spacing of tie plates shall be such that the slenderness ratio of any component in the length between tie plates shall not exceed 300.

# NF-3322.6 Webs, Flanges, and Stiffeners.

(a) Webs. The ratio of the clear distance between flanges to the web thickness shall not exceed

(U.S. Customary Units)

$$14,000 / \sqrt{S_y(S_y + 16.5)}$$
 (36)

(SI Units)

96 500 / 
$$\sqrt{S_y(S_y + 114)}$$
 (36)

except that when transverse stiffeners are provided,

spaced not more than  $1\frac{1}{2}$  times the girder depth, the limiting ratio may be  $2,000/\sqrt{\mathit{S}_{y}}$  (for SI units, use 5 300 /  $\sqrt{S_y}$ ), where  $S_y$  is the yield stress of the compression flange.

(1) Combined Shear and Tension Stress. Plate girder webs which depend upon tension field action, as provided in eq. (e)(2)(45), shall be so proportioned that bending tensile stress due to moment in the plane of the girder web shall not exceed  $0.6S_v$ , or

$$\left(0.825 - 0.375 \frac{f_{\nu}}{F_{\nu}}\right) S_{\nu} \tag{37}$$

where

 $F_v$  = allowable web shear stress, ksi (MPa), according to eq. (e)(2)(45)

The allowable shear stress in the webs of girders having flanges and webs with yield values greater than 65 ksi (450 MPa) shall not exceed the values given by eq. (e)(2)(44) if the flexural stress in the flange  $f_b$  exceeds  $0.75F_{b}$ .

## (2) Web Crippling

(-a) Local Web Yielding. Webs of beams and welded plate girders shall be so proportioned that the compressive stress at the web toe of the fillets, resulting from concentrated loads not supported by bearing stiffeners, shall not exceed the value of  $0.66S_v$ ; otherwise, bearing stiffeners shall be provided. The governing equations shall be

(-1) for interior loads

$$\frac{R}{t(N+5k)} \le 0.66S_y \tag{38}$$

(-2) for end reactions

$$\frac{R}{t(N+2.5k)} \leq 0.66S_y \tag{39}$$

where N is greater than or equal to k for end reactions.

(-b) Stress on Compression Edge of Web Plate. Bearing stiffeners shall be provided in the webs of members under concentrated loads, when the compressive force exceeds the following limits:

when the concentrated load is applied at a distance not less than d/2 from the end of the member

(U.S. Customary Units)

$$R = 67.5 t_w^2 \left[ 1 + 3 \left( \frac{N}{d} \right) \left( \frac{t_w}{t_f} \right)^{1.5} \right] \sqrt{S_y t_f / t_w}$$
 (40)

(SI Units)

$$R = 177t_{w}^{2} \left[ 1 + 3 \left( \frac{N}{d} \right) \left( \frac{t_{w}}{t_{f}} \right)^{1.5} \right] \sqrt{S_{y}t_{f}/t_{w}}$$
(40)

The concentrated load is applied less than a distance in the end of the member to the many Units)

$$R = 34t^{2} \left[ 1 + 3 \left( \frac{N}{d} \right) \left( \frac{t_{w}}{t_{f}} \right)^{1.5} \right] \sqrt{S_{x}t_{f}/t_{w}}$$
(41)

when the concentrated load is applied less than a distance d/2 from the end of the member

(U.S. Customary Units)

$$R = 34t_{w}^{2} \left[ 1 + 3 \left( \frac{N}{d} \right) \left( \frac{t_{w}}{t_{f}} \right)^{1.5} \right] \sqrt{S_{y}t_{f}} \sqrt{w}$$
 (41)

(SI Units)

$$R = 89t_w^2 \left[ 1 \frac{\sqrt{3}}{d} \left( \frac{t_w}{t_f} \right)^{1.5} \right] \sqrt{S_y t_f / t_w}$$
 (41)

where

d = overall depth of the member, in. (mm)

 $S_v =$  specified minimum yield stress of beam web, ksi (MPa) flange thickness, in. (mm)

= web thickness, in. (mm)

If stiffeners are provided and extend at least onehalf the web depth, eqs. (40) and (41) need not be checked.

Bearing stiffeners shall be provided in the webs of members with flanges not restrained against relative movement by stiffeners or lateral bracing and subject to concentrated compressive loads, when the compressive force exceeds the following limits:

If the loaded flange is restrained against rotation and  $(d_c/t_w)/(l/b_f)$  is less than 2.3

(U.S. Customary Units)

$$R = \frac{6,800t_w^3}{h} \left[ 1 + 0.4 \left( \frac{d_c / t_w}{l / b_f} \right)^3 \right]$$
 (41a)

(SI Units)

$$R = \frac{46 882 t_w^3}{h} \left[ 1 + 0.4 \left( \frac{d_c / t_w}{l / b_f} \right)^3 \right]$$
 (41a)

If the loaded flange is not restrained against rotation and  $(d_c/t_w)/(l/b_f)$  is less than 1.7

(U.S. Customary Units)

$$R = \frac{6,800t_w^3}{h} \left[ 0.4 \left( \frac{d_c / t_w}{l / b_f} \right)^3 \right]$$
 (41b)

(SI Units)

$$R = \frac{46 882 t_w^3}{h} \left[ 0.4 \left( \frac{d_c / t_w}{I / b_f} \right)^3 \right]$$
 (41b)

where

 $b_f$  = flange width, in. (mm)

 $d_c$  = web depth clear of fillets, in. (mm)

= d - 2k

l = largest laterally unbraced length along either flange at the point of load, in. (mm)

Equations (41a) and (41b) need not be checked providing  $(d_c/t_w)/(l/b_f)$  exceeds 2.3 or 1.7, respectively, or for webs subject to uniformly distributed load.

- (b) Flanges
- (1) The thickness of outstanding parts of flanges shall conform to the requirements of NF-3322.2(d)(1).
- (2) Flanges of welded plate girders may be varied in thickness or width by splicing a series of plates or by the use of cover plates.
- (3) The total cross-sectional area of cover plates of bolted girders shall not exceed 70% of the total flange area
  - (c) Flange Development
- (1) High-strength bolts or welds connecting flange to web or cover plate to flange shall be proportioned to resist the total horizontal shear resulting from the bending forces on the girder. The longitudinal distribution of these bolts or intermittent welds shall be in proportion to the intensity of the shear, but the longitudinal spacing shall not exceed the maximum permitted, respectively, for compression or tension members in NF-3322.4(a)(3) or NF-3322.5(a). Additionally, bolts or welds connecting flange to web shall be proportioned so as to transmit to the web any loads applied directly to the flange unless provision is made to transmit such loads by direct bearing.
- (2) Partial length cover plates shall be extended beyond the theoretical cut-off point and the extended portion shall be attached to the beam or girder by high-strength bolts for friction-type joints or fillet welds adequate at the applicable stresses allowed in NF-3324.6(a), NF-3324.5(d), or NF-3332.4, to develop the cover plate portion of the flexural stresses in the beam or girder at the theoretical cut-off point. In addition, for

welded cover plates, the welds connecting the cover plate termination to the beam or girder in the length  $a^\prime$ , defined in (-a) through (-c), shall be adequate at the allowed stresses to develop the cover plate portion of the flexural stresses in the beam or girder at the distance  $a^\prime$  from the end of the cover plate. The length  $a^\prime$  measured from the end of the cover plate, shall be

- (-a) a distance equal to the width of the coverplate when there is a continuous weld equal to or larger than three-fourths of the plate thickness across the end of the plate and continuous welds along both edges of the cover plate in the length  $a^{\prime}$
- (-b) a distance equal to  $1\frac{1}{2}$  times the width of the cover plate when there is a continuous weld smaller than three-fourths of the plate thickness across the end of the plate and continuous welds along both edges of the cover plate in the length a'
- (-c) a distance equal to two times the width of the cover plate when there is no weld across the end of the plate, but continuous welds along both edges of the cover plate in the length  $\alpha'$
- (d) Reduction in Flange Stress. When the web depth-thickness ratio exceeds  $760 / \sqrt{F_b}$  (for SI units, use 2 000 /  $\sqrt{F_b}$ ), the maximum stress in the compression flange shall not exceed

(U.S. Customary Units)

$$F'_b \le F_b \left[ 1.0 - 0.0005 \frac{A_W}{A_f} \left( \frac{h}{t} - \frac{760}{\sqrt{F_b}} \right) \right]$$
 (42)

(SI Units)

$$F'_b \le F_b \left[ 1.0 - 0.0005 \frac{A_W}{A_f} \left( \frac{h}{t} - \frac{2\ 000}{\sqrt{F_b}} \right) \right]$$
 (42)

where  $F_b$  is the applicable bending stress, ksi (MPa), given in NF-3322.1(d). The maximum stress in either flange of a hybrid girder shall not exceed the value given in eq. (42) or

$$F'_b \le F_b \left[ \frac{12 + (A_w / A_f)(3\alpha - \alpha^3)}{12 + 2(A_w / A_f)} \right]$$
 (43)

- (e) Stiffeners
- (1) Bearing. Bearing stiffeners shall be placed in pairs at unframed ends on the webs of plate girders and, where required by (a)(2), at points of concentrated loads. Such stiffeners, when the load normal to the flange is tensile, shall be welded to the loaded flange and when the load normal to the flange is compressive may be welded or bear on the loaded flange. They shall be designed as columns subject to the provisions of NF-3322.1, assuming the column section to comprise the pair of stiffeners and a centrally located strip of the web whose width is

equal to not more than 25 times its thickness at interior stiffeners, or a width equal to not more than 12 times its thickness when the stiffeners are located at the end of the web. The effective length shall be taken as not less than three-fourths of the length of the stiffeners in computing the ratio l/r. Only that portion of the stiffener outside of the flange angle fillet or the flange to web welds shall be considered effective in bearing.

(2) Average Web Shear. Except as provided herein, the largest average web shear  $f_{\nu}$ , ksi (MPa), computed for any condition of complete or partial loading, shall not exceed the value given by eq. (44)

$$F_{\nu} = (S_y / 2.89) C_{\nu} \le 0.4 S_y$$
 (44)

where, when  $C_v$  is less than 0.8,

(U.S. Customary Units)

$$C_{V} = \frac{45,000k}{S_{y} \left(\frac{h}{t}\right)^{2}}$$

(SI Units)

(17)

$$C_{V} = \frac{310,000k}{S_{y} \left(\frac{h}{t}\right)^{2}}$$

when  $C_v$  is greater than 0.8,

(U.S. Customary Units)

$$C_{V} = \left(\frac{190}{\frac{h}{t}}\right) \sqrt{\frac{k}{S_{y}}}$$

(SI Units)

$$C_{\nu} = \left(\frac{500}{\frac{h}{t}}\right) \sqrt{\frac{k}{s_{y}}}$$

where

$$k = 4.00 + [5.34/(a/h)^2]$$
, when  $a/h$  is less than 1.0  
= 5.34 +  $[4.00/(a/h)^2]$ , when  $a/h$  is more than 1.0

Alternatively, for girders other than hybrid girders and tapered members, if intermediate stiffeners are provided and spaced to satisfy the provisions of (3) and if  $C_v \le 1$ , the allowable shear given by eq. (45) may be used in lieu of the value given in eq. (44)

$$F_{\nu} = \frac{S_{y}}{2.89} \left[ C_{\nu} + \frac{1 - C_{\nu}}{1.15 \sqrt{1 + (a/h)^{2}}} \right] \le 0.4S_{y}$$
 (45)

NOTE: Equation (45) recognizes the contribution of tension field action.

## (3) Intermediate Stiffeners

- (-a) Subject to the limitations of (a), intermediate stiffeners are not required when the ratio h/t is less than 260 and the maximum web shear stress  $f_{\nu}$  is less than that permitted for  $F_{\nu}$  by eq. (2)(44).
- (-b) The spacing of intermediate stiffeners, when stiffeners are required, shall be such that the web shear stress will not exceed the value for  $F_v$  given by eq. (2)(44) or eq. (2)(45), as applicable, and the ratio a/h shall not exceed  $[260/(h/t)]^2$  or 3.
- (-c) In girders designed on the basis of tension field action, the spacing between stiffeners at end panels, at panels containing large holes, and at panels adjacent to panels containing large holes shall be such that  $f_{\nu}$  does not exceed the value for  $F_{\nu}$  given by eq. (2)(44).

# (4) Moment of Inertia and Gross Area

- (-a) The moment of inertia, in.<sup>4</sup> (mm<sup>4</sup>), of a pair of intermediate stiffeners or a single intermediate stiffener, with reference to an axis in the plane of the web, shall not be less than  $(h/50)^4$ .
- (-b) The gross area, in.<sup>2</sup> (mm<sup>2</sup>), of intermediate stiffeners spaced as required for eq. (2)(45) (total area, when stiffeners are furnished in pairs), shall be not less than that computed by eq. (46)

$$A_{St} = \frac{1 - C_{v}}{2} \left[ \frac{a}{h} - \frac{(a/h)^{2}}{\sqrt{1 + (a/h)^{2}}} \right] YDht$$
 (46)

whore

- D = 1.0 for stiffeners furnished in pairs
  - = 1.8 for single angle stiffeners
  - = 2.4 for single plate stiffeners
- (-c) When the greatest shear stress  $f_{\nu}$  in a panel is less than that permitted by eq. (2)(45), this gross area requirement may be reduced proportionately.
- (-d) Intermediate stiffeners required by eq. (2)(45) shall be connected for a total shear transfer, kips/linear in. (N/linear mm) of single stiffener or pair of stiffeners, not less than that computed by eq. (47)

(U.S. Customary Units)

$$f_{vs} = h \left( \frac{S_y}{340} \right)^{3/2} \tag{47}$$

(SI Units)

$$f_{VS} = h \left( \frac{S_y}{647} \right)^{3/2} \tag{47}$$

This shear transfer may be reduced in the same proportion that the largest computed shear stress  $f_{\nu}$  in the adjacent panels is less than that permitted by eq. (2)(45). However, bolts and welds in intermediate stiffeners which are required to transmit to the web an applied concentrated load or reaction shall be proportioned for not less than the applied load or reaction.

- (-e) Intermediate stiffness may be stopped short of the tension flange, provided bracing is not needed to transmit a concentrated load or reaction; the weld by which intermediate stiffeners are attached to the web shall be terminated not closer than four times nor more than six times the web thickness from the near toe of the web to flange welds. When single stiffeners are used, they shall be attached to the compression flange, if it consists of a rectangular plate, to resist any uplift tendency due to torsion in the plate. When lateral bracing is attached to a stiffener or a pair of stiffeners, these, in turn, shall be connected to the compression flange to transmit 1% of the total flange stress, unless the flange is composed only of angles.
- (-f) Bolts connecting stiffeners to the girder web shall be spaced not more than 12 in. (300 mm) on center. If intermittent fillet welds are used, the clear distance between welds shall not be more than 16 times the web thickness, or more than 10 in. (250 mm).
- (f) Splices. Groove welded splices in the plate girders and beams shall develop the full strength of the smaller spliced section; other types of splices in cross sections of plate girders and in beams shall develop the strength required by the stresses at the point of splice.
- (g) Rotational Restraint at Points of Support. Beams, girders, and trusses shall be restrained against rotation about their longitudinal axis at points of support.

# NF-3322.7 Simple and Continuous Spans, and Provision for Expansion.

- (a) Design Requirements for Spans
- (1) Simple Spans. Beams, girders, and trusses shall ordinarily be designed on the basis of simple spans whose effective length is equal to the distance between centers of gravity of the members to which they deliver their end reactions.
- (2) Continuous Spans. When designed on the assumption of full or partial end restraint, due to continuous, semi-continuous, or cantilever action, the beams, girders, and trusses, as well as the sections of the members to which they connect, shall be designed to carry the shears and moments so introduced, as well as all other forces, without exceeding at any point the unit stresses prescribed in NF-3321.1, except that some nonelastic but self-limiting deformation of a part of the connection may be permitted when this is essential to the avoidance of overstressing of fasteners.
- (3) Provision for Expansion. Adequate provision shall be made for expansion and contraction appropriate to the function of the support structure.

#### NF-3322.8 Gross and Net Areas.

- (a) Determination of Gross and Net Areas. The gross area of a member at any point shall be determined by summing the products of the thickness and the gross width of each element as measured normal to the axis of the member. The net area shall be determined by substituting the net width, computed in accordance with (c) for the gross width.
- (b) Net Area and Effective Net Area. Unless otherwise specified, tension members shall be designed on the basis of net area. Compression members shall be designed on the basis of gross area. Beams and girders shall be designed in accordance with NF-33223(a). The effective net area of threaded members shall be the tensile stress area, with the exception that threaded hanger rods shall comply with NF-3412.3.
  - (c) Calculation of Net Area
    - (1) Parts With Holes
- (-a) In the case of a chain of holes extending across a part in any diagonal or zigzag line, the net width of the part shall be obtained by deducting from the gross width the sum of the diameters of all the holes in the chain and adding, for each gage space in the chain, the quantity  $s^2/4g$ .
- (-b) The critical net area  $A_n$  of the part is obtained from that chain which gives the least net width.
- (-c) In determining the net area across plug or slot welds, the weld metal shall not be considered as adding to the net area.
- (-d) The effective net area  $A_e$  of axially loaded tension members, where the load is transmitted by bolts through some but not all of the cross-sectional elements of the member, shall be computed from the following equation:

$$A_e = C_t A_n$$

Unless a larger coefficient can be justified by tests or other recognized criteria, the following values of  $\mathcal{C}_t$  shall be used in computations:

(-1) W, M, or S shapes with flange widths not less than two-thirds of the depth, and structural tees cut from these shapes, provided the connection is to the flanges and has no fewer than three fasteners per line in the direction of applied stress

$$C_t = 0.90$$

(-2) W, M, or S shapes not meeting the conditions of (-1), structural tees cut from these shapes, and all other shapes, including built-up cross sections, provided the connection has not less than three fasteners per line in the direction of applied stress

$$C_t = 0.85$$

(-3) all members whose connections have only two fasteners per line in the direction of applied stress

$$C_t = 0.75$$

When the load is transmitted by welds through some but not all of the cross-sectional elements of the member, the effective net area  $A_e$  shall be computed as

$$A_e = U A_g$$

where

 $A_g = \text{gross area of member, in.}^2 \text{ (mm}^2\text{)}$ 

When load is transmitted by transverse welds to some but not all of the cross-sectional elements of W, M, or S shapes and structural tees cut from these shapes,  $A_e$  shall be taken as the area of the directly connected elements.

When the load is transmitted to a plate by longitudinal welds along both edges at the end of the plate, the length of the welds shall not be less than the width of the plate. The effective net area  $A_e$  shall be computed as above.

Unless a larger coefficient can be justified by tests or other criteria, the following values of U shall be used: when l > 2w, U = 1.0; when 2w > l > 1.5w, U = 0.87; when 1.5w > l > w, U = 0.75

where

**(17)** 

l = weld length, in. (mm) w = plate width (distance between welds), in. (mm)

- (-e) Bolted splice and gusset plates, and other connection fittings subject to tensile force shall be designed in accordance with the provisions of NF-3322.1(a) where the effective net area shall be taken as the actual net area, except that for the purpose of design calculations it shall not be taken as greater than 85% of the gross area.
- (2) Size of Bolt Holes. In computing net area, the diameter of a bolt hole shall be taken as  $\frac{1}{16}$  in. (1.5 mm) greater than the nominal dimension of the hole normal to the direction of applied stress.
- (3) Angles. For angles, the gross width shall be the sum of the widths of the legs less the thickness. The gage for holes in opposite legs shall be the sum of the gages from the back of angles less the thickness.
  - (4) Pin-Connected Members

(a) Eyebars shall be of uniform thickness without reinforcement at the pinholes. They shall have circular heads in which the periphery of the head beyond the pinhole is concentric with the pinhole. The radius of transition between the circular head and the body of the eyebar shall be equal to or greater than the diameter of the head. The width of the body of the eyebar shall not exceed eight times its thickness and the thickness shall not be less than  $\frac{1}{2}$  in. (13 mm). The minimum net area beyond the pin hole, parallel to the axis of the member, shall

not be less than  $^2/_3$  of the net area across the pin hole. The thickness may be less than  $^1/_2$  in. (13 mm) only if external nuts are provided to tighten pin plates and filler plates into snug contact. The diameter of the pin shall not be less than seven-eighths of the width of the body of the eyebar. The diameter of the pinhole shall not be more than  $^1/_{32}$  in. (0.8 mm) greater than the diameter of the pin. For steels having a yield stress greater than 70.0 ksi (480 MPa), the diameter of the pinhole shall not exceed five times the plate thickness.

NOTE: Members having a different thickness at the pinhole location are termed built-up.

- (-b) In pin-connected plates other than eyebars, the tensile stress on the net area transverse to the axis of the member shall not exceed the stress allowed in NF-3322.1(a), and the bearing stress on the projected area of the pin shall not exceed the stress allowed in NF-3322.1(f)(1). The minimum net area beyond the pinhole, parallel to the axis of the member, shall not be less than two-thirds of the net area across the pinhole.
- (-c) The distance used in calculations, transverse to the axis of pin-connected plates or any individual element of a built up member, from the edge of the pinhole to the edge of the member or element shall not exceed 4 times the thickness at the pinhole. For calculation purposes, the distance from the edge of the pinhole to the edge of the plate or to the edge of a separated element of a built-up member at the pinhole, shall not be assumed to be more than 0.8 times the diameter of the pinhole.
- (-d) The corners beyond the pinhole may be cut at 45 deg to the axis of the member, provided the net area beyond the pinhole, on a plane perpendicular to the cut, is not less than that required beyond the pinhole parallel to the axis of the member.
- (-e) Thickness limitations on both eyebars and pin-connected plates may be waived whenever external nuts are provided so as to tighten pin plates and filler plates into snug contact. When the plates are thus contained, the allowable stress in bearing shall be no greater than as specified in NF-3322.1(f)(1).
- **NF-3322.9 Rectangular Tubular Sections.** For electric resistance welded rectangular tubular sections, the designer shall use a wall thickness of 0.93 times the nominal wall thickness for calculating section properties.

# NF-3323 Design Requirements for Steel Castings and Forgings

**NF-3323.1 Allowable Stresses.** Allowable stresses for steel castings and forgings shall be the same as those provided in NF-3322 with the applicable values for yield strength of Section II, Part D, Subpart 1, Table Y-1.

# NF-3324 Design Requirements for Connections and Joints

#### NF-3324.1 General Requirements.

(a) Types of Connections Permitted. Structural Connections of a welded, bolted, or welded and bolted type shall be used.

Types of bolted connections include

- (1) Friction-type connections that are mechanically fastened structural connections that use high-strength bolts (such as SA-325 and SA-490) and are sufficiently tightened to reliably produce a high clamping force that prevents slip between faying surfaces. Load is transferred at the faying surfaces by friction [Figure NF-3324.1(a)(1)].
- (2) Bearing-type connections that are mechanically fastened structural connections that depend on direct shear of the fastener to transfer the load from one connected part to another.
- (3) Other connections that are not covered by (1) and (2), such as clamps and U-bolts shown in Figure NF-1214-1, are not friction-type connections and shall meet the requirements of NF-3380, unless otherwise indicated in the design documents.
- (b) Provision for Eccentric Connections. Axially stressed members meeting at a point shall have their gravity axes intersect at a point if practicable; if not, provision shall be made for bending stresses due to the eccentricity.
- (c) Placement of Bolts and Welds. Except as otherwise provided, groups of bolts or welds at the ends of any member transmitting axial stress into that member shall have their centers of gravity on the gravity axis of the member, unless provision is made for the effect of the resulting eccentricity. Except in members subject to repeated variation in stress, as stipulated in NF-3330, disposition of fillet welds to balance the forces about the neutral axes for end connections of single angle, double angle, and similar type members is not required.

Eccentricity between the gravity axes of such members and the gage lines for their bolted end connections may be neglected in statically loaded members, but should be considered in members subjected to fatigue loading.

#### NF-3324.2 Connection Design.

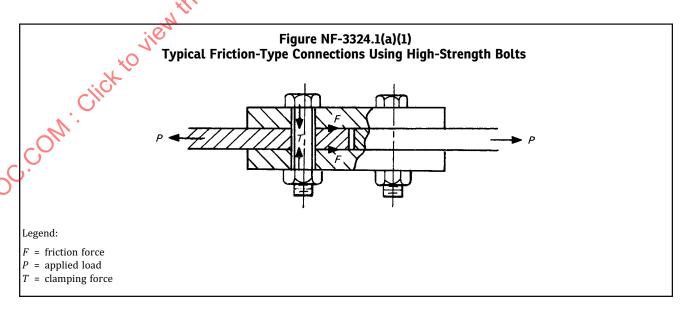
- (a) Connections for Unrestrained Members
- (1) Except as otherwise indicated by the design documents, connections of beams, girders, or trusses shall be designed as flexible and may ordinarily be proportioned for the reaction shears only.
- (2) Flexible beam connections shall accommodate end rotations of unrestrained (simple) beams. To accomplish this, inelastic action in the connection is permitted.
  - (b) Connections for Restrained Members
- (1) Fasteners or welds for end connections of beams, girders, and trusses shall be designed for the combined effect of forces resulting from moment and shear induced by the rigidity of the connections.
- (2) When flanges or moment connection plates for end connections of beams and girders are welded to the flange of an L or H-shaped column, a pair of column web stiffeners having a combined cross-sectional area  $A_{st}$  not less than that computed from eq. (48) shall be provided whenever the calculated value of  $A_{st}$  is positive

$$A_{st} = \frac{P_{bf} - F_{yc}t(t_b + 5k)}{F_{yst}} \tag{48}$$

(3) Notwithstanding the requirements of (2), a stiffener or a pair of stiffeners shall be provided opposite the compression flange when the column web depth clear of fillets  $d_c$  is greater than

(U.S. Customary Units)

$$\frac{4,100t^3~\times~\sqrt{F_{yc}}}{P_{bf}}$$



(SI Units)

$$\frac{10.765t^3 \times \sqrt{F_{yc}}}{P_{bf}}$$

and a pair of stiffeners shall be provided opposite the tension flange when the thickness of the column flange  $t_f$  is less than

$$0.4 \sqrt{P_{bf}/F_{yc}}$$

- (4) Stiffeners required by the provisions of (2) and (3) shall comply with the following criteria:
- (-a) The width of each stiffener plus one-half the thickness of the column web shall not be less than one-third the width of the flange or moment connection plate delivering the concentrated force.
- (-b) The thickness of stiffeners shall not be less than  $t_h/2$ .
- (-c) When the concentrated force delivered occurs on only one column flange, the stiffener length need not exceed one-half the column depth.
- (-d) The weld joining stiffeners to the column web shall be sized to carry the force in the stiffener caused by unbalanced moments on opposite sides of the column.
- (5) The thickness of column webs within the boundaries of rigid connections of two or more members whose webs lie in a common plane shall not be less than

(U.S. Customary Units)

$$t_{\min} = \frac{32M}{d_b \cdot d_c \cdot S_y}$$

(SI Units)

$$t_{\min} = \frac{2666M}{d_b \cdot d_b \cdot S_y}$$

where

 $d_b$  = the beam depth (n. (mm)

 $d_c$  = the column depth, in. (mm)

M = the algebraic sum of clockwise and counterclockwise moments, kip-ft (N·mm), applied on opposite sides of the connection boundary

 $t_{\min}$  = the web thickness, in. (mm)

When the web thickness is less than  $t_{\min}$  doubler plates or longitudinal stiffeners may be provided to reinforce the column web.

(c) Connections of Tension and Compression Members in Trusses. The connections at ends of the tension or compression member in trusses shall develop the force due to the Design Load but not less than 50% of the effective strength of the member, based upon the kind of stress that governs the selection of the member.

- (d) Connections for Compression Members With Bearing Joints
- (1) For members subjected to compression forces only, which bear on bearing plates, sufficient bolts or welds shall be provided to hold all parts securely in place.
- (2) For members subjected to compression forces only, which are finished to bear at splices, the splice material and its bolting or welding shall be arranged to hold all parts in line and shall be proportioned for 50% of the computed stress.
- (3) All of the foregoing joints shall be proportioned to resist any tension that would be developed by design moments acting in conjunction with 75% of the axial compression forces.

#### NF-3324.3 Design of Lap Joints and Fillers.

- (a) Lap Joint Design. The minimum amount of lap on lap joints shall be five times the thickness of the thinner part joined but not less than 1 in. (25 mm). Lap joints subjected to axial stress shall be fillet welded along the end of both lapped parts, except where the deflection of the lapped parts is sufficiently restrained to prevent opening of the joint under maximum loading.
  - (b) Filler Design
- (1) *Bolted Construction.* When bolts carrying computed stress pass through fillers thicker than  $\frac{1}{4}$  in. (6 mm), except in friction-type connections assembled with high-strength bolts, the fillers shall be extended beyond the splice material and the filler extension shall be secured by enough bolts to distribute the total stress in the member uniformly over the combined section of the member and the filler or an equivalent number of fasteners shall be included in the connection. Fillers between  $\frac{1}{4}$  in. (6 mm) and  $\frac{3}{4}$  in. (19 mm) thick, inclusive, need not be extended and developed, provided the allowable shear stress in the bolts is reduced by the factor, 0.4 (t-0.25) [for SI units, use 0.016 (t-6)], where t is the total thickness of the fillers, up to  $\frac{3}{4}$  in. (19 mm).
- (2) Welded Construction. In welded construction, any filler  $^1/_4$  in. (6 mm) or more in thickness shall extend beyond the edges of the splice plate and shall be welded to the part on which it is fitted with sufficient weld to transmit the splice plate stress, applied at the surface of the filler as an eccentric load. The welds joining the splice plate to the filler shall be sufficient to transmit the splice plate stress and shall be long enough to avoid overstressing the filler along the toe of the weld. Any filler less than  $^1/_4$  in. (6 mm) thick shall have its edges made flush with the edges of the splice plate, and the weld size shall be the sum of the size necessary to carry the splice plate stress plus the thickness of the filler plate.

#### NF-3324.4 Joint Combinations.

(a) Combinations of Welds. If two or more of the general types of weld (groove, fillet, plug, or slot) are combined in a single joint, the effective capacity of each shall be

separately computed with reference to the axis of the group, in order to determine the allowable capacity of the combination.

(b) Bolts in Combination With Welds. SA-307 bolts or high-strength bolts used in bearing-type connections shall not be considered as sharing the stress in combination with welds. Welds, if used, shall be provided to carry the entire stress in the connection. High-strength bolts installed as a friction-type connection prior to welding may be considered as sharing the stress with the welds.

#### NF-3324.5 Design of Welded Joints.

- (a) Permissible Types of Welded Joints in Linear Supports. All welded joints in Linear Supports shall be as described in NF-3256.1 except that intermittent or lap joints using fillet welds may not be used for the support of Class 1 Components or Class 2 vessels designed to NC-3200. The allowable stress limits shall be as defined in (b) and (c).
- (b) Design Limits. The allowable stress limits that must be satisfied for welds for Design Loadings stated in the Design Specification shall be as follows:
- (1) Full Penetration Groove Welds. The stress limits for full penetration groove welds shall not exceed the applicable stress value for the base metal being joined, as specified in NF-3321.1 and Table NF-3324.5(a)-1.
  - (2) Partial Penetration Groove Welds
- (-a) Compression Normal to Effective Throat or Shear on Effective Throat. The stress limits shall be the same as those for the base metal, as required by NF-3321.1.

- (-b) Tension Normal to the Axis on the Effective Throat. The stress limits shall be as specified in Table NF-3324.5(a)-1.
- (3) Fillet Welds. The allowable stress limits for fillet welds shall be as specified in Table NF-3324.5(a)-1.
- (c) Service Limits, Level A Through D, and Test. The rules and stress limits which must be satisfied for welds for any Level A through D Service and Test Loading stated in the Design Specification are those given in (b) multiplied by the appropriate base material stress limit factor given in Table NF-3312.1(b)-1.

#### (d) Fillet Welds

- (1) Minimum Size of Fillet Welds and Partial Penetration Welds. Fillet and partial penetration welds shall not be specified less than  $\frac{1}{8}$  in. (3 mm) (NF-4427). When fillet or partial penetration welds less than  $\frac{1}{4}$  in. (6 mm) are used to join heavy section members, the designer shall consider specifying preheat and special requirements for fit-up of members to ensure adequate weld deposition.
- (2) Maximum Effective Size of Fillet Welds. The maximum size of a fillet weld that may be assumed in the design of a connection shall be such that the stresses in the adjacent base material do not exceed the values allowed in Tables NF-3324.5(a)-1 and NF-3321.1. The maximum size that may be used along edges of connected parts shall be as stipulated in (-a) and (-b)
- (-a) along edges of material less than  $\frac{1}{4}$  in. (6 mm) thick, the maximum size may be equal to the thickness of the material

# Table NF-3324.5(a)-1 Allowable Stress Limits for Class 1 Linear-Type Support Welds

Kind of Stress	Base Metal TS Range, ksi (MPa)	Weld Metal Min. TS, ksi (MPa) [Note (1)]	Stress Limits, ksi (MPa)
Tension and compression parallel to axis of any full penetration groove weld			Same as for base metal
Tension normal to effective throat of full penetration groove weld			Same as allowable tensile stress for base metal
Compression normal to effective throat of full penetration groove weld and partial penetration groove weld	:		Same as allowable compressive stress for base metal
Shear stress on effective throat of fillet weld, regardless of direction of application of load; tension normal to the axis on the effective throat of a partial penetration groove weld and shear stress on effective area of a plug or slot weld. The given stresses shall also apply to such welds made with the specified electrode on steel having a yield stress greater than that of the matching base metal.	45-60 (310-410) 61-70 (411-480) 71-80 (481-550) 81-90 (551-620) 91-100 (621-690) 101-120 (691-830)	60 (410) 70 (480) 80 (550) 90 (620) 100 (690) 120 (830)	0.30 × minimum tensile strength of weld metal [ksi (MPa)], except shear stress on base metal shall not exceed 0.40 × yield stress of base metal

#### NOTE:

(1) Weld metal one nominal strength level stronger than shown will be permitted (e.g., E70xx for E60xx applications).

(-b) along edges of material  ${}^{1}/_{4}$  in. (6 mm) or more in thickness, the maximum size shall be  ${}^{1}/_{16}$  in. (1.5 mm) less than the thickness of the material, unless the weld is especially designated on the drawings to be built up to obtain full throat thickness

#### (3) Length of Fillet Welds

- (-a) The effective length of a fillet weld shall be the overall length of a full-size fillet, including returns.
- (-b) The minimum effective length of a strength fillet weld shall be not less than four times the nominal size, or else the size of the weld shall be considered not to exceed one-fourth of its effective length.
- (-c) If longitudinal fillet welds are used alone in end connections of tension members, the length of each fillet weld shall be not less than the perpendicular distance between them. The transverse spacing of longitudinal fillet welds used in end connections shall not exceed 8 in. (200 mm), unless the design otherwise prevents excessive transverse bending in the connection.
- (4) Effective Throat Thickness of Fillet Welds. The effective throat thickness of a fillet weld joining members at angles between 60 deg and 135 deg, inclusive (Figure NF-4427-1), shall be the shortest distance from the root to the face of the diagrammatic weld, except that for fillet welds made by the submerged arc process the effective throat thickness shall be taken equal to the leg size for  $\frac{3}{8}$  in. (10 mm) and smaller fillet welds, and equal to the theoretical throat plus 0.11 in. (2.8 mm) for fillet welds over  $\frac{3}{8}$  in. (10 mm). For fillet welds joining members at angles less than 60 deg use the rules for partial penetration groove welds, (f)(3).
- (5) Effective Area of Fillet Welds. The effective area of fillet welds shall be considered as the effective length of the weld times the effective throat thickness
- (6) Effective Area of Fillet Welds in Holes and Slots. The effective area of fillet welds in holes and slots shall be computed as specified in (5) for fillet welds using for effective length the length of centerline of the weld through the center of the plane through the throat. However, in the case of overlapping fillets the effective area shall not exceed the nominal cross-sectional area of the hole or slot in the plane of the faying surface.
- (7) Intermittent Fillet Welds. Intermittent fillet welds may be used to transfer calculated stresses across a joint or faying surface, when the strength required is less than that developed by a continuous fillet weld of the smallest permitted size, and to join elements of built-up members. The effective length of any segment of intermittent fillet welding shall not be less than four times the weld size with a minimum of  $1\frac{1}{2}$  in. (38 mm).
- (8) End Returns to Fillet Welds. Side or end fillet welds terminating at ends or sides, respectively, of parts or members shall, when required by the designer, be returned continuously around the corners for a distance not less than twice the nominal size of the weld. This provision shall apply to side and top fillet welds connecting brackets, beam seats, and similar connections, on the

plane about which bending moments are computed. For framing angles and simple end-plate connections which depend upon flexibility of the outstanding legs for connection flexibility, end returns shall not exceed four times the nominal size of the weld. Fillet welds which occur on opposite sides of a common plane shall be interrupted at the corner common to both welds. End returns, when required by the designer, shall be indicated on the drawings.

(9) Fillet Welds in Holes and Slots. Fillet welds in holes or slots may be used to transmit shear in lap joints or to prevent the buckling or separation of lapped parts and to join elements of built-up members. Such fillet welds may overlap, subject to the provisions of (6). Fillet welds in holes or slots are not to be considered plug or slot welds.

#### (e) Plug and Slot Welds

- (1) Use of Plug and Slot Welds. Plug and slot welds may be used to transmit shear in a lap joint or to prevent buckling of lapped parts and to join component parts of built-up members.
- (2) Diameter of Holes for Plug Welds. The diameter of the holes for a plug weld shall be not less than the thickness of the part containing it plus  $\frac{5}{16}$  in. (8 mm), rounded to the next greater odd  $\frac{1}{16}$  in. (1.5 mm), nor greater than the minimum diameter plus  $\frac{1}{8}$  in. (3 mm) or  $2\frac{1}{4}$  times the thickness of the weld metal.
- (3) Spacing of Plug Welds. The minimum center-tocenter spacing of plug welds shall be four times the diameter of the hole.
- (4) Length of Slot Welds. The length of slot for a slot weld shall not exceed 10 times the thickness of the weld. The width of the slot shall be not less than the thickness of the part containing it plus  $^5/_{16}$  in. (8 mm), rounded to the next greater odd  $^1/_{16}$  in. (1.5 mm), nor shall it be greater than  $2^1/_4$  times the thickness of the weld. The ends of the slot shall be semicircular or shall have the corners rounded to a radius not less than the thickness of the part containing it, except for those ends which extend to the edge of the part.
- (5) Spacing of Slot Welds. The minimum spacing of lines of slot welds in a direction transverse to their length shall be four times the width of the slot. The minimum center to center spacing in a longitudinal direction on any line shall be two times the length of the slot.
- (6) Thickness of Plug and Slot Welds. The thickness of plug and slot welds in material  $\frac{5}{8}$  in. (16 mm) or less in thickness shall be equal to the thickness of the material. In material over  $\frac{5}{8}$  in. (16 mm) in thickness, it shall be at least one-half the thickness of the material but not less than  $\frac{5}{8}$  in. (16 mm).
- (7) Effective Shearing Area of Plug and Slot Welds. The effective shearing area of plug and slot welds shall be considered as the nominal cross-sectional area of the hole or slot in the plane of the faying surface.
- (f) Full Penetration and Partial Penetration Joints. The effective area shall be the effective weld length multiplied by the effective throat thickness.

- (1) The effective weld length for any groove weld, square or skewed, shall be the length of weld throughout which the correct proportioned cross section exists. In a curved weld it shall be its true length measured along its curvature.
- (2) The effective throat thickness of a full penetration groove weld which shall conform to the requirements of Article NF-4000 shall be the thickness of the thinner part joined. No increase is permitted for weld reinforcement.
- (3) The effective throat of partial penetration groove welds is dependent upon the type of groove.
- (-a) For square, U, and J groove welds, the effective throat is equal to the depth of preparations.
- (-b) For V and bevel groove welds with an included angle at the root equal to or greater than 60 deg, the effective throat shall be the minimum distance from the root to the face of the weld.
- (-c) For V and bevel groove welds with an included angle at the root less than 60 deg but equal to or greater than 45 deg, the effective throat shall be the minimum distance from the root to the face of the weld less  $\frac{1}{8}$  in. (3 mm).
- (-d) For V and bevel groove welds, with an included angle at the root less than 45 deg but equal to or greater than 30 deg, the effective throat shall be the minimum distance from the root to the face of the weld less  $\frac{1}{8}$  in. (3 mm) and multiplied by 0.75. The required effective throat must be specified on the drawing.
- (-e) For V and bevel groove welds, angles less than 30 deg at the root are not allowed.
- (-f) For flare bevel groove welds, when filled flush to the surface, the effective throat shall be 0.31 times the outside radius of the curved section forming the groove. For formed rectangular tubing, the outside radius may be considered as two times the wall thickness.
- (-g) For flare V groove welds, when filled flush to the surface, the effective throat shall be 0.5 (except use 0.375 for GMAW when  $R \ge \frac{1}{2}$  in (13 mm) times the outside radius when the outside radius is less than 1 in. (25 mm). For flare bevel groove welds, the effective throat shall be 0.312 times the outside radius.
- (-h) Consideration of Lamellar Tearing. Welded joint configurations causing significant through-thickness tensile stress [as defined in NF-1215(b)] during fabrication and/or service on rolled product forms should be avoided. However, if this type of construction is used, the designer should consider one or several of the following factors that may reduce the susceptibility of the joint to experience lamellar tearing and provide documentation, including fabrication requirements, in the Design Output Documents:
- (-1) Reduce volume of weld metal to the extent practical.
- (-2) Select materials that are resistant to lamellar tearing.
- (-3) Invoke any of the special fabrication requirements of NF-4441.

#### NF-3324.6 Design Requirements for Bolted Joints.

The rules and stress limits for bolting shall be as given in this paragraph. The stress limits which must be satisfied for any Design, Levels A through D, and Test Loadings, shall be those given in this paragraph, multiplied by the appropriate stress limit factors given in Table NF-3225.2-1 for the particular Loading specified in the Design Specification (NCA-3250). This product shall not exceed the yield strength of the material at temperature.

- (a) Allowable Stresses. Allowable tensile, shearing, and bearing stresses in bolts and threaded parts shall be as given in the paragraphs below. All allowables are expressed in ksi (MPa) acting on the actual bolt area available in the shear planes of the connected parts. All are expressed in terms of the ultimate tensile strength at temperature (Section II, Part D, Subpart 1, Table U). The shear capacity of bolts is directly proportional to the shear area available in the shear planes. Shear strength is unaffected by shear plane location.
- (1) Tensile Stress Only. Bolts loaded in direct tension shall be so proportioned that their average tensile stress  $F_{tb}$ , computed on the basis of the actual tensile stress area available (independent of any initial tightening force), shall not exceed

For ferritic steels

$$F_{tb} = \frac{S_u}{2}$$

For austenitic steels

$$F_{tb} = \frac{S_u}{3.33}$$

The applied load shall be the sum of the external load and any tension resulting from prying action produced by deformation of the connected parts.

- (2) Shearing Stress Only
  - (-a) Bearing-Type Joints
- (-1) Threads Excluded From Shear Planes. The allowable shear  $F_{vb}$  in bolts and threaded parts loaded in direct shear, expressed in ksi (MPa) of actual shear stress area available (applicable to the total nominal bolt area in the shear planes in this case), shall not exceed

For ferritic steels

$$F_{vb} = \frac{0.62S_u}{3}$$

For austenitic steels

$$F_{vb} = \frac{0.62S_u}{5}$$

(-2) Threads Not Excluded From Shear Planes. The allowable shear stress  $F_{vb}$  in bolts and threaded parts loaded in direct shear, expressed in ksi (MPa) of actual shear stress area available (applicable to the total bolt root area in the shear planes in this case), shall not exceed

For ferritic steels

$$F_{vb} = \frac{0.62S_u}{3}$$

For austenitic steels

$$F_{vb} = \frac{0.62S_u}{5}$$

#### (3) Combined Tensile and Shear Stresses

(-a) Bearing-Type Joints. Bolts subjected to combined shear and tension shall be so proportioned that either the shear or the tensile stress, ksi (MPa) of actual cross-sectional area, shall not exceed the value derived from the ellipse equation below when the corresponding computed tensile or shearing stress is substituted

$$\frac{f_t^2}{F_{th}^2} + \frac{f_v^2}{F_{vh}^2} = 1$$

The allowable tensile and shear stress values shall be those derived from the equations given in (1) and (2).

- (-b) Friction-Type Joints. A bolt in a connection designed as a friction-type joint is not subjected to shear (provided the joint does not slip into bearing); it experiences tension only. Friction-type joints shall be designed as given in (4).
- (4) Slip Resistance Friction-Type Joints. The maximum slip resistance to which a friction-type joint may be designed shall not exceed the value of  $P_s$ , calculated in the following equation [see Table NF-3324.6(a)( $\frac{1}{4}$ )- $\frac{1}{4}$ ]:

$$P_S = mnT_ik_S$$

If the joint clamping force will be reduced by any direct tension load on the joint, the T<sub>k</sub> value shall be reduced by an equivalent amount before substituting in the above equation. SA-307 and austenitic steel bolting shall not be used for friction-type joints.

#### Table NF-3324.6(a)(4)-1 **Effective Slip Coefficient Versus Surface** Condition

Surface Condition	Slip Coefficient, $k_s$
Clean mill scale	0.25
Grit-blasted carbon and low alloy high-strength steel	0.41
Grit-blasted, heat-treated steel	0.25
Hot dip galvanized wire, brushed, scored, or blasted	0.31
Blast cleaned, zinc rich paint	0.31
Blast cleaned, zinc silicate paint	0.45

(5) Bearing Stress. Allowable bearing stress on the projected area of bolts in bearing type connections shall

$$F_p = \frac{LS_u}{2d} \le 1.5S_u$$

#### (b) Minimum Edge Distance

(1) Minimum Edge Distance in Line of Load. In both bearing- and friction-type joints the minimum distance from the center of the end bolt in a connection to that edge of the connected part toward which the load is directed shall be determined in accordance with either (-a) or (-b).

(-a) The edge distance shall satisfy all of the following:

$$\begin{array}{l} f \\ (-1) \ L/d \ge 0.5 + 1.43 \ (f_p/S_u) \\ (-2) \ L/d \ge 1.2 \\ (-3) \ f_p/S_u \le 1.5 \end{array}$$

$$(-2) L/d \ge 1.2$$

$$(-3) f_p/S_u \le 1.5$$

where

$$f_p = P/dt$$

- (-b) The edge distance shall be as shown in Table NF-3324.6(b)(1)-1.
- (2) Minimum Edge Distance in Line of Load for Level C (17) and Level D. For Level C and Level D Loadings, the edge distance shall satisfy all of the following:

$$(-a) L/d \ge 0.5 + 1.2(f_p/S_u)$$

(-b) 
$$L/d \ge 1.2$$

$$(-c) f_p / S_u \le 2.1$$

(c) Maximum Edge Distance. The maximum distance from the center of any rivet or bolt to the nearest edge of parts in contact shall be 12 times the thickness of the

#### Table NF-3324.6(b)(1)-1 Minimum Edge Distances

Minimum Edge Distance for Punched, Reamed, or Drilled Holes, in. (mm)

		At Rolled Edges of Plates,
Bolt Diameter,		Shapes, or Bars, or Gas
in. (mm)	At Sheared Edges	Cut Edges [Note (1)]
<sup>1</sup> / <sub>2</sub> (12.7)	<sup>7</sup> / <sub>8</sub> (22.2)	<sup>3</sup> / <sub>4</sub> (19.1)
<sup>5</sup> / <sub>8</sub> (15.9)	$1\frac{1}{8}$ (28.6)	<sup>7</sup> / <sub>8</sub> (22.2)
<sup>3</sup> / <sub>4</sub> (19.1)	$1^{1}/_{4}$ (31.8)	1 (25.4)
$\frac{7}{8}$ (22.2)	$1\frac{1}{2}$ (38.1) [Note (2)]	$1\frac{1}{8}$ (28.6)
1 (25.4)	1 <sup>3</sup> / <sub>4</sub> (44.5) [Note (2)]	$1\frac{1}{4}$ (31.8)
$1\frac{1}{8}$ (28.6)	2 (50.8)	$1\frac{1}{2}$ (38.1)
$1^{1}/_{4}$ (31.8)	$2^{1}/_{4}$ (57.2)	$1\frac{5}{8}$ (41.3)
>11/4 (31.8)	$1^3/_4 \times diam$ .	$1^{1}/_{4} \times diam$ .

- (1) All edge distances in this column may be reduced  $\frac{1}{8}$  in. (3.2 mm) when the hole is at the point where stress does not exceed 25% of the maximum allowed stress in the
- (2) These may be  $1\frac{1}{4}$  in. (32 mm) at the ends of beam connection angles.

connected part under consideration, but shall not exceed 6 in. (150 mm). Bolted joints in unpainted steel exposed to atmospheric corrosion require special limitations on pitch and edge distance.

For unpainted, built-up members made of weathering steel which will be exposed to atmospheric corrosion, the spacing of fasteners connecting a plate and a shape or two-plate components in contact shall not exceed 14 times the thickness of the thinnest part nor 7 in. (175 mm), and the maximum edge distance shall not exceed eight times the thickness of the thinnest part, or 5 in. (125 mm).

(d) Minimum Spacing. The distance between centers of standard, oversized, or slotted fastener holes shall be not less than three times the nominal diameter of the bolt. Along a line of transmitted force, the distance between centers of holes shall be not less than the following:

#### Standard Holes: $2P/S_ut + d/2$

- (e) Effective Bearing Area. The effective bearing area of bolts shall be the diameter multiplied by the length in bearing, except that for countersunk bolts one-half the depth of the countersink shall be deducted.
- (f) Long Grips. SA-307 bolts, which carry calculated stress and the grip of which exceeds five diameters, shall have their number increased 1% for each additional  $\frac{1}{16}$  in. (1.5 mm) in the grip.
- (g) Anchor Bolts. Anchor bolts shall be designed to provide resistance to all conditions of tension and shear at the bases of columns, including the net tensile components of any bending moments which may result from fixation or partial fixation of columns.

#### NF-3324.7 Design Requirements for Column Bases.

Proper provision shall be made to transfer the column loads and moments, if any, to the footings and foundations.

# NF-3330 HIGH CYCLE FATIGUE DESIGN FOR CLASS 1

#### NF-3331 Introduction

**NF-3331.1 Scope.** Members and their connections, subject to a number of cycles (>20,000) of fatigue loading resulting in damage as defined in NF-3331.2, shall be proportioned to satisfy the stress range limitations provided therein.

**NF-3331.2 Definitions.** High cycle fatigue, as used in this subsubarticle, is defined as the damage that may result in fracture after a sufficient number of fluctuations of stress. *Stress range* is defined as the numerical sum of maximum repeated tensile and compressive stresses or the sum of maximum shearing stresses of opposite direction at a given point, resulting from differing arrangements of live load.

#### NF-3332 Design Requirements

**NF-3332.1 Design Considerations.** In the design of members and connections subject to repeated variation of live load stress, consideration shall be given to the number of stress cycles, the expected range of stress, and the type and location of member or detail.

NF-3332.2 Classification of Loading Conditions. Loading conditions shall be classified as shown in Table NF-3332.2-1.

**NF-332.3 Stress Categories.** Stress categories for use in establishing the maximum range of stress in relation to the type of member or part, material, and location shall be as stipulated in Table NF-3332.3-1. For illustrations of the members and parts, see Figure NF-3332.3-1.

**NF-332.4 Allowable Stresses.** The maximum stress shall not exceed the basic allowable stress permitted by NF-3322 and the maximum range of stress shall not exceed that given in Table NF-3332.4-1.

NF-332.5 Provisions for Mechanical Fasteners. Range in tensile stress in properly tightened ASTM A325 or ASTM A490 bolts need not be considered, but the maximum computed stress, including prying action, shall not exceed the values given in NF-3332.4 subject to the following stipulations:

- (a) Connections subject to more than 20,000 cycles, but not more than 500,000 cycles of direct tension may be designed for the stress produced by the sum of applied and prying loads if the prying load does not exceed 10% of the externally applied load. If the prying force exceeds 10%, the allowable tensile stress given in NF-3324.6(a)(1) shall be reduced 40%, applicable to the external load alone.
- (b) Connections subject to more than 500,000 cycles of direct tension may be designed for the stress produced by the sum of applied and prying loads if the prying load does not exceed 5% of the externally applied load. If the

#### Table NF-3332.2-1 Classification of Loading Conditions

Loading	Number of Loading Cycles		
Condition	From	To	
1	20,000 [Note (1)]	100,000 [Note (2)]	
2	100,000	500,000 [Note (3)]	
3	500,000	2,000,000 [Note (4)]	
4	>2.000.000		

#### NOTES

- (1) Approximately equivalent to two applications every day for 25 yr.
- (2) Approximately equivalent to ten applications every day for 25 yr.
- (3) Approximately equivalent to 50 applications every day for 25 vr
- (4) Approximately equivalent to 200 applications every day for 25 yr.

	IF-3332.3-1 Categories		
Situation	Kind of Stress [Note (1)]	Stress Category (See Table NF-3332.4-1)	Illustrative Example [Note (2)] Sketch Nos. (See Figure NF-3332.3-1)
General Condition — Plain material			
Base metal with rolled or cleaned surfaces	T or Rev.	A	(1), (2)
General Condition — Built-up members			<u></u>
Base metal and weld metal in members, without attachments, built up plates or shapes connected by continuous full or partial penetration groove welds, or continuous fillet welds parallel to the direction of applied stress	T or Rev.	В	(3) through (6)
Base metal in members without attachments, built-up plates or shapes connected by continuous full-penetration groove welds or by continuous fillet welds parallel to the direction of applied stress	T or Rev.	В′	(3) through (6)
Calculated flexural stress, $f_b$ , in base metal at toe of welds on girder webs or flanges adjacent to welded transverse stiffeners	T or Rev.	C	(7)
Base metal at ends of partial length welded cover plates narrower than the flange having square or tapered ends, with or without welds across the ends or wider than flange with welds across the ends		if ASM.	
Flange thickness ≤ 0.8 in. (20 mm) Flange thickness > 0.8 in. (20 mm)	T or Rev. T or Rev.	E E'	(5) (5)
Base metal at end of partial length welded cover plates wider than the flange without welds across the ends	ري (۲	E'	(5)
General Condition — Mechanically fastened connections	N		
Base metal at gross section of high-strength bolted slip-critical connections, except axially loaded joints which induce out-of-plane bending in connected material	T or Rev.	В	(8)
Base metal at net section of other mechanically fastened joints [Note (3)]	T or Rev.	D	(8), (9)
Base metal at net section of fully tensioned high-strength, bolted-bearing connections	T or Rev.	В	(8), (9)
General Condition — Fillet welded connections			
Base metal at intermittent fillet welds	T or Rev.	E	
Base metal at junction of axially loaded members with fillet-welded end connections. Welds shall be disposed about the axis of the member so as to balance weld stresses.			
$b \le 1$ in. (25 mm)	T or Rev.	Е	(17), (18)
<i>b</i> > 1 in. (25 mm)	T or Rev.	E'	(17), (18)
Base metal at members connected with transverse fillet welds $b \le \frac{1}{2}$ in. (13 mm) $b > \frac{1}{2}$ in. (13 mm)	T or Rev.	C [Note (4)]	(20), (21) (20), (21)
General Condition — Fillet welds			(20), (21)
Weld metal of continuous or intermittent longitudinal or transverse fillet welds	S	F [Note (4)]	(15), (17), (18), (20), (21)
General Condition — Groove welds			
Base metal and weld metal at full penetration groove welded splices of parts of similar cross section ground flush, with grinding in the direction of applied stress and with weld soundness established by radiographic or ultrasonic inspection in accordance with the requirements of Table 9.25.3 of AWS D1.1-77	T or Rev.	В	(10)
Base metal and weld metal at full penetration groove welded splices at transitions in width or thickness, with welds ground to provide slopes no steeper than 1 to $2^1\!/_2$ , with grinding in the direction of applied stress, and with weld soundness established by radiographic or ultrasonic inspection in accordance with the requirements of Table 9.25.3 of AWS D1.1-77	T or Rev.	В	(12), (13)

	NF-3332.3-1 egories (Cont'o	1)	
Situation	Kind of Stress [Note (1)]	Stress Category (See Table NF-3332.4-1)	Illustrative Example [Note (2)] Sketch Nos. (See Figure NF-3332.3-1)
General Condition — Groove welds (Cont'd)			7:2
Base metal and weld metal at full penetration groove welded splices, with or without transitions having slopes no greater than 1 to $2\frac{1}{2}$ , when reinforcement is not removed and with weld soundness established by radiographic or ultrasonic inspection in accordance with the requirements of Table 9.25.3 of AWS D1.1-77	T or Rev.	С	(10) through (13)
General Condition — Partial penetration groove welds			50
Weld metal of partial penetration transverse groove welds, based on effective throat area of the weld or welds	T or Rev.	F [Note (4)]	(16)
General Condition — Plug or slot welds		- C2X	
Base metal at plug or slot welds	T or Rev.	E	(27)
Shear on plug or slot welds	S	R	(27)
General Condition — Attachments		\S'.	
Base metal at details attached by full-penetration groove welds subject to longitudinal and/or transverse loading when the detail embodies a transition radius <i>R</i> with the weld termination ground smooth and for transverse loading, the weld soundness established by radiographic or ultrasonic inspection in accordance with 9.25.2 or 9.25.3 of AWS D1.1  Longitudinal loading	,C,III.	N. C.	
R > 24 in. (600 mm)	Tor Rev.	В	(14)
24 in. (600 mm) > R > 6 in. (150 mm)	Tor Rev.	С	(14)
6 in. (150 mm) > R > 2 in. (50 mm) 2 in. (50 mm) > R  Detail base metal for transverse loading: equal thickness and	T or Rev. T or Rev.	D E	(14) (14)
reinforcement removed		_	
R > 24 in. (600 mm) 24 in. (600 mm) > R > 6 in. (150 mm)	T or Rev. T or Rev.	B C	(14) (14)
6 in. (150 mm) > R > 2 in. (50 mm)	T or Rev.	D	(14)
2 in. (50 mm) > R  Detail base metal for transverse loading: equal thickness and reinforcement not removed	T or Rev.	Е	(14), (15)
R > 24 in. (600 mm)	T or Rev.	С	(14)
24 in. (600 mm) > R > 6 in. (150 mm)	T or Rev.	C	(14)
6 in. (150 mm) > R > 2 in. (50 mm) 2 in. (50 mm) > R  Detail base metal for transverse loading: unequal thickness and reinforcement removed	T or Rev. T or Rev.	D E	(14) (14), (15)
R > 2 in. (50 mm)	T or Rev.	D	(14)
2 in. (50 mm) > R Detail base metal for transverse loading: unequal thickness and reinforcement not removed	T or Rev.	Е	(14), (15)
all R  Detail base metal for transverse loading	T or Rev.	Е	(14), (15)
R > 6 in. (150 mm)	T or Rev. T or Rev.	C D	(19)
6 in. (150 mm) > R > 2 in. (50 mm) 2 in. (50 mm) > R	T or Rev.	E E	(19) (19)
Base metal at detail attached by full-penetration groove welds subject to longitudinal loading  2 < a < 12b or 4 in. (100 mm)	T or Rev.	D	(15)
$a > 12b$ or 4 in. (100 mm) when $b \le 1$ in. (25 mm)	T or Rev.	Е	(15)
a > 12b or 4 in. (100 mm) when $b > 1$ in. (25 mm)  Base metal at detail attached by fillet welds or partial-penetration groove welds subject to longitudinal loading	T or Rev.	E'	(15)
$a \le 2$ in. (50 mm)	T or Rev.	С	(15), (23) through (26)

Table NF-3332.3-1	
Stress Categories (Cont'd	)

	-		
Situation	Kind of Stress [Note (1)]	Stress Category (See Table NF-3332.4-1)	Illustrative Example [Note (2)] Sketch Nos. (See Figure NF-3332.3-1)
General Condition — Attachments (Cont'd)			
2 in. (50 mm) $< a \le 12b$ or 4 in. (100 mm) $a > 12b$ or 4 in. (100 mm) when $b \le 1$ in. (25 mm) a > 12b or 4 in. (100 mm) when $b > 1$ in. (25 mm)	T or Rev. T or Rev. T or Rev.	D E E'	(15), (23), (24), (26) (15), (23), (24), (26) (15), (23), (24), (26)
Base metal attached by fillet welds or partial-penetration groove welds subjected to longitudinal loading when the weld termination embodies a transition radius with the weld termination ground smooth: $R > 2 \text{ in. (50 mm)}$ $R \le 2 \text{ in. (50 mm)}$	T or Rev. T or Rev.	D E	Caso Section
Fillet-welded attachments where the weld termination embodies a transition radius, weld termination ground smooth, and main material subject to longitudinal loading:  Detail base metal for transverse loading $R > 2$ in. (50 mm) $R \le 2$ in. (50 mm)	T or Rev. T or Rev.	L E ASME	(19) (19)
Base metal at stud-type shear connector attached by fillet weld or automatic end weld	T or Rev.	1,7, c	(22)
Shear stress on nominal area of stud-type shear connectors	S	F	

#### NOTES:

- (1) T signifies range in tensile stress only; Rev. signifies a range involving reversal of tensile or compressive stress; S signifies range in shear, including shear stress reversal.
- (2) These examples are provided as guidelines and are not intended to exclude other reasonably similar situations.
- (3) When stress reversal is involved, use of SA-307 bolts is not recommended.
- (4) Allowable fatigue stress range for transverse partial penetration and transverse fillet welds is a function of the effective throat, depth of penetration and plate thickness. See Frank and Fisher, "Fatigue Strength of Fillet Welded Cruciform Joints," Journal of the Structural Division, ASCE, Vol. 105, No. 519, 1979.

prying force exceeds 5%, the allowable tensile stress given in NF-3324.6(a)(1) shall be reduced 50%, applicable to the external load alone. The use of other bolts and threaded parts subject to tensile fatigue loading is not recommended. Bolts and threaded parts subjected to cyclic loading in shear may be designed for the bearing-type shear stresses given in NF-3324.6(a)(2) insofar as the fatigue strength of the fasteners is concerned.

# NF-3340 LIMIT ANALYSIS DESIGN FOR CLASS 1 NF-3341 Introduction

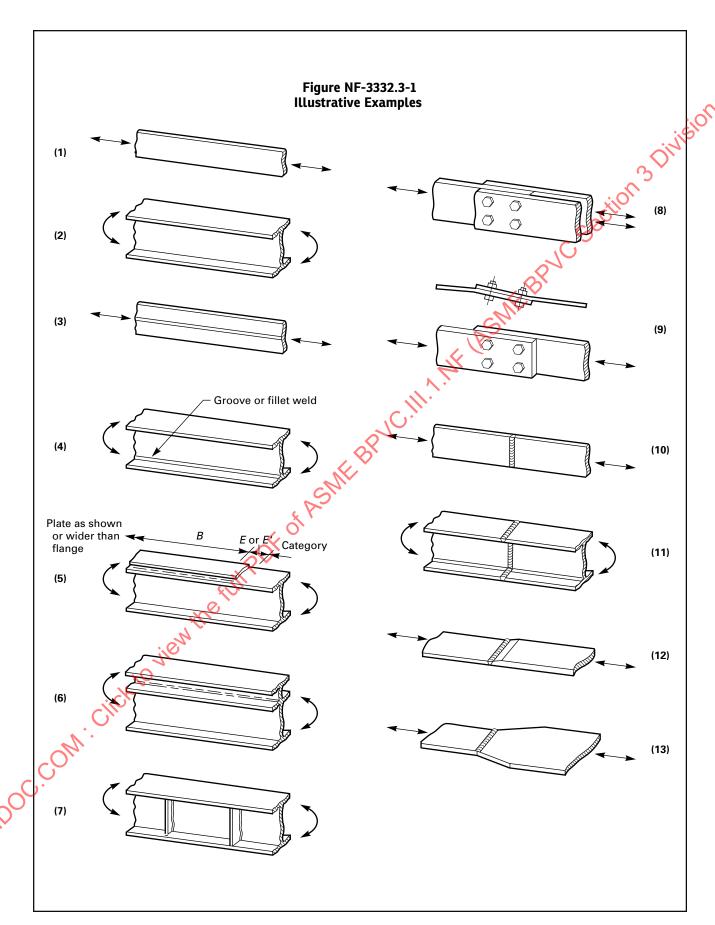
#### (17) NF-3341.1 Scope.

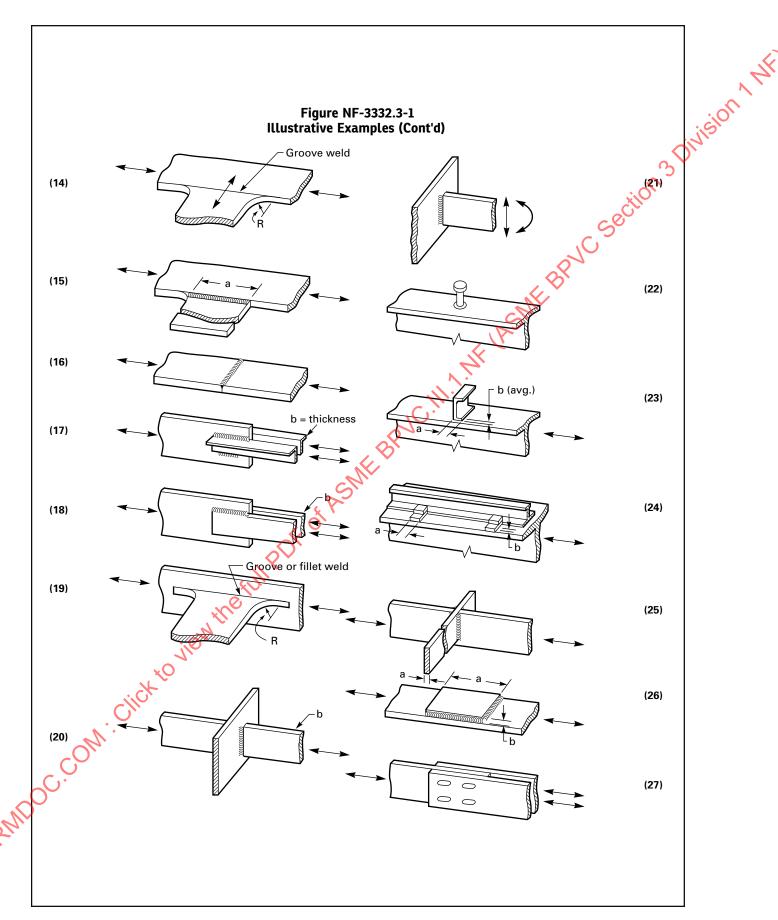
(a) Subject to the limitation of this subsubarticle, simple or continuous beams, rigid frames, and similar portions of structures rigidly constructed so as to be continuous over at least one interior support may be proportioned on the basis of plastic design, namely, on the basis of the lower bound collapse load. This strength, as determined by rational analysis, shall not be less than that required to support a factored load equal to 1.7 times those of the Level A and Level B Service Limits, 1.3 times

that of the Level C Service Limits, or 1.1 times that of the Level D Service Limits. For Service Level D, the lower bound collapse load may be determined using a yield stress in the analysis that is the greater of  $1.2S_y$  and  $1.5S_m$ , but not larger than  $0.7S_u$ .

NOTE: As used here, *interior support* may be taken to include a rigid frame knee formed by the junction of a column and a sloping or horizontal beam or girder.

- (b) Connections joining a portion of a structure designed on the basis of plastic behavior with a portion not so designed need be no more rigid than ordinary seat and cap angle or ordinary web connections.
- (c) When plastic design is used as the basis for proportioning continuous beams and structural frames, the provisions relating to allowable working stress contained in NF-3320 are waived. Except as modified by these rules, all other provisions of NF-3320 shall govern.





#### Table NF-3332.4-1 Allowable Stress Ranges

_		Allowable Range of	Stress, <i>F<sub>sr</sub></i> , ksi (MPa)	
Stress Category Used in Table	Loading Condition 1,	Loading Condition 2,	Loading Condition 3,	Loading Condition 4,
NF-3332.3-1	$F_{sr1}$	$F_{sr2}$	$F_{sr3}$	F <sub>sr4</sub>
A	63 (430)	37 (255)	24 (165)	24 (165)
В	49 (340)	29 (200)	18 (125)	16 (110)
B'	39 (270)	23 (160)	15 (100)	12 (85)
С	35 (240)	21 (145)	13 (90)	10 (70) [Note (1)]
D	28 (195)	16 (110)	10 (70)	7 (50)
E	22 (150)	13 (90)	8 (55)	5 (35)
E'	16 (110)	9 (60)	6 (40)	3 (20)
F	15 (100)	12 (85)	9 (60)	8 (55)

NOTE

(1) Flexural stress range of 12 ksi (83 MPa) permitted at toe of stiffener welds on webs or flanges.

#### NF-3342 Design Requirements

#### NF-3342.1 General Design Requirements.

(a) Shear. Unless reinforced by diagonal stiffeners or a doubler plate, the webs of columns, beams, and girders, including areas within the boundaries of the connections, shall be so proportioned that

$$V_{u} \leq 0.555S_{y}td \tag{1}$$

where  $V_u$  is the shear, kips (N), that would be produced by the required factored loading.

- (b) Combined Loads. When the interaction method of calculation (Section III Appendices, Nonmandatory Appendix A, Article A-9000) is used, members shall be so proportioned that the interaction of factored loads at a section shall satisfy the interaction equations of Section III Appendices, Nonmandatory Appendix A, Table A-9210 (d)-1, except that columns subjected to combined bending and compression shall be governed by the requirements of NF-3342.2(b)(3).
- (c) Depth-Thickness Ratio of Webs. The depth-thickness ratio of webs of members subjected to plastic bending shall not exceed the value given by eqs. (2) and (3), as applicable

(U.S. Customary Units)

$$\frac{d}{t} = \frac{412}{\sqrt{S_y}} \left[ 1 - \left( 1.4 \frac{P}{P_y} \right) \right] \tag{2}$$

(SI Units)

$$\frac{d}{t} = \frac{1080}{\sqrt{S_y}} \left[ 1 - \left( 1.4 \frac{P}{P_y} \right) \right] \tag{2}$$

 $\frac{P}{P_v} \leq 0.27$ 

(U.S. Customary Units)

$$\frac{d}{t} = \frac{257}{\sqrt{S_V}} \tag{3}$$

when

$$\frac{P}{P_v} > 0.27$$

(SI Units)

when

$$\frac{d}{t} = \frac{673}{\sqrt{S_v}} \tag{3}$$

$$\frac{P}{P_V} \leq 0.27$$

$$\frac{P}{P_y} > 0.27$$

(d) Web Crippling. Web stiffeners are required on a member at a point of load application where a plastic hinge would form. At points on a member where the concentrated load delivered by the flanges of a member framing into it would produce web crippling opposite the compression flange or high tensile stress in the tension flange, web stiffeners are required in accordance with the provisions of NF-3324.2(b).

#### (e) Width-Thickness Ratios of Flanges

(1) The width-thickness ratio for flanges of rolled W shapes and similar built-up single web shapes that would be subjected to compression involving hinge rotation under ultimate loading shall not exceed the following values:

$S_{\nu}$	$b_f/2t_f$
36 (248)	8.5
42 (290)	8.0
45 (310)	7.4
50 (345)	7.0
55 (380)	6.6
60 (413)	6.3
65 (448)	6.0

The thickness of sloping flanges may be taken as their average thickness.

(2) The width-thickness ratio of similarly compressed flange plates in box sections and cover plates shall not exceed  $190/\sqrt{S_y}$  ( $500/\sqrt{S_y}$ ). For this purpose the width of a cover plate shall be taken as the distance between longitudinal lines of connecting high-strength bolts or welds.

#### NF-3342.2 Specific Design Requirements.

(a) Beams. The maximum bending strength of a flexural member shall be

$$M_p = S_y Z_\chi \tag{4}$$

(b) Columns

(1) In the plane of bending of columns which would develop a plastic hinge at ultimate loading, the slenderness ratio l/r shall not exceed  $C_c$  defined in NF-3322.1(c).

(2) The maximum strength of an axially loaded compression member shall be taken as

$$P_{cr} = 1.7AF_a \tag{5}$$

where A is the gross area of the member, and  $F_a$ , as defined by eq. NF-3322.1(c)(1)(-a)(4), is based upon the applicable slenderness ratio.

(3) Members subject to combined axial load and bending moment, including moment resulting in secondary stress, shall be proportioned so as to satisfy the following interaction equations:

$$\frac{P}{P_{cr}} + \frac{C_m M}{\left[1 - \left(P/P_e\right)\right] M_m} \le 1.0 \tag{6}$$

$$\frac{P}{P_y} + \frac{M}{1.18M_p} \le 1.0; M \le M_p$$
 (7)

where

 $C_m$  = coefficient defined in NF-3322.1(e)  $P_e$  = 1.92 $AF'_e$ , where  $F'_e$  is defined in NF-3322.1(e)

The right-hand side of eq. (7) may be replaced by 1.5 when it can be shown that the bending moment results only in secondary stress.

(4) For columns braced in the weak direction

$$M_m = M_p$$

(5) For columns unbraced in the weak direction

(U.S. Customary Units)

$$M_m = M_p \left[ 1.070 \frac{(1/r_y)\sqrt{S_y}}{3,160} \right] \le M_p$$
 (8)

(SI Units)

$$M_m = M_p \left[ 1.07 - \frac{(l/r_y)\sqrt{S_y}}{8280} \right] \le M_p$$
 (8)

(c) Basis for Maximum Strength Determination. For one or two story frames, the maximum strength may be determined by a routine plastic analysis procedure, and the frame instability effect  $P\Delta$  may be ignored. For braced multistory frames, provisions should be made to include the frame instability effect in the design of the bracing system and frame members. For unbraced multistory frames, the frame instability effect should be included directly in the calculations for maximum strength. The vertical bracing system for a plastically designed braced multistory frame shall be shown to be adequate, as determined by a rational analysis, to prevent buckling of the structure under factored gravity load, and maintain the lateral stability of the structure, including consideration of the overturning effects of drift, under factored horizontal and gravity loads.

(1) Stability of Braced Frames. The vertical bracing system may be considered to function together with inplane shear-resisting exterior and interior walls, floor slabs, and roof decks, if these walls, slabs, and decks are secured to the structural frames. The columns, girders, beams, and diagonal members, when used as the vertical bracing systems, may be considered to comprise a cantilever, simply connected truss in the analysis for frame buckling and lateral stability. Axial deformation of all members in the vertical bracing system shall be included in the lateral stability analysis. The axial force in these members, caused by factored horizontal and gravity loads, shall not exceed  $0.85P_y$ , where  $P_y$  is the product of yield stress times area of the member. Girders and

beams included in the vertical bracing system of a braced multistory frame shall be proportioned for axial force and moment caused by the concurrent factored horizontal and gravity loads, in accordance with eq. (b)(2)(5) with  $P_{cr}$  taken as the maximum axial strength of the beam, based on the actual slenderness ratio between braced points in the plane of bending.

(2) Stability of Unbraced Frames. The strength of an unbraced multistory frame shall be determined by a rational analysis which includes the effects of frame instability and column axial deformation. Such a frame shall be designed to be stable under the following conditions:

- (-a) factored gravity loads
- (-b) factored gravity plus factored horizontal loads

The axial force in the column at factored load levels shall not exceed  $0.75P_{\nu}$ .

#### (d) Lateral Bracing

(1) Members shall be adequately braced to resist lateral and torsional displacements at the plastic hinge locations associated with the failure mechanism. The laterally unsupported distance  $l_{cr}$  from such braced hinge locations to similarly braced adjacent points on the member or frame shall not exceed the value determined from eq. (9) or eq. (10), as applicable

$$l_{cr}/r_y = 1,375/S_y + 25$$

(SI Units)

$$l_{cr}/r_y = 9470/S_y + 25$$
 (9)

when

$$+1.0 > M' / M^2 > -0.5$$

or

(U.S. Customary Units)

$$l_{cr}/r_y = 1,375/S_y$$
 (10)

(SI IInits)

$$l_{cr}/r_y = 9 \, 470 \, / S_y \tag{10}$$

when

$$-0.5 \ge M' / M_p > -1.0$$

where

 $M'/M_p$  = the end moment ratio, positive when the segment is bent in reverse curvature and negative when bent in single curvature

- (2) The provisions of (1) need not apply in the region of the last hinge to form, in the failure mechanism assumed as the basis for proportioning a given member, nor in members oriented with their weak axis normal to the plane of bending. However, in the region of the last hinge to form and in regions not adjacent to a plastic hinge, the maximum distance between points of lateral support shall be such as to satisfy the requirements of eq. NF-3322.1(d)(5)(-b)(17), eq. NF-3322.1(d)(5)(-b)(18), eq. NF-3322.1(d)(5)(-b)(19), eq. NF-3322.1(e)(1)(20), or eq. NF-3322.1(e)(1)(21). For this case, the value of  $f_a$  and  $f_b$  shall be computed from the moment and axial force at factored loading, divided by the applicable load factor.
- (3) Members built into a masonry wall and having their web perpendicular to this wall can be assumed to be laterally supported with respect to their weak axis of bending.

#### (e) Connections

- (1) General Requirements. All connections, the rigidity of which is essential to the continuity assumed as the basis for the analysis, shall be capable of resisting the moments, shears, and axial loads to which they would be subjected by the full factored loading, or any probable partial distribution thereof.
- (2) Corner Connections. Corner connections (haunches) that are tapered or curved for architectural reasons shall be so proportioned that the full plastic bending strength of the section adjacent to the connection can be developed, if required.
- (3) Stiffeners. Stiffeners shall be used, as required, to preserve the flange continuity of interrupted members at their junction with other members in a continuous frame. Such stiffeners shall be placed in pairs on opposite sides of the web of the member which extends continuously through the joint.
- (4) Stress in Bolts and Welds. High-strength bolts, SA-307 bolts, and welds shall be proportioned to resist the forces produced at factored load, using stresses equal to 1.7 times those given in NF-3324.6(a) for bolts and Table NF-3324.5(a)-1 for welds. In general, groove welds are preferable to fillet welds, but their use is not mandatory.
- (5) Joints With Painted Contact Surfaces. Highstrength bolts may be used in joints having painted contact surfaces when these joints are of such size that the slip required to produce bearing would not interfere with the formation, at factored loading, of the plastic hinges assumed in the design.

# NF-3350 DESIGN BY ANALYSIS FOR CLASS 2 AND MC

The design by analysis of Class 2 and MC supports shall be in accordance with NF-3320 and NF-3340.

#### NF-3360 DESIGN BY ANALYSIS FOR CLASS 3

The design by analysis of Class 3 supports shall be in accordance with NF-3320 and NF-3340.

#### (17) NF-3370 EXPERIMENTAL STRESS ANALYSIS

Supports of all types may be designed by experimental stress analysis in accordance with Section III Appendices, Mandatory Appendix II. When experimental stress analysis is used, the appropriate stress acceptance criteria and other limits of NF-3300 shall apply. For Level D Service Limits, the static or equivalent static loads shall not exceed 100% of the test collapse load.

# NF-3380 DESIGN BY LOAD RATING NF-3381 Procedure for Load Rating

The procedure for load rating shall consist of imposing a total load on one or more duplicate full-size samples for a support equal to or less than the load under which the support fails to perform its required function. Full-size samples composed of various parts may have each part or a number of parts load rated provided that all parts in the load path are either load rated or otherwise qualified per NF-3300 or by experimental stress analysis. When parts are connected by bolting or welding, the connection shall be either load rated or qualified per NF-3225 or NF-3226. Should more than one part be load rated in a single load test, then the load rating equations of NF-3380 shall be evaluated for each part using the part's  $S_{y(act)}$  and  $S_{u(act)}$  values. The part having the lowest load rating shall establish the load rating for the combination of parts. A single test sample is permitted, but in that case, the load ratings shall be decreased by 10%. Otherwise, tests shall be run on a statistically significant number of samples. The permissible types of welded joints shall be as permitted for the specific class of construction in NF-3226.1, NF-3256.1, NF-3266, and NF-3324. The requirements of NF-3324.5(b) for size of fillet and partial penetration welds do not apply, provided the full-size sample is fabricated for testing using the minimum weld stipulated in the Design Drawings. Bolted joints in the test sample shall be made up using the lowest strength bolt material and minimum edge distance allowed by the specification.

#### (17) NF-3382 Load Ratings in Relation to Design Service and Test Loadings

The load rating for Design Loadings shall be determined in accordance with the requirements for Service Level A limits. The load ratings for Service Loadings for which Level A through D Limits have been designated shall be determined by means of the equations in the following subparagraphs. The load rating for Test Loadings shall be determined in accordance with the requirements for Service Level B limits.

**NF-3382.1 Nomenclature.** The symbols used in this paragraph are defined as follows:

 $F_{\text{all}}$  = allowable value for the type of stress in NF-3310, ksi (MPa)

Kl = load rating coefficient for supports in compression

 $S_T$  = specified minimum tensile strength of the material used in the support as given in the applicable tables of Section II, Part D, Subpart 1, ksi (MPa)

 $S_u = \text{see NF-3313.1}$ 

 $S_{u(act)}$  = actual tensile strength of the material used in the part or support which had reached ultimate capacity during the test, ksi (MPa)

 $S_Y$  = specified minimum yield strength of the material used in the support as given in the applicable tables of Section II, Part D, Subpart 1, ksi (MPa)

 $S_y = \text{see NF-3313.1}$ 

 $S_{y(act)}$  = actual yield strength of the material used in the part or support which yielded during the test, ksi (MPa)

 $TL_u$  = support test load at which a substantial increase in load displacement results in zero or negative increase in actual support load  $TL_v$  = support test load at or below yield

**NF-3382.2 Linear-Type Supports.** The load ratings for Linear-Type Supports for the Service Loadings shall be determined by the following equations:

Design and Level A Limits (lower of the two values)

load rating<sub>(yield)</sub> = 
$$TL_y \times 1.0 \frac{F_{all}}{S_y(act)}$$
 (1)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 1.0 \frac{F_{all}}{S_{u[act]}}$$
 (2)

Test and Level B Limits (lower of the two values)

load rating<sub>(yield)</sub> = 
$$TL_y \times 1.33 \frac{F_{all}}{S_{y(act)}}$$
 (3)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 1.33 \frac{F_{all}}{S_u(act)}$$
 (4)

Level C Limits (lower of the two values)

load rating<sub>(yield)</sub> = 
$$TL_y \times 1.5 \frac{F_{all}}{S_y(act)}$$
 (5)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 1.5 \frac{F_{all}}{S_{u[act]}}$$
 (6)

Level D Limits (lowest of the three values)

load rating(yield) = 
$$TL_y \times 2.0 \frac{F_{\text{all}}}{S_y(\text{act})}$$
 (7)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 0.7 \frac{S_u}{S_u(act)}$$
 (8)

load rating<sub>(ult.)</sub> = 
$$TL_u \times 0.7$$
 (9)

**NF-3382.3 Supports Loaded in Compression.** The load rating for the supports loaded in compression shall be determined for Service Level A through D by the methods of NF-3382.2. In addition, for cases where buckling governs,  $TL_u$  shall be corrected for the ratio of Young's Modulus at the Design Temperature to Young's Modulus at the test temperature. The load rating shall be determined by the following equations:

Design and Level A Limits

load rating = 
$$0.52Kl \times TL_u$$
 (10)

Test and Level B Limits

load rating = 
$$0.67Kl \times TL_u$$
 (11)

Level C Limits

load rating = 
$$0.67Kl \times TL_u$$

Kl = 1.0 for elastic buckling

$$KI = \frac{S_y}{S_y(act)}$$
 for inelastic buckling

Level D Limits

load rating 
$$\stackrel{>}{=} 0.67 \text{Kl} \times TL_u$$
 (13)

NF-3382.4 Alternate Load Rating Method Using  $TL_y$  or  $TL_u$  Test Results Only. Either  $TL_y$  or  $TL_u$  test results may be used to establish a support load rating provided the load rating as determined by the method of NF-3382.2 or NF-3382.3 is modified as follows:

- (a) For load rating based on  $TL_{\nu}$  test results only
- (1) for supports constructed of materials having definitely determinable yield points and specified minimum  $S_Y/S_T$  ratios not greater than 0.625, the load rating shall be multiplied by 0.83. If  $S_{y(act)}$  of the material is not known, 0.67 shall be used in place of multiplier 0.83
- (2) for supports constructed of carbon steel with specified minimum tensile strengths of not over 70,000 psi (480 MPa),  $S_{y(act)}$  shall be increased by 5,000 psi (34 MPa) prior to determining the load rating

(3) for all other materials the determined load rating shall be multiplied by 0.67

For supports loaded in compression, a  $TL_u$  test shall be performed with the compressive load rating determined in accordance with NF-3382.3.

(b) For load rating based on  $TL_u$  test results, the determined load rating shall be multiplied by 0.50.

# NF-3400 DESIGN RULES FOR STANDARD SUPPORTS

#### NF-3410 GENERAL REQUIREMENTS

The following requirements apply in addition to the requirements of NF-3100, NF-3200, and NF-3300.

#### NF-3411 General Design Requirements

Standard Supports (NF 1214) shall conform to the design considerations of NF-3120, the design rules of NF-3200 or NF-3300, and the following additional requirements.

NF-3411.1 Standard Supports Used as Component Supports. Standard Supports may be used as component supports as defined in NF-3500.

NF-3411.2 Standard Supports Used as Piping Supports. Standard Supports may be used as piping supports, as defined in NF-3600.

#### NF-3411.3 Loads.

- (a) Standard Supports shall be capable of carrying all concurrently acting loads listed in NF-3111 and stipulated in the Design Specification. The supports shall be designed to provide the required supporting effort and to permit the designed piping or component movement.
- (b) Spring supports shall be capable of exerting a supporting force equal to the load, as determined by weight-balance calculations, plus the weight of all hanger parts such as clamps and rods that will be supported by a spring at the point of attachment to a pressure-retaining component or to an integral attachment.
- (1) The design shall be such as to prevent complete release of the piping or component load in the event of spring failure or misalignment.
- (2) When springs are to be enclosed in spaces where high ambient temperatures occur, the spring material shall be such that the spring constants will not be significantly changed at the temperature to be experienced.
- (c) Snubbers and dampers shall meet the requirements of (a), and (1) and (2) below.
- (1) The following occasional loads shall be considered:
  - (-a) seismic inertial
  - (-b) seismic anchor displacement
- (-c) hydraulic transient loads resulting from, but not limited to, water hammer, steam hammer, pump startup, pump shutdown, and safety and safety relief valve discharges, as specified in the Design Specification

(2) Thermal Expansion and Thermal Anchor Displacement. The snubber and damper shall not resist the effects of thermal growth of the component, the piping system, or the anchorage to the degree where it imposes a significant load or stress on the piping or component.

#### NF-3412 Standard Support Items

**NF-3412.1 Constant Support Spring Hangers.** Constant support spring hangers may be used to support piping or components at those locations requiring a substantially uniform supporting force throughout the travel range but which, while subject to thermal movement, cannot tolerate appreciable variability of the supporting force. They shall have a mean variability (deviation), including friction, of no more than 6% throughout the travel range. Deviation is the sum of kinematic friction and manufacturing tolerance factors. Determination of deviation is by load test machine and is calculated as follows:

deviation = (maximum reading moving down

- minimum reading moving up)
- + (maximum reading moving down
- + minimum reading moving up)

**NF-3412.2 Variable Support Spring Hangers.** Variable support spring hangers may be used to support piping or components at those locations subject to vertical movement due to temperature differences. The variability factor of the supporting force is calculated by the following equation:

variability factor =  $(travel \times spring rate)$ 

The recommended maximum variability factor is 0.25.

- (a) The variability factor of the supporting force resulting from movement of the piping or component shall be considered in the loadings used in the stress analysis of the piping or component.
- (b) Variable support spring hangers shall be provided with means to limit misalignment, buckling, and eccentric loading and to prevent overstressing of the spring.
- (c) It is recommended that all hangers employing springs be provided with means to indicate at all times the compression of the spring with respect to the appropriate hot and cold positions of the piping or component.
- **NF-3412.3 Hanger Rods.** Design loads for threaded hanger rods shall be based on the root area of the threads. In no case shall hanger rods less than  $\frac{3}{8}$  in. (10 mm) diameter be used for supporting pipe NPS 2 (DN 50) and smaller, or less than  $\frac{1}{2}$  in. (13 mm) diameter rod for supporting pipe NPS  $2^{1}/_{2}$  (DN 65) and larger. Pipe, structural shapes, or bars may be used instead of hanger rods. Hanger rods, structural shapes, etc., shall be designed to permit the free movement of piping as indicated in the thermal stress analysis. The possibility of moment loading of

hanger rods or shapes as a result of pipe motion shall be considered and avoided unless the support is specifically designed for such loading.

#### NF-3412.4 Snubbers.<sup>6</sup>

- (a) Snubbers may be incorporated in the system design to accommodate Design Mechanical Loads or conditions of a vibratory or dynamic nature. Snubbers may be attached to a piping or component to protect it against dynamic type loading. Snubbers shall allow essentially free movement of the piping or component to which they are attached during nondynamic application of load such as that imposed by expansion and contraction.
- (b) Snubbers shall be carefully applied to ensure that they will perform their intended function without placing unacceptable loads on the piping system or other components.
- (c) The Design Specification shall contain, as a minimum, the following:
- (1) the Design Loadings for loading conditions and transients, and combinations of loadings for which the snubber is designed to accommodate
- (2) required force, time, and displacement relationship
- (3) the environmental conditions that the snubber will be exposed to, such as
  - temperature
  - irradiation
  - corrosive atmosphere
  - moisture
  - airborne particles
  - (4) consideration of material characteristics, such as
    - compatibility
    - stability
    - · fire resistance
    - wear
    - aging
  - (5) tests which are required prior to installation
- (d) Design of functional members such as interconnections, tubing and fittings reservoirs, and flow distributors shall consider the effect of internal pressure, thermal expansion, and vibration loading.
- **NF-3412.5 Dampers.** Dampers provide similar system behavior as snubbers and can be used to mitigate dynamic events. Dampers shall meet the requirements of NF-3412.4.

# NF-3420 DESIGN BY ANALYSIS FOR CLASS 1 NF-3421 General Design Requirements

The design of Class 1 Standard Supports shall be in accordance with the requirements of NF-3420 using one of the design procedures indicated in Table NF-3131(a)-1 for Class 1 construction.

# NF-3422 Design of Plate- and Shell-Type Standard Supports

The requirements of NF-3200 shall be met.

# NF-3423 Design of Linear-Type Standard Supports

The requirements of NF-3300 shall be met.

#### NF-3425 Design of Bolting

The requirements of NF-3225 and NF-3324.6 shall be met.

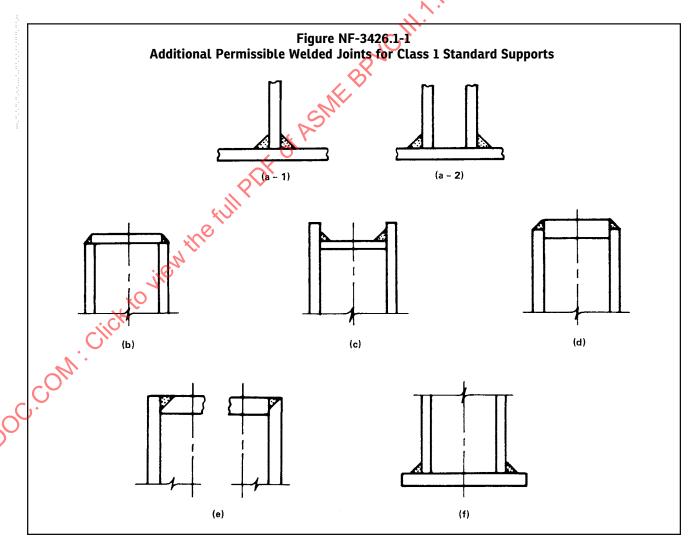
#### NF-3426 Design of Welded Joints

**NF-3426.1 Permissible Types.** The permissible types of welded joints for Standard Supports are described in paragraphs NF-3226.1 and NF-3256.1. Additional types of joints are as follows. Typical examples of the types are shown in Figure NF-3426.1-1.

- (a) Tee Joints. Tee joints shall be one of the following:
  - (1) fillet, double welded, sketch (a-1).
- (2) fillet, single welded, when double members are used, sketch (a-2).
- (3) fillet, single welded, between a flat surface and the end surface of a closed tubular section or a closed formed section, sketch (f). Partially closed tubular sections or partially closed formed sections and the total

length of their respective welds shall encompass a minimum of 270 deg of the circumference and provide symmetrical loadings on closure plates.

- (b) Corner Joints. Corner joints shall be one of the following:
- (1) Partial penetration, without a fillet weld between the edge of a plate and the end surface of a closed tubular section or partially closed formed section, sketch (e). Partially closed tubular sections or partially closed formed sections as well as the total length of their respective welds shall encompass a minimum of 270 deg of the circumference and provide symmetrical loadings on closure plates.
- (2) Fillet welds between the edge of a plate and the end surface of a closed tubular section or partially closed tubular section or partially closed formed section, sketches (b), (c), and (d). In addition, weld joints (b) and (d) may be used only if the lengths of the respective legs of the fillet weld are equal to the exposed thickness of the plate and shell elements. Partially closed tubular sections or partially closed formed sections as well as the



total length of their respective welds shall encompass a minimum of 270 deg of the circumference and provide symmetrical loadings on closure plates.

(3) The use of welded joints permitted by (a)(3) and (b) is limited to housings for spring encapsulation of standard supports.

NF-3426.2 Design Stress Intensity and Allowable Stress Limits for Welded Joints. The limit of design stress intensity or of allowable stress for welded joints for Standard Supports shall not exceed the applicable design stress intensity value or allowable stress value for the base metal being joined.

#### NF-3450 DESIGN BY ANALYSIS FOR CLASS 2 NF-3451 General Design Requirements

The design of Class 2 Standard Supports shall be in accordance with the requirements of NF-3450 using one of the design procedures indicated in Table NF-3131(a)-1.

# NF-3452 Design of Plate- and Shell-Type Standard Supports

The requirements of NF-3200 shall be met.

# NF-3453 Design of Linear-Type Standard Supports

The requirements of NF-3300 shall be met.

#### NF-3455 Design of Bolting

The requirements of NF-3225 and NF-3324.6 shall be met.

#### NF-3456 Design of Welded Joints

The requirements of NF-3426 shall be met, except that for groove welded and fillet welded T-joints, the welds may be intermittent instead of continuous.

#### NF-3460 DESIGN BY ANALYSIS FOR CLASS 3

The design of Class 3 Standard Supports shall be in accordance with the requirements of NF-3450, using one of the design procedures indicated in Table NF-3131(a)-1.

# NF-3470 DESIGN BY EXPERIMENTAL STRESS ANALYSIS

Standard Supports may be designed by experimental stress analysis in accordance with the requirements of NF-3270 for Plate- and Shell-Type Standard Supports and NF-3370 for Linear-Type Standard Supports.

#### NF-3480 PROCEDURE FOR LOAD RATING

Standard Supports may be designed by the procedure for load rating in accordance with the requirements of NF-3280 for Plate- and Shell-Type Standard Supports and NF-3380 for Linear-Type Standard Supports.

# NF-3500 DESIGN RULES FOR COMPONENT SUPPORTS

#### NF-3510 GENERAL REQUIREMENTS

The design of component supports shall be in accordance with this subarticle and the applicable general requirements of NF-3110, NF-3210, NF-3310, and NF-3410.

# NF-3520 DESIGN BY ANALYSIS FOR CLASS 10 NF-3521 General Design Requirements

This subsubarticle provides stress limits for elements of Class 1 component supports. For general requirements as to stress determinations, definitions, derivations of stress intensities, and classification of stresses, refer to NF-3120.

# NF-3522 Design of Plate- and Shell-Type Component Supports

- (a) The design rules and stress intensity limits which must be satisfied for the Design and Service Loadings are given in NF-3220.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3221.2-1.

# NF-3523 Design of Linear-Type Component Supports

- (a) The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3320.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3312.1(b)-1.

#### NF-3524 Design of Standard Component Supports

- (a) The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3420.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3221.2-1 or Table NF-3312.1(b)-1, as applicable.

#### NF-3525 Design of Bolting

The number and cross-sectional area of bolts required for the Design Loadings of NF-3112 shall be determined in accordance with the procedures of NF-3324.6.

#### NF-3526 Design of Welded Joints

- (a) Welded joints in Plate- and Shell-Type Component Supports shall be as stipulated in NF-3226.
- (b) Welded joints used in Linear-Type Component Supports shall be as stipulated in NF-3324.5.

## NF-3550 DESIGN BY ANALYSIS FOR CLASS 2, 3, AND MC

# NF-3552 Design of Plate- and Shell-Type Component Supports

- (a) The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3250 and NF-3260.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3251.2-1.

# NF-3553 Design of Linear-Type Component Supports

The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3523.

# NF-3554 Design of Class 2 and 3 Standard Component Supports

The design of Standard Component Supports shall be in accordance with NF-3524.

#### NF-3555 Design of Bolting

The provision of NF-3525 shall be met.

#### NF-3556 Design of Welded Joints

- (a) Welded joints in Plate- and Shell-Type Component Supports shall be as stipulated in NF-3256 for Class 2 and MC and NF-3266 for Class 3.
- (b) Welded joints in Linear-Type Component Supports shall be as stipulated in NF-3324.5.

#### NF-3570 EXPERIMENTAL STRESS ANALYSIS

Component supports of all types may be designed by experimental stress analysis in accordance with Section III Appendices, Mandatory Appendix II.

#### NF-3580 DESIGN BY LOAD RATING

Component supports of all types may be designed by load rating in accordance with the requirements of NF-3280 for Plate- and Shell-Type, NF-3380 for Linear-Type and the applicable subsubarticle for Standard Component Supports.

# NF-3600 DESIGN RULES FOR PIPING SUPPORTS

#### **NF-3610 GENERAL REQUIREMENTS**

The design of piping supports shall be in accordance with this subarticle and the applicable general requirements of NF-3110, NF-3210, NF-3310, and NF-3410.

#### NF-3611 Spacing of Piping Supports

Supports for piping with a longitudinal axis in approximately a horizontal position shall be spaced to prevent excessive shear stresses resulting from sag and bending in the piping, with special consideration given when components such as pumps and valves impose concentrated loads. The suggested maximum spans for spacing of weight supports for standard weight and heavier piping are given in Table NF-3611-1.

#### NF-3612 Vibration

Piping shall be arranged and supported so that vibration will be minimized.

#### NF-3613 Anchors, Guides, Pivots, and Restraints

Anchors, guides, pivots, and restraints shall be designed to secure the desired points of piping in a relatively fixed position. They shall permit the piping to expand and contract freely as directed from the anchored or guided point and shall be structurally capable of withstanding the thrusts, moments, and other imposed loads.

#### NF-3614 Support Adjustments

(a) Screwed adjustments shall have threaded parts to conform to ASME B1.1 coarse threaded series, Class 2 fit. Alternatively, other thread forms, series, and fits

#### Table NF-3611-1 Suggested Piping Support Spacing

Nominal	Suggested Maxim	um Span, ft (m)
Pipe Size,		Steam, Gas, or
in. (DN)	Water Service	Air Service
1 (25)	7 (2.1)	9 (2.7)
2 (50)	10 (3.0)	13 (3.9)
3 (80)	12 (3.6)	15 (4.5)
4 (100)	14 (4.2)	17 (5.1)
6 (150)	17 (5.1)	21 (6.4)
8 (200)	19 (5.7)	24 (7.3)
12 (300)	23 (7.0)	30 (9.1)
16 (400)	27 (8.2)	35 (10.6)
20 (500)	30 (9.1)	39 (11.8)
24 (600)	32 (9.7)	42 (12.8)

#### GENERAL NOTES:

- (a) Suggested maximum spacing between piping supports for horizontal straight runs of standard and heavier piping with a maximum operating temperature of 750°F (400°C).
- (b) Does not apply where span calculations are made or where there are concentrated loads between supports such as flanges, valves, specialties, etc.
- (c) The spacing is based on a maximum combined bending and shear stress of 1.5 ksi (10 MPa) and insulated piping filled with water or the equivalent weight of steel pipe for steam, gas, or air service, and the pitch of the line is such that a sag of 0.1 in. (2.5 mm) between supports is permissible.

may be used which permit adjustment after erection, and are designed to withstand the forces, moments, and other imposed loads acting on the support, considering the worst combination of thread tolerances.

(b) Turnbuckles and adjusting nuts shall have the full length of thread in engagement. Means shall be provided for determining that full thread length is engaged. All screw and equivalent adjustments shall be provided with suitable locking devices.

#### NF-3615 Piping Clamps

- (a) When clamps are used to support vertical piping lines, it is recommended that shear lugs be welded to the piping to prevent slippage.
- (b) In addition to the provision of (a), clamps to support vertical piping lines with rigid connections shall be designed to support the total load on either arm in the event the load shifts because of piping or hanger movement, or both.

#### NF-3616 Hanger Rods

The provisions for hanger rods of NF-3412.3 apply.

# NF-3620 DESIGN BY ANALYSIS FOR CLASS 1 NF-3622 Design of Plate- and Shell-Type Piping Supports

- (a) The design rules and stress intensity limits which must be satisfied for the Design and Service Loadings are given in NF-3220.
- (b) When design by analysis is used, the stress intensity limit factors for each loading and stress category are specified in Table NF-3221.2-1.

#### NF-3623 Design of Linear-Type Piping Supports

- (a) The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3320.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3312.1(b)-1.

#### NF-3624 Design of Standard Piping Supports

- (a) The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3420.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3221.2-1 or Table NF-3312.1(b)-1, as applicable.

#### NF-3625 Design of Bolting

The number and cross-sectional area of bolts required for the Design Loadings of NF-3112 shall be determined in accordance with the procedures of NF-3324.6. The allowable bolt design stress value shall be determined in accordance with NF-3324.6.

#### NF-3626 Design of Welded Joints

- (a) Welded joints in Plate- and Shell-Type Piping Supports shall be as stipulated in NF-3226. In addition, tee joints, double fillet welded, sketch (a) [Figure NF-3626(a)-1] and single welded, when double members, sketch (b), are permitted.
- (b) Welded joints used in Linear-Type Piping Supports shall be as stipulated in NF-3324.5.

#### NF-3650 DESIGN BY ANALYSIS FOR CLASS-2 AND 3

# NF-3652 Design of Plate- and Shell-Type Piping Supports

- (a) The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3250 and NF-3260.
- (b) When design by analysis is used, the stress limit factors for each loading and stress category are specified in Table NF-3251.2-1.

#### NF-3653 Design of Linear-Type Piping Supports

The design rules and stress limits which must be satisfied for the Design and Service Loadings are given in NF-3623.

#### NF.3654 Design of Standard Piping Supports

The design of Standard Piping Supports shall be in accordance with NF-3624.

#### NF-3655 Design of Bolting

The provisions of NF-3625 shall be met.

# Figure NF-3626(a)-1 Additional Permissible Welded Joints for Class 1 Piping Supports (a)

And the state of t

# ARTICLE NF-4000 FABRICATION AND INSTALLATION

#### **NF-4100 GENERAL REQUIREMENTS**

**NF-4110 INTRODUCTION** 

#### NF-4111 Fabrication and Installation

Supports shall be fabricated and installed in accordance with the requirements of this Article and shall be manufactured from material which meets the requirements of Article NF-2000.

#### NF-4112 Reassembly of Subsection NF Supports

The Certificate Holder may reassemble supports from completed supports that have not been in operating service or from parts and material of disassembled supports that have not been in operating service, provided all required documentation is available and the applicable Code requirements are met.

The program for maintaining identification of material and parts, including material documentation and certification documents (Certificate of Compliance and NS-1 Certificate of Conformance) shall be described in a written procedure.

# NF-4120 CERTIFICATION OF MATERIALS AND FABRICATION BY SUPPORT CERTIFICATE HOLDER

#### NF-4121 Means of Certification

The NS Certificate Holder for a support shall certify Code compliance by the furnishing of an NS-1 Certificate of Conformance (NCA-3687) for welded supports or a Certificate of Compliance (NCA-3689) for nonwelded supports.

NF-4121.1 Certification of Treatments, Tests, and Examinations. If the Certificate Holder performs treatments, tests, repairs, or examinations required by other Articles of this Section, he shall certify that he has fulfilled that requirement [NCA-3861(c)]. Reports of all required treatments and the results of all required tests, repairs, and examinations performed by the NS Certificate Holder shall be maintained as quality assurance records in accordance with NCA-4134.17.

**NF-4121.2 Repetition of Tensile or Impact Tests.** If during the fabrication or installation of the support the material is subjected to heat treatment that has not been covered by treatment of the test coupons (NF-2200) and that may reduce either the tensile or impact properties below the required values, the tensile and impact tests

shall be repeated by the Certificate Holder on test specimens taken from test coupons which have been taken and treated in accordance with the requirements of Article NF-2000.

#### NF-4122 Material Identification

Material for supports shall carry identification markings which will remain distinguishable until the support is fabricated or installed. If the original identification markings are cut off or the material is divided, the marks shall be accurately transferred to the parts or a coded marking shall be used to assure identification of each piece of material during subsequent fabrication or installation, unless otherwise provided by NF-2150. Material supplied with a Certificate of Compliance and welding and brazing material shall be identified and controlled so that they can be traced to each support, or else a control procedure shall be employed which ensures that the specified material is used.

#### NF-4123 Visual Examinations

Visual examination activities that are not referenced for examination by other specific Code paragraphs, and are performed solely to verify compliance with requirements of Article NF-4000, may be performed by the persons who perform or supervise the work. These visual examinations are not required to be performed by personnel and procedures qualified to NF-5500 and NF-5100, respectively, unless so specified.

# NF-4125 Testing of Welding and Brazing Materials

All welding and brazing materials shall meet the requirements of NF-2400.

# NF-4130 REPAIR OF MATERIAL NF-4131 Elimination and Repair of Defects

Material originally accepted on delivery in which defects exceeding the limits of NF-2500 are known or discovered during the process of fabrication or installation is unacceptable. The material may be used provided the condition is corrected in accordance with the requirements of NF-2500 for the applicable product form, except

(a) weld repair is not required if the defect is removed by mechanical means and does not reduce the section below the minimum thickness required by Article NF-3000

- (b) when weld repair is performed
- (1) the weld repair restores the section to the minimum thickness required by Article NF-3000
- (2) the limitation on the depth of weld repair does not apply
- (3) the time of examination of weld repairs to weld edge preparations and to material shall be in accordance with NF-5120
- (4) the provisions of NF-2610(e) for weld repair of structural material apply

# NF-4200 FORMING, FITTING, AND ALIGNING

# NF-4210 CUTTING, FORMING, AND BENDING NF-4211 Cutting

Material may be cut to shape and size by mechanical means such as machining, shearing, or chipping, or by grinding or thermal cutting. Thermally cut edges that are to be welded shall be smooth and free of all loose scale and slag accumulations.

#### NF-4211.1 Preheating Before Thermal Cutting.

When thermal cutting is performed to prepare weld joints or edges, to remove attachments or defective material, or for any other purpose, consideration shall be given to preheating the material using preheat schedules, such as suggested in Section III Appendices, Nonmandatory Appendix D.

#### NF-4212 Forming and Bending Processes

Any process may be used to hot or cold form or bend materials, including weld metal, provided the required dimensions are attained (see NF-4214 and NF-4220), and provided the specified impact properties of the material, when required, are not reduced below the minimum specified values or they are effectively restored by heat treatment following the forming operation. *Hot forming* is defined as forming with the material temperature higher than 100°F (56°C) below the lower transformation temperature of the material. When required, the process shall be qualified for impact properties as outlined in NF-4213.

#### NF-4213 Qualification of Forming Processes for Impact Property Requirements

When impact testing is required by the Design Specifications, a procedure qualification test shall be conducted using specimens taken from material of the same material specification, grade or class, heat treatment, and with similar impact properties as required for the material in the support. These specimens shall be subjected to the equivalent forming or bending process and heat treatment as the material in the support. Applicable tests shall be conducted to determine that the required impact properties of NF-2300 are met after straining.

**NF-4213.1 Exemptions.** Procedure qualification tests are not required for material listed in (a) through (f)

- (a) hot-formed material, such as forgings, in which the hot forming is completed by the Material Organization prior to removal of the impact test specimens
- (b) hot-formed material represented by test coupons which has been subjected to heat treatment representing the hot forming procedure and the heat treatments to be applied to the parts
- (c) material which does not require impact tests in accordance with NF-2300
  - (d) material which has final strain less than 0.5%
- (e) material where the final strain is less than that of a previously qualified procedure for that material
- (f) material from which the impact testing required by NF-2300 is performed on each heat and lot, as applicable, after forming

**NF-4213.2 Procedure Qualification Test.** The procedure qualification test shall be performed in the manner stipulated in (a) through (f).

- (a) The tests shall be performed on three different heats of material both before and after straining to establish the effects of the forming and subsequent heat treatment operations.
- (b) Specimens shall be taken in accordance with the requirements of Article NF-2000 and shall be taken from the tension side of the strained material.
- (c) The percent strain shall be established by the following equations:

For cylinders:

% strain = 
$$\left(50t/R_f\right)\left[1 - \left(R_f/R_o\right)\right]$$

For spherical or dished surfaces

% strain = 
$$\left(75t/R_f\right)\left[1 - \left(R_f/R_o\right)\right]$$

For pipe

$$\%$$
 strain =  $100r/R$ 

where

R =nominal bending radius to the centerline of the pipe, in. (mm)

r = nominal radius of the pipe, in. (mm)

 $R_f$  = final radius to centerline of shell, in. (mm)

 $R_o$  = original radius (equal to infinity for a flat part), in. (mm)

t = nominal thickness, in. (mm)

- (d) The procedure qualification shall simulate the maximum percent surface strain, employing a bending process similar to that used in the fabrication of the material or by direct tension on the specimen.
- (e) Sufficient Charpy V-notch specimens shall be taken from each of three heats of material to establish a transition curve showing in both the upper and lower shelves.

On each of these three heats, tests consisting of three impact specimens shall be conducted at a minimum of five different temperatures distributed throughout the transition region. The upper and lower shelves may be established by the use of one test specimen each, depending on the product form.

- (f) Using the results of the impact test data from each of three heats, taken both before and after straining, determine either
- (1) the maximum change in NDT temperature along with
- (-a) the maximum change of lateral expansion and energy of the temperature under consideration or
- (-b) the maximum change of temperature at the lateral expansion and energy levels under consideration or
- (2) when lateral expansion is the acceptance criterion (NF-2300), either the maximum change in temperature or the maximum change in lateral expansion

#### NF-4213.3 Acceptance Criteria for Formed Material.

To be acceptable, the formed material used in the support shall have impact properties before forming sufficient to compensate for the maximum loss of impact properties due to the qualified forming procedure used.

**NF-4213.4 Requalification.** A new procedure qualification test is required when any of the following changes are made:

- (a) the actual postweld heat treatment time at temperature is greater than previously qualified considering NF-2211; if the material is not postweld heat treated the procedure must be qualified without postweld heat treatment
- (b) the maximum calculated strain of the material exceeds the previously qualified strain by more than 0.5%
- (c) where preheat over 250°F (120°C) is used in the forming or bending operation but not followed by a subsequent postweld heat treatment

# NF-4214 Minimum Thickness of Fabricated Material

If any fabrication operation reduces the thickness below the minimum required to satisfy the rules of Article NF-3000, the material may be repaired in accordance with NF-4130.

#### NF-4220 SUPPORT TOLERANCES

Tolerances not specified in this subsection shall be specified by the designer in accordance with NF-3134.

# NF-4221 Tolerances for Plate- and Shell-Type Supports

(a) The outer surface of a plate- and shell-type support shall not deviate from the specified shape by more than  $1^{11}/_{4}\%$  of the overall design dimension. Such deviations shall not include abrupt changes.

(b) For plate- and shell-type supports which are skirts, the difference between the maximum and minimum outside diameters shall not exceed 1% of the nominal outside diameter.

# NF-4230 FITTING AND ALIGNING NF-4231 Fitting and Aligning Methods

Parts that are to be joined may be fitted, aligned, and retained in position during the joining operation by the use of bars, jacks, clamps, drift pins, tack welds, or temporary attachments. Mechanical devices shall be carefully used to avoid damage to surfaces of the parts and to avoid enlargement of bolt holes.

**NF-4231.1 Tack Welds.** Tack welds used to secure alignment shall either be removed completely, when they have served their purpose, or their stopping and starting ends shall be properly prepared by grinding or other suitable means so that they may be satisfactorily incorporated into the final weld. Tack welds shall be made by qualified welders using qualified welding procedures. When tack welds are to become part of the finished weld, they shall be visually examined and defective tack welds removed.

#### NF-4231.2 Column Bases.

- (a) Column bases shall be set level and to correct elevation with full bearing on the masonry.
- (b) Column bases shall be finished in accordance with the requirements of (1) through (3).
- (1) Rolled steel bearing plates 2 in. (50 mm) or less in thickness may be used without milling, provided a satisfactory contact bearing is obtained. Rolled steel bearing plates over 2 in. (50 mm) but not over 4 in. (100 mm) in thickness may be straightened by pressing or, if presses are not available, by milling for all bearing surfaces except those noted in (3) to obtain satisfactory contact bearing. Rolled steel bearing plates over 4 in. (100 mm) in thickness shall be milled for all bearing surfaces except as noted in (3).
- (2) Column bases other than rolled steel bearing plates shall be planed for all bearing surfaces, except as noted in (3).
- (3) The bottom surfaces of bearing plates and column bases which are grouted to ensure full bearing contact on foundation need not be planed.

#### NF-4232 Maximum Offset of Aligned Sections

Alignment of butt joints shall be such that the maximum offset of the finished weld will not be greater than the applicable amount listed in Table NF-4232-1, where *t* is the nominal thickness of the thinner section of the joint.

#### Table NF-4232-1 Maximum Allowable Offset in Final Butt-Welded Joints

Section Thickness, in. (mm)	Maximum Allowable Offset
Up to $\frac{3}{4}$ (19), incl.	<sup>1</sup> / <sub>4</sub> t
Over $\frac{3}{4}$ to $1\frac{1}{2}$ (19 to 38), incl.	<sup>3</sup> / <sub>16</sub> in. (5 mm)
Over $1^{1}/_{2}$ to 6 (38 to 150), incl.	$^{1}/_{8}t$
Over 6 (150)	<sup>3</sup> / <sub>4</sub> in. (19 mm)

**NF-4232.1 Fairing of Offsets.** Any offset within the allowable tolerance of Table NF-4232-1 shall be blended uniformly over the width of the finished weld or, if necessary, by adding additional weld metal beyond what would otherwise be the edge of the weld.

#### NF-4240 REQUIREMENTS FOR WELDED JOINTS

Butt welds may be made with or without backing or consumable insert rings. When the use of permanent backing rings is undesirable [NF-3226.1(a) or NF-3256.1(a)]

- (a) the backing ring shall be removed and the inside of the joint ground smooth
  - (b) the joint shall be welded without backing rings or
  - (c) consumable insert rings shall be used

#### NF-4245 Complete Joint Penetration Welds

Complete joint penetration is considered to be achieved when the acceptance criteria for the examinations specified by this Subsection have been met. No other examination is required to assess that complete penetration has been achieved.

# NF-4300 WELDING QUALIFICATIONS NF-4310 GENERAL REQUIREMENTS NF-4311 Types of Processes Permitted

Only those welding processes which are capable of producing welds in accordance with the welding procedure qualification requirements of Section IX and this Subsection shall be used for welding support material or attachments thereto. Any process used shall be such that the records required by NF-4320 can be prepared except that records for stud welds shall be traceable to the welders and welding operators, and not necessarily to each specific weld.

**NF-4311.1 Stud Welding Restrictions.** Stud welding is acceptable for insulation supports, nameplates, and locating lugs. Studs shall be limited to 1 in. (25 mm) maximum diameter for round studs or an equivalent cross-sectional area for studs of other shapes when welding in the flat position and  $\frac{3}{4}$  in. (19 mm) diameter for all other welding positions. When studs are  $\frac{1}{2}$  in. (13 mm) in

diameter or less, postweld heat treatment, qualified welding procedures, and certified welding materials are not required.

**NF-4311.2 Capacitor Discharge Welding.** Capacitor discharge welding may be used for welding temporary attachments and permanent nonstructural attachments provided

- (a) temporary attachments are removed in accordance with the provisions of NF-4435(b)
- (b) the energy output for permanent nonstructural attachments such as strain gages and thermocouples is limited to 125 W-sec and the minimum thickness of the material to which the attachment is made is greater than 0.09 in. (2.3 mm)
- (c) a Welding Procedure Specification is prepared describing the capacitor discharge equipment, the combination of materials to be joined, and the technique of application; qualification of the welding procedure is not required

# NF-4311.4 Inertia and Continuous Drive Friction Welding.

- (a) Inertia and continuous drive friction welding shall not be used for fabrication of Class 1 Plate- and Shell-Type Component Supports.
- (b) The weld between the two members shall be a full penetration weld.

# NF-4320 WELDING QUALIFICATIONS AND RECORDS

#### NF-4321 Required Qualifications

- (a) Each Certificate Holder is responsible for the welding done by his organization and shall establish the procedure and conduct the tests required by this Article and Section IX in order to qualify both the welding procedures and the performance of welders and welding operators who apply these procedures.
- (b) Procedures, welders, and welding operators used to join permanent or temporary attachments to support elements and to make permanent or temporary tack welds used in such welding shall also meet the qualification requirements of this Article.
- (c) When making procedure test plates for butt welds, consideration shall be given to the effect of angular, lateral, and end restraint on the weldment. This applies particularly to material and weld metal of 80 ksi (550 MPa) tensile strength or higher and heavy sections of both low and high tensile strength material. The addition of restraint during welding may result in cracking difficulties that otherwise might not occur.
- (d) NCA-3131 provides specific additional requirements when welding services are subcontracted to or through organizations not holding an appropriate Certificate of Authorization.

## NF-4322 Maintenance and Certification of Records

The Certificate Holder shall maintain a record of his qualified welding procedures and of the welders and welding operators qualified by him, showing the date and results of tests and the identification mark assigned to each welder. These records shall be reviewed, verified, and certified by the Certificate Holder by signature or some other method of control in accordance with the Certificate Holder's Quality Assurance Program.

# NF-4322.1 Identification of Joints for Class 1 Component Supports.

(a) For Class 1 Plate- and Shell-Type, and Linear-Type Component Supports, the welder or welding operator shall apply the identification mark assigned to him by the Certificate Holder on or adjacent to all permanent welds, including fillet welds, at 3 ft (1 m) intervals or less, except as noted in (b). The marking shall be done with either blunt nose continuous or blunt nose interrupted dot die stamps. As an alternative, the Certificate Holder shall keep a record of permanent welded joints in a component support, and of the welders and welding operators used in making each of the joints.

(b) For partial penetration welds with a depth less than 1 in. (25 mm) and fillet welds with a throat dimension less than 1 in. (25 mm) in primary members, and for all welds in secondary members, the Certificate Holder need not identify the welder or welding operator who welded each joint provided

(1) the Certificate Holder maintains a system that will identify the welders or welding operators who made such welds on each item

(2) the welds in each category are all of the same type and configuration and are welded with the same Welding Procedure Specification

**NF-4322.2 Identification of Other Joints.** For all types of Class 2, 3, and MC component supports and for all classes of piping supports and Standard Supports, the Certificate Holder shall certify that only welders and welding operators qualified in accordance with NF-4321 were used in making all welds.

**NF-4322.3 Identification of Tack Welds.** The identification of welder or welding operator is not required for tack welds.

### NF-4323 Welding Prior to Qualification

No welding shall be undertaken until after the welding procedures which are to be used have been qualified. Only welders and welding operators who are qualified in accordance with NF-4320 and Section IX shall be used.

#### NF-4324 Transferring Qualifications

The welding procedure qualifications and the performance qualification tests for welders and welding operators conducted by one Certificate Holder shall not qualify

welding procedures and shall not qualify welders or welding operators to weld for any other Certificate Holder, except as provided in Section IX.

# NF-4330 GENERAL REQUIREMENTS FOR WELDING PROCEDURE QUALIFICATION TESTS

# NF-4331 Conformance to Section IX Requirements

All welding procedure qualification tests shall be in accordance with the requirements of Section IX as supplemented by the requirements of this Article.

#### NF-4334 Preparation of Test Coupons and Specimens

(a) Removal of test coupons from the test weld and the dimensions of specimens made from them shall conform to the requirements of Section IX, except that the removal of impact test coupons and the dimensions of impact test specimens shall be in accordance with (b).

(b) Weld deposit of each process in a multiple process weld shall, where possible, be included in the impact test specimens. When each process cannot be included in the full-size impact test specimen at the  ${}^1\!\!/_4 t$  location required by this Section, additional full-size specimens shall be obtained from locations in the test weld that will ensure that at least a portion of each process has been included in full-size test specimens. As an alternative, additional test welds can be made with each process so that full-size specimens can be tested for each process.

#### NF-4334.1 Coupons Representing the Weld Depos-

**its.** Impact test specimen and testing methods shall conform to NF-2321. The impact specimen shall be located so that the longitudinal axis of the specimen is at least  ${}^{1}\!/_{4}t$ , and where the thickness of the test assembly permits, not less than  ${}^{3}\!/_{8}$  in. (10 mm) from the weld surface of the test assembly. In addition, when the postweld heat treatment temperature exceeds the maximum temperature specified in NF-4620, and the test assembly is cooled at an accelerated rate, the longitudinal axis of the specimen shall be a minimum of t from the edge of the test assembly. The specimen shall be transverse to the longitudinal axis of the weld with the area of the notch located in the weld. The length of the notch of the Charpy V-notch specimen shall be normal to the surface of the weld.

**NF-4334.2 Coupons Representing the Heat-Affected Zone.** Where impact tests of the heat-affected zone are required by NF-4335.2, specimens shall be taken from the welding procedure qualification test assemblies in accordance with (a) through (c).

(a) If the qualification test material is in the form of a plate or a forging, the axis of the weld shall be oriented either parallel to or perpendicular to the principal direction of rolling or forging.

(b) The heat-affected zone impact test specimens and testing methods shall conform to NF-2321. The specimens shall be removed from a location as near as practical to a depth midway between the surface and center thickness. The coupons for heat-affected zone impact specimens shall be taken transverse to the axis of the weld and etched to define the heat-affected zone. The notch of the Charpy V-notch specimen shall be cut approximately normal to the material surface in such a manner as to include as much heat-affected zone as possible in the resulting fracture. Where the material thickness permits, the axis of a specimen may be inclined to allow the root of the notch to align parallel to the fusion line. When a grain refining heat treatment is not performed on welds made by the electroslag or electrogas welding process, the notch for the impact specimens shall be located in the grain coarsened region.

(c) For the comparison of heat-affected zone values with base material values [NF-4335.2(b)], Charpy V-notch specimens shall be removed from the unaffected base material at approximately the same distance from the base material surface as the heat-affected zone specimens. The axis of the unaffected base material specimens shall be parallel to the axis of the heat-affected zone specimens, and the axis of the notch shall be normal to the surface of the base material.

#### NF-4335 Impact Test Requirements

When materials are required to be impact tested per NF-2300, impact tests of the weld metal and heat-affected zone shall be performed in accordance with the following subparagraphs. Exemptions from impact testing under NF-2311(b)(9) and NF-2311(b)(10) do not apply to weld metal unless the specific weld metal used is included in Table NF-2311(b)-1 (weld metal exemptions are being developed). Exemption from impact testing of the heat-affected zone of those base materials which are exempted by NF-2311(b)(9) and NF-2311(b)(10) is not permitted. The welding procedure qualification impact test specimens shall be prepared and tested in accordance with the applicable requirements of NF-2330 and NF-4334. Retests in accordance with the provisions of NF-2350 are permitted.

#### NF-4335.1 Impact Tests of Weld Metal.

(a) Impact tests of the weld metal shall be required for welding procedure qualification tests for production weld joints exceeding  $\frac{5}{8}$  in. (16 mm) in thickness when the weld is made on the surface or penetrates the base material that requires impact testing in accordance with NF-2310. In addition, such testing of the weld metal is required for the welding procedure qualification tests for any weld repair to base material that requires impact testing in accordance with NF-2310, regardless of the depth of the repair. Exemption from impact testing under NF-2311(b)(9) and NF-2311(b)(10) does not apply to weld metal of the welding procedure qualification test

for either production weld joints or base material repairs unless the specific weld metal used is included in Table NF-2311(b)-1 (weld metal exemptions are being developed).

(b) The impact test requirements and acceptance standards for welding procedure qualification weld metal shall be the same as specified in NF-2330 for the base material to be welded or repaired. Where two materials which have different fracture toughness requirements are to be joined by welding, the test requirements and acceptance standards of either material may be used for the weld metal, except where otherwise specified by NCA-1280 or other parts of this Section

(c) A Welding Procedure Specification (WPS) qualified to the impact testing requirements of Subsection NB, NC, or NE may be accepted as an alternative to the WPS impact testing requirements of this Subsection.

#### NF-4335.2 Impact Tests of Heat-Affected Zone.

(a) Charpy V-notch tests of the heat-affected zone of the welding procedure qualification test assembly are required whenever the thickness of the weld exceeds  $\frac{5}{8}$  in. (16 mm), and either of the base materials requires impact testing in accordance with the rules of NF-2310. Exemption of base materials by NF-2311(b)(9) or NF-2311(b)(10) does not apply to the welding procedure qualification of the heat-affected zone or unaffected base material for such materials. The only exceptions to the requirements are the following:

(1) the qualification for welds in P-Nos. 1 and 3 and SA-336 F12 materials that are postweld heat treated and are made by any process other than electroslag, electrogas, or thermit

(2) the qualification for weld deposit cladding or hard-facing on any base material

(3) that portion of the heat-affected zone associated with GTAW root deposits with a maximum of two layers or  $\frac{3}{16}$  in. (5 mm) thickness, whichever is less

(b) Charpy V-notch testing shall be performed as specified in (1) through (6).

(1) Charpy V-notch test specimens representing both the heat-affected zone and the unaffected base material shall be tested. The unaffected base material shall be tested at a temperature equal to or below that specified in NF-2311(c).

(2) The Charpy V-notch tests of the unaffected base material shall meet the applicable requirements of NF-2330 for the applicable Class and acceptance category. If the requirements are not met at the test temperature, additional testing shall be performed at higher temperatures until the above requirements are met.

(3) The heat-affected zone specimens shall be tested at the test temperature determined in (2). If the average applicable toughness value of the heat-affected zone specimens equals or exceeds the average applicable toughness value of the unaffected base material, the qualification test is acceptable for the essential and

supplemental essential variables recorded on the Welding Procedure Qualification Record. If the heat-affected zone average applicable toughness value is less than the unaffected base material average applicable toughness value, the adjustment given in (4) through (6) shall be determined and applied as provided in (c). Alternatively, another test coupon may be welded and tested.

(4) Additional Charpy V-notch tests shall be performed on either the heat-affected zone or the unaffected base material, or both, at temperatures where the applicable toughness values of all three specimens tested are not less than that specified in (2). The average applicable toughness value for each test meeting this requirement shall be plotted on a property-temperature graph. The difference in temperature  $T_{\rm HAZ}$  and  $T_{\rm UBM}$  where the heat-affected zone and the unaffected base material average applicable toughness values are the same and not less than that specified in (2) shall be used to determine the adjustment temperature  $T_{\rm ADI}$  where

$$T_{\text{ADJ}} = T_{\text{HAZ}} - T_{\text{UBM}}$$

If  $T_{\text{ADJ}} \leq 0$ , then  $T_{\text{ADJ}} = 0$ .

- (5) As an alternative to (4), if the applicable toughness values of the heat-affected zone are no less than those specified in NF-2330 for the applicable Class and acceptance category and the average applicable toughness value of the heat-affected zone specimens is not less than 7 ft-lb (10 J) or 5 mils (0.13 mm) below the average applicable toughness value of the unaffected base material,  $T_{\rm ADI}$  may be taken as 15°F (8°C).
- (6) As a second alternative to (4), if the applicable toughness values of the heat-affected zone are no less than those specified in NF-2330 for the applicable Class and acceptance category, the difference between the average applicable toughness value of the heat-affected zone and the unaffected base material shall be calculated and used as described in (c)(3).
- (c) At least one of the following methods shall be used to compensate for the heat-affected zone toughness decrease due to the welding procedure.
- (1) The lowest service temperature specified in the Design Specification for all of the material to be welded in production Welding Procedure Specifications supported by this Welding Procedure Qualification Record shall be increased by the adjustment temperature  $T_{\rm ADJ}$ .
- (2) The specified testing temperature for the production material may be reduced by  $T_{\rm ADJ}$ .
- (3) The materials to be welded may be welded using the Welding Procedure Specification, provided they exhibit toughness values that are no less than the minimum required toughness values specified in NF-2300 plus the difference in the average applicable toughness values established in (b)(6).
- (d) The Charpy V-notch testing results shall be recorded on the Welding Procedure Qualification Record and any offsetting  $T_{\rm ADJ}$  or increased toughness requirements shall be noted on the Welding Procedure

Qualification Record and on the Welding Procedure Specification. More than one compensation method may be documented on the Welding Procedure Qualification Record.

(e) A Welding Procedure Specification qualified to the impact testing requirements of Subsection NB, NC, or NE may be accepted as an alternative to the Welding Procedure Specification impact testing requirements of this Subsection.

# NF-4400 RULES GOVERNING MAKING AND REPAIRING WELDS

# NF-4410 PRECAUTIONS TO BE TAKEN BEFORE WELDING

# NF-4411 Identification, Storage, and Handling of Welding Materials

Each Certificate Holder is responsible for control of the welding electrodes and other materials which are used in the fabrication and installation of supports (NF-4120). Suitable identification, storage, and handling of electrodes, flux, and other welding material shall be maintained. Precautions shall be taken to minimize absorption of moisture by electrodes and flux.

## NF-4412 Cleanliness and Protection of Weld Surfaces

The method used to prepare the base metal shall leave the weld preparation with reasonably smooth surfaces. The surfaces for welding shall be free of scale, rust, oil, grease, and other deleterious material. The work shall be protected from deleterious contamination and from rain, snow, and wind during welding. Welding shall not be performed on wet surfaces.

#### NF-4420 RULES FOR MAKING WELDED JOINTS NF-4421 Backing Strips

The materials for backing strips, when used, shall be compatible with the base metal.

#### NF-4422 Peening

The weld metal may be peened when it is deemed necessary or helpful to control distortion.

#### NF-4423 Double-Welded Joints

Before applying weld metal on the second side to be welded, the root of full penetration double-welded joints shall be prepared by suitable methods, such as chipping, grinding, or thermal gouging, except for those processes of welding by which proper fusion and penetration are otherwise obtained and demonstrated to be satisfactory by welding procedure qualification.

- (a) The surface condition of the finished weld shall be sufficiently free from coarse ripples, grooves, overlaps, abrupt ridges, and valleys for the proper interpretation of radiographic and other required nondestructive examinations of the welds. In those cases where there is a question regarding the surface condition on the interpretation of a radiographic film, the film shall be compared to the actual weld surface for interpretation and determination of acceptability.
- (b) Concavity on the root side of a single-welded circumferential butt weld is permitted when the resulting thickness of the weld meets the requirements of Article NF-3000.
- (c) Fusion shall exist between adjacent layers of weld metal and between weld metal and base metal except as provided in NF-5360(c).
- (d) For inertia and continuous drive friction welding, the weld upset shall meet the specified amount within ±10%. Flash shall be removed to sound metal.

#### NF-4427 Shape and Size of Welds

- (a) Fillet welds may vary from convex to concave. The shape and size of the weld shall be in accordance with the requirements of Figure NF-4427-1. Convexity of fillet welds is not a criteria for acceptance and need not be measured.
- (b) The faces of groove welds may be flat or convex. The thickness of groove welds shall be no less than  $\frac{1}{32}$  in. (0.8 mm) less than the thickness of the thinner of the members joined.
- (c) On web-to-flange welds on girders, no underrun is permitted at the ends for a length equal to twice the width of the flange.

#### NF-4429 Plug Welds

When plug welds are used on supports, a fillet weld shall first be deposited around the circumference at the bottom of the hole.

#### NF-4430 WELDING OF ATTACHMENTS

#### NF-4435 Welding of Nonstructural and Temporary Attachments, and Their Removal

- (a) Nonstructural attachments such as insulation supports, nameplates, locating lugs, and temporary attachments may be noncertified material and may be welded to the support or parts thereof with continuous or intermittent fillet or partial penetration welds, provided the requirements of (1) through (5) are met
- (1) the welding procedure and the welders have been qualified in accordance with NF-4321
- (2) the material is identified and is suitable for welding
- (3) the material is compatible with the material to which it is attached

- (4) the welding material is identified and compatible with the materials joined
- (5) the welds are postweld heat treated when required by NF-4620
- (b) Removal of nonstructural temporary attachments shall be accomplished as follows:
- (1) the temporary attachment shall be completely removed in accordance with the procedures of NF-4211
- (2) as an alternative to (a)(5), postweld heat treatment may be deferred until after removal of the attachment
- (3) the surface of the support shall be visually examined after removal of the temporary attachment

# NF-4440 SPECIAL REQUIREMENTS FOR WELDING

#### NF-4441 Through-Thickness Loading

Welded joint configurations causing significant through-thickness tensile stress [as defined in NF-1215(b)] during fabrication or service on rolled product forms should be avoided. When this type of construction is used, weld volume and welding heat input on the rolled surfaces should be limited to the extent practical. When identified by the Design Output Documents [NF-3226.3, NF-3256.4, and NF-3324.5(f)(3)(-h)], weld joints in primary members 1 in. (25 mm) or greater in thickness in component supports subjected to significant through-thickness tensile loads and any other weld joints identified in the Design Output Documents shall meet one of the following requirements:

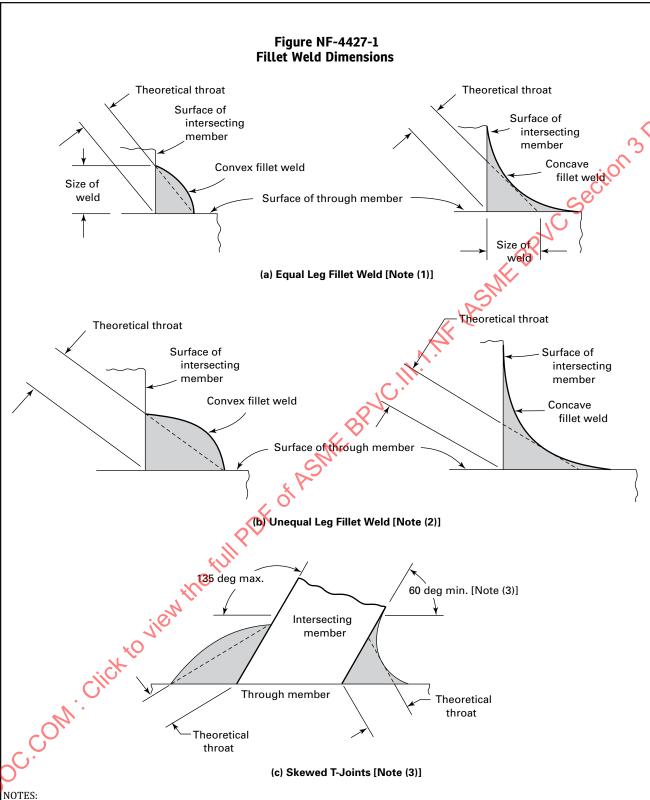
- (a) After completion of welding, the base material directly underneath the attachment shall be ultrasonically examined in accordance with the requirements of NF-5214, NF-5224, or NF-5234, as appropriate to the component support class.
- (b) A weld metal inlay or overlay shall be made in accordance with Figure NF-4441-1 and ultrasonically examined in accordance with NF-5214, NF-5224, or NF-5234, prior to making the weld joining the attachment member to the inlay or overlay.
- (c) The material shall meet the acceptance standards of SA-770, Through-Thickness Tension Testing of Steel Plates for Special Applications.

# NF-4450 REPAIR OF WELD METAL DEFECTS NF-4451 General Requirements

Defects in weld metal detected by the examinations required by Article NF-5000 shall be eliminated or repaired.

#### NF-4452 Elimination of Surface Defects

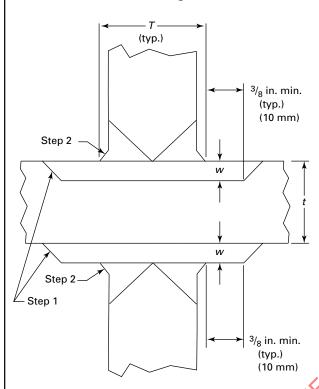
Weld metal surface defects may be removed by grinding or machining and need not be repaired by welding, provided that the requirements of (a) through (c) are met.



- (1) The "size" of an equal leg fillet weld is the leg length of the largest inscribed right isosceles triangle. Theoretical throat = 0.7 × size of weld.
- (2) The "size" of an unequal leg fillet weld is the shorter leg length of the largest right triangle, which can be inscribed within the fillet weld cross section.
- (3) When the intersecting member is less than 60 deg, the weld shall be considered a partial penetration groove weld [NF-3324.5(d)(4)].

**(17)** 

#### Figure NF-4441-1 **Weld Joint Producing Through-Thickness** Loading



Step 1: Inlay or overlay weldment

Step 2: Completion of weldment

GENERAL NOTES:

- (a) T is the width of the welded zone measured on the primary member surface.
- Members may be welded on one side only or from both sides.
- Inlay or overlay thickness shall be 1/4 in. (6 mm) min. for t = 1 in. (25 mm) and shall be increased  $\frac{1}{16}$  in. (1.5 mm) for each additional  $\frac{3}{8}$  in. (10 mm) of t but not exceed  $\frac{1}{2}$  in.
- When T exceeds t, w shall be increased by an additional  $\frac{1}{16}$  in. (1.5 mm) for each  $\frac{3}{8}$  in (10 mm) of T over t, to a maximum of  $\frac{1}{2}$  in. (13 mm).
- The minimum specified tensile strength of the electrodes used for inlay or overlay shall not exceed the minimum specified tensile strength of the base metal by more than 12 ksi (83 MPa))
- (a) The remaining thickness of the section is not reduced below that required by Article NF-3000.
- (b) The depression, after defect elimination, is blended uniformly into the surrounding surface.
- (c) The area is examined after blending by a magnetic particle or liquid penetrant method in accordance with NF-5110 and meets the acceptance standards of NF-5300 to ensure that the defect has been removed or the indication reduced to an acceptable limit. Defects detected by the visual or volumetric method and located on

an interior surface need only be reexamined by the method which initially detected the defect when the interior surface is inaccessible for surface examination.

#### Requirements for Making Repairs to NF-4453

Excavations in weld metal, when repaired by welding shall meet the requirements of the following subparagraphs.

NF-4453.1 Defect Removal. Defects may be removed by mechanical means or by thermal gouging processes. The area prepared for repair shall be examined by a liquid penetrant or magnetic particle method in accordance with NF-5110 and shall meet the acceptance standards of NF-5340 or NF-5350. This examination is not required where defect elimination removes the full thickness of the weld and where the backside of the weld joint is not accessible for removal of examination materials.

Requirements for Welding Materials, NF-4453.2 **Procedures, and Welders.** The weld repair shall be made using welding material, welders, and welding procedures qualified in accordance with NF-4125 and NF-4300.

**NF-4453.3 Blending of Repaired Areas.** After repair, the surface shall be blended uniformly into the surrounding surface.

NF-4453.4 Examination of Repair Welds. The examination of a weld repair shall be repeated as required for the original weld, except that it need only be reexamined by the liquid penetrant or magnetic particle method when the unacceptable indication was originally detected by the liquid penetrant or magnetic particle method and when the repair cavity does not exceed the following:

- (a)  $\frac{1}{3}t_w$  for  $t_w \le \frac{3}{4}$  in. (19 mm) (b)  $\frac{1}{4}$  in. (6 mm) for  $\frac{3}{4}$  in. (19 mm)  $< t_w \le 2^{\frac{1}{2}}$  in.
- (c) the lesser of  $\frac{3}{8}$  in. (10 mm) or 10%  $t_w$  for  $t_w > 2^1/_2$  in. (64 mm)

where  $t_w$  equals the nominal thickness of the original weld.

NF-4453.5 Heat Treatment of Repaired Areas. The repaired area shall be heat treated when required by NF-4620.

#### NF-4500 **BRAZING**

#### NF-4510 **RULES FOR BRAZING**

#### NF-4511 Where Brazing May Be Used

Furnace brazing is permitted for the construction of supports in accordance with the rules of this Subsection provided the following additional requirements are met:

(a) Brazing procedures and brazing operators shall be qualified in accordance with Section IX.

- (b) Clearance or interference between mating parts shall be in accordance with the Brazing Procedure Specification.
- (c) Base material shall be limited to material permitted by NF-2120 and Section IX, P-Nos. 101 and 102 ferritic material.
- (d) Brazing alloys shall be limited to Section IX, F-No. 105.
- (e) The allowable shear stress shall be 15 ksi (105 MPa) or 0.4 times the yield strength of the base material [NF-3322.1(b)], whichever is less, and the joint shall be loaded principally in shear only.
- (f) Postbrazing heat treatment to develop or restore base material mechanical properties shall be qualified by test specimens which meet the thickness requirements of Section IX, QB-451. The base material test specimens shall be exposed to the same thermal treatment as a brazed part but need not contain a brazed joint. Postbrazing heat treatment of actual parts or test specimens may be performed in the same heat cycle as the brazing operation or in a separate heat cycle. Postbrazing heat treatment shall not exceed 1,850°F (1 010°C).
- (g) Brazing alloy shall be preplaced on one side of the joint only. Each production joint shall be visually examined after brazing. Brazing metal must be continuous on the preplaced brazing metal side and 85% continuous on the other side.

#### NF-4512 Brazing Material

Where brazing is permitted, the brazing filler material and fluxes shall conform to the rules covering identification in NF-2150 and to the following requirements:

- (a) The filler material used in brazing shall be a nonferrous metal or alloy with a solidus temperature above 800°F (425°C) and at least 500°F (278°C) above the highest temperature of the joint in service.
- (b) The filler material shall melt and flow freely by capillary action within the desired temperature range, and in conjunction with a suitable flux or controlled atmosphere the filler material shall wet and adhere to the surfaces to be joined.
- (c) Fluxes that are fluid and chemically active at the brazing temperature shall be used, when necessary, to prevent oxidation of the filler metal and the surfaces to be joined and to promote free flowing of the filler material.

#### NF-4600 HEAT TREATMENT

# NF-4610 WELDING PREHEAT AND INTERPASS REQUIREMENTS

#### NF-4611 When Preheat Is Necessary

The need for and temperature of preheat are dependent on a number of factors, such as the chemical analysis, degree of restraint of the parts being joined, elevated temperature, physical properties, and material thicknesses.

Some practices used for preheating are given in Section III Appendices, Nonmandatory Appendix D as a general guide for the materials listed by P-Numbers of Section IX. It is cautioned that preheating suggested in Section III Appendices, Nonmandatory Appendix D does not necessarily ensure satisfactory completion of the welded joint and that the preheat requirements for individual materials within the P-Number listing may be more or less restrictive. The Welding Procedure Specification for the material being welded shall specify the minimum preheating requirements under the welding procedure qualification requirements of Section IX.

#### NF-4612 Preheating Methods

Preheat for welding or thermal cutting, when employed, may be applied by any method which does not harm the base material or any weld metal already applied or which does not introduce deleterious material into the welding area which is harmful to the weld.

#### NF-4613 Interpass Temperatures

Consideration should be given to limitations of interpass temperatures for quenched and tempered material to avoid detrimental effects on the mechanical properties.

# NF-4620 POSTWELD HEAT TREATMENT NF-4621 Heating and Cooling Methods

Postweld heat treatment (PWHT) may be accomplished by any suitable methods of heating and cooling, provided the required heating and cooling rates, metal temperature, metal temperature uniformity, and temperature control are maintained.

# NF-4622 PWHT Time and Temperature Requirements

NF-4622.1 General Requirements.¹ Except as otherwise permitted in NF-4622.7, all welds, including repair welds, shall be postweld heat treated. During postweld heat treatment, the metal temperature shall be maintained within the temperature ranges and for the minimum holding times specified in Table NF-4622.1-1, except as otherwise permitted in NF-4622.4(c). P-Number groups in Table NF-4622.1-1 are in accordance with Section IX, QW-420. Except as provided in NF-4624.3, PWHT shall be performed in temperature-surveyed and -calibrated furnaces, or PWHT shall be performed with thermocouples in contact with the material or attached to blocks in contact with the material. In addition, the requirements of the following subparagraphs shall apply.

**NF-4622.2 Time-Temperature Recordings.** Time-temperature recordings of all postweld heat treatments shall be maintained as quality assurance records in accordance with NCA-4134.17. Identification on the time-temperature recording shall be to the weld, part, or

Table NF-4622.1-1					
Mandatory Requirements for Postweld Heat Treatment (PWHT) of Welds					

	Holding	Minimum Holding Time at Temperature for Weld Thickness (Nominal)			
P-Number (Sect. IX, QW-420)	Temperature Range, °F (°C) [Note (1)]	<sup>1</sup> / <sub>2</sub> in. (13 mm) or less	Over ½ in. (13 mm) to 2 in. (50 mm)	Over 2 in. (50 mm) to 5 in. (125 mm)	Over 5 in. (125 mm)
1, 3	1,100–1,250 (595–675)	30 min	1 hr/in. (2 min/mm)	2 hr plus 15 min each additional inch (2 h plus 0.5 min/mm) over 2 in. (50 mm)	2 hr plus 15 min each additional inch (2 h plus 0.5 min/mm) over 2 in. (50 mm)
4	1,100–1,250 (595–675)	30 min	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)	5 hr plus 15 min each additional inch (5 h plus 0.5 min/mm) over 5 in. (125 mm)
5A, 5B, 5C, 6 except P-No. 6 Gr. 4 6 Gr. 4	1,250-1,400 (675-760) 1,050-1,150 (565-620)	30 min.	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)	5 hr plus 15 min each additional inch (5 h plus 0.5 min/mm) over 5 in. (125 mm)
7	1,300–1,400 (705–760)	30 min	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)	5 hr plus 15 min each additional inch (5 h plus 0.5 min/mm) over 5 in. (125 mm)
9A Gr.1 9B Gr. 1	1,100–1,250 (595–675) 1,100–1,175	30 min	1 hr/in. (2 min/mm)	1 hr/in (2 min/mm)	5 hr plus 15 min each additional inch (5 h plus 0.5 min/mm) over 5 in. (125 mm)
10A Gr. 1, 10C Gr. 1, 10F Gr. 1 10I Gr. 1	(595-635) 1,100-1,250 (595-675) 1,300-1,400 (705-760)	30 min	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)	5 hr plus 15 min each additional inch (5 h plus 0.5 min/mm) over 5 in. (125 mm)
11A Gr. 1	1,025-1,085 (550-585) 1,000-1,050 (540-565)	30 min	Or/in. (2 min/mm)	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)
11A Gr. 5 11B Gr. 1 through 10	1,100–1,150 (595–620) 1,000–1,100 (540–595)	30 min	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)
15E Gr. 1	1,350-1,425 (730-775)	30 min	1 hr/in. (2 min/mm)	1 hr/in. (2 min/mm)	5 hr plus 15 min each additional inch (5 h plus 0.5 min/mm) over 5 in. (125 mm)
P-Nos. 8, 10H Gr. 1, 34, 42, 43, 45 and hard surfacing on P-No. 1 base metal whose reported carbon	ilentit	PWHT neither required nor prohibited			

GENERAL NOTE: Exemptions to the mandatory requirements of this Table are defined in NF-4622.7. NOTE:

(1) All temperatures are metal temperatures.

content is not more than 0.30%

support. A summary of the time-temperature recording may be provided for permanent records in accordance with NCA-4134.17.

NF-4622.3 Definition of Nominal Thickness Governing PWHT. The nominal thickness in Table NF-4622.7(b)-1 is the thickness of the weld, the base material, or the thinner of the base materials being joined, whichever is least. It is not intended that nominal thickness include material provided for forming allowance, thinning, or mill overrun when the excess material does not exceed  $\frac{1}{8}$  in. (3 mm). For fillet welds, the nominal thickness is the throat thickness and, for partial penetration and material repair welds, the nominal thickness is the depth of the weld groove or preparation.

#### NF-4622.4 Holding Times at Temperature.

- (a) The holding time at temperature as specified in Table NF-4622.1-1 shall be based on the nominal thickness of the weld. The holding time need not be continuous. It may be an accumulation of the times of multiple postweld heat treatment cycles.
- (b) Holding time at temperature in excess of the minimum requirements of Table NF-4622.1-1 may be used, provided that specimens so heat treated are tested in accordance with NF-2200, NF-2400, and NF-4300.
- (c) Alternatively, when it is impractical to postweld heat treat at the temperature range specified in Table NF-4622.1-1, it is permissible to perform the postweld heat treatment of certain materials at lower temperatures for longer periods of time in accordance with Table NF-4622.4(c)-1 and (1) through (3).
- (1) Except for P-No. 1 materials, when welds in the materials listed in Table NF-4622.4(c)-1 are to be postweld heat treated at these lower minimum temperatures, the impact test specimens for the welding procedure qualification required by NF-4300 shall be made using the same minimum temperatures and increased minimum holding time. Welding procedures, qualified at the temperature range and minimum holding time specified in Table NF-4622.1-1 and at the lower minimum temperature and increased minimum holding time permitted by Table NF-4622.4(c)-1, are also qualified for any

Table NF-4622.4(c)-1 Alternative Holding Temperatures and Times					
Material P-Numbers	Alternative Minimum Holding Temperatures, °F (°C)	Alternative Minimum Holding Times			
1, 3, 9A Gr. 1, 9B Gr. 1	1,050 (565)	2 hr/in. (4 min/mm) thick			
1, 3, 9A Gr. 1, 10A Gr. 1, 9B Gr. 1	1,000 (540)	4 hr/in. (8 min/mm) thick			
SENERAL NOTE:	All other requiremen	nts of NF-4622 shall apply			

temperature in between. When such an in-between temperature is used, the minimum holding time shall be interpolated from Table NF-4622.1-1 and the alternative requirements of Table NF-4622.4(c)-1.

(2) Except for P-No. 1 materials, when welds in the materials listed in Table NF-4622.4(c)-1 are to be postweld heat treated at these lower minimum temperatures, the welding material certification required by NF-2400 shall be made using the same minimum temperatures and increased minimum holding time. Welding material certified at the temperature range and minimum holding time specified in Table NF-4622.1-1, and at the lower minimum temperatures and increased minimum holding time permitted by Table NF-4622.4(c)-1 is also certified for any temperature in between.

(3) Base material certified in accordance with NF-2210 may be postweld heat treated at these lower minimum temperatures and increased minimum holding times without recertification. Postweld heat treatment at these lower minimum temperatures and increased minimum holding times may also be the tempering operation, provided a higher tempering temperature is not required by the material specification.

NF-4622.5 PWHT Requirements When Different P-Number Materials Are Joined. When materials of two different P-Number groups are joined by welding, the applicable postweld heat treatment shall be that specified in Table NF-4622.1-1 for the material requiring the higher PWHT temperature range.

**NF-4622.6 PWHT Requirements for Non-pressure-Retaining Parts.** When non-pressure-retaining material is welded to pressure-retaining material, the postweld heat treatment temperature range of the pressure-retaining material shall control.

**NF-4622.7 Exemptions to Mandatory Require-** (**ments.** Postweld heat treatment in accordance with this subarticle is not required for

- (a) nonferrous materials
- (b) welds exempted in Table NF-4622.7(b)-1 or Table NF-4622.7(b)-2
- (c) welds subjected to temperatures above the PWHT temperature range specified in Table NF-4622.1-1, provided the Welding Procedure Specification is qualified in accordance with Section IX and the base material and the deposited weld material has been heat treated at the higher temperature
- (d) postweld heat treatment is not required for supports constructed of Type 405 material or of Type 410 material with carbon content not to exceed 0.08%, welded with electrodes that produce an austenitic chromium-nickel weld deposit or a non-air-hardening nickel-chromium-iron weld deposit, provided the plate nominal thickness at the welded joint does not exceed  $\frac{3}{8}$  in. (10 mm), and for thicknesses over  $\frac{3}{8}$  in. to  $1\frac{1}{2}$  in.

# Table NF-4622.7(b)-1 Exemptions to Mandatory PWHT

P-Number (Section IX, QW-420)	Type of Weld [Note (1)]	Nominal Thickness (NF-4622.3)	Max. Reported Carbon, % [Note (2)]	Min. Prehea Required, °F (°C)
1	All welds where the materials being joined are $1\frac{1}{2}$ in. (38 mm) and less	$1\frac{1}{4}$ in. (32 mm) and less Over $1\frac{1}{4}$ in. (32 mm) to $1\frac{1}{2}$ in. (38 mm)	0.30 or less 0.30 or less	 200 (95)
		<sup>3</sup> / <sub>4</sub> in. (19 mm) or less Over <sup>3</sup> / <sub>4</sub> in. (19 mm) to 1 <sup>1</sup> / <sub>2</sub> in. (38 mm)	Over 0.30 Over 0.30	 200 (95)
	All welds in material over $1\frac{1}{2}$ in. (38 mm)	$\frac{3}{4}$ in. (19 mm) or less		200 (95)
3 except Gr. 3	All welds	$\frac{1}{2}$ in. (13 mm) or less	0.25 or less	200 (95)
4	All welds in pipes NPS 4 (DN 100) or less or tubes with nominal O.D. 4.5 in. (114 mm) or less and attachment welds	$\frac{1}{2}$ in. (13 mm) or less	0.15 or less	250 (120)
5A, 5B, 5C	All welds in pipes or tubes with maximum specified chromium 3.00% or less, NPS 4 (DN 100) or less pipe, and nominal O.D. 4.5 in. (114 mm) or less tubes	<sup>1</sup> / <sub>2</sub> in. (13 mm) or less	0.15 or less	300 (150)
6 (for Type 410S) or 7 Gr. 1 (for Type 405)	Type 405 and 410S welded with A-No. 8, A-No. 9, or F-No. 43 filler metal	<sup>3</sup> / <sub>8</sub> in. (10 mm) or less	0.08 or less	
9A Gr. 1	All welds, provided the procedure qualification is made using equal or greater thickness base material than the production weld [Note (3)]	5/ <sub>8</sub> in. (16 mm) or less		200 (95)
	Attachment welds joining non-pressure-retaining material to pressure-retaining material over 5/8 in. (16 mm)	½ in. (13 mm) or less		200 (95)
	All welds in pipes NPS 4 (DN 100) or less of tubes with nominal O.D. 4.5 in. (114 mm) and less	½ in. (13 mm) or less	0.15 or less	250 (120)
9B Gr. 1	All welds, provided the procedure qualification is made using equal or greater thickness base material than the production weld [Note (3)]	5/8 in. (16 mm) or less		200 (95)
	Attachment welds joining non-pressure-retaining material to pressure-retaining material over 5/8 in. (16 mm)	$\frac{1}{2}$ in. (13 mm) or less		200 (95)
10A Gr. 1	All welds, provided the procedure qualification is made using equal or greater thickness base material than the production weld	<sup>5</sup> / <sub>8</sub> in. (16 mm) or less		200 (95)
	Attachment welds joining non-pressure-retaining material to pressure-retaining material over 5% in. (16 mm)	½ in. (13 mm) or less	0.25 or less	200 (95)
You	All welds in pipes or tubes to pressure-retaining material over $\frac{5}{8}$ in. (16 mm)	½ in. (13 mm) or less	0.25 or less	200 (95)
10C Gr. 1	All welds in materials $1\frac{1}{2}$ in. (38 mm) and less	1½ in. (32 mm) or less	0.30 or less	
10C Gr. 1	Attachment welds joining non-pressure-retaining material to pressure-retaining material over	Over $1\frac{1}{4}$ in. (32 mm) to $1\frac{1}{2}$ in. (38 mm)		200 (95)
	1 <sup>1</sup> / <sub>4</sub> in. (32 mm)	½ in. (13 mm) or less		200 (95)
	Groove or fillet welds attaching nozzle connections not over 2 in. (50 mm) finished I.D. in material over 1½ in. (32 mm), provided ligaments do not require an increased shell or head thickness	½ in. (13 mm) or less		200 (95)
10I Gr. 1	All welds in material $\frac{1}{2}$ in. (13 mm) and less	<sup>1</sup> / <sub>2</sub> in. (13 mm) or less	l	
11A Gr. 1	All welds in material $\frac{1}{2}$ in. (13 mm) and less	$\frac{1}{2}$ in. (13 mm) or less		
	All welds in material $\frac{1}{2}$ in. (13 mm) and less	$\frac{1}{2}$ in. (13 mm) or less		250 (120)

**17**)

# Table NF-4622.7(b)-1 **Exemptions to Mandatory PWHT (Cont'd)**

P-Number (Section IX, QW-420)	Type of Weld [Note (1)]	Nominal Thickness (NF-4622.3)	Max. Reported Carbon, % [Note (2)]	Min. Preheat Required, °F (°C)
11B Gr. 1	All welds in materials $\frac{1}{2}$ in. (13 mm) and less	$\frac{1}{2}$ in. (13 mm) or less		
through Gr. 10				

GENERAL NOTE: The exemptions noted in this table do not apply to the following:

- (1) electron beam welds in ferritic materials over  $\frac{1}{8}$  in. (3 mm) in thickness
- (2) inertia and friction welds in material of any thickness of P-No. 3, P-No. 4, P-No. 5, P-No. 7 (except for Types 405 and 4108), P-No. 10, and P-No. 11 materials

### NOTES:

- (1) Where the thickness of material is identified in the Type of Weld column, it is the thickness of the base material at the welded joint.
- (2) Carbon level of the materials being joined.
- (3) Weld Procedure Qualification coupon need not exceed 1.5 in. (38 mm) in thickness.

# Table NF-4622.7(b)-2 Additional Exemptions to Mandatory PWHT

		<u>-</u>			
P-Number (Section IX, QW-420)	Type of Weld	Nominal Thickness (NF-4622.3)	Maximum Reported Carbon % [Note (1)]	Minimum Preheat Required, °F (°C)	Minimum Lateral Expansion, mils (mm) [Note (2)]
1 Gr. 1	All welds	1½ in. (38 mm) or less	. 📎	50 (10)	NR
	All welds	Over $1\frac{1}{2}$ in. (38 mm) to 4 in. (100 mm)		200 (95)	25 (0.64)
1 Gr. 2 and 3	All welds	1 in. (25 mm) or less	0.30 or less	50 (10)	NR
	All welds	Over 1 in. (25 mm) to 4 in. (100 mm)	0.30 or less	200 (95)	25 (0.64)
3 Gr. 1	All welds	<sup>5</sup> / <sub>8</sub> in. (16 mm) or less	0.25 or less	150 (65)	NR
	All welds	Over $\frac{5}{8}$ in. (16 mm) to 1 in. (25 mm)	0.25 or less	150 (65)	15 (0.38)
	All welds	Over 1 in. (25 mm) to $1\frac{1}{2}$ in. (38 mm)	0.25 or less	150 (65)	25 (0.64)
	All welds	Over $1\frac{1}{2}$ in. (38 mm) to 4 in. (100 mm)	0.25 or less	250 (120)	25 (0.64)
3 Gr. 3	All welds	<sup>5</sup> / <sub>8</sub> in. (16 mm) or less	0.25 or less	200 (95)	NR
	All welds	Over <sup>5</sup> / <sub>8</sub> in. (16 mm) to 1 in. (25 mm)	0.25 or less	200 (95)	15 (0.38)
	All welds	Over 1 in. (25 mm) to $1^{1}/_{2}$ in. (38 mm)	0.25 or less	200 (95)	25 (0.64)
Ciic	All welds	Over $1\frac{1}{2}$ in. (38 mm) to 4 in. (100 mm)	0.25 or less	300 (150)	25 (0.64)

### GENERAL NOTES:

- (a) In addition to the requirements of NF-5120, magnetic particle or liquid penetrant examination shall be performed at ambient temperature after sufficient time (no sooner than 48 hr after reaching ambient temperature).
- the exemptions noted in this table do not apply to electron beam welds in ferritic materials over  $\frac{1}{16}$  in. (3 mm) in thickness.
- The exemptions noted in this table do not apply to inertia or friction welds in P-No. 3 materials of any thickness.
- Carbon and low alloy steel electrodes shall be supplied with a diffusible hydrogen limit of H4 for SMAW electrodes and H8 for flux cored electrodes.

- (1) Carbon level of the materials being joined.
- (2) Impact testing is required where lateral expansion values are listed, and shall be performed in accordance with the requirements of NF-2300, NF-2400, and NF-4335. When impact testing is required, the base material, heat affected zone, and weld shall be tested. "NR" indicates impact testing is not required.

(10 mm to 38 mm), provided a preheat of 450°F (230°C) is maintained during welding and the joints are completely radiographed

# NF-4623 PWHT Heating and Cooling Rate Requirements

Above 800°F (425°C), the rate of heating and cooling in any hourly interval shall not exceed 400°F (220°C) divided by the maximum thickness in inches of the material being heat treated, but shall not exceed 400°F (220°C) and need not be less than 100°F (56°C) in any hourly interval. During the heating and cooling period there shall not be a greater variation in temperature than 250°F (140°C) within any 15 ft (4.5 m) interval of weld length. The exceptions of (a) and (b) are permitted.

- (a) P-No. 6 material may be cooled in air from the postweld heat treatment holding temperature specified in Table NF-4622.1-1.
- (b) For P-No. 7 materials, the cooling rate at temperatures above 1,200°F (650°C) shall not exceed 100°F/hr (56°C/h) after which the rate of cooling shall be sufficiently rapid to prevent embritlement.

### NF-4624 Methods of Postweld Heat Treatment

The postweld heat treatment shall be performed in accordance with the requirements of one of the following subparagraphs.

**NF-4624.1 Furnace Heating** — **One Heat.** Heating the support or item in a closed furnace in one heat is the preferred procedure and should be used whenever practical. The furnace atmosphere shall be controlled so as to avoid excessive oxidation, and direct impingement of flame on the support or item is prohibited.

**NF-4624.2 Furnace Heating** — **More Than One Heat.** The support or item may be heated in more than one heat in a furnace, provided the furnace atmosphere control requirements of NF-4624.1 apply and overlap of the heated sections of the support or item is at least 5 ft (1.5 m). When this procedure is used, the portion of the support or item outside the furnace shall be shielded so that the temperature gradient is not harmful. The cross section where the support or item projects from the furnace shall not intersect a structural discontinuity.

NF-4624.3 Local Heating. Welds may be locally postweld heat treated when it is not practical to heat treat the entire support or item. Local postweld heat treatment shall consist of heating a circumferential band around the support or item at temperatures within the ranges specified in this subarticle. The minimum width of the controlled band at each side of the weld, on the face of the greatest weld width, shall be the thickness of the weld or 2 in. (50 mm), whichever is less. The temperature of the support or item from the edge of the controlled band outward shall be gradually diminished so as to avoid harmful thermal gradients. This procedure may also be used for postweld heat treatment after repairs.

**NF-4624.4 Internal Heating.** The support or item may be heated internally by any appropriate means and with adequate indicating and recording temperature devices to aid in the control and maintenance of a uniform distribution of temperature in the support or item. Previous to this operation, the support or item should be fully enclosed with insulating material.

# NF-4660 HEAT TREATMENT OF ELECTROSLAG WELDS

Electroslag welds in ferritic material over  $1\frac{1}{2}$  in. (38 mm) in nominal thickness at the joints shall be given a grain refining heat treatment.

# NF-4700 REQUIREMENTS FOR BOLTED CONSTRUCTION

# NF-4710 BOLTING AND THREADING

# NF-4711 Thread Engagement

The threads of all bolts or studs shall be engaged for the full length of thread in the load carrying nut unless otherwise specified in the Design Documents.

# NF-4712 Thread Lubricants

Any lubricant or compound used in threaded joints shall be suitable for the service conditions and shall not react unfavorably with any support element material. Contact surfaces within friction-type joints shall be free of lubricants.

### NF-4713 Removal of Thread Lubricants

All threading lubricants or compounds shall be removed from surfaces which are to be welded.

# NF-4720 BOLTING NF-4721 Bolt Holes

For the purpose of this Article, high-strength bolts shall be considered those with specified minimum yield strength greater than 80 ksi (550 MPa). Bolt holes shall meet the requirements of (a) through (e).

- (a) Holes for nonfitted bolts shall meet the requirements of Table NF-4721(a)-1. For anchor bolts set in concrete or concrete expansion anchor, the hole sizes indicated in this Subsection may be increased by  $\frac{1}{16}$  in. (1.5 mm). When the bolt hole size is  $\frac{1}{8}$  in. (3 mm) larger than the bolt, and the bolt is  $\frac{1}{2}$  in. (13 mm) or smaller, standard washers shall be used.
- (b) Oversized or slotted bolt holes may be used with high-strength bolts  $\frac{1}{2}$  in. (13 mm) in diameter and larger, except as restricted in (1), (2), and (3).
- (1) Oversized holes shall not exceed the requirements of Table NF-4721(b)-1. They may be used in any or all plies of friction-type connections. Hardened washers shall be installed over exposed oversized holes.

Table NF-47 Bolt Hole	
Bolt Size	Hole Size
≤1 in. (25 mm)	Bolt diameter plus $\frac{1}{16}$ in. (1.5 mm)
$1\frac{1}{8}$ in. to 2 in. (28 mm to 50 mm)	Bolt diameter plus $\frac{1}{8}$ in. (3 mm)
>2 in. (50 mm)	Bolt diameter plus $\frac{3}{16}$ in. (5 mm)

(2) Short-slotted holes shall not be wider than permitted by (a) and shall not have a length exceeding the oversize diameter allowed in (1) by more than  $^{1}/_{16}$  in. (1.5 mm). They may be used in any or all plies of friction-type or bearing-type connections. The slots may be used without regard to direction of loading in friction-type connections, but shall be normal to the direction of the load in bearing-type connections. Hardened washers shall be installed over exposed short-slotted holes.

(3) Long-slotted holes shall not be wider than permitted by (a) and shall not have a length which exceeds  $2\frac{1}{2}$  times the bolt diameter. In friction-type connections, the long-slotted holes may be used without regard to direction of loading, provided the stress on the bolts does not exceed 75% of the allowable working stress given in Article NF-3000. In bearing-type connections, the long diameter of the slot shall be normal to the direction of loading. Long-slotted holes may be used in only one of the connected parts of either a friction-type or bearingtype connection at an individual faying surface. Structural plate washers or a continuous bar not less than  $\frac{5}{16}$  in. (8 mm) in thickness shall be used to cover long slots that are in the outer plies of joints. These washers or bars shall have a size sufficient to cover the slot completely after installation and shall meet the requirements of Article

(c) Except as specified in (d), holes may be punched, provided the thickness of the material is not greater than the nominal diameter of the bolt plus  $\frac{1}{8}$  in. (3 mm). When the thickness of the material is greater than the nominal diameter of the bolt plus  $\frac{1}{8}$  in. (3 mm), holes shall be drilled, subpunched, and reamed, or thermally cut. Thermal cutting shall not be used unless the load-bearing

# Table NF-4721(b)-1 Tolerances on Oversized Holes

Bolt Size	Hole Size
$\leq \frac{7}{8}$ in. (22 mm)	Bolt diameter plus $\frac{3}{16}$ in. (5 mm)
1 in. (25 mm)	Bolt diameter plus $\frac{1}{4}$ in. (6 mm)
> 1 in. (25 mm)	Bolt diameter plus $\frac{5}{16}$ in. (8 mm)

surfaces are machined or ground smooth. For subpunched holes, the die shall be at least  $\frac{1}{16}$  in. (1.5 mm) smaller than the nominal diameter of the bolt.

(d) Bolt holes in material over  $\frac{1}{2}$  in. (13 mm) thick having a specified minimum yield strength greater than 80 ksi (550 MPa) shall be drilled.

(e) For bolts not subjected to shear, the limits for oversized and slotted holes in (d) may be increased if structural plate washers or continuous bars which meet the requirements of Article NF-3000 are provided.

# NF-4722 Bolted Connections

(a) Surfaces of bolted parts in contact with the bolt head and nut shall not have a slope of more than 1:20 with respect to a plane normal to the bolt axis. Where the surface of high-strength bolted part has a slope of more than 1:20, a beveled washer shall be used to compensate for the lack of parallelism.

(b) Bolts loaded in pure shear shall not have threads located in the load-bearing part of the shank unless permitted by the Design Specifications.

# NF-4723 Precautions Before Bolting

All parts assembled for bolting shall have contact surfaces free from scale, chips, or other deleterious material. Surfaces and edges to be joined shall be smooth, uniform, and free from fins, tears, cracks, and other defects which would degrade the strength of the joint.

### NF-4724 Bolt Tension

All high-strength structural bolts shall be preloaded to a value not less than that given in the Design Specifications. Preloading shall be monitored by the turn of nut method, by properly calibrated wrenches, by load indicating washers, or by direct extension indicators. Bolts preloaded by means of a calibrated wrench shall be installed with a hardened washer under the nut or bolt head, whichever is the element turned in preloading. Hardened washers are required under the bolt head and the nut when the direct extension or load indicating washer method is used. Hardened washers are not required when bolts are preloaded by the turn of nut method, except that hardened washers are required under the nut and bolt head when the bolts are used to connect material having a specified yield strength less than 40 ksi (270 MPa).

# NF-4725 Locking Devices

**NF-4725.1** Types of Locking Devices. Threaded fasteners shall be provided with locking devices to prevent loosening during service. Elastic stop nuts and thread locking compounds (when compatible with service conditions), lock nuts, including full or jam, slotted, drilled and wired, free spinning and prevailing torque are all acceptable locking devices. Upset threads by cold working or

tack welding may serve as locking devices. Internally and externally toothed washers, and disk and helical spring lock washers shall not be used as locking devices.

Preloading of High-Strength Fasteners. Preloading threaded fasteners made of material with a yield strength of 80 ksi (550 MPa) or greater, loaded in tension, to a value at least 20% above the maximum load on the fastener for the specified loading conditions, limited to 70% of the specified minimum tensile strength of the fastener material satisfies the requirement for

fasteners in an assembly with fastener material of a yield strength below 80 ksi (550 MPa) is an acceptable method for locking provided the resulting preload is at least 20% above the maximum load on the fastener for the specified loading conditions, but is limited to 70% of the specified minimum tensile strength of the fastener. The threaded assembly shall be tested for the dynamic loading condication tions specified in the Design Specification, and the estab-A control of the state of the s lished preload shall be verified on the assembly by properly calibrated wrenches, direct extension indicators, or the turn of the nut method. The results of the test, required preload, and specified thread lubrication shall be

# ARTICLE NF-5000 EXAMINATION

# NF-5100 GENERAL REQUIREMENTS FOR EXAMINATION

# NF-5110 PROCEDURES, QUALIFICATIONS, AND EVALUATIONS

# NF-5111 General Requirements

Nondestructive examinations shall be conducted in accordance with the examination methods of Section V, except as they may be modified by the requirements of this Article. Radiographic examination shall be in accordance with Section V, Article 2, except that the geometric unsharpness shall not exceed the limits of Section V, Article 2, T-274.2. Ultrasonic examination shall be in accordance with Section V, Article 4; magnetic particle examination shall be in accordance with Section V, Article 7; liquid penetrant examination shall be in accordance with Section V, Article 6; and visual examination shall be in accordance with Section V, Article 9. The examinations required by this Article or by reference to this Article shall be performed by personnel who have been qualified as required by this Article. The results of the examinations shall be evaluated in accordance with the acceptance standards of this Article.

# NF-5112 Nondestructive Examination Procedures

All nondestructive examinations required by this Article shall be performed in accordance with detailed written procedures which have been proven by actual demonstration to the satisfaction of a Level III Examiner. The procedures shall comply with the appropriate Article of Section V for the particular examination method. The digitization of radiographic film and radioscopic images shall meet the requirements of Section V, Article 2, Mandatory Appendix (II) "Digital Image Acquisition, Display and Storage for Radiography and Radioscopy." Written procedures and records of demonstration of procedure capability and personnel qualification shall be maintained on file as detailed in the Certificate Holder's Quality Program. At least one copy of the procedure shall be readily available to all applicable nondestructive examination personnel for reference and use.

### NF-5113 Post-Examination Cleaning

Following any nondestructive examination in which examination material is applied to the piece, the piece shall be thoroughly cleaned in accordance with applicable material or procedure specifications.

# NF-5114 Rounding-Off of Weld Measurements

In order to properly evaluate specified dimensional sizes, measurements shall be rounded off to the nearest  $\frac{1}{32}$  in. (0.8 mm) for undercut and  $\frac{1}{16}$  in. (1.5 mm) for weld size. Other dimensions of welds shall be rounded off to the nearest  $\frac{1}{16}$  in. (3 mm) or as otherwise specified in the design documents.

# NF-5120 TIME OF EXAMINATION OF WELDS

Examinations of welds required by NF-5200 shall be performed at the times stipulated in (a) and (b) during fabrication and installation.

- (a) Radiography of welds may be performed prior to any postweld heat treatment.
- (b) Magnetic particle or liquid penetrant examination shall be performed after any postweld heat treatment, except that welds in P-No. 1 material may be examined before or after postweld heat treatment.

# NF-5200 REQUIRED EXAMINATION OF WELDS

# NF-5210 EXAMINATION OF CLASS 1 SUPPORT WELDS

### NF-5211 Scope

The requirements of this subsubarticle shall apply to all Class 1 supports except that supports for Class 1 piping, NPS 2 (DN 50) and less, may be examined by the rules of NF-5220.

### NF-5212 Primary Member Welded Joints

- (a) All full penetration butt-welded joints in primary members shall be examined by the radiographic method.
- (b) All other welded joints in primary members shall be examined by the liquid penetrant or magnetic particle method, except that the exposed ends of welds need only be visually examined.

# NF-5213 Secondary Member Welded Joints

All welded joints in secondary members shall be examined by the visual method.

# NF-5214 Special Requirements

For weldments that impose loads in the through thickness direction of primary members 1 in. and greater in thickness, the base material beneath the weld shall be ultrasonically examined, when required by NF-4440, over 100% of the referenced area using the procedure of SA-577 or SA-578, as detailed in Section V to the acceptance standards of NF-5332, except that a calibration block representative of the primary member shall be used. The block for straight beam examination shall have 1/4 in. (6 mm) diameter flat-bottomed holes at one-fourth, one-half, and three-fourths of the thickness of the member being welded, from which a distance amplitude curve shall be established.

# NF-5220 EXAMINATION OF CLASS 2 AND MC SUPPORT WELDS

# NF-5221 Primary Member Welded Joints

- (a) All butt-welded joints in primary members shall be examined by the liquid penetrant or magnetic particle method.
- (b) All partial penetration or fillet welds in primary members that have a groove depth or throat dimensions greater than 1 in. (25 mm) and T-welded joints welded with throat dimensions of  $\frac{1}{2}$  in. (13 mm) or greater shall be examined by the liquid penetrant or magnetic particle method, except that the exposed ends of welds need only be visually examined.
- (c) All primary welds exclusive of those described in (a) and (b) shall be examined by the visual method.

# NF-5222 Secondary Member Welded Joints

All welded joints in secondary members shall be examined by the visual method.

# NF-5224 Special Requirements

For weldments that impose loads in the through thickness direction of primary members 1 in. (25 mm) and greater in thickness, the base material beneath the weld shall be ultrasonically examined, when required by NF-4440, over 100% of the referenced area using the procedure of SA-577 or SA-578, as detailed in Section V to the acceptance standards of NF-5332, except that a calibration block representative of the primary member shall be used. The block for straight beam examination shall have  $\frac{1}{4}$  in. (6 mm) diameter flat-bottomed holes at one-

fourth, one-half, and three-fourths of the thickness of the member being welded, from which a distance amplitude curve shall be established.

# NF-5230 EXAMINATION OF CLASS 3 SUPPORT WELDS

# NF-5231 Primary Member Welded Joints

- (a) Primary member welded joints that have a groove depth or throat dimension greater than 1 in. (25 mm) shall be examined by the liquid penetrant or magnetic particle method, except that the exposed ends of welds need only be visually examined.
- (b) Primary welded joints exclusive of those described in (a) shall be examined by the visual method.

# NF-5232 Secondary Member Welded Joints

All welded joints in secondary members shall be examined by the visual method.

# NF-5234 Special Requirements

For weldments that impose loads in the through thickness direction of primary members 1 in. (25 mm) and greater in thickness, the base material beneath the weld shall be ultrasonically examined, when required by NF 4440, over 100% of the referenced area using the procedure of SA-577 or SA-578, as detailed in Section V to the acceptance standards of NF-5332, except that a calibration block representative of the primary member shall be used. The block for straight beam examination shall have ½ in. (6 mm) diameter flat-bottomed holes at one-fourth, one-half, and three-fourths of the thickness of the member being welded, from which a distance amplitude curve shall be established.

# NF-5240 INERTIA AND CONTINUOUS DRIVE FRICTION WELDS

- (a) When radiographic examination is required by this Article, inertia and continuous drive friction welds shall also be examined by the ultrasonic method to verify bonding over the entire area.
- (b) The materials used shall be those assigned a P-Number by Section IX, but shall not include rimmed or semikilled steel.
- (c) One of the two parts to be joined must be held in a fixed position and the other part rotated. The two faces to be joined must be symmetrical with respect to the axis of rotation.
- (d) The weld between the two members shall be a full penetration weld.

# NF-5300 ACCEPTANCE STANDARDS

# NF-5320 RADIOGRAPHIC ACCEPTANCE STANDARDS

### NF-5321 Evaluation of Indications

Indications shown on the radiographs of welds and characterized as imperfections are unacceptable under the following conditions:

- (a) any indication characterized as a crack or zone of incomplete fusion or penetration
- (b) any other elongated indication which has a length greater than
  - $(1)^{1}/_{4}$  in. (6 mm) for t up to  $^{3}/_{4}$  in. (19 mm), inclusive
- $(2)^{1}/_{3}t$  for t from  $^{3}/_{4}$  in. (19 mm) to  $2^{1}/_{4}$  in. (56 mm), inclusive
- (3)  $\frac{3}{4}$  in. (19 mm) for t over  $2\frac{1}{4}$  in. (56 mm) where t is the thickness of the thinner portion of the weld
- (c) internal root weld conditions are acceptable when the density change as indicated in the radiograph is not abrupt; elongated indications on the radiograph at either edge of such conditions shall be unacceptable, as provided in (b)
- (d) any group of aligned indications having an aggregate length greater than t in a length of 12t unless the minimum distance between successive indications exceeds 6L, in which case the aggregate length is unlimited, L being the length of the largest indication
- (e) rounded indications are not a factor in the acceptability of welds that are radiographed

# NF-5330 ULTRASONIC ACCEPTANCE STANDARDS

All imperfections which produce a response greater than 20% of the reference level shall be investigated to the extent that the operator can determine the shape, identity, and location of all such imperfections and evaluate them in terms of the acceptance standards as given in (a) and (b).

- (a) Imperfections are unacceptable if the indications exceed the reference level amplitude and have lengths exceeding
  - (1)  $\frac{1}{4}$  in. (6 mm) for t up to  $\frac{3}{4}$  in. (19 mm), inclusive
- (2)  $\frac{7}{3}t$  for t from  $\frac{3}{4}$  in. (19 mm) to  $2\frac{1}{4}$  in. (56 mm), inclusive
- (3)  $\frac{3}{4}$  in. (19 mm) for t over  $2\frac{1}{4}$  in. (56 mm) where t is the thickness of the weld being examined; if a weld joins two members having different thicknesses at the weld, t is the thinner of these two thicknesses.
- (b) Indications characterized as cracks, lack of fusion, or incomplete penetration are unacceptable regardless of length.

# NF-5332 Acceptance Standards for Laminar Indications

Any indication detected in the base material beneath the weld which is of a laminar type is unacceptable if the indication cannot be contained within a circle having a diameter equal to one-half of the thickness of the thinner of the members joined.

# NF-5340 MAGNETIC PARTICLE ACCEPTANCE STANDARDS

# NF-5341 Evaluation of Indications

- (a) Mechanical discontinuities at the surface are revealed by the retention of the examination medium. All indications are not necessarily defects, however, since certain metallurgical discontinuities and magnetic permeability variations may produce similar indications which are not relevant.
- (b) Any indication which is believed to be nonrelevant shall be reexamined by the same or other nondestructive examination methods to verify whether or not actual defects are present. Surface conditioning may precede the reexamination. After an indication has been verified to be nonrelevant, it is not necessary to reinvestigate repetitive nonrelevant indications of the same type. Nonrelevant indications which would mask defects are unacceptable.
- (c) Relevant indications are indications which result from imperfections. Linear indications are indications in which the length is more than three times the width. Rounded indications are indications which are circular or elliptical with the length equal to or less than three times the width.

### NF-5342 Acceptance Standards

(a) Only imperfections producing indications with major dimensions greater than  $^{1}/_{16}$  in. (1.5 mm) are required to be evaluated for acceptance.

**(17)** 

- (b) Imperfections producing the following indications are unacceptable:
- (1) linear indications with dimensions greater than  $\frac{1}{16}$  in. (1.5 mm)
- (2) rounded indications with dimensions greater than  $^{3}\!/_{\!16}$  in. (5 mm)
- (3) four or more rounded indications in a line separated by  $^{1}/_{16}$  in. (1.5 mm) or less edge to edge
- (4) ten or more rounded indications in any 6 in.<sup>2</sup> (4 000 mm<sup>2</sup>) of surface with the major dimension of this area not to exceed 6 in. (150 mm), with the area taken in the most unfavorable location relative to the indications being evaluated

# NF-5350 LIQUID PENETRANT ACCEPTANCE STANDARDS

### NF-5351 Evaluation of Indications

- (a) Mechanical discontinuities at the surface are revealed by bleeding out of the penetrant; however, localized surface discontinuities such as may occur from machining marks or surface conditions may produce similar indications which are not relevant.
- (b) Any indication which is believed to be nonrelevant shall be reexamined to verify whether or not actual defects are present. Surface conditioning may precede the reexamination. Nonrelevant indications and broad areas of pigmentation which would mask defects are unacceptable.
- (c) Relevant indications are indications which result from imperfections. Linear indications are indications in which the length is more than three times the width. Rounded indications are indications which are circular or elliptical with the length equal to or less than three times the width.

### (17) NF-5352 Acceptance Standards

- (a) Only imperfections producing indications with major dimensions greater than  $\frac{1}{16}$  in. (1.5 mm) are required to be evaluated for acceptance.
- (b) Imperfections producing the following indications are unacceptable:
- (1) linear indications with dimensions greater than  $\frac{1}{16}$  in. (1.5 mm)
- (2) rounded indications with dimensions greater than  $\frac{3}{16}$  in. (5 mm)
- (3) four or more rounded indications in a line separated by  $\frac{1}{16}$  in. (1.5 mm) or less edge to edge
- (4) ten or more rounded indications in any 6 in.<sup>2</sup>  $(4\,000\,\text{mm}^2)$  of surface with the major dimensions of this area not to exceed 6 in. (150 mm), with the area taken in the most unfavorable location relative to the indications being evaluated

# NF-5360 VISUAL ACCEPTANCE STANDARDS

When visual examination is performed in accordance with NF-5200, the acceptance standards shall be in accordance with the following:

- (a) Cracks are unacceptable.
- (b) A fillet weld is permitted to be less than the size specified by  $\frac{1}{16}$  in. (1.5 mm) for one-fourth the length of the weld. Oversized fillet welds are acceptable if the oversized weld does not interfere with mating parts.
- (c) In fillet welds, incomplete fusion of  $^3/_8$  in. (10 mm) in any 4 in. (100 mm) segment, and  $^1/_4$  in. (6 mm) in welds less than 4 in. (100 mm) long, is acceptable. For groove welds, incomplete fusion is not acceptable. For fillet and groove welds, rounded end conditions that occur in welding (starts and stops) shall not be considered indications of incomplete fusion and are irrelevant.

- (d) Overlap is acceptable provided the criteria for weld size and fusion can be satisfied. When fusion in the overlap length cannot be verified, an overlap length of  $^3/_8$  in. (10 mm) in any 4 in. (100 mm) segment, and  $^1/_4$  in. (6 mm) in welds less than 4 in. (100 mm) long, is acceptable.
- (e) Craters are acceptable when the criteria for weld size are met. Craters that occur outside the specified weld length are irrelevant provided there are no cracks.
- (f) Requirements for acceptability of undercuts are as follows:
- (1) For material  $\frac{3}{8}$  in. (10 mm) and less nominal thickness, undercut depth of  $\frac{1}{32}$  in. (0.8 mm) on one side of the member for the full length of the weld, or  $\frac{1}{32}$  in. (0.8 mm) on one side for one-half the length of the weld, and  $\frac{1}{16}$  in. (1.5 mm) for one-fourth the length of the weld on the same side of the member is acceptable. For members welded on both sides where undercut exists in the same plane of a member, the cumulative lengths of undercut are limited to the lengths of undercut allowed on one side. Melt-through that results in a hole in the base metal is unacceptable.
- (2) For material greater than  $\frac{3}{16}$  in. (10 mm) nominal thickness, undercut depth of  $\frac{1}{16}$  in. (0.8 mm) for the full length of the weld and  $\frac{1}{16}$  in. (1.5 mm) for one-fourth the length of the weld on both sides of the member is acceptable. When either welds or undercut exist only on one side of the member or are not in the same plane, the allowable undercut depth of  $\frac{1}{16}$  in. (0.8 mm) may be increased to  $\frac{1}{16}$  in. (1.5 mm) for the full length of the weld.
- (g) Only surface porosity whose major surface dimension exceeds  $^{1}/_{16}$  in. (1.5 mm) shall be considered relevant. Fillet and groove welds that contain surface porosity are unacceptable if
- (1) the sum of diameters of random porosity exceeds  $\frac{3}{8}$  in. (10 mm) in any linear inch of weld or  $\frac{3}{4}$  in. (19 mm) in any 12 in. (300 mm) of weld or
- (2) four or more pores are aligned and the pores are separated by  $^{1}\!/_{16}$  in. (1.5 mm) or less, edge to edge
- (h) The length and location of welds shall be as specified on the detail drawing, except that weld lengths may be longer than specified. For weld lengths less than 3 in. (75 mm), the permissible underlength is  $^{1}/_{8}$  in. (3 mm), and for welds 3 in. (75 mm) and longer, the permissible underlength is  $^{1}/_{4}$  in. (6 mm). Intermittent welds shall be spaced within 1 in. (25 mm) of the specified location.
- (i) Arc strikes and associated blemishes on the weld or in the base material are acceptable provided no cracking is visually detected.
- (j) Slag whose major surface dimension is  $\frac{1}{8}$  in. (3 mm) or less is irrelevant. Isolated surface slag that remains after weld cleaning and does not exceed  $\frac{1}{4}$  in. (6 mm) in its major surface dimension is acceptable. [Slag is considered to be isolated when it does not occur more frequently than once per weld or more than once in a 3 in. (75 mm) weld segment.] Spatter remaining after the cleaning operation is acceptable.

# NF-5500 QUALIFICATIONS AND CERTIFICATION OF NONDESTRUCTIVE EXAMINATION PERSONNEL

# NF-5510 GENERAL REQUIREMENTS

Organizations performing Code required nondestructive examinations shall use personnel competent and knowledgeable to the degree specified by NF-5520. When these services are subcontracted by the Certificate Holder or Quality System Certificate Holder, he shall verify the qualification of personnel to the requirements of NF-5520. All nondestructive examinations required by this Subsection shall be performed by and the results evaluated by qualified nondestructive examination personnel.

# NF-5520 PERSONNEL QUALIFICATION, CERTIFICATION, AND VERIFICATION

### NF-5521 Qualification Procedure

- (a) Personnel performing nondestructive examinations shall be qualified in accordance with the recommended guidelines of SNT-TC-1A. The ACCP Level II and III provisions for qualification and certification and the ASNT administered Level II certification provision for qualification and certification of NDE personnel shall not be used for Section III. The Employer's written practice required by paragraph 5 of SNT-TC-1A shall identify Employer requirements relative to the recommended guidelines. The recommended guidelines of SNT-TC-1A shall be considered minimum requirements except as modified in (1) through (5).
- (1) Qualification of level III nondestructive examination personnel shall be by examination.
- (-a) The basic and method examinations, paras. 8.8.1 and 8.8.2 of SNT-TC-1A, may be prepared and administered by the Employer, ASNT, or an outside agency.
- (-b) The specific examinations, para. 8.8.3 of SNT-TC-1A, shall be prepared and administered by the Employer or an outside Agency. The employer or outside agency administering the specific examination shall identify the minimum grade requirement in a written program when the basic and method examinations have been administered by ASNT which issues grades on a pass/fail basis. In this case, the minimum grade for the specific examination may not be less than 80%.
- (2) The written practice identified in paragraph 5 of SNT-TC-1A and the procedures used for examination of personnel shall be referenced in the Employer's Quality Program.
- (3) The number of hours of training and experience for nondestructive examination personnel who perform only one operation of a nondestructive examination method that consists of more than one operation, or perform a nondestructive examination of limited scope, may be less than that recommended in Table 6.3.1 A of SNT-

- TC-1A. The training and experience times shall be described in the written practice and any limitations or restrictions placed on the certification shall be described in the written practice and on the certificate.
- (-a) The minimum classroom training times for visual examination personnel identified in Table 6.3.1 A of SNT-TC-1A for Level II certification may be reduced from 16 hr to 8 hr.
- (4) For the near-vision acuity examination, the Jaeger Number 1 letters shall be used in lieu of the Jaeger Number 2 letters specified in paragraph 8.2.1 of SNTC-1A. The use of equivalent type and size letters is permitted.
- (5) An NDE Level I individual shall be qualified to properly perform specific setups, specific calibrations, specific NDE, and specific evaluations for acceptance or rejection determinations according to written instructions and to record results. The NDE Level I individual shall receive the necessary instruction and supervision from a certified NDE Level II or Level III individual. A Level I individual may independently accept the results of nondestructive examinations when the specific acceptance criteria are defined in the written instructions.
- (b) For nondestructive examination methods not covered by SNT-TC-1A documents, personnel shall be qualified to comparable levels of competency by subjection to comparable examinations on the particular method involved.
- (c) The emphasis shall be on the individual's ability to perform the nondestructive examination in accordance with the applicable procedure for the intended application.
- (d) For nondestructive examination methods that consist of more than one operation or type, it is permissible to use personnel qualified to perform one or more operations. As an example, one person may be used who is qualified to conduct radiographic examination and another may be used who is qualified to interpret and evaluate the radiographic film.

### NF-5522 Certification of Personnel

- (a) The Employer retains responsibility for the adequacy of the program and is responsible for certification of Level I, II, and III nondestructive examination personnel.
- (b) When ASNT is the outside agency administering the Level III basic and method examinations [NF-5521(a)(1)(-a)], the Employer may use a letter from ASNT as evidence on which to base the certification.
- (c) When an outside agency is the examining agent for Level III qualification of the Employer's personnel, the examination results shall be included with the Employer's record.

JRDS

malification records identified in para. 9.4 of of shall be retained by the Employer.

shall be retained by the Employer.

shall be retained by the Employer.

And the Republic of Again, the shall be retained by the Employer.

And the Republic of Again, the shall be retained by the Employer.

And the Republic of Again, the shall be retained by the Employer.

And the Republic of Again, the shall be retained by the Employer.

And the Republic of Again, the shall be retained by the Employer.

And the Republic of Again, the shall be retained by the Employer.

And the Employer

# **ARTICLE NF-8000 CERTIFICATES OF AUTHORIZATION AND CERTIFICATION DOCUMENTS**

### NF-8100 **GENERAL REQUIREMENTS**

3 Division NE Ay to to 300 Qual As and Cert.

Is and Cert.

Is a separate full poly of Ashir Bay C. III. 1. H. Lashir Hay C. II. 1. H Article NCA-8000 and shall apply only to those items fabricated under an Article NCA-4000 Quality Assurance Program. Inspection by an ANI and Certification Mark

# MANDATORY APPENDIX NF-I ADDITIONAL MATERIAL FOR SUPPORTS

# **ARTICLE NF-I-1000 INTRODUCTION**

# NF-I-1100 SCOPE

ed by this App.

ed by

# C Section 3 Division 1 MK **GENERAL REQUIREMENTS** NF-I-1200

The requirements of Subsection NF shall be met except as modified by this Appendix.

# **ARTICLE NF-I-2000 MATERIALS**

# NF-I-2100 GENERAL

The additional materials, design stress intensity and allowable stress values, yield strength, and tensile strength values, listed in Table NF-I-2100-1 (U.S. Customary Units) and Table NF-I-2100-1M (SI Units) of this Appendix may be used in the construction of Class 1, 2, 3, and MC supports for Section III, Division 1, in addition to those listed in Section II, Part D. Other than the units used, Tables NF-I-2100-1 and NF-I-2100-1M are the same. The tabulated values of tensile strength and yield strength are those suitable for use in design calculations required by Section III, Division 1. At temperatures above room temperature, the values of tensile strength trend toward an average or expected value that may be as much as 10% above the tensile strength trend curves adjusted to the minimum specified room temperature tensile strength. At temperatures above room temperature, the yield strength values correspond to the yield strength trend curve adjusted to the minimum specified room temperature yield strength. Neither the tensile strength nor the vield strength values correspond exactly to either "average" or "minimum" as these terms are applied to a statistical treatment of a homogenous set of data.

Neither the ASME or ASTM material specifications nor the rules of Section III, Division 1 require elevated temperature testing for tensile or yield strengths of production material for use in Code components. It is not intended that results of such tests, if performed, be compared with these tabulated tensile and yield strength values for ASME Code acceptance/rejection purposes for materials. If some elevated temperature test results on production material appear lower than the tabulated values by a large amount (more than the typical variability of material, suggesting the possibility of some error), further investigation by retest or other means should be considered.

The applicable revisions of the ASTM specifications in Tables NF 2100-1 and NF-I-2100-1M are listed in Table NF-I-2100-2.

### NF-I-2200 **REQUIREMENTS**

### NF-I-2210

3N3 Division 1 NF When the Material Grade column Tables NF-I-2100-1 and NF-I-2100-1M references AISI grades, only materials meeting the chemical composition requirements of the specific AISI grades listed shall be used, with the exception that 0.60% maximum silicon is permitted for castings. Free machining modifications of the specific AISI grades listed may be used at the same design stress intensities, allowable stresses, and yield strengths of the reference grades but their use is limited to 400°F (205°C) maximum temperature.

# NF-I-2220

Unless specifically exempted by Tables NF-I-2100-1 and NF-1-2100-1M, the maximum measured tensile strength of the support material should not exceed 170 ksi (1 172 MPa), in view of the susceptibility of highstrength materials to brittleness and stress corrosion cracking. Certain applications may exist where a tensile strength value of up to 190 ksi (1 310 MPa) could be considered acceptable for a material. Under this condition, the Design Specification shall specify impact testing per NF-2300 for the material, and shall include requirements for the consideration of the effects of sustained loads, environment, and heat treatment on the susceptibility to stress corrosion cracking of these high-strength materials.

### NF-I-2230

When the ASTM specification referenced in Tables NF-I-2100-1 and NF-I-2100-1M does not specify minimum tensile and yield strengths, the material shall meet the values listed under the appropriate columns.

### NF-I-2240

Materials in Tables NF-I-2100-1 and NF-I-2100-1M that are listed as an AISI composition may be accepted as satisfying the requirements of the ASTM specification provided the chemical requirements of the AISI specification are within the specified range of the designated ASTM specification.

	Design 5	Stress Int	ensity	. (S <sub>m</sub> ), N		Table NF-I-2100-1 Design Stress Intensity ( $S_m$ ), Maximum Allowable Stress ( $S$ ), Yield Strength ( $S_y$ ), and Tensile Strength ( $S_u$ ) Values	Table NF-I-2100-1 ble Stress (S), Yielc	1-2100-1 (S), Yield	Strength	(S <sub>y</sub> ),	and	Tens	le St	rengt	th (S	") Va	ılues			
	Material	Condition	Jick		į			Min. Yield Strangth	Min. Tensile			( for M	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) Metal Temperatures, °F, Not to Exceed	Stress and Strength, ksi, Itiply by 1,000 to Obtain Temperatures, °F, Not to	Streng 300 to ( ures, °F	th, ksi, Obtain F, Not to	psi)	_		
Spec. No.	5	Type	Class	P/S No.	No.	Product Form	Notes	su engui, ksi	su engur, ksi		100	200	300 4	400 50	200 60	9 009	650 700	0 750		800
<b>Carbon Steels</b> A108	els 1015, 1018, 1020	CW	ŧ	P-No-	- 4K	Bar	(4), (29), (31)	40	09	$S_m$ $S_y$ $S_u$	16.7 14.3 32.0 50.0	16.7 14.3 29.2 50.0	16.7 1 14.3 1 28.3 2 50.0 5	16.7 14.3 27.4 50.0	1111	1111			1111	
A108	1015, 1018, 1020	CW	i.	:		<b>WILL</b>	(2), (4), (31)	40	09	S S S Z	20.0 17.1 40.0 60.0	20.0 17.1 36.5 60.0	20.0 2 17.1 1 35.4 3 60.0 6	20.0					1111	
A108	1045	CW	:	i	i	Bar and bolt	(1), (31)	100	120	S S S 1	40.0 34.3 100.0 120.0	40.0 34.3 91.2 120.0	40.0 4 34.3 3 88.4 8 120.0 12	40.0					1 1 1 1	
A108	1050	CW	÷	ŧ	:	Bar and bolt	(1)	125	140	S S S 1	46.7 40.0 125.0 1 140.0 1	46.7 40.0 1114.0	46.7 4 40.0 4 110.5 10 140.0 14	46.7 40.0 107.0 140.0					1 1 1 1	
A108	1117	:	÷	ŧ	:	Bar	(1), (31)	P. C.	6 VO.III.	$S_m$	23.3 20.0 60.0 70.0	23.3 20.0 54.7 70.0	23.3 2 20.0 2 53.0 5 70.0 7	23.3 20.0 51.4 70.0					1 1 1 1	
A108	1141, 12L14	Cold drawn and tempered	:	i:	:	Bolt 3 in. max.	(1), (31)	81	1050 I	~ ~ ~ ~ ~	 81.0 105.0	 73.9 105.0	 71.6 6 105.0 10	 69.3 105.0					1 1 1 1	
A108	1144	ŧ	Ē	i	ŧ	Bar	(1), (31)	100	115	S S S S S S S S S S S S S S S S S S S	38.3 32.9 100.0 115.0	38.3 32.9 91.2 150.1	38.3 3 32.9 3 88.4 8 115.0 111	38.3 32.9 85.6	1 1 1 1	1 1 1 1			1 1 1 1	
A108	1144	Cold drawn and tempered	:	i.	:	Bolt 3 in. max.	(1), (31)	105	125	$S_m$ $S_y$ $S_u$	  105.0 125.0	 95.8 125.0 13	92.8	: :					1 1 1 1	
A108	1144	i	:	i i	:	Bolt over 3 in. to 10 in. max.	(1), (31)	81	105	S S S S S S S S S S S S S S S S S S S	 81.0 105.0	73.9	 71.6 6 105.0 10	69.3	<i>C C C C C C C C C C</i>				: : : :	

	Design Stres	s (ntensis	ty (S <sub>m</sub>	,), Maxin	mnr	Table NF-I-2100-1 Design Stress ( $S_{\mu}$ ), Maximum Allowable Stress ( $S_{\nu}$ ), and Tensile Strength ( $S_{\mu}$ ) Values (Cont'd)	Table NF-I-2100-1 tress (S), Yield Str	-1-2100-1 Yield Stre	ngth (S <sub>y</sub> ),	, and	Ten	lile S	treng	th (S	, Va	ılues	(Con	t'd)		
	Material	Condition	Š		Grp.			Min. Yield Strength,	Min. Tensile Strength,			for M	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ss and y by 1,0	Stress and Strength, ksi, ttiply by 1,000 to Obtain Temperatures, °F, Not t	h, ksi, btain p Not to	psi) Excee	=		
Spec. No. Carbon Ste	Spec. No. Grade or AISI Carbon Steels (Cont'd)	Type	Class	Class P/S No.	No.	Product Form	Notes	ksi	ksi		100	200	300 4	400 50	200 600	0 650	0 200	0 750	0 800	<u>e</u> l
A108	12L14	i	ŧ	Oilen	: 4	Bar	(1)	55	65	$S_m$ $S_y$ $S_y$	21.7 18.6 55.0 65.0	21.7 18.6 50.2 65.0	21.7 2 18.6 1 48.6 4 65.0 6	21.7 18.6 47.1 65.0						
A228	i	i	E	į	is in the second	Wire	(1), (6), (30)	250	270	S <sub>m</sub> S <sub>y</sub> 2	 250.0 2 270.0 2	 220.5 20 270.0 2	 202.5 18 270.0 27	 187.5 270.0						
A381	i.	i	Y35	P-No. 1	Н	Pipe	101	35	09	$S_m$ $S_y$	 35.0 60.0	 31.9 60.0	  31.0 3 60.0 6	30.0 28	 28.3 25.	6	9	25.2	: : : :	
A500	ш	i	E	P-No. 1	1	Tube (shaped)	SME	SAIK P	28	S S S S S S S S S S S S S S S S S S S	 46.0 58.0	 41.9 4	 40.8 3	 39.4 3'	 37.2 34.	0	83	33.1		
A500	Ω	:	:	P-No. 1	н	Tube (round)	÷	24 C.	28	S S S S S S S S S S S S S S S S S S S	 42.0 58.0	38.3	 37.2 3 58.0 5	35.9 3:	 33.9 31.	0	4	30.2	1 1 1 1	
A500	U	Ĭ	:	P-No. 1	1	Tube (shaped)	÷	20	, N. 922	S S S 2	 50.0 62.0	 45.6	 44.3 4 62.0 6	 42.9 40	 40.4 37.	0	es	36.0	1 1 1 1	
A500	Ú	i	ŧ	P-No. 1	П	Tube (round)	÷	46	62	S S S S S	46.0 62.0	41.9	 40.8 3 62.0 6	 39.4 3.7 62.0	 37.2 34.	0	83	 33.1	1 1 1 1	
A501	:	ï	ŧ	P-No. 1	1	Tube	÷	36	28	$S_m$ $S_y$ $S_u$	 36.0 58.0	32.8	31.9 3	30.8 2.58.0	 29.1 26. 	9		 25.9	1 1 1 1	
A513	1015	CW	:	P-No. 1	11	Tube	(4), (29)	55	65	S S S <sub>y</sub>	16.7 14.3 32.0 50.0	16.7 14.3 29.2 50.0	16.7 14.3 28.3 50.0 5	167 143 27.4 50.0					: : : :	

	Table NF-I-2100-1 Tesign Stress Intensity ( $S_m$ ), Maximum Allowable Stress ( $S$ ), Yield Strength ( $S_y$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	s Intensi	ty (S	m), Maxim	E F	Allowable St	Table NF-I-2100-1 ress (S), Yield Str	-I-2100-1 Yield Stre	ngth (S <sub>y,</sub>	), and	Ten	sile 5	treng	gth (3	S <sub>u</sub> ) V	/alue	) s (Co	nt'd)		
	Material	Condition	Click	\ \	E			Min. Yield Strenoth	Min. Tensile			for N	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ess and ly by 1	Streng ,000 to tures, °	Stress and Strength, ksi, Itiply by 1,000 to Obtain Temperatures, °F, Not t	, ı psi) to Exce	ed		
Spec. No.	5 3	Type	Class	P/S No.	Š č	Product Form	Notes	ksi	ksi ksi		100	200	300 4	400	200	9 009	650 7		3 052	800
A513	1020	CW	ŧ	P-No.T	" the	Tube	(4), (29)	09	70	$S_m$	16.7 14.3 32.0 50.0	16.7 14.3 29.2 50.0	16.7 14.3 28.3 50.0	16.7 14.3 27.4 50.0	1111	: : : :	1111		: : : :	
A513	1015	CW	i	i	ŧ	PO P	(2), (4)	55	65	$S_m$	21.7 18.6 55.0 65.0	21.7 18.6 50.2 65.0	21.7 18.6 48.6 65.0	21.7 18.6 47.1 65.0	: : : :	: : : :	1 1 1 1		: : : :	: : : :
A513	1020	CW	i	i	ŀ		(4) (Z) (4)	09	20	$S_{n}$	23.3 20.0 60.0 70.0	23.3 20.0 54.7 70.0	23.3 20.0 53.0 70.0	23.3 20.0 51.4 70.0	i i i i	: : : :	1111		: : : :	
A513	1025	CW	i	P-No. 1	7	Tube	(5), (29)	59 BR	75	$S_m$	18.3 15.7 35.0 55.0	18.3 15.7 31.9 55.0	18.3 15.7 31.0 55.0	18.3 15.7 30.0 55.0	: : : :	: : : :	1111		: : : :	: : : :
A513	1026	CW	i	P-No. 1	т	Tube	(5), (29)	6 0.	80	$S_m$	18.3 15.7 35.0 55.0	18.3 15.7 31.9 55.0	18.3 15.7 31.0 55.0	18.3 15.7 30.0 55.0	: : : :	: : : :	1111		: : : :	: : : :
A513	1025	CW	i	÷	i	Tube	(2), (5)	65	75	S S S	25.0 21.4 65.0 75.0	25.0 21.4 59.3 75.0	25.0 21.4 58.5 75.0	25.0 21.4 55.6 75.0	1 1 1 1	1 1 1 1	1 1 1 1		1 1 1 1	
A513	1026	CW	i	:	i	Tube	(2), (5)	70	80	S S S S S S S S S S S S S S S S S S S	26.7 22.9 70.0 80.0	26.7 22.9 63.8 80.0	26.7 22.9 61.9 80.0	26.7 22.9 59.8 80.0	1 1 1 1	: : : :	1 1 1 1		1 1 1 1	::::
A519	1018, 1020, 1022	HR	Ē	P-No. 1	П	Tube	÷	32	20	S S S S S S S S S S S S S S S S S S S	16.7 14.3 32.0 50.0	16.7 14.3 29.2 50.0	16.7 14.3 28.3 50.0	16.7 14.3 27.4 50.0	1111	1 1 1 1	1 1 1 1		1 1 1 1	1 1 1 1
A519	1018, 1020, 1022	CW	÷	P-No. 1	7	Tube	(4), (29)	09	70	$S_m$ $S_y$ $S_u$	16.7 14.3 32.0 50.0	16.7 14.3 29.2 50.0	16.7 14.3 28.3 50.0	16.7 14.3 27.4 50.0	C, S		1 1 1 1		1 1 1 1	: : : :

	O Design Stres	s Intensil	ty (S <sub>m</sub>	), Maxim	mn /	Table NF-I-2100-1 Design Stress intensity ( $S_m$ ), Maximum Allowable Stress ( $S_p$ ), $S_p$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	Table NF-I-2100-1 :ress (S), Yield Str	-I-2100-1 Yield Stre	ingth (S <sub>y</sub> ),	and	Tens	ile S	treng	th (S	<sup>2</sup> \ ("	alues	(Coi	ıt'd)		
	Material	Condition	C)		Ę			Min. Yield	Min. Tensile			() for Me	Stre Multipl stal Ter	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	Strengt 100 to ( 1res, °F	th, ksi, Obtain ', Not to	psi)	ă		
Spec. No. Carbon Ste	Gr els (	Type	Class	Class P/S No.	No.	Product Form	Notes	ksi	ksi		100 2	200 3	300 4	400 50	200 60	9 009	650 70	700 7	750 8	800
A519	1018, 1020, 1022	CW	i	Ojjew	: 1	Tube	(2), (4)	09	70	S S S S S S S S S S S S S S S S S S S	23.3 20.0 60.0 70.0	23.3 20.0 54.7 70.0	23.3 2 20.0 2 53.0 5 70.0 7	23.3 20.0 51.4	: : : :				: : : :	1111
A519	1025, 1026	HR	ŧ	P-No. 1	ce.	Tube	ŧ	35	55	S S S S S S S S S S S S S S S S S S S	18.3 1 15.7 1 35.0 3	18.3 15.7 31.9 55.0	18.3 1 15.7 1 31.0 3 55.0 5	18.3 15.7 30.0 55.0					1 1 1 1	1111
A519	1025, 1026	CW	÷	P-No. 1	7	Tube	(3), (5), (29)	65	75	S. S.	18.3 1 15.7 1 35.0 3	18.3 15.7 31.9 55.0	18.3 1 15.7 1 31.0 3 55.0 5	18.3 15.7 30.0 55.0					1 1 1 1	1111
A519	1025, 1026	CW	i	i	:	Tube	(2), (8), (2)	65	75	S S S S S S S S S S S S S S S S S S S	25.0 221.4 265.0 575.0 7	25.0 221.4 25.9 25.9 275.0 75.0 7	25.0 2 21.4 2 58.5 5 75.0 7	25.0 21.4 55.6 75.0					1 1 1 1	1111
A521	1020, 1025, 1030	i	22	S-No. 1	$\leftarrow$	Forging	(31)	Se Cili	09	S <sub>m</sub> S <sub>y</sub> :	 30.0 2 60.0 6	 27.3 2 60.0	 26.6 2 60.0 6	 25.7 2 <sup>4</sup> 60.0	 24.3 22	22.2 2	21.8 2	 21.6		1111
A521	1020, 1025, 1030	ŧ	CE	S-No. 1	7	Forging	(31)	37	. A.		 37.0 75.0	 33.7 75.0	 32.8 3 75.0 7	 31.7 29 75.0	 29.9 27	27.4 20	 26.9 2 	 26.6	: : : :	1111
A521	1035, 1040	i	50	i	÷	Forging 4 in. and under	(1), (31)	55	06	\$ 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	.:. :55.0 90.0	50.2	 48.6 4 90.0 9	 47.1 4 <sup>4</sup> 90.0	44.5 40		39.9 3	39.6	: : : :	1111
A521	1035, 1040	i	90	i	i	Forging over 4 in. to 10 in. incl.	(1), (31)	50	85	S <sub>m</sub> S <sub>y</sub>	 50.0 4 85.0 8	 45.6 85.0	 44.3 4 85.0	 42.9 4( 85.0	40.4 37	37.0 36	36.3 3	36.0	: : : :	1111
A653	33	A	i	S-No. 1	-	Sheet	(13)	33	45	S S S	15.0 1 12.9 1 33.0 3	15.0 112.9 130.1 245.0	15.0 12.9 29.2 29.2 45.0	12.9 28.3 45.0	5.5	: : : :	: : : :	: : : :	: : : :	: : : :

	Table NF-I-2100-1 Design Stress Intensity ( $S_m$ ), Maximum Allowable Stress ( $S$ ), Yield Strength ( $S_v$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	s Intensi	ty (S,		unu	Allowable Si	Table NF-I-2100-1 tress (S), Yield Str	-I-2100-1 Yield Stre	ngth (S.,)	, and	Ten	 	 treng	th (S	) (a	 	(Cor	ıt'd)		
	Material	Condition	Click		Grp.			Min. Yield Strength,	Min. Tensile Strength,			for M	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ss and y by 1,( nperat	Stress and Strength, ksi, tiply by 1,000 to Obtain Temperatures, °F, Not to	h, ksi, Obtain , Not to	psi)	و ا		
Spec. No. Carbon Ste	Gr els (	Type	Class	CP/S No.	No.	Product Form	Notes	ksi	ksi		100	200	300 40	400 50	200 (	9 009	650 700		750 8	800
A668	1020, 1022, 1025, 1026, 1030	:	В	S-No.	- the	Forging	(8), (31)	30	09	$S_m$	20.0 17.1 30.0 60.0	20.0 17.1 27.3 60.0	20.0 2 17.1 1 26.6 2 60.0 6	20.0 2 17.1 25.7 2 60.0	20.0 20.0 20.0 20.0 20.0 20.0 20.0 20.0	20.0 20.0 20.0 20.0 20.0 20.0 20.0 20.0	20.0 21.8 2	21.6		1111
A668	1035	ŧ	ш	ŧ	:	Forging	(1), (8), (9), (31)	30	09	$S_m$	 30.0 60.0	 27.3 60.0			24.5 2.	22.22	21.8 2	21.6		
A668	1020, 1022, 1025, 1026, 1030	ŧ	U	S-No. 1	1	Forging	(31)	33	99	$S_m$	22.0 18.9 33.0 66.0	22.0 18.9 30.1 66.0	22.0 2 18.9 1 29.2 2 66.0 6	22.0 2 18.9 28.3 2 66.0	22.0 2:  26.5 2·	22.0 2: 24.4 2:	22.0 24.1 2.	23.8		1111
A668	1035	ŧ	ပ	Ē	i	Forging	(1), (8), (9)	33	99	S S S	 33.0 66.0	 30.1 66.0	29.2 2 66.0 6	28.3 2 66.0	26.5 2,	24.4 2.	24.1 2:	23.8	: : : :	1111
A668	1020, 1022, 1025, 1026, 1030	ŧ	Ω	S-No. 1	7	Forging	(8), (31)	37.5	75	S S S	25.0 21.4 37.5 75.0	25.0 21.4 34.2 75.0	25.0 2 21.4 2 33.2 3 75.0 7	25.0 2 21.4 32.1 3 75.0	25.0 22 30.3 2'	25.0 22	25.0 27.2 2	 27.0		
A668	1035	i	Ω	i	i	Forging	(1), (8), (9), (31)	37.5	752	S & S S	37.5	 34.2 75.0		 32.1 3	30.3 2	27.77 27	27.2 2.	 27.0		
A668	1030, 1330	ŧ	ĹT.	S-No. 1	ю	Forging 4 in. and under	(7), (8), (31)	55	06	ς ς ς, ς, ς	30.0 25.7 55.0 90.0	30.0 25.7 50.2 90.0	30.0 3 25.7 2 48.6 4 90.0 9	30.0	44.5 40		39.9	39.6		
A668	1030, 1330	i	Ĩ.	S-No. 1	т	Forging over 4 in. to 10 in. incl.	(8), (31)	20	82	S S S	28.3 24.3 50.0 85.0	28.3 24.3 45.6 85.0	28.3 2 24.3 4 44.3 4 85.0 8	28.3	40.4	37.0 36	36.3 3.	36.0		1111
A668	1035, 1040 1045	:	ĨĪ.	i	:	Forging 4 in. and under	(1), (7), (8), (9), (31)	55	06	$S_m$ $S_y$ $S_u$	 55.0 90.0	 50.2 90.0	 48.6 4 90.0	 47.1 4 90.0	44.5		39.9 3	39.6		: : : :

	5																			
	Design Stress (Mensity (S <sub>m</sub> ), Maximu	o (ntensit	ty (S <sub>m</sub>	,), Maxin	, mun	Table NF-I-2100-1 m Allowable Stress (S), Yield Strength ( $S_{ u}$ ), and Tensile Strength ( $S_{ u}$ ) Values (Cont'd)	Table NF-I-2100-1 tress (S), Yield Str		ngth (S <sub>y</sub> )	, and	l Ten	sile 9	Stren	gth (	S <sub>u</sub> ) \	/alue	) (Co	nt'd)		
	Medical		C					Min. Yield	Min. Tensile			for [	Str (Multip	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures. °F. Not to Exceed	d Stren 1,000 to	gth, ksi Obtain °F. Not	l, n psi) to Exce	ed		
Spec. No.	Material Grade or AISI	Condition	Class	P/S No.	No.	Product Form	Notes	strengtn, ksi	strengtn, ksi		100	200	300	400	500	009	650 7		750	800
Carbon Ste	Carbon Steels (Cont'd)		,	*0			3	1	ı	,										
A668	1035, 1040, 1045	:	T.	ilen	1 4	Forging over 4 in. to 10 in. incl.	(1), (8), (9), (31)	20	822	$S_{m}$	 50.0 85.0	 45.6 85.0	 44.3 85.0	 42.9 85.0	 40.4 	37.0	36.3	36.0	: : : :	1 1 1 1
A675	75	i	i	S-No. 1	2	2 Bar	(28)	37.5	75	$S_m$ $S_y$ $S_u$	25.0 21.4 37.5 75.0	25.0 21.4 34.2 75.0	25.0 21.4 33.2 75.0	25.0 21.4 32.1 75.0	25.0	25.0 27.7	25.0	 27.0 	1111	1111
LOW Alloy Steels A148 Gra 41	Steels Grade 90-60: 4130, 4320, 4330	:	:	S-No. 4	ĸ	Casting	(11), (19), (31)	09	06	$S_m$	30.0 25.7 60.0 90.0	30.0 25.7 58.2 90.0	30.0 25.7 55.9 90.0	30.0 25.7 52.8 90.0	30.0 25.7 50.1 90.0	30.0 25.7 49.8 90.0	30.0	30.0	1 1 1 1	1 1 1 1
A148	Grade 105-85: 4130, 4140, 4320, 4340	i	i	į	ŧ	Casting	(1), (40)31)	85	105	S S S S 2	35.0 30.0 85.0 105.0	35.0 30.0 82.5 105.0	35.0 30.0 79.2 105.0	35.0 30.0 74.8 105.0	35.0 30.0 71.0 105.0	35.0 30.0 70.5 105.0 1	35.0 30.0 70.5 105.0	35.0 30.0 70.5 105.0	: : : :	1111
A148	Grade 115-95: 4130, 4140, 4320, 4340	i	i	i.	i	Casting	(1), (18), (31)		115	S S S S S S S S S S S S S S S S S S S	38.3 32.9 95.0 115.0	38.3 32.9 92.1 115.0	38.3 32.9 88.5 115.0	38.3 32.9 83.5 115.0	38.3 32.9 79.3 115.0	38.3 32.9 78.9 115.0	38.3 32.9 78.9 115.0	38.3 32.9 78.9 115.0	: : : :	1111
A148	Grade 135-125: 4130, 4140, 4320, 4340	ï	:	Ē	ŧ	Casting	(1), (31)	125	135	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	45.0 38.6 125.0 135.0	45.0 38.6 121.1 135.0	45.0 38.6 1116.5 1 135.0 1	45.0 38.6 110.0 135.0	45.0 38.6 104.3 1 135.0 1	45.0 38.6 103.9 1 135.0 1	45.0 38.6 103.9 10 135.0 13	45.0 38.6 103.9 135.0	: : : :	1111
A322	4150	ï	ŧ	ŧ	ŧ	Bar	(1), (31)	100	115	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	115.0	 93.5 115.0	 90.2 115.0	 87.8 115.0	  85.1 115.0	 81.4 115.0	 78.9 115.0	 76.0 115.0	: : : :	1 1 1 1
A434	4130, 4140, 4145, 4320, 4340, 8260	ï	BB	ŧ	ŧ	Bar $1_{1}^{4}$ z in. and under	(1), (31)	06	110	S S S S S S S S S S S S S S S S S S S	 90.0 1110.0	84.1 110.0 1	81.3 110:0	 79.0	  76.6 110.0	 73.3 110.0 1	 71.0 110.0	 68.4 110.0	: : : :	1 1 1 1
A434	4130, 4140, 4145, 4320, 4340, 8260	:	BB	:	i	Bar over $1\frac{1}{2}$ in. to $2\frac{1}{2}$ in incl.	(1), (31)	80	105	S S S S S S S S S S S S S S S S S S S	 80.0 105.0	 74.8 105.0	 72.3 105.0	70.3		 65.1 105.0	 63.1 105.0	 60.0 105.0	: : : :	1 1 1 1
																)_				

	Design Stress Intensity (S <sub>m</sub> ), Maximu	s Intensity	ty (S <sub>m</sub> )	', Maxin	unu	Table NF-I-2100-1 m Allowable Stress (S), Yield Strength ( $S_{\nu}$ ), and Tensile Strength ( $S_{u}$ ) Values (Cont'd)	Table NF ress (S),	Table NF-I-2100-1 ress (S), Yield Stre	ngth (S <sub>y</sub> )	and ,	Ten	 	trenç	jth (S	) V.	alues	(Co	nt'd)		
	Material	Condition	click		r.			Min. Yield	Min. Tensile			for M	Stre Multipl etal Te	ss and y by 1,0	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	th, ksi, Obtain	psi)	<u>.</u>		
Spec. No.	Grade or AISI	Type	Class	P/S No.	No.	Product Form	Notes	su engui, ksi	su engui, ksi		100	200	300 4	400 5	200 60	9 009	650 7		750 8	800
Low Alloy S A434	Low Alloy Steels (Cont'd) A434 4130, 4140, 4145, 4320, 4340, 8260	i	BB	lien	711	Bar over $2\frac{1}{2}$ in. to 4 in. incl.	(1), (31)	75	100	$S_m$		70.1				 61.0 5	59.1 5	57.0	: : :	
A434	4130, 4140, 4145, 4320, 4340, 8260	į	BB	i	<b>&amp;</b>	Bar over 44in.to 7 in. incl.	(1), (31)	75	95		75.0	70.1	67.7 6	  65.8 6	63.8 6 95.0 99	61.0 57.0 95.0 995.0	59.1 5	57.0 95.0		
A434	4130, 4140, 4145, 4320, 4340, 8260	÷	BB	i	i	Bar over 7 in. to 9½ in. incl.	(I) (II)	65	06	$S_m$ $S_y$	65.0	60.7	 58.6 90.0	 57.1 5 90.0 9	555.3 5:	 52.9 5	 51.2 4 90.0 9	 49.4	: : : :	
A434	4130, 4140, 4145, 4320, 4340	Ē	ВС	i	ŧ	Bar $1_{1}^{4}$ z in. and under	(1), (31)	110	130	$S_m$ $S_y$ $S_u$ $S_u$	 110.0 1 130.0 1	 102.9 130.0	 99.4 130.0	 96.6 9 130.0 13	 93.6 8' 130.0 13	 89.5 8 130.0 13	 86.8 8 130.0 13	 83.6 130.0	1 1 1 1	1 1 1 1
A434	4130, 4140, 4145, 4320, 4340	÷	ВС	÷	i	Bar over $1/2$ in. to $2^1/2$ in incl.	(1), (31)	105	125	$S_m$ $S_y$ 1	 105.0 125.0	 98.1 125.0	 94.8 125.0	 92.2 8 125.0 12	 89.4 8 125.0 12	 85.5 8 125.0 12	 82.9 7 125.0 12	 79.8 125.0	1 1 1 1	1 1 1 1
A434	4130, 4140, 4145, 4320, 4340	÷	ВС	÷	i	Bar over $2\frac{1}{2}$ in. to 4 in. incl.	(1), (31)	95	115	S. 2 2. 2.	 95.0 115.0 1	 88.5 1115.0	 85.4 8 115.0 11	 83.0 8 115.0 11	 80.6 7 115.0 11.	 77.0 7. 115.0 11.	 74.6 7 115.0 111	 72.0	1 1 1 1	
A434	4130, 4140, 4145, 4320, 4340	÷	ВС	÷	i	Bar over 4 in. to 7 in. incl.	(1), (31)	82	110	S. S. 1	85.0 110.0 1	10.05	 76.8	 74.6 7 110.0 11	 72.4 6' 1110.0 11	 69.1 6 110.0 11	 67.0 6 110.0 11	64.6	1 1 1 1	1 1 1 1
A434	4130, 4140, 4145, 4320, 4340	÷	ВС	÷	i	Bar over 7 in. to $9\frac{1}{2}$ in. incl.	(1), (31)	80	105	S <sub>m</sub> S <sub>y</sub>	 80.0 105.0 1	 74.8 105.0	72.3	 70.3 6 105.0 10	 68.1 6. 105.0 10.	 65.1 6 105.0 10	 63.1 6 105.0 10	 60.9	1 1 1 1	1 1 1 1
A434	4130, 4140, 4145, 4320, 4340	i.	BD	i	÷	Bar $1^{1}/_{2}$ in. and under	(1), (31)	130	155	S <sub>m</sub> S <sub>y</sub> 1	 130.0 1 155.0 1	 121.5 1 155.0 1	 117.2 11 155.0 18	 114.1 11 155.0 15	 110.7 195.7 102.5 155.0 155.0 155.0	 5.7 10 5.0 15	<u></u>	 98.8 155.0	: : : :	: : : :

	Table NF-I-2100-1 Design Stress intensity ( $S_m$ ), Maximum Allowable Stress ( $S_p$ ), $S_p$ ), and Tensile Strength ( $S_n$ ) Values (Cont'd)	S (ntensi	ty (S <sub>m</sub>	), Maxim	Ium A	Ilowable St	Table NF-I-2100-1 :ress (S), Yield Str	-l-2100-1 Yield Stre	ingth (S <sub>y</sub> ),	, and	Ten	ile S	treng	) jth (S	N ("	Ilues	(Cor	ıt'd)		
	Material	Condition	Ċ		Ę			Min. Yield Strength	Min. Tensile Strength			for M	Stre Multipl etal Ter	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	Strengi 300 to ( ures, °F	th, ksi, Obtain ; , Not to	psi)			
Spec. No.	Grade or AISI	Type	class	P/S No.	- 1	Product Form	Notes	ksi ksi	ksi		100	200 3	300 4	400 50	200 60	600 650	20 200		750 8	800
Low Alloy S A434	Low Alloy Steels (Cont'd) A434 4130, 4140, 4145, 4320, 4340	i	BD	to jien		Bar over $1\frac{1}{2}$ in. to $2\frac{1}{2}$ in. incl.	(1), (31)	120	150	$S_m$ $S_y$ $S_u$ $S_u$	 120.0 1 150.0 1	 112.1 10 150.0 15	 108.4 10 150.0 15	 105.2 10 150.0 15	 102.1 97 150.0 150	 97.6 94 150.0 150	 94.6 91.1 150.0 150.0	 91.1 50.0		
A434	4130, 4140, 4145, 4320, 4340	į	BD	ŧ	Bar	Bar Over 2½ in. 10 4 in. incl.	(1), (31)	110	140	$S_m$ $S_y$ $S_u$ $S_u$	 110.0 1 140.0 1	 102.9 140.0	 99.4 9 140.0 14	 96.6 9 140.0 14	 93.6 89 140.0 146	 89.5 86 140.0 140	 86.8 83.6 140.0 140.0	 83.6		
A434	4130, 4140, 4145, 4320, 4340	÷	BD	i .	:	Bar over 4 in. to 7 in. incl.	(1), (31)	105	135	$S_m$ $S$ $S_y$ $S_u$ $S_u$	 105.0 135.0	 98.1 135.0	 94.8 9 135.0 13	 92.2 8 135.0 13	 89.4 8E 135.0 13E	 85.5 82 135.0 135	 82.9 79.8 135.0 135.0	 79.8 35.0		
A434	4130, 4140, 4145, 4320, 4340	÷	BD	i	:	Bar over 7 in. to $9^{1}/_{2}$ in. incl.		100	130	$S_m$ $S_y$ $S_u$ $S_u$	 100.0 130.0	 93.5 130.0	  90.2 8 130.0 13	 87.8 8 130.0 13	 85.1 87 130.0 130	 81.4 78	 78.9 76.0 130.0 130.0	 76.0 30.0		
A471	3Ni-Cr-Mo-V	÷	7	i .	:	Forging	(1), (10)	Se C.	105	$S_m$ $S_y$ $S_u$ $S_u$	 90.0 105.0	 84.8 105.0	 82.1 8 105.0 10	 80.3 7 104.0 10	 79.0 77 103.3 101	 77.6 76.4 101.9 100.4		 75.4 98.8		
A471	3Ni-Cr-Mo-V	÷	м	i	:	Forging	(1), (10)	100	1. 7h	S S 1	 100.0 110.0	 94.2 110.0	 91.2 8 110.0 10	 89.2 8 108.9 10	 87.8 86.2 108.1 106.7	_		 83.8 03.5		
A471	3Ni-Cr-Mo-V	i	4	ŧ	:	Forging	(1), (10)	110	120	\$ 5 S 1	 110:0 103.7 120:0 120:0		 100.4 9 120.0 11	 98.2 9 118.8 11	 96.5 92 118.0 116	 94.9 93.5 116.4 114.7	7			
A471	3Ni-Cr-Mo-V	i	rv	÷	:	Forging	(1), (10)	120	130	$S_m$ $S_y$ $S_u$ $S_u$	 120.0 1 130.0 1	 113.1 130.0	 109.5 10 130.0 43	 107.1 10	 105.3 103.5 127.8 126.1	10	10			
A471	3Ni-Cr-Mo-V	÷	9	i	:	Forging	(1), (10)	130	140	$S_m$ $S$ $S_y$ $S_u$ $S_u$	 130.0 122.5 118.6 140.0 140.0 140.0	 122.5 140.0	 118.6 11 140.0 13			 112.1 110.5 135.8 133.8	13	 19.0		: : : :

\_\_\_\_\_120

	Design Stress Intensity ( $S_m$ ), Maximul	Constant of the state of the st	ty (5,	"), Maxim	m m	Table NF-I-2100-1 m Allowable Stress (S), Yield Strength ( $S_{ u}$ ), and Tensile Strength ( $S_{u}$ ) Values (Cont'd)	Table NF tress (S),	Table NF-I-2100-1 ress (S), Yield Stre	ngth (S <sub>y</sub> )	, anc	1 Ten	sile	Stren	gth (	S <sub>u</sub> ) v	/alue	S (Co	ont'd)		
	Material	Condition	Hick	NL.	Grb.			Min. Yield Strength,	Min. Tensile Strength,			for !	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ess and ly by 1 empera	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) fetal Temperatures, °F, Not to Ex	gth, ksi Obtair F, Not	i, n psi) to Exce	pee		
Spec. No.	Steel	Type	Class	P/S No.	No.	Product Form	Notes	ksi	ksi		100	200	300	400	200	009	029	200	750	800
A471	3Ni-Cr-Mo-V	į	۲	ien	the	Forging	(1), (10)	135	150	$S_m$	 135.0 150.0	 127.1 150.0	  123.0 1 150.0 1	  120.3 1 148.5 1	  118.3 1 147.5 1	  116.2 1 145.5 1	  114.8 1 143.4 1	  113.1 141.1	: : : :	::::
A471	3Ni-Cr-Mo-V	:	ω	:	i	Forging	(1), (10)	145	160	$S_m$	 145.0 160.0	 136.6 160.0	 132.2 1 160.0 1	 129.3 1 158.4 1	 127.1 1 157.3 1	 124.9 1 155.2 1	 123.2 1 152.9 1	 121.8 150.5	1 1 1 1	: : : :
A471	3Ni-Cr-Mo-V	ŧ	6	:	i	Forging	(a), (10)	155	170	$S_{m}$	 155.0 170.0	 146.0 170.0	 141.2 1 170.0 1	  138.1 1 168.3 1	  136.0 1 167.2 1	  133.5 1 164.9 1	  131.8 1 162.5 1	  130.0 159.9	1111	: : : :
A490	4137, 4140, 4142, 4145, 4340	11	:	:	i	Bolt	(1)	130	150	$S_m$	 130.0 150.0	 121.5 150.0	 117.2 1 150.0 1	  114.1 1 150.0 1	 110.7 1 150.0 1	  105.7 1 150.0 1	 102.5 150.0	 98.8 150.0	1111	::::
A514	٧	:	:	P-No. 11B	17	Plate $1^{1}/_{4}$ in. and less	÷	001	110	$S_{u}$	36.7 31.4 100.0 1110.0	36.7 31.4 95.5 110.0	36.7 31.4 92.5 110.0	36.7 31.4 89.8 1110.0	36.7 31.4 87.6 110.0 1	36.7 31.4 85.5 110.0 1	36.5 31.4 84.3 110.0 1	35.9 30.8 83.0 107.7	1111	: : : :
A514	Ω	ŧ	:	P-No. 11B	4	Plate 1 <sup>1</sup> / <sub>4</sub> in. and less	÷	100	110	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	36.7 31.4 100.0	36.7 31.4 95.5 110.0	36.7 31.4 92.5 110.0	36.7 31.4 89.8 1110.0	36.7 31.4 87.6 1110.0	36.7 31.4 85.5 110.0 1	36.5 31.4 84.3 1110.0	35.9 30.8 83.0 107.7	1111	::::
A514	ш	:	÷	P-No. 11B	7	Plate $2\frac{1}{2}$ in. and less	÷	100	110	ς, ς, ς, α,	36.7 31.4 31.4 110.0	36.7 31.4 95.5 110.0	36.7 31.4 92.5 110.0	36.7 31.4 89.8 1110.0	36.7 31.4 87.6 1110.0	36.7 31.4 85.5 110.0 1	36.5 31.4 84.3 110.0 1	35.9 30.8 83.0 107.7	1111	::::
A514	Ľ.	ŧ	÷	P-No. 11B	ю	Plate 2½ in. and less	į	100	110	S S S	36.7 31.4 100.0 1110.0	36.7 31.4 95.5 110.0	36.7 31.4 92.5 110.0	36.7 31.4 89.8 110.0	36.7 31.4 87.6 1110.0	36.7 31.4 85.5 110.0 1	36.5 31.4 84.3 110.0 1	35.9 30.8 83.0 107.7	1111	1111
A514	C.	÷	i	P-No. 11B	œ	Plate $2\frac{1}{2}$ in. and less	i	100	110	s s s s	36.7 31.4 100.0 1110.0	36.7 31.4 95.5 110.0	36.7 31.4 92.5 110.0	36.7 31.4 89.8 110.0 1	36.7 36.7 31.4 31.4 87.6 85.5 110.0 110.0		36.5 31.4 84.3 110.0 1	35.9 30.8 83.0 107.7	: : : :	1 1 1 1

	Design Stress (ntensity $(S_m)$ , Maximu	of Organity	(S <sub>m</sub> )	, Maximu	mr A	Table NF-I-2100-1 m Allowable Stress (S), Yield Strength ( $S_{ u}$ ), and Tensile Strength ( $S_{u}$ ) Values (Cont'd)	Table NF-I-2100-1 ress (S), Yield Str	-1-2100-1 Yield Stre	ngth (S <sub>y</sub> ),	and	Ten	ile S	tren	gth (c	\ ("S	/alue	CC	ont'd,		
	Material	Condition	نہ		E.			Min. Yield Strength	Min. Tensile Strength			for M	Str Multip etal Te	ess and ly by 1	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	gth, ksi Obtail F, Not	i, n psi) to Exce	pea		
Spec. No.	Grade or AISI	,	class P/S No.		No.	Product Form	Notes	ksi ksi	ksi		100	200	300 4	400	200	009	650	1 1	750	800
Low Alloy A514	Low Alloy Steels (Cont'd) A514 Q	ŧ		S-No. 11B	6	Plate 2½ in. and less	÷	100	110	$S_m$ $S$ $S_y$ $S_u$ $S_u$	36.7 31.4 100.0 1110.0	36.7 31.4 95.5 110.0	36.7 31.4 92.5 1110.0	36.7 31.4 89.8 1110.0	36.7 31.4 87.6 1110.0	36.7 31.4 85.5 110.0	36.5 31.4 84.3 1110.0	35.9 30.8 83.0 107.7	: : : :	: : : :
A514	ы	ŧ	:	P-No. 11B	Ca .	Plate $5 \times 12 = 10$ in. $5 \times 10 = 10$	÷	06	100	$S_m$ $S_y$ $S_u$ $S_u$	33.3 28.6 90.0 100.0	33.3 28.6 86.0 100.0 1	33.3 28.6 83.3 100.0 1	33.3 28.6 80.8 100.0 1	33.3 28.6 78.8 100.0	33.3 28.6 77.0 100.0 1	33.2 28.6 75.9 100.0	32.6 28.0 74.7 97.9	1 1 1 1	: : : :
A514	<u>a.</u>	ŧ	:	P-No. 11B	ω	Plate $2^{1/2}_{1/2}$ in. to 6 in. incl.	: 51	06	100	$S_m$ $S_y$ $S_u$ $S_u$	33.3 28.6 90.0 100.0	33.3 28.6 86.0 100.0 1	33.3 28.6 83.3 100.0 1	33.3 28.6 80.8 100.0 1	33.3 28.6 78.8 100.0	33.3 28.6 77.0 100.0 1	33.2 28.6 75.9 100.0	32.6 28.0 74.7 97.9	1 1 1 1	: : : :
A514	O	ŧ	:	P-No. 11B	6	Plate over $2\frac{1}{2}$ in. to 6 in. incl.	SME	90 100 AME	100	$S_m$ $S_y$ $S_u$ $S_u$	33.3 28.6 90.0 100.0	33.3 28.6 86.0 100.0 1	33.3 28.6 83.3 100.0 1	33.3 28.6 80.8 100.0	33.3 28.6 78.8 100.0	33.3 28.6 77.0 100.0 1	33.2 28.6 75.9 100.0	32.6 28.0 74.7 97.9	1 1 1 1	: : : :
A519	4140	4140SR	i	ŧ	i	Tube	(1)		120	$S_m$ $S_y$ $S_y$ $S_u$ $S_u$	40.0 34.3 100.0 120.0	40.0 34.3 93.5 120.0	40.0 34.3 90.2 120.0	40.0 34.3 87.8 120.0	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1111	: : : :
A519	4142	4142SR	i	:	÷	Tube	(1)	100	750 AF	_	40.0 34.3 100.0 120.0	40.0 34.3 93.5 120.0 1	40.0 34.3 90.2 120.0 1	40.0 34.3 87.8 120.0	1111	1 1 1 1	1 1 1 1	1 1 1 1	1111	: : : :
A572	42	ŧ	ŧ	P-No. 1	Н	Plate and shapes	(14)	42	09	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	20.0 47.1 42.0 60.0	20.0 17.1 40.0 60.0	20.0 17.1 38.3 60.0	20.0 17.1 36.8 60.0	20.0 17.1 35.2 60.0	20.0 17.1 33.5 60.0	20.0 17.1 32.7 60.0	20.0 17.1 31.8 60.0	1111	1 1 1 1
A572	20	ŧ	i	P-No. 1	₩	Plate and shapes	(14)	50	65	S S S S S S S S S S S S S S S S S S S	21.7 18.6 50.0 65.0	21.3 118.6 47.5 65.0	21.7 18.6 45.6 65.0	21.7 18.6 43.8 65.0	21.7 18.6 41.8 65.0	21.7 18.6 39.9 65.0	21.7 18.6 38.9 65.0	21.7 18.6 37.9 65.0	1111	: : : :
A572	വ	i	i	P-No. 1	7	Plate and shapes	i	55	70	S <sub>m</sub> S <sub>y</sub>	23.3 20.0 55.0 70.0	23.3 20.0 52.3 70.0	23.3 20.0 50.2 70.0	23.3 20.0 48.2 70.0	23.3	23.3  43.9 70.0	23.3  42.8 70.0	23.3  41.7 70.0	1 1 1 1	: : : :

	Design Stres	Contraction of the second of t	ty (S <sub>m</sub>	,), Maxim	/ Wni	Table NF-I-2100-1 Terisity ( $S_m$ ), Maximum Allowable Stress ( $S_p$ ), Yield Strength ( $S_p$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	rable NF	Table NF-I-2100-1 :ress (S), Yield Stre	ngth (S <sub>y</sub> .),	and	Tens	sile S	trenç	Jth (5	\ (n)	alue	) co	nt'd)		
	Material	Condition	Jick	1	i.			Min. Yield Strength	Min. Tensile Strength			for M	Stre Multipl etal Te	ess and y by 1, mperat	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	th, ksi, Obtain F, Not t	psi)	eq		
Spec. No.	Grade or AISI	Type	Class	A/S No.	No.	Product Form	Notes	su engui, ksi	su engun, ksi		100 2	200	300 4	400 5	200	9 009	650 7	H	750 8	800
Low Alloy 9 A579	Low Alloy Steels (Cont'd) A579 5Ni-Cr-Mo-V	12a	ŧ	lien	The	Forging	$\Xi$	140	150	$S_m$ $S$ $S_y$ $S_u$ $S_u$	50.0 42.9 140.0 1 150.0 1	49.5 42.4 138.6 1 148.5 1	48.0 41.1 134.0 12	47.0 40.3 41.29.5 12141.0 14	47.0 4 40.3 4 127.7 12 141.0 14	47.0 4 40.3 3 126.3 12	46.0 439.4 3123.5 1138.0 13	44.0 .37.7 .117.6 .132.0	1 1 1 1	1111
A588	А, В	ŧ	ŧ	P-No. 3	Н	Pate and bar 4 m. and less, all shapes	(7), (15)	20	70	$S_m$ $S_y$	23.3 20.0 50.0 70.0	23.3 20.0 47.5 70.0	23.3 20.0 45.6 70.0	23.3 20.0 43.0 70.0	23.3 2 20.0 2 41.8 3	23.3 2 20.0 2 39.9 3	23.3 20.0 38.9 70.0	23.3 20.0 37.9 70.0		1 1 1 1
A588	A, B	ŧ	i	P-No. 3	П	Plate and bar over 4 in. to 5 in. incl.	(14) (24)	46	29	S S S 2	22.3 19.1 46.0 67.0	22.3 19.1 43.8 67.0	22.3 19.1 41.9 67.0	22.3 2 19.1 1 1 40.3 3 67.0 6	22.3 2 19.1 1 38.6 3 67.0 6	22.3 2 19.1 1 36.7 3 67.0 6	22.3 2 19.1 1 35.8 3 67.0 6	22.3 19.1 34.8 67.0	: : : :	
A588	А, В	i.	i	P-No. 3	1	Plate and bar over 5 in. to 8 in. incl.	(14)	457 BR	63	S S S	21.0 18.0 42.0 63.0	21.0 18.0 40.0 63.0	21.0 18.0 38.3 63.0	21.0 21.0 21.0 21.0 21.0 21.0 21.0 21.0	21.0 2 18.0 1 35.2 3 63.0 6	21.0 2 18.0 1 33.5 3 63.0 6	21.0 2 18.0 1 32.7 3 63.0 6	21.0	: : : :	
A618	Mn-Cu-V	ŧ	Ξ	P-No. 1	7	Tube $\frac{3}{4}$ in. and less	ŧ	05.	70	S S S 3	 50.0 70.0	 45.4 70.0	 41.7 70.0	 38.0	 34.6 3 70.0 7	 33.9 3 70.0 7	 33.6	 33.1 69.3	: : : :	
A618	Mn-Cu-V	ŧ	Ħ	P-No. 1	7	Tube over $\frac{3}{4}$ in. to $1\frac{1}{2}$ in. incl.	ŧ	46	12.5	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	46.0	 41.7 67.0	 38.4 67.0	 35.0	  31.8 3 67.0 6	 31.2 3 67.0 6	 30.9	30.5 66.3	1 1 1 1	
A618	Mn-V	ŧ	≣	P-No. 1	$\leftarrow$	Tube	ŧ	50	65	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	50.0	 45.4 65.0	41.7	 38.0	  34.6 3 65.0 6	 33.9 3 65.0 6	 33.6 65.0	33.1 64.4	: : : :	
A633	<b>V</b>	:	i	P-No. 1	-	Plate and shapes 4 in. and less	ŧ	42	63	S S S	21.0 18.0 42.0 63.0	21.0 18.0 38.2 63.0	21.0 18.0 35.1 63.0	21.0 18.0 131.9 63.0	21.0 2 18.0 1 29.1 2 63.0 6	21.0 2 18.0 1 28.5 2 63.0 6	21.0 21.0 21.0 21.0 21.0 21.0 21.0 21.0	20.8 17.8 27.8 62.4	: : : :	
A633	C, D	ŧ	:	P-No. 1	7	Plate and shapes $2\frac{1}{2}$ in. and less	ŧ	50	70	S S S S S S S S S S S S S S S S S S S	23.3 20.0 50.0 70.0	23.3 20.0 45.4 70.0	23.3 20.0 41.7 70.0	23.3 20.0 38.0 70.0	23.3 2 20.0 2 34.6 3	23.3 2 20.0 2 33.9 3	23.3 2 20.0 3 33.6 3 70.0 6	23.1 19.8 33.1 69.3	: : : :	: : : :

	Table NF-I-2100-1 Design Stress ( $S_p$ ), and Tensile Strength ( $S_p$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	S (ntensity	ty (S <sub>m</sub>	), Maxim	/ mni	Allowable Si	Table NF-I-2100-1 tress (S), Yield Str		ingth (S <sub>y</sub> ),	and	Ten	sile !	Stren	gth (	(S <sub>u</sub> )	Valu	) sa	ont'o	<b>1</b>	
	Material	Condition	Ċ		Grb.			Min. Yield Strength.	Min. Tensile Strength.			for A	Stı (Multiț fetal Tu	ress an oly by empera	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ngth, k o Obta °F, No	si, in psi) t to Ex	pəəɔ		
Spec. No. Low Alloy S	Steel	Type	Class	P/S No.	No.	Product Form	Notes	ksi	ksi		100	200	300	400	200	009	650	700	750	800
A633	С, D	Ē	i	P-No. 1	- 1	Plate, bar, and shapes over $2^{1}/_{2}$ in. to 4 in. incl.	ŧ	46	65	$S_m$ $S_y$	21.7 18.6 46.0 65.0	21.7 18.6 41.8 65.0	21.7 18.6 38.4 65.0	21.7 18.6 35.0 65.0	21.7 18.6 31.8 65.0	21.7 18.6 31.2 65.0	21.7 18.6 30.9 65.0	21.5 18.4 30.4 64.4	1 1 1 1	1 1 1 1
A633	п	i	i	P-No. 1	C.	Plate 4 in. and less, all-shapes	(3)	09	80	$S_m$ $S_y$ $S_u$	26.7 22.9 60.0 80.0	26.7 22.9 54.5 80.0	26.7 22.9 50.1 80.0	26.7 22.9 45.6 80.0	26.7 22.9 41.4 80.0	26.7 22.9 40.7 80.0	26.7 22.9 40.3 80.0	26.4 22.3 39.7 78.2	: : : :	1 1 1 1
A668	4130, 4320, 4330, 8620, 8630	i	×	S-No. 4	ю	Forging 7 in. and less	(7), (8), (31)	80	105	$S_m$ $S_y$ $S_u$ $S_u$	35.0 30.0 80.0 105.0	35.0 30.0 74.8 105.0	35.0 30.0 72.3 105.0	35.0 30.0 70.3 105.0	35.0 30.0 68.1 105.0	35.0 30.0 65.4 105.0	35.0 30.0 63.1 105.0	35.0  60.9 105.0	: : : :	1 1 1 1
A668	4130, 4320, 4330, 8620, 8630	ŧ	×	S-No. 4	ю	Forging over 7 in. to 10 in. incl.	(E) NF	75	100	$S_m$ $S_y$ $S_u$ $S_u$	33.3 28.6 75.0 100.0	33.3 28.6 70.1 100.0	33.3 28.6 67.7 100.0	33.3 28.6 65.8 100.0	33.3 28.6 63.8 100.0	33.3 28.6 61.0 100.0	33.3 28.6 59.0 100.0	33.3 28.6 57.0 100.0	: : : :	1 1 1 1
A668	4140, 4340	÷	×	i .	ŧ	Forging 7 in. and less	(1), (7), (8), (9), (16), (31)		105	$S_m$ $S_y$ $S_u$ $S_u$	35.0 30.0 80.0 105.0	35.0 30.0 74.8 105.0	35.0 30.0 72.3 105.0	35.0 30.0 70.3 105.0	35.0 30.0 68.1 105.0	35.0 30.0 65.1 105.0	35.0 30.0 63.1 105.0	35.0 30.0 60.9 105.0	: : : :	1 1 1 1
A668	4140, 4340	:	×	i .	ŧ	Forging over 7 in. to 10 in. incl.	(1), (8), (9), (16), (31)	75	1. Mr.	S S 2 1	33.3 28.6 75.0 100.0	33.3 28.6 70.1 100.0	33.3 28.6 67.7 100.0	33.3 28.6 65.8 100.0	33.3 28.6 63.8 100.0	33.3 28.6 61.0 100.0	33.3 28.6 59.0 100.0	33.3 28.6 57.0 100.0	: : : :	1 1 1 1
A668	4130, 4320, 4330, 8620, 8630	:	J	S-No. 4	ю	Forging 4 in. and less	(7), (8), (11), (31)	105	125	S, S, 1	41.7 35.7 105.0 125.0	41.7 35.7 98.1 125.0	41.7 35.7 94.8 125.0	41.7 35.7 92.2 125.0	41.7 35.7 89.4 125.0	41.7 35.7 85.5 125.0	 82.9 	79.8	: : : :	1 1 1 1
A668	4130, 4320, 4330, 8620, 8630	:	J	S-No. 4	т	Forging over 4 in. to 7 in. incl.	(7), (8), (11), (31)	95	115	$S_m$ $S_y$ $S_u$ $S_u$	38.3 32.9 95.0 115.0	38.3 32.9 88.5 1115.0	38.3 32.9 85.4 115:0	38.3 32.9 83.0	38.3 32.9 80.0 1115.0	38.3 32.9 77.0 1115.0	 32.9 74.6 115.0	32.9 72.0 115.0	: : : :	1 1 1 1
A668	4130, 4320, 4330, 8620, 8630	:	J	S-No. 4	8	Forging over 7 in. to 10 in. incl.	(8), (11), (31)	82	110	$S_m$ $S_y$ $S_u$ $S_u$	36.7 31.4 85.0 110.0 1	36.7 31.4 79.5 110.0	36.7 31.4 76.8 110.0 1	38.7 31.4 74.6 110.0	36.7 31.4 110.0	36.7 31.4 69.1 110.0	36.7 31.4 67.0 110.0	36.7 31.4 64.6 110.0	: : : :	: : : :

	J. J	C																		
	Table NF-I-2100-1 Design Stress Intensity ( $S_m$ ), Maximum Allowable Stress ( $S$ ), Yield Strength ( $S_{ u}$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	s Intensi	ty (5	), Maxin	, mbr	Allowable Si	Table NF-I-2100-1 tress (S), Yield Str	-I-2100-1 Yield Stre	ngth (S <sub>y</sub> )	, and	Ten:	sile S	trenç	jth (S	) ("s	alues	s (Co	nt'd)		
	Motoriol	Condition	Jick		į.			Min. Yield	Min. Tensile			for M	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	Stress and Strength, ksi, tiply by 1,000 to Obtain Temperatures, °F, Not to	Streng 000 to ures, °l	th, ksi, Obtain F, Not t	, ı psi) to Exce	eq		
Spec. No.	Ę.	Type	Class	P/S No.	No.	Product Form	Notes	surengun, ksi	surengui, ksi		100	200	300 4	400 5	500 6	9 009	650 7	_	750	800
Low Alloy	Low Alloy Steels (Cont'd)			i																
A668	4135	ŧ	J	Sh	THE	Forging 4 in. and less	(1), (7), (8), (9), (31)	105	125	$S_m$ $S_y$ $S_u$ $S_u$	 105.0 125.0	 98.1 125.0	 94.8 925.0	 92.2 8 125.0 12	 89.4 8 125.0 12	 85.5 8 125.0 12	 82.9 7 125.0 12	 79.8 125.0		: : : :
A668	4135	÷	J	i	i	Forging over 4 in. to 7 in. incl.	(1), (7), (8), (9), (31)	95	115	$S_m$ $S_y$ $S_u$ $S_u$	 95.0 115.0	 88.5 115.0	 85.4 8 115.0 11	 83.0 8 115.0 11	 80.6 7 115.0 11	 77.0 7 115.0 11	 74.6 7 115.0 11	 72.0 115.0	1 1 1 1	: : : :
A668	4135	ŧ	J	i	ŧ	Forging over 7 in. to 10 in. incl.	(31)	85	110	$S_m$ $S_y$ $S_u$ $S_u$	 85.0 110.0	 79.5 110.0	 76.8 7 110.0 11	 74.6 7 110.0 111	 72.4 6 110.0 11	 69.1 6 110.0 11	 67.0 6 1110.0 11	 64.6 110.0	1111	: : : :
A668	4140, 4340	÷	J	i .	:	Forging 4 in. and less	(1), (7), (8) (9), (16), (31)	105	125	$S_m$ $S_y$ $S_u$ $S_u$	41.7 35.7 105.0 125.0	41.7 35.7 98.1 125.0	41.7 4 35.7 3 94.8 9	41.7 4 35.7 3 92.2 8 125.0 12	41.7 4 35.7 3 89.4 8 125.0 12	41.7 4 35.7 3 85.5 8 125.0 12	41.7 4 35.7 3 82.9 7 125.0 12	41.7 35.7 79.8 125.0	: : : :	: : : :
A668	4140, 4340	÷	J	i .	:	Forging over 4 in. to 7 in. incl.	(1), (7), (8), (9), (16), (31)	0.56	115	$S_m$ $S_y$ $S_u$ 1	38.3 32.9 95.0 115.0 1	38.3 32.9 88.5 1115.0	38.3 32.9 85.4 8115.0	38.3 3 32.9 3 83.0 8 115.0 11	38.3 3 32.9 3 80.6 7 1115.0 111	38.3 3 32.9 3 77.0 7 115.0 11	38.3 32.9 74.6 7115.0	38.3 32.9 72.0 115.0	: : : :	: : : :
A668	4140, 4340	ŧ	J	i:	÷	Forging over 7 in. to 10 in. incl.	(1), (8), (9), (16), (31)	82	110	S. S. S. S. S. S. C. J. S. C.	36.7 31.4 85.0	36.7 31.4 79.5 1110.0	36.7 3 31.4 3 76.8 7 110.0 11	36.7 3 31.4 3 74.6 7 110.0 11	36.7 3 31.4 3 72.4 6 1110.0 11	36.7 3 31.4 3 69.1 6	36.7 3 31.4 3 67.0 6	36.7 31.4 64.6 1110.0	: : : :	: : : :
A668	4135, 4330	Ē	Σ	i	÷	Forging 4 in. and less	(1), (7), (8), (9), (12), (31)	120	145	S <sub>m</sub> S <sub>y</sub> 1	120.0		 108.4 10 145.0 14	 105.2 10 145.0 14	 102.1 9 145.0 14	 97.6 9 145.0 14	 91.6 945.0	 91.1 145.0	1 1 1 1	: : : :
A668	4135, 4330	÷	Σ	i	÷	Forging over 4 in. to 7 in. incl.	(1), (7), (8), (9), (12), (31)	115	140	$S_m$ $S_y$ $S_u$ $S_u$	 115.0 1 140.0 1	 107.5 1 140.0 1	 103.8 140.0	 101.0 9 1400 14	  98.0 9 140.0 14	 93.6 9 140.0 14	 90.7 8 140.0 14	 87.5 140.0	1 1 1 1	: : : :
A668	4135, 4330	;	Σ	:	:	Forging over 7 in. to 10 in. incl.	(1), (8), (9), (12), (31)	110	135	$S_m$ $S_y$ $S_y$ $S_u$ $S_u$	 110.0 1 135.0 1	 102.9 135.0	 99.4 135.0	 96.6 93.6 89.5 86.8 135.0 135.0 135.0 135.0	93.6	 89.5 8 135.0 13		 83.6 135.0	: : : :	: : : :

	Table NF-I-2100-1 Design Stress intensity ( $S_m$ ), Maximum Allowable Stress ( $S_p$ ), $S_p$ ), and Tensile Strength ( $S_p$ ) Values (Cont'd)	S (ntensi	ty (Sm	), Maxim	/ wn	Allowable Si	Table NF-I-2100-1 tress (S), Yield Str	-1-2100-1 Yield Stre	ingth (S <sub>y</sub> ),	and	Ten	sile 5	tren	gth (	\\	/alue	es (C	ont'd		
	Material	Condition	Ċ		Grb.			Min. Yield Strength.	Min. Tensile Strength.			for M	Str (Multip Tetal Te	ess and like the second	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ngth, ks o Obtai °F, Not	si, in psi) t to Exc	eed		
Spec. No.	Spec. No. Grade or AISI	Type	Class	class P/S No.	No.	Product Form	Notes	ksi	ksi		100	200	300	400	200	009	650	200	750	800
A668	4140, 4340	ŧ	Σ	rojien	: X	Forging 4 in. and less	(1), (7), (8), (9), (16), (31)	120	145	$S_m$ $S$ $S_y$ $S_u$ $S_u$	48.3 41.4 120.0 1 145.0 1	48.3 41.4 112.1 145.0	48.3 41.4 108.4 145.0	48.3 41.4 105.2 1 145.0 1	48.3 41.4 102.1 145.0	48.3 41.4 97.6 145.0 1	48.3 41.4 91.6 145.0	48.3 41.4 91.1 145.0	1 1 1 1	1111
A668	4140, 4340	ŧ	Σ	ŧ	ue.	Forging over 4 in. to 7 in incl.	(1), (7), (8), (9), (16), (31)	115	140	$S_m$ $S$ $S_y$ $S_u$ $S_u$	46.7 40.0 115.0 1	46.7 40.0 107.5 1 140.0 1	46.7 40.0 103.8 1 140.0 1	46.7 40.0 101.0 140.0	46.7 40.0 98.0 140.0 1	46.7 40.0 93.6 140.0 1	46.7 40.0 90.7 140.0	46.7 40.0 87.5 140.0	1 1 1 1	1 1 1 1
A668	4140, 4340	:	Σ	ŧ	i	Forging over 7 in to 10 in. incl.	(1), (8), (9), (16), (31)	110	135	$S_m$ $S_y$ $S_u$ $S_u$	45.0 38.6 110.0 135.0	45.0 38.6 102.9 135.0	45.0 38.6 99.4 135.0	45.0 38.6 96.6 135.0 1	45.0 38.6 93.6 135.0	45.0 38.6 89.5 135.0	45.0 38.6 86.8 135.0	45.0 38.6 83.6 135.0	1 1 1 1	1 1 1 1
A668	4340	:	Z	ŧ	ŧ	Forging 4 in. and less	(1), (7), (8), (9), (19), (31)	140	170	$S_m$ $S_y$ $S_u$ $S_u$	56.7 48.6 140.0 170.0	56.7 48.6 131.0 1	56.7 48.6 126.3 1 170.0 1	56.7 48.6 123.0 1 170.0 1	56.7 48.6 119.1 170.0	56.7 48.6 114.0 1 170.0 1	56.7 48.6 110.4 170.0	56.7 48.6 106.3 170.0	1 1 1 1	1 1 1 1
A668	4340	:	Z	ŧ	E	Forging over 4 in. to 7 in. incl.	(1), (7), (8), (9), (16), (31)	\$ 10.°	165	$S_m$ $S$ $S_y$ $S_u$ $S_u$	55.0 47.1 135.0 165.0	55.0 47.1 126.1 165.0	55.0 47.1 121.9 1 165.0 1	55.0 47.1 118.5 1 165.0 1	55.0 47.1 115.0 1 165.0 1	55.0 47.1 109.9 1 165.0 1	55.0 47.1 106.5 165.0	55.0 47.1 102.7 165.0	1 1 1 1	1 1 1 1
A668	4340	:	Z	ŧ	i	Forging over 7 in. to 10 in. incl.	(1), (8), (9), (16), (31)	130	N. A.	S S 1	53.3 45.7 130.0 160.0	53.3 45.7 121.5 1 160.0 1	53.3 45.7 117.2 1 160.0 1	53.3 45.7 114.1 1 160.0 1	53.3 45.7 110.7 1	53.3 45.7 105.7 160.0	53.3 45.7 102.5 160.0	53.3 45.7 98.8 160.0	1 1 1 1	1 1 1 1
A992	i	i	Ē	P-No. 1	н	Shapes	(14)	20	92	2, 2, 2, 2, 2, 2, 2, 2, 2, 2, 2, 2, 2, 2	21.7 18.6 50.0 65.0	21.7 18.6 45.8 65.0	21.7 18.6 44.2 65.0	21.7 18.6 42.7 65.0	21.7 18.6 40.7 65.0	21.7 18.6 38.3 65.0	21.7 18.6 37.1 65.0	1 1 1 1	1 1 1 1	1 1 1 1
Age-Hardening Steels A710 A	ning Steels A	i	1	S-No. 12	17	Plate, bar, and shapes $\frac{7}{16}$ in. and less	ij	82	06	$S_m$ $S_y$ $S_u$	30.0 25.7 85.0 90.0	30.0 25.7 80.0 90.0	30.0 25.7 77.1 90:0	30.0 25.7 74.1	30.0 25.7 71.5 90.0	30.0 25.7 69.6 90.0	1 1 1 1	: : : :	1 1 1 1	1 1 1 1
A710	A	:	1	S-No. 12	1	Plate, bar, and shapes over $\frac{5}{1_{16}}$ in. to $\frac{3}{4}$ in.	i	80	06	$S_m$ $S_y$	30.0 25.7 80.0 90.0	30.0 25.7 75.3 90.0	30.0 25.7 72.6 90.0	30.0 25.7 69.4 90.0	30.0 25.7 67.6 90.0	30.0 25.7 65.5 90.0	1 1 1 1	: : : :	: : : :	: : : :

	Design Stress Intensity ( $S_m$ ), Maximul	Ss Intensit	(y (S <sub>m</sub> )	, Maxim	mm 4	Table NF-I-2100-1 m Allowable Stress (S), Yield Strength ( $S_{ u}$ ), and Tensile Strength ( $S_{u}$ ) Values (Cont'd)	Table NF-I-2100-1 ress (S), Yield Str	-1-2100-1 Yield Stre	ngth (S <sub>y</sub> )	and,	Ten:	sile S	treng	)th (S	, (n)	alue:	) s (Co	nt'd)		
	Material	Condition	Hick	•	Grp.			Min. Yield Strength.	Min. Tensile Strenøth.			for M	Stress and Strength, ksi, (Multiply by 1,000 to Obtain psi) for Metal Temperatures, °F, Not to Exceed	ess and by 1, mperat	Stress and Strength, ksi, tiply by 1,000 to Obtain Temperatures, °F, Not t	th, ksi Obtain F, Not t	psi)	ed		
Spec. No.	ji Gr	Type	Class	P/S No.	No.	Product Form	Notes	ksi	ksi		100	200	300 4	400 5	200 60	9 009	650 7	700 7	750	800
A710	⋖	:	2	S-No. 12	- The	Plate, bar, and shapes 1 in. and less	÷	65	72	S, S	24.0 20.6 65.0 72.0	24.0 20.6 61.1 72.0	24.0 20.6 56.5 72.0	24.0 2 20.6 2 56.8 5 72.0 7	24.0 2 20.6 2 54.9 5 72.0 7	24.0 20.6 53.2 72.0	1111	1111	: : : :	1111
A710	A	i	7	S-No. 12	н	Plate, bar, and shapes over 1 in to 2 in.	i	09	72	$S_m$ $S_y$	24.0 20.6 60.0 72.0	24.0 20.6 56.5 72.0	24.0 20.6 54.1 72.0	24.0 2 20.6 2 52.4 5 72.0 7	24.0 2 20.6 2 50.7 4 72.0 7	24.0 20.6 49.1 72.0	1 1 1 1	: : : :	: : : :	1111
A710	A	i	7	S-No. 12	т	Plate, bar, and shapes over 2 in. to 4 in.	KAS	55	65	$S_m$ $S_y$	21.7 18.6 55.0 65.0	21.7 18.6 51.8 65.0	21.7 2 18.6 1 47.6 4	21.7 2 18.6 1 48.1 4 65.0 6	21.7 2 18.6 1 46.5 4 65.0 6	21.7 18.6 45.0 65.0	1 1 1 1	: : : :	1111	1111
A710	A	i	ĸ	S-No. 12	н	Plate, bar, and shapes 2 in. and less		75	85	$S_m$	28.3 24.3 75.0 85.0	28.3 24.3 70.6 85.0	28.3 24.3 268.0 685.0 8	28.3 2 24.3 2 66.0 6 85.0 8	28.3 2 24.3 2 65.4 6 85.0 8	28.3 24.3 61.4 85.0	1111	: : : :	1111	1111
A710	∢ .	i	ю	S-No. 12	т	Plate, bar, and shapes over 2 in. to 4 in.	į	65	75	$S_m$ $S_y$	25.0 21.4 65.0 75.0	25.0 21.4 61.2 75.0	25.0 21.4 57.0 75.0	25.0 2 21.4 2 56.6 5	25.0 2 21.4 2 54.7 5 75.0 7	25.0 21.4 53.2 75.0	1111	: : : :	: : : :	1111
Martensitic A276	Martensitic Stainless Steels A276	440C	÷	÷	Ē	Bar	(1), (20), (21), (24), (30)	210	275	S & S S S S S S S S S S S S S S S S S S	 210.0 2 275.0 2	 205.9 2 275.0 2	 205.2 20 275.0 27	 205.2 20 274.2 26	 204.4 20 269.2 26	 200.0 19 261.2 25	 195.8 255.9	: : : :		: : : :
A276	i	440C	i	i	i	Bar	(1), (20), (22), (24), (25), (30)	275	285	S S S 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	275.0	· P	1111	1111		1 1 1 1	1111	: : : :	1111	1111
A276	Ē	420	i	i	i	Bar	(1), (23)	65	95	$S_m$	 65.0 95.0	 60.0 95.0	57.5 95.0	 56.1 54.7	55.3	54.5	53.8	: : : :	1111	1111
A582	i	416, 416Se	i	i	i	Bar	(1), (26)	40	70	$S_m$ $S_y$ $S_u$	 40.0 70.0	38.1	36.9	35.7	(C)		1111	: : : :	: : : :	1 1 1 1

GENERAL NOTE: The allowable stress values (S) are based on the revised criterion of tensile strength divided by 3.5, where applicable. NOTES:

Welding is not permitted on these materials.
 These values are for material that is not weld
 These materials are limited for use only for si

These values are for material that is not welded.

These materials are limited for use only for standard supports.

# Design Stress Intensity ( $S_m$ ), Maximum Allowable Stress (S), Yield Strength ( $S_y$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd) Table NF-I-2100-1

NOTES (CONT'D):

Max. BHN 215. (4)

Max. BHN 225. (2)

This material may be used only in fully constrained applications, such as thread inserts, so that failure of the wire would not affect the function of the support. This material is exempt from the requirements of NF-I-2220. 9

By agreement between Purchaser and Material Manjacturer, these materials may be procured to the lower specified minimum tensile strength and minimum yield strength values given in this table. 9

For each forging 250 lb and less, the marking requirements of A668 shall be met by a suitable code or symbol identified by the manufacturer in his Certificate of Compliance or Certified Material Test Report. 8

The hardness test requirement may be performed only on the tensile specimen. 6

The minimum specified yield strength shall be taken at 0.2% offset. (10)

(11) There is no standard AISI composition 4330. By agreement with the manufacturer, the carbon content of AISI 4320 can be ordered as 0.27% to 0.33%.

(12) There is no standard AISI composition 4330. By agreement with the material manufacturer, the carbon content of AISI 4340 can be ordered as 0.27% to 0.33%, the molybdenum as 0.40% to 0.60%, the vanadium as 0.04% to 0.10%, and the silicon to 0.35% max

(13) Proper ventilation for the welder is required when welding on galvanized stee (Ref. ANSI Z49.1).

(14) Maximum tensile strength shall not exceed specified minimum tensile strength by more than 30 ksi, per SA-6, para. S18.

15) Maximum tensile strength shall not exceed specified minimum tensile strength by More than 25 ksi, per SA-6, para. S18.

16) E4340H may be used with the molybdenum range increased by agreement to 0.40% may. E4340H may be used for linear design only.

17) The elongation and reduction of area requirements for Grade 105-85 may be specified as 14% and 30%, respectively. The elongation and reduction of area requirements for Grade 115-95 may be specified as 42% and 25%, respectively.

19) The elongation and reduction of area requirements for this material may be specified as 17% and 35% minimum, respectively. (18)

20) This material may be used only in fully constrained applications, such as valve disk or flow restrictor halls or mechanical snubber parts such as gears, so that failure of these parts does not significantly affect the function of the item. Design stresses are limited to  $0.4S_{\nu}$ .

(21) Austenitize 1,850°F to 1,950°F, oil quench, temper at 700°F min.

(22) Austenitize 1,850°F to 1,950°F, oil quench, temper at 600°F min.

(23) Austenitize at 1,800°F minimum, rapid cool by forced air or oil, temper at 1,300°F minimum.

(24) For compressive applications, tensile testing is not required and each size and lot shall be accepted on the basis of hardness testing of three pieces. Each test specimen shall meet RC53

min. in the final heat-treated condition.

(25)

(26) For bar sizes under  $\frac{1}{2}$  in. nominal size, this material may be ordered to A581-95b. Hot finished, heat treated.

(27) Solution heat treated and hardened (1,325°F for 8 hr, furnace cooled to 1,150°F, held to a total of 18 hr, air cooled).

(28) Max. carbon 0.35% for welded connections.

(29) These values are for material that is welded. The Max. BNH must be met prior to welding. The stress and strength values at temperalure are based on the equivalent hot rolled material.

(30) These materials are exempt from the requirements of NF-I-2220.

These materials grades are AISI grades. See NF-I-2210

  -  -	jaci	ر. ن		, in	ي	Table NF-I-2100-1M  Table Strongth (C.) and Tongilo Strongth (C.) Values		Table N	Table NF-I-2100-1M	0-1N			1	<u> </u>	2			5	, t	ע אַ	-		ي			
	ביים ביים ביים ביים ביים ביים ביים ביים				2	), Maxillul	- Attowar	Min. Yield	Min. Tensile		7	בו בו	<b>;</b>	1/2	g   g	Stress and Strength, MPa, for Metal Tenneratures. °C. Not to Exceed	S and S	rength	, MPa,	5   S	<u>.</u>		<u>.</u>			1
Spec. No.	Material Grade or AISI	Condition Type	class	No.	Grp. No.	Product Form	Notes	Strength, MPa	Strength, MPa		38	20	75 10	100 125	5 150	0 175	200	225	250	275	8	325	350 3	375 4	400 4	425
Carbon A108	<b>Carbon Steels</b> A108 1015, 1018, 1020	CW	:	16.5 1.00 1.00 1.00 1.00 1.00 1.00 1.00 1.	7,40	Bar	(4), (29), (31)	276	414	S <sub>m</sub> S <sub>y</sub>	115 98.6 221 345	1115 98.6 217 345	1115 1 98.6 208 2 345 3	115 11 98.6 9 201 19 345 34	115 11 98.6 9 198 19 345 34	115 115 98.6 98. 195 192 345 345	115 115 98.6 98.6 192 189 345 345	9.	1 1 1 1	1111	::::	::::	::::	::::	1 1 1 1	1111
A108	1015, 1018, 1020	CW	i	ŧ	<b>*</b> !	jen!	(2), (4), (31)	276	414	S S S S S S S S S S S S S S S S S S S	138 118 276 414	138 118 271 414	138 1 118 1 260 2 414 4	138 13 118 11 251 24 414 41	138 138 118 118 248 244 414 414	138 138 118 118 244 240 414 414	8 138 8 118 0 237 4 414	1 1 1 1	1 1 1 1	1 1 1 1	: : : :	: : : :	: : : :	: : : :	: : : :	1111
A108	1045	CW	i	ŧ	:	Bar and bolt		689	827	s s s s s	276 236 689 827	276 2 236 2 677 6 827 8	276 2 236 2 650 6 827 8	276 23 236 23 627 61 827 82	276 27 236 23 618 61 827 82	276 276 236 236 610 601 827 827	6 276 6 236 1 592 7 827	1111	1111	1111	: : : :	: : : :	: : : :	: : : :	: : : :	1111
A108	1050	CW	i	ŧ	i	Bar and bolt	(1).(g)	862	965	S S S S S S S S S S S S S S S S S S S	322 276 862 965	322 276 276 3846 965	322 3 276 2 813 7 965 9	322 32 276 27 784 77 965 96	322 32 276 25 773 76 965 96	322 322 276 276 762 751 965 965	2 322 6 276 1 740 5 965	1 1 1 1	: : : :	1 1 1 1	: : : :	: : : :	: : : :	: : : :	: : : :	1111
A108	1117	i.	i	ŧ	:	Bar	(1), (31)	2414 2414 2414	414 483	S S S S S S S S S S S S S S S S S S S	161 138 414 483	161 138 406 483	161 1 138 1 390 3 483 4	161 16 138 13 381 38 483 48	161 161 138 138 381 381 483 483	1 161 8 138 1 380 3 483	1 161 8 138 0 380 3 483	1111	: : : :	: : : :	: : : :	: : : :	: : : :	: : : :	: : : :	1111
A108	1141, 12L14	Cold drawn and tempered	:	ŧ	:	Bolt 75 mm max.	(1), (31)	558	425	S S S	 558 724	 548 724	 527 5 724 7	 508 50 724 72	 501 49 724 72	 494 486 724 724	 6 479 4 724	1111	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	: : : :	1111
A108	1144	:	i	ŧ	ŧ	Bar	(1), (31)	689	793	S. S. S.	264 227 689 793	264 227 677 677 793	264 2 227 2 650 6 793 7	264 26 227 22 627 61 793 79	264 264 227 227 618 610 793 793	7 227 0 601 3 793	4 264 7 227 1 592 3 793	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	: : : :	1 1 1 1	: : : :	1111
A108	1144	Cold drawn and tempered	÷	ŧ	i	Bolt 75 mm max.	(1), (31)	724	862	S S S	 724 862	7111 6	683	659 64 862 86	 649 64 862 86	 640 630 862 862	 80 621 52 862	1 1 1 1	1 1 1 1	1 1 1 1	: : : :	: : : :	: : : :	: : : :	: : : :	1111
A108	1144	i	i	i	÷	Bolt over 75 mm to 250 mm max.	(1), (31)	558	724	S S S S S S S S S S S S S S S S S S S	 558 724	 548 724	 527 5	 508 508 57	501	 494 486 724 724	 16 479 14 724	1 1 1 1	: : : :	1 1 1 1	1 1 1 1	: : : :	: : : :	::::	: : : :	1111
A108	12L14	i	i	ŧ	E	Bar	(1)	379	448	S. S. S.	150 128 379 448	150 128 372 448	150 1 128 1 358 3 448 4	150 15 128 12 345 34	150 15 128 12 340 33 448 44	150 150 128 128 335 330 448 448	150 150 128 128 330 326 448 448	1111	1 1 1 1	1 1 1 1	1 1 1 1	: : : :	: : : :	1111	: : : :	
A228	:	:	i	ŧ	ŧ	Wire	(1), (6), (30)	1724	1862	S <sub>m</sub> S <sub>v</sub> 1		 681 15 862 18	 592 15 362 18	 09 1453 62 1862	 53 1396 52 1862	 16 134 12 186	 1349 1302 1862 1862			1111	1 1 1 1	1 1 1 1	: : : :	1 1 1 1	: : : :	
A381	:	i	Y35	P-No. 1	#	Pipe	ŧ	240	415	S S S S S S S S S S S S S S S S S S S	 240 415	236 2	 227 2 415 4	 219 21 415 41	 217 21 415 41	 214 211 415 415	 1 207 5 415	200	198	<b>₩</b>	: : : : :	178	176	1 1 1 1	: : : :	: : : :

Second   S		Design S	Design Stress Intensity (S <sub>m</sub> ), Maxim	ensit	y (S <sub>m</sub> ),	, Ma	ximum A	\llowable	Table NF-I-2100-1M um Allowable Stress (S), Yield Strength ( $S_{m  u}$ ), and Tensile Strength ( $S_{m u}$ ) Values (Cont'd)	Table NF-I-2100-1M tress (S), Yield Stre	-1M Stre	ngth	(S)	,), a		ens	le S	tre	l gth	(S <sub>u</sub>	%	alue	) s	Cont	Ę.		
Second Control   Control		Topic and Marketine	To State	() 	9,6	,			Min. Yield	Min. Tensile						for Met	Stress al Tem	and St	rength,	MPa, Not to	Exceed						ı
B	No. Carbon	Grade or AISI Steels (Cont'd)		Class	\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	No.	Form	Notes	Surengui, MPa	MPa MPa			1 1	11	125	150	175	200	225	250	275 3	008	1 1	1 1	1 1	1 1	52
Part	A500	ш	i	Ē	P-No. 1	The	Tube (shaped)	į	31.7	400									 263 	 261 				223	1111	1 1 1 1	1111
C PANG I I Tuble  C PANG I I T	A500	ш	÷	i	P-No. 1	1	Tube	્	290	400	S, S	::000							240	238				502	: : : :	: : : :	1111
C PMo. I Tube	A500	O	÷	i	P-No. 1	1	Tube (shaped)	MPDY	345	427	S, S	45							286	283				.: 520	: : : :	1111	1111
1015 CW	A500	O	÷	i	P-No. 1	1	Tube (round)	:	Or 317	427	S <sub>m</sub> S <sub>y</sub> ::	17							263	 261 				223	: : : :	: : : :	
1015 CW PNo.1 1 Tube	A501	ŧ	÷	i	P-No. 1	1	Tube	÷	250	400	S, S, ,	: :: 20							206	204	196	188		:: ::80	: : : :	: : : :	1111
1020 CW P-No. 1 2 Tube (4), (20) 414 483 S <sub>n</sub> 115 115 115 115 115 115 115 115 115 11	A513	1015	CW	i	P-No. 1	Ħ	Tube	(4), (29)	379	448	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	115 1 98.6 21 2 45 3	6 64 63	6 (4 (6)	3 7 7	9 1	3 7 7			1 1 1 1	: : : :	1111	: : : :	1 1 1 1	1 1 1 1	::::	
1015 CW Tube (2), (4) 379 448 5, 150 150 150 150 150 150 150 150 150	A513	1020	CW	i	P-No. 1	7	Tube	(4), (29)		483			.5 11. 8.6 99 15 34		9			115 6 98. 189 345		1 1 1 1	: : : :	1111	: : : :	1 1 1 1	1111	1111	1111
1020 CW Tube (2), (4) 414 483 5, 161 161 161 161 161 161 161 161 161 16	A513	1015	CW	i	i	i	Tube	(2), (4)	379	448				11					1 1 1 1	1 1 1 1	: : : :	: : : :	: : : :	1 1 1 1	: : : :	: : : :	1111
1025 CW P-No. 1 2 Tube (5), (29) 448 517 5, 126 126 126 126 126 126 126 126 126 126	A513	1020	CW	i	ŧ	i	Tube	(2), (4)	414	483					•				1 1 1 1	1111	: : : :	1111	: : : :	1111	1111	: : : :	
1026 CW P-No.1 3 Tube (5), (29) 483 552 S <sub>m</sub> 126 126 126 126 126 126 126 126	A513	1025	CW	ŧ	P-No. 1	7	Tube	(5), (29)	448	517								( )	C-0	1111	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1 1 1 1	1111
	A513	1026	CW	i	P-No. 1	ю	Tube	(5) (5)		552										30%	:::6		: : : :	: : : :	1111	1111	1 1 1 1

# ASME BPVC.III.1.NF-2017

Table NF-1-2100-1M  Design Stress (Increase)  See - Material Stress (Increase)  See - Material Control Design Stress (Increase (Increase)  See - Material Control Design Stress (Increas		_	_											
Table NFI-1200-1M   Street of Type   S			1 1					1111	1111	1 1 1 1	1 1 1 1	1111	1 1 1 1	1 1 1 1
Design Stress Intensity (S_m), Maxim    Carde or AlSi			1 1											
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	l t													
Design Stress Intensity (S_m), Maxim    Carde or AlSi	ပိ		1 1											
Design Stress Intensity (S_m), Maxim    Carde or AlSi	nes													
Design Stress Intensity (S_m), Maxim    Carde or AlSi	Val	g	300											
Design Stress Intensity (S_m), Maxim    Carde or AlSi	(")	l, to Exce	275											
Design Stress Intensity (S_m), Maxim    Carde or AlSi	) u	h, MPa	250											
Design Stress Intensity (S_m), Maxim    Carade or AlSi   Type   Class   No.   Forgit	ngt	trengt ires, °(	225											
Design Stress Intensity (S_m), Maxim    Crade or AlSi   Type   Crass   No.   No.   No.     1025   1026   CW     P.No. 1   1   Tube     1018, 1020, 1022   CW     P.No. 1   1   Tube     1025, 1026   CW     P.No. 1   2   Tube     1020, 1025, 1030     CC   S.No. 1   1   Forgit     1020, 1025, 1030     CG   S.No. 1   2   Forgit     1035, 1040     CG         Forgit     1035, 1040     CG         Forgit     1035, 1040     CG         Forgit     1035, 1040     CG               1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG                     1035, 1040     CG	) tre	and S	500			115 3.6 98 189 345	115 3.6 98 189 345							
Design Stress Intensity (S_m), Maxim    Crade or AlSi   Type   Crass   No.   No.   No.     1025   1026   CW     P.No. 1   1   Tube     1018, 1020, 1022   CW     P.No. 1   1   Tube     1025, 1026   CW     P.No. 1   2   Tube     1020, 1025, 1030     CC   S.No. 1   1   Forgit     1020, 1025, 1030     CG   S.No. 1   2   Forgit     1035, 1040     CG         Forgit     1035, 1040     CG         Forgit     1035, 1040     CG         Forgit     1035, 1040     CG               1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG                     1035, 1040     CG	le 3	Stress al Tem	175				9						$\sim$	
Design Stress Intensity (S_m), Maxim    Crade or AlSi   Type   Crass   No.   No.   No.     1025   1026   CW     P.No.   1   Tube     1018, 1020, 1022   CW     P.No.   1   Tube     1025, 1026   CW     P.No.   2   Tube     1025, 1026   CW     P.No.   1   Forgin     1020, 1025, 1030     CE   S.No.   2   Forgin     1035, 1040     CG     Rengin     1035, 1040     CG     Rengin     1035, 1040     CG       Forgin     1035, 1040     CG         Forgin     1035, 1040     CG         Forgin     1035, 1040     CG         Forgin     1035, 1040     CG             1035, 1040     CG               1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG                     1035, 1040     CG	ensi	r Meta	150	172 148 403 517	184 158 427 552			161 138 365 483	126 108 214 379	126 108 214 379	172 148 403 517	 1183 415	226	335
Design Stress Intensity (S_m), Maxim    Carde or AlSi	Ĕ	4	125	172 148 406 517	184 158 433 552		T T C	161 138 371 483	126 108 217 379	126 108 217 379	172 148 406 517		 229 520	 340 620
Design Stress Intensity (S_m), Maxim    Carde or AlSi	a a		100	172 148 408 517	184 158 439 552		- (4 t)	161 138 376 483	126 108 219 379	126 108 219 379	172 148 408 517	 1188 415	 232 520	 345 620
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Cass   No.   F.     1025   1026   CW     P-No. 1   1   Tube     1018, 1020, 1022   CW     P-No. 1   1   Tube     1025, 1026   CW     P-No. 1   1   Tube     1025, 1026   CW     P-No. 1   2   Tube     1025, 1026   CW     P-No. 1   2   Tube     1025, 1026   CW     P-No. 1   2   Tube     1020, 1025, 1030     CC   S-No. 1   1   Forgin     1030, 1025, 1030     CG   S-No. 1   2   Forgin     1031, 1040     CG       Forgin     1031, 1040     CG       Forgin	ا (پرS			172 148 423 517	184 158 455 552	115 98.6 208 345	- (4()	161 138 390 483	126 108 227 379	126 108 227 379	172 148 423 517	 194 415	 240 520	 358 620
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Cass   No.   F.     1025   1026   CW     P-No. 1   1   Tube     1018, 1020, 1022   CW     P-No. 1   1   Tube     1025, 1026   CW     P-No. 1   1   Tube     1025, 1026   CW     P-No. 1   2   Tube     1025, 1026   CW     P-No. 1   2   Tube     1025, 1026   CW     P-No. 1   2   Tube     1020, 1025, 1030     CC   S-No. 1   1   Forgin     1030, 1025, 1030     CG   S-No. 1   2   Forgin     1031, 1040     CG       Forgin     1031, 1040     CG       Forgin	=		20	172 148 440 517	184 158 474 552	115 98.6 217 345	- 1413	161 138 406 483	126 108 237 379	126 108 237 879	172 148 440 517	 201 415	 250 520	 373 620
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Cass   No.   F.     1025   1026   CW     P-No. 1   1   Tube     1018, 1020, 1022   CW     P-No. 1   1   Tube     1025, 1026   CW     P-No. 1   1   Tube     1025, 1026   CW     P-No. 1   2   Tube     1025, 1026   CW     P-No. 1   2   Tube     1025, 1026   CW     P-No. 1   2   Tube     1020, 1025, 1030     CC   S-No. 1   1   Forgin     1030, 1025, 1030     CG   S-No. 1   2   Forgin     1031, 1040     CG       Forgin     1031, 1040     CG       Forgin	- Bud,		38	172 148 448 517	184 158 483 552	9	115 98.6 221 345	161 138 414 483	126 108 241 379	126 108 241 379	172 148 448 517	 205 415	 255 520	380
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	-1N Str			S S S 2	$S_m$ $S_y$	$S_m$	S S S S S S S S S S S S S S S S S S S	$S_m$	2 S S	~ v v, v,	S S y S	S . S	$S_m$ $S_y$	S S S S S
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	) 100 eld	ensile	a iii	17	22	45	83	83	827	17	17	15	20	20
Design Stress Intensity (S_m), Maxim    Carde or AlSi	`; ;	Min. T	MI	rv	ហ	m	4	4		ın	ro	4	rv	9
Design Stress Intensity (S_m), Maxim    Carde or AlSi	N (S)							CM						
Design Stress Intensity (S_m), Maxim    Carde or AlSi	able	lin. Yie	MPa	448	483	221	414	4	241	448	448	205	255	380
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	Sti	Σ.	<u> </u>				10							
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	- aple		otes	(2)	(5)		2	4	i	5), (29)	(3), (5)	31)	31)	(31)
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	OW		ž	(2)	(2)		4	[2]		(3)	(2), (	2	<u> </u>	Ξ)
Design Stress Intensity (S_m), Maxim    Carde or AlSi   Type   Case   No.   No.   No.     1025   1026   CW     P-No.   1   Tube     1018, 1020, 1022   CW     P-No.   1   Tube     1025, 1026   CW     P-No.   1   Tube     1020, 1025, 1030     CC   S-No.   1   Forgin     1030, 1025, 1030     CG   S-No.   2   Forgin     1031, 1040     CG     P-No.   1   Forgin     1035, 1040     CG       P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG         P-Rogin     1035, 1040     CG               1035, 1040     CG                 1035, 1040     CG	All	Ι.	.			ENI.								pun
Design Stress Intensity (S_m), Maxim    Crade or AlSi   Type   Crass   No.   No.   No.     1025   1026   CW     P.No.   1   Tube     1018, 1020, 1022   CW     P.No.   1   Tube     1025, 1026   CW     P.No.   1   Tube     1020, 1025, 1030     CC   S.No.   1   Forgin     1030, 1025, 1030     CG   S.No.   2   Forgin     1031, 1040     CG     Rengin     1031, 1040     CG       Rengin     1031, 1040     CG       Rengin     1031, 1040     CG       Rengin     1031, 1040     CG       Rengin     1031, 1040     CG       Rengin     1031, 1040     CG       Rengin     1031, 1040     CG       Rengin     1031, 1040     CG         Rengin     1031, 1040     CG         Rengin     1031, 1040     CG         Rengin     1031, 1040     CG         Rengin     1031, 1040     CG         Rengin     1031, 1040     CG             1031, 1040     CG               1031, 1040     CG               1031, 1040     CG                 1031, 1040     CG                 1031, 1040     CG	E		Form		1/4	S						po po	ρū	ıg O mm ö der
Design Stress Intensity (S_m), N    Material   Condition   Type   Clade or AlSi   Type   Clade or AlSi   Type   Clade or AlSi   Clade   Clade or AlSi   Clade   Clade	ĸ	'	`	Tube	Jube	Tube	Tube	Tube	Tube	Tube	Tube	Forgin	Forgin	Forgin 100 unc
	<b>⊠</b>		No.	: 3	No.	<b></b>	7	:	1					
	m),	١.	ا ہ	10		1.	1.		1.	1.		<del>.</del> .	Ħ.	
	(S	}	ŽŽ	CA.	:	P-No	P-Nc	i	P-Nc	P-Nc	i	S-No	S-No	:
	  sity		Sal	:	:	:	:	i	i	÷	i	S	CE	50
	te	1												
			Type	CW	CW	HR	CW	CW	HR	CW	CW	:	:	:
	S E	3	AISI			), 1022	), 1022	), 1022	920	920	920	6, 1030	5, 1030	040
	  esig		rade or	1025	1026	8, 102(	8, 102(	8, 102(	025, 1	.025, 1	.025, 1	0, 1025	0, 1025	.035, 1
			Gr Steels			101	101	101	1	т	Т	102	102	7
NS1: NS1: NS1: NS1: NS1: NS1: NS1: NS1:			No. Carbon	A513	A513	A519	A519	A519	A519	A519	A519	A521	A521	A521

	Design Str	ess Int	ensit	y (S <sub>m</sub> )		Table NF-I-2100-1M Design Stress Intensity ( $S_m$ ), Maximum Allowable Stress ( $S$ ), Yield Strength ( $S_y$ ), and Tensile Strength ( $S_u$ ) Values (Cont'd)	llowable	Table NF-I-2100-1M Stress (S), Yield Stre	I-2100 ), Yield		l angt		, (v	pue	Ten	sile	Stre	ingtl	S) h	\ \frac{\circ}{\circ}	alu	) sa	Con	r'd)		
Suga	Material	Condition	O,	9/d		Droding		Min. Yield	Min. Tensile						for M	Stress and Strength, MPa, for Metal Temperatures, °C, Not to Exceed	s and s	Stress and Strength, MPa, Il Temperatures, °C, Not to	ı, MPa, , Not to	Excee	_					<u> </u>
No.	Spec. Grade or AISI Carbon Steels (Cont'd)	Type	Class	ک	No.		Notes	MPa MPa	MPa MPa		38	50 7	75 100	0 125	5 150	1 1	175 200	225	250	275	300	325	350 3	375 4	400 4	425
A521	1035, 1040	į	99		0	Forging over 100 mm to 250 mm incl.	(1), (31)	345	585	S S S 2	345	338 3	 325 314 585 585	31		 305 301 585 585	29	 7 286 5	283	272	262	253	250	1 1 1 1	1 1 1 1	1111
A653	33	∢	i	S-No. 1	П	Sheet	(13)	230	310	s s s s	103 1 88.9 230 2 310 3	103 1 88.9 225 2 310 3	103 103 88.9 88 215 207 310 310	6. 28.	9 3 3	103 103 88.9 88. 201 199 310 310	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	.03 88.9 96 188 110	186	1111	1111	: : : :	1111	1111	1111	1111
A668	1020, 1022, 1025, 1026, 1030	÷	ш	S-No. 1	₩	Forging		205	415	s, s	138 1 119 1 205 2 415 4	138 1 119 1 201 1 415 4	138 13 119 11 194 18 415 41	138 138 119 119 188 186 415 415		138 138 119 119 183 181 415 415	138 138 119 119 181 178 415 415	8 138 9 8 172 5	138  170	138  164	138 157	138  152	150	1111	1111	1111
A668	1035	i	ш	i	i	. Forging	(1), (8), (9), (31)	205	415	S <sub>m</sub> S <sub>y</sub>	 205 2 415 4	 201 1 415 4	  194 18 415 41	 188 186 415 415		 183 181 415 415	17	 8 172 5	171	165	157	152	150	1 1 1 1	::::	1111
A668	1020, 1022, 1025, 1026, 1030	÷	C	S-No. 1	1	Forging	(8), (31)	230	455	S S S S S S S S S S S S S S S S S S S	152 130 230 2455	152 1 130 1 225 2 455 4	152 152 130 130 215 207 455 455	152 152 130 130 207 204 455 455		152 152 130 130 201 199 455 455	152 152 130 130 199 196 455 455	2 152 0 6 188 5	152  186	152  179	152 172	152  168	166	1111	1 1 1 1	1 1 1 1
A668	1035	i	C	i	i	. Forging	(1), (8), (9), (31)	230	455	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	230 2	 225 2 455 4	 215 21 455 45	 210 210 455 455		 210 210 455 455	212		186	 179 	172	168	166	1 1 1 1	1 1 1 1	1111
A668	1020, 1022, 1025, 1026, 1030	÷	Ω	S-No. 1	2	Forging	(8), (31)	260	515	2 S S S S S S S S S S S S S S S S S S S	172 147 260 515	172 1 147 1 255 2 515 5	172 172 147 147 244 235 515 515	172 172 147 147 235 232 515 515		172 172 147 147 229 225 515 515	72 172 17 147 15 222 15 515	2 172 7 2 214 5	172 212	172  204	172  196	172  190	187	1 1 1 1	1 1 1 1	1111
A668	1035	÷	Ω	ŧ	į	. Forging	(1), (8), (9), (31)	260	515	S S S S S S S S S S S S S S S S S S S	 260 2 515	 255 2 515 5	244 23 515 51	 235 232 515 515		 229 225 515 515	22	 2 214 5	212	204	196	190	187	1111	1 1 1 1	::::
A668	1030, 1330	÷	í±.	S-No. 1	т	Forging 100 mm and under	(7), (8), (31)	380	620	S S S S S S S S S S S S S S S S S S S	207 2 1177 1 380 3	207 2 177 1 373 3 620 6	207 20 177 17 358 34 620 62	207 207 177 177 345 340 620 620	( ,	207 207 177 177 335 330 620 620	207 207 177 177 330 326 620 620	7 7 6 315 0	312	300	588	 279 	27.5	1111	1 1 1 1	1111
A668	1030, 1330	į	ít.	S-No. 1	т	Forging over 100 mm to 250 mm incl.	(8), (31)	345	585	S S S S S S S S S S S S S S S S S S S	195 1 167 1 345 3	195 1 167 1 338 3 585 5	195 19 167 16 325 31 585 58	195 195 167 167 314 310 585 585		195 195 167 167 305 301 585 585	195 195 167 167 301 297 385 585	25 25 286	283	272	262	253	250	1111	1111	1111
A668	1035, 1040 1045	i	Ĺt.	:	i	. Forging 100 mm and under	(1), (7), (8), (9), (31)	380	620	S S S S S S S S S S S S S S S S S S S	380 3	 373 3 620 6	 358 345 620 620	 15 340 20 620		 335 330 620 620	32	315	, Og	300		279	275	1 1 1 1	1 1 1 1	1 1 1 1

	Design St	ress	tensit	.y (S,	, . N	Design Stress Intensity (S <sub>m</sub> ), Maximum Allowable	llowable	Table NF-I-2100-1M Strength $(S_y)$ , and Tensile Strength $(S_u)$ Values (Cont'd)	Table NF-I-2100-1M itress (S), Yield Stre	)-1M Stre	ingt	) h	\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	pu.	len:	ile	Stre	ngtl	(S)	\ \frac{1}{2}	alue	) sa	Coni	ŢĠ		
3043	Motouiol	Condition		9,4		topond		Min. Yield	Min. Tensile						for Me	Stress and Strength, MPa, for Metal Temperatures, °C, Not to Exceed	and S	Stress and Strength, MPa, I Temperatures, °C, Not to	, MPa, Not to	Exceed	_					ı
No.	No. Grade or AISI	Type	Class	No.		No. Form	Notes	мРа МРа	MPa MPa		38 5	50 75	2 100	0 125	150	0 175	200	225	250	275	300	325	350 3	375 4	400 47	425
Carbon	steels (Cont a)			(0)																						
A668	1035, 1040, 1045	ŧ	Ľ.	F.	x0	Forging over 100 mm to 250 mm incl.	(1), (8), (9), (31)	345	585	S, S	345 3	338 33	 325 314 585 585	 .4 310 35 585	305	 5 301 5 585	 1 297 5 585	286	283	272	262	253	250	: : : :	: : : :	
A675	75	i	i	S-No. 1		ZZ ZZ	(28)	260	515	S, S, S,	172 1 147 1 260 2 515 5	172 17 147 14 255 24 515 55	172 172 147 147 244 235 515 515	2 172 7 147 5 232 5 515	172 7 147 2 229 5 515	2 172 7 147 9 225 5 515	2 172 7 147 5 222 5 515	172 214	172 212	172 204	172 196	172 190	188	1111	: : : :	
Low Alle A148	Low Alloy Steels A148 Grade 90-60: 4130, 4320, 4330	ŧ	i	S-No. 4		3 Casting	(11), (19),	415	620	S S S S S S S S S S S S S S S S S S S	207 2 177 1 415 4 620 6		207 207 177 177 406 400 620 620	7 207 7 177 0 393 0 620	7 207 7 177 3 385 0 620	7 207 7 177 5 376 0 620	7 207 7 177 6 366 0 620	207 177 353 620	207 177 351 620	207 177 345 620	207 177 344 620	207 620	207 620	::::	: : : :	
A148	Grade 105-85: 4130, 4140, 4320, 4340	÷	i	i		Casting	(1). (7). (8)		725	S S S 2	241 2 207 2 585 5 725 7	241 2/ 207 20 582 5/ 725 7/	241 241 207 207 574 567 725 725	1 241 7 207 7 556 5 725	1 241 7 207 5 546 5 725	1 241 7 207 6 532 5 725	1 241 7 207 2 518 5 725	241 207 501 725	241 207 497 725	241 207 489 725	241 207 487 725	241 207 486 725	241 207 486 725	1111	: : : :	
A148	Grade 115-95: 4130, 4140, 4320, 4340	i	ŧ	i		Casting	(1), (18), (31)	ES IN	795	S <sub>w</sub>	265 2 227 2 655 6 795 7	265 20 227 23 651 64	265 265 227 227 642 633 795 795	5 265 7 227 3 621 5 795	265 7 227 1 610 5 795	5 265 7 227 0 595 5 795	5 265 7 227 5 579 5 795	265 227 559 795	265 227 555 795	265 227 546 795	265 227 545 795	265 227 544 795	265 227 544 795	: : : :	: : : :	
A148	Grade 135-125: 4130, 4140, 4320, 4340	:	i	i		Casting	(1), (31)	860	BRY	S S S S	310 3 266 2 860 8 930 9	310 3. 266 24 855 84 930 99	310 310 266 266 844 832 930 930	0 310 6 266 2 818 0 930	310 5 266 8 803 930	0 310 6 266 3 783 0 930	0 310 6 266 3 763 0 930	310 266 736 930	310 266 730 930	310 266 718 930	310 266 717 930	310 266 716 930	310 266 716 930	1111	: : : :	
A322	4150	:	i	i	•	Bar	(1), (31)	689	793	\$ 00 00° 00°	793 6	680 66	 660 643 793 793	  13 632 13 793	 2 622 3 793	19	 4 607 3 793	595	 592 793	 580 793	 568 793	 555 793	 540 793	: : : :	: : : :	
A434	4130, 4140, 4145, 4320, 4340, 8260	÷	BB	i	•	Bar 38 mm and under	(1), (31)	620	760	S S S S S S S S S S S S S S S S S S S	 620 6 760 7	612 59	 594 578 760 760	 78 569 760 760	36	 51 553 50 760	 3 546 0 760	535	 533 760	 522 760	 512 760	200	 486 760	1 1 1 1	: : : :	1111
A434	4130, 4140, 4145, 4320, 4340, 8260	ŧ	BB	ŧ	•	Bar over 38 mm to 63 mm incl.	(1), (31)	550	720	S S S S S S S S S S S S S S S S S S S	 550 5 720 7	 543 5.77	 528 514 720 720	4 4 5 6 6 7 2 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	49	 18 492 10 720	 2 486 0 720	476	 474 720	 464 720	 454 720	 444 720	 43.1 72.0	1 1 1 1	1 1 1 1	1111
A434	4130, 4140, 4145, 4320, 4340, 8260	ŧ	BB	ŧ	•	Bar over 63 mm to 100 mm incl.	(1), (31)	520	069	S S S S S S S S S S S S S S S S S S S	 520 690 6	 512 46 690 69	 496 482 690 690	47	44.7 0 690	0 4 8		446	 444 690	 435 690	 426 690	 416 690	 405 690	1 1 1 1	: : : :	
A434	4130, 4140, 4145, 4320, 4340, 8260	:	BB	:		Bar over 100 mm to 175 mm incl.	(1), (31)	520	099	S S S S S S S S S S S S S S S S S S S	520 5	 512 49 660 60	 496 482 660 660	 12 474 10 660	96	46		C (499	:: 444 660	 435 660	 426 660	416	405	1 1 1 1	: : : :	: : : :

Crade or AISI   Type   Class   W6   No.   Form   Notes   MFa	Spec.		ress Int	ensity	(S <sub>m</sub> ),	Ma G	Design Stress Intensity $(S_m)$ , Maximum Allowable	T wable St	\(\frac{1}{2}\)	I-2100 ), Yield 3	-1M Stre	ngth	(S <sub>y</sub>	), aı	Pe   "	Tensile Strength (Su) V.  Stress and Strength, MPa.  for Metal Temperatures, "C, Not to Exceed	le Strength (S	tren	igth	(S <sub>u</sub>	Va Va	Ine	ا ای	ontí		
1130 4444445	No. Low All	Grade or AISI loy Steels (Cont'd)	Type	Class	×(	No.	Form		MPa	MPa	8			100	125	150	175									
4134 4444 4456	A434	4130, 4140, 4145, 4320, 4340, 8260		BB	<b>)</b>	ije	Bar over 175 mm to 238 mm find.	(1), (31)	450	620						 404 620	 399 620	 395 620	 387 620							
4130,4140,4145, BE Bit	A434	4130, 4140, 4145, 4320, 4340	:	BC	ŧ	ŧ	Bar 38 mm and under control	(1), (31)	760	006						 685 900	 677 900	 	 654 900							
4130,4440,4445 BC Bar over 15 mm bat look 10,911	A434	4130, 4140, 4145, 4320, 4340	i.	ВС	i	i	Bar over 38 mm to 63 mm incl.		720	860						 654 860	 645 860	 637 860	 625 860							
4130,4140,4145, BC Bur (1), (31) S59 7720 S, Bur (1), (31) S	A434	4130, 4140, 4145, 4320, 4340	i	ВС	ŧ	ŧ	Bar over 63 mm to 100 mm incl.	0,								 589 790	 581 790	 574 790	 563 790							
4130,4140,4145	A434	4130, 4140, 4145, 4320, 4340	ŧ	BC	ij	i	5 Z 2		290							 530 760	 523 760	 516 760	 506 760							
4130,4140,4145, BD Bar maid and sover 100 mm incl.  4320,4340  4320,4	A434	4130, 4140, 4145, 4320, 4340	÷	BC	i	i	Bar over 175 mm to 238 mm incl.	(1), (31)	550	720	~ /					 498 720	 492 720	 486 720	 476 720							
4320, 4340  4320,	A434	4130, 4140, 4145, 4320, 4340	i	BD	i	i	Bar 38 mm and under	(1), (31)	006	1070				 835 1070	 822 1070	 808 1 070	 798 070	 789 1 070	₩.	 770 070 1.0	  754 7 070 10	 38 72 70 100	-			
4320, 4146, 4145, BD Bar over 63 mm to over 175 mm incl.  4320, 4146, 4145, BD Bar incl.  4320, 4146, 4145, Bar incl.  4320, 4146, 4146, 4146, Bar incl.  4320, 4146, 4146, 41	A434	4130, 4140, 4145, 4320, 4340	i	BD	i	i	Bar over 38 mm to 63 mm incl.	(1), (31)	830	1 030		 30 813 30 1 03	 3 793 0 1030	1030		 747 1030	 737 1030	 727 1 030	 713 1 030 1	 710 030 1 (	  595 6 130 10	 81 66 30 103				
4320,4145, BD Bar over 100 mm to 238 m	A434	4130, 4140, 4145, 4320, 4340	:	BD	ŧ	ŧ	Bar over 63 mm to 100 mm incl.	(1), (31)	760	096						960	 677 960	  	 654 960							
4320, 4340 BD Bar (1), (31) 690 900 5 <sub>n</sub>	A434	4130, 4140, 4145, 4320, 4340	:	BD	i	ŧ	Bar over 100 mm to 175 mm incl.	(1), (31)	720	930						 654 930	645	930	625							
	A434	4130, 4140, 4145, 4320, 4340	:	BD	i	i	Bar over 175 mm to 238 mm incl.	(1), (31)	069	006		ı		ı		 622 900	 614 900	607	595	$\sim$						

# ASME BPVC.III.1.NF-2017

		425											
ਉ		2 400	1111		1111								1 1 1
ont'		0 375	 525 690	 584 722	643	 701 854	 760 920	986				251 216 579	251 216 579
၁		5 350	 532 52 699 69	 592 58 732 72	  651 64 799 78	 710 70 865 85	 769 76 932 92	 798 78 99 899	7	7	-	253 25 216 21 587 57 758 75	253 25 216 21 587 57
<u>I</u> ne		00 325	 538 53 705 69	 597 59 738 73	 657 68 806 79	 717 7. 873 80	 777 76 940 93	 805 79	-	-		253 23 216 23 593 58 758 73	253 22 216 2 593 58
Va	ceed	275 300	  542 5: 710 71	 602 5	 662 61 811 81	 723 7 878 8	 783 77 946 9	 812 8 013 10	 872 865 081 1 074	-	 54 7. 34 10.	253 21 216 2: 600 5: 758 7:	253 2: 216 2:
(S <sub>u</sub> )	Pa, t to Ex	250 27	 547 5 714 7	608 6	 669 6 815 8	 730 7 883 8	 790 7 951 9	 820 8 1 019 1 0	 881 8 1 087 1 0	 942 933 1155 1148	 70 7 34 10	253 2 216 2 608 6 758 7	253 2 216 2 608 6
퇸	Stress and Strength, MPa, Il Temperatures, °C, Not to	25 2	 549 5 714 7	 610 6 748 7	 670 6 816 8	 731 7 884 8	7792 7	 822 8 1 020 1 0	 883 8 1088 10	 944 9 11156 11	  773 7	253 2 216 2 610 6 758 7	253 2
eng	1 Stren atures,	200 225	 555 5	 616 6 752 7	 678 6 820 8	 740 7 889 8	 801 7 956 9	 831 8 1025 10	 893 8 1093 10	 954 9 1161 11	 .89 7 34 10	253 2 216 2 621 6 758 7	253 2 216 (2
Str	ess and	175 2	560 5	623 6	685 6	 747 7 894 8	 810 8 961 9	 840 1030 10	 902 8 1098 10	 964 9 1167 11	 798 7 34 10	253 2 216 2 629 6 758 7	253 2
Table NF-I-2100-1M um Allowable Stress (S), Yield Strength ( $S_{ u}$ ), and Tensile Strength ( $S_{ u}$ ) Values (Cont'd)	Stress and Strength, MPa, for Metal Temperatures, °C, Not to Exceed	150 1	566 5	629 6	 692 6 830 8	755	 818 8	 848 1034 10	 911 9 1103 10	 974 9 1172 11	_	253 216 638 758	253 2
Ten	for M	5 13						-		-	(F.	•	
and Tu		125	 3 575 5 725	 18 638 10 760	 3 703 0 830	 8 766 0 900	 12 830 55 965	103	 89 925 13 1103	98	896 884 858 835 822 1034 1034 1034 1034	3 253 6 216 7 647 8 758	3 253 6 216
, (v		100	 7 583 5 725	97	 1 713 0 830	 96 778 00 900	96	1 03	93		58 58 84 103	3 253 6 216 9 657 8 758	3 253 6 216
. (S		75	  .3 597 .5 725	 681 664 760 760	 11 731 10 830	5. 6	88	 9 895 4 1 034	 88 962 33 1103	 1069 1056 1028 1172 1172 1172	 34 85 34 103	3 253 6 216 13 669 18 758	253 253 216 216
ıgt		3 20	 620 613 725 725	 690 68 760 76	 760 751 830 830	 825 815 900 900	 895 884 965 965	 931 919 1034 1034		 59 105 72 117	 36 88 34 103	253 253 216 216 689 683 758 758	253 25
IM tre		38	S. S. 62 S. 72	S. S. 69	S, 76 S, 76 S, 83	, E S (A, E	, E S , A, L	. E S . A . B	S S Sy. 1000 Su. 1103	$S_m$ $S$ $S_y$ $S_y$ $S_u$ $S_u$ $S_u$	S <sub>m</sub> S <sub>y</sub> 89 S <sub>u</sub> 103	S <sub>m</sub> 25 S 21 S <sub>y</sub> 68 S <sub>u</sub> 75	S <sub>m</sub> 25
-00 P1	sile	!   ≟				8 4 8 8	8 8 8	2,28					
-21  ≺ie	Min. Tensile	MPa	725	760	830	006	895 965 8	\$	1 103	1 172	1 034	758	758
Table NF-I-2100-1M tress (S), Yield Stre							N						
ble ess (	Min. Yield	MPa	620	069	760	825	P.S.	931	1 000	1 069	968	689	689
_ Stre	Min	F .				, 0	, ,			<del>(</del> -)			
ble		Notes	(1), (10)	(1), (10)	(1) (10) 760	(3)	(10)	(1), (10)	(1), (10)	(10)	<u>(T)</u>	i	ŧ
owa		No	(1)	(1)	Ê	(1)	(1)	(1)	(1)	(1), (10)	E	•	•
All					FULL							p	P
E	1	Form	pro	ging X	<b>O</b>	គ្	p0	ēō.	ρ.	<b>8</b>		mm and s	mm and
× Ë	-		Forgin	Forging	Forging	Forgin	Forgin	Forgin	Forgin	Forgin	Bolt	Plate 32 mi less	Plate
Ma	į	Š Š	1	16	÷	i	i	i	÷	i	:	н	4
(m)	١	No.	1, XO	į	:	:	:	i	i	i	ŧ	P-No. 11B	P-No. 11B
S) >	-	Z	CK	·	•	·	·	•	•	•	•	P-No	P-No
nsit		Class	71	ю	4	ιn	9	<b>^</b>	80	6	:	÷	:
<u>Inte</u>													
SSS	on difficult	Type	i	i	÷	i	i	i	÷	i	1	÷	÷
Design Stress Intensity (S <sub>m</sub> ), Maxim			` >	5	5	5	5	5	>	5	42, 0		
ign	- I cinc	or AIS	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	3Ni-Cr-Mo-V	7, 4140, 4142 4145, 4340	<b>4</b>	В
Des	3	Grade or AISI Steels (Cont'd)	3Ni-C	3Ni-C	3Ni-C	3Ni-C	3Ni-C	3Ni-C	3Ni-C	3Ni-C	4137, 4140, 4142, 4145, 4340		
		No. Grade or AISI Low Alloy Steels (Cont'd)	1	T.	Ē.	T.	1.	F.	Ħ	T.		4	4
	3	5 Z	A471	A471	A471	A471	A471	A471	A471	A471	A490	A514	A514

			425	1111	: : : :	1111	1111	1111	1111	1111	1111	1111	1111	: : : :
			400	1 1 1 1	: : : :	1111	1 1 1 1	::::	: : : :	::::	: : : :	::::	::::	: : : :
	ıt'd		375	1111	1111	1111	1111	1111	1111	1111	1111	1111	1111	: : : :
	[Co		320	251 216 579 755	251 216 579 755	251 216 579 755	251 216 579 755	228 196 522 687	228 196 522 687	228 196 522 687	1111	1111	138 118 224 414	150 128 267 448
	es		325	253 216 587 758	253 216 587 758	253 216 587 758	253 216 587 758	229 197 528 689	229 197 528 689	229 197 528 689	: : : :	: : : :	138 118 229 414	150 128 273 448
	/alu	l e	300	253 216 593 758	253 216 593 758	253 216 593 758	253 216 593 758	230 197 534 689	230 197 534 689	230 197 534 689	: : : :	1111	138 118 234 414	150 128 279 448
	("	Excee	275	253 216 600 758	253 216 600 758	253 216 600 758	253 216 600 758	230 197 540 689	230 197 540 689	230 197 540 689	1 1 1 1	1111	138 118 240 414	150 128 285 448
	(S	gth, MPa, °C, Not to Exceed	250	253 216 608 758	253 216 608 758	253 216 608 758	253 216 608 758	230 197 547 689	230 197 547 689	230 197 547 689	1 1 1 1	1111	138 118 246 414	150 292 248 448
	ngth			253 216 610 758	253 216 610 758	253 216 610 758	253 216 610 758	230 197 549 689	230 197 549 689	230 197 549 689	1 1 1 1	1111	138 118 247 414	150 128 294 448
	tre	and St	200	253 216 621 758	253 216 621 758	253 216 621 758	253 216 621 758	230 197 559 689	230 197 559 689	230 197 559 689	276 236 607 827	276 236 607 827	138 118 255 414	150 128 303 448
	le S	tress :	175	253 216 629 758	253 216 629 758	253 216 629 758	253 216 629 758	230 197 567 689	230 197 567 689	230 197 567 689	276 236 614 827	276 236 614 827	138 118 259 414	150 128 309 448
	ensil	Stress and Stren for Metal Temperatures,	150	253 216 638 758	253 216 638 758	253 216 638 758	253 216 638 758	230 197 574 689	230 197 574 689	230 197 574 689	276 236 622 827	276 236 622 827	138 118 264 414	150 128 314 448
	) P	] º	125	253 216 647 758	253 216 647 758	253 216 647 758	253 216 647 758	230 197 583 689	230 197 583 689	230 197 583 689	276 236 632 827	276 236 632 827	138 118 269 414	150 128 320 448
	, a		100	253 216 657 758	253 216 657 758	253 216 657 758	253 216 657 758	230 197 591 689	230 197 591 689	230 197 591 689	276 236 643 827	276 236 643 827	138 118 275 414	150 128 326 448
	(, 2)		75	253 216 669 758	253 216 669 758	253 216 669 758	253 216 669 758	230 197 603 689	230 197 603 689	230 197 603 689	276 236 660 827	276 236 660 827	138 118 281 414	150 128 334 448
			20	253 216 683 758	253 216 683 758	253 216 683 758	253 216 683 758	230 197 615 689	230 197 615 689	230 197 615 689	276 236 680 827	276 236 680 827	138 118 287 414	150 128 341 448
	eng /		38	253 216 689 758	253 216 689 758	253 216 689 758	253 216 689 758	230 197 621 689	230 197 621 689	230 197 621 689	276 236 689 827	276 236 689 827	138 118 290 414	150 128 345 448
	0-11   Str			$S_{n}$	$S_m$ $S_y$	$S_m$ $S_y$	S S S N N N N N N N N N N N N N N N N N	$S_m$	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	S S S S S S S S S S S S S S S S S S S	S S S N	$\mathop{\mathcal{S}}_{x_{y}}$	$S_m$	S <sub>m</sub>
	Table NF-I-2100-1M tress (S), Yield Strength ( $S_{ u}$ ) Values (Cont'd)	Min. Tensile	Strength, MPa	758	758	758	758	689	689	689	827	827	414	448
	NF.	1						Mr						
	Table Stress	Min. Yield	MPa	689	689	689	689	621	621	621	689	689	290	345
	ble S		es			n and	O.							
	Design Stress Intensity ( $S_m$ ), Maximum Allowable		Notes	i	•	"6\	i	ŧ	i.	:	(1)	( <del>1</del> )	(14)	(14)
	All	١.	_	pu	ا کی	E .	pu	r 63 mm to mm incl.	late over 63 mm to 150 mm incl.	nm to incl.			es	es
	E		Form	mm and		Ē	Ē	er 63 mm 0 mm incl	er 63 r	ite over 63 mm to 150 mm incl.			d shapes	d Shapes
	xi			2 Plate 63 n	Plate 63 mm and less	Plate 63 r less	Plate 63 1 less	Plate ovel 150	Plate over 150	Plate over 150	Tube	Tube	Plate and	Plate and
	¥a	,	No.	alle	т	∞	6	73	ω	6	i	÷	₩	11
	(m)	پ	2 6	P-No. 118	P-No. 11B	P-No. 11B	P-No. 11B	P-No. 11B	P-No. 11B	P-No. 11B	:	:	P-No. 1	P-No. 1
	S)	~\i		P-No	P-No	P-No	P-No	P-No	P-No	P-No			P-N	P-N
	ensit	<u>(</u>	Class	i	i	i	i	i	i	i	i	i	i	i
į.	OF	100	Type	i	ŧ	i	i	i	:	i	4140SR	4142SR	i	:
	· C	]	J.	•	•	•	•	•		•	414	414	•	
	n St		AISI	nt a)										
2,2,111,21	sign		Grade or AISI	E E	í.	۵.	0	ш	۵	0	4140	4142	42	20
	ے ا		Gra	oy ster										
Y		]	No.	Low Alloy Steels (cont d) A514 E	A514	A514	A514	A514	A514	A514	A519	A519	A572	A572
	I	-			•	•	•	-	•	•	•	•	•	,

Design Street The centure of the control of the con		<b>,</b>																										
The continue of the continue o	ign	<b>₹</b>	ressTh	tensi	ty (S,	1 '("	Maxin	num Allo	T wable Si	rable Nitress (S	F-I-2100 ), Yield	Stre	l engtl	h (S	(^	and	Te T	sile	Stre	engt.	£ €	("9	Vali	ues	္ဌ	nt'd	_	
Page   Care   March	;	1			,						Min. Tensile						for M	Stre	s and	Streng ures. °	th, MPa	t,	l p					
Howard I Private	Material Grade or AISI		Condition	1	6		Grp. No.	Product Form		1	Strength, MPa		П	П	1 1	1 1	5 15	0 17	5 20	0 22	5 250	275	300		1 1	375	1 1	425
The control of the co	Low Alloy Steels (Cont'd)	_		,	jic																							
1.   1.   1.   1.   1.   1.   1.   1.	55		Ē	i	P-No.	_x0	2 Plata		i	379	483	S S S 2														1 1 1 1	: : : :	: : : :
Pake 3   Pake and lower   Pake 3   Pa	5Ni-Cr-Mo-V	_	12a	i	i			ging W	Ξ	965	1 034	S <sub>m</sub> S <sub>y</sub>	-	7	-	7										1111	: : : :	: : : :
Pales   Pale	A, B		ŧ	i	P-No.	m		e and bar 00 mm and ess, all shapes	(7), (15)	345	483	S S S S S S S S S S S S S S S S S S S															: : : :	: : : :
Part	А, В		ŧ	i	P-No.	ε.		Ē,	(A)	317	462	S. S. S.														1111	: : : :	: : : :
Factor   F	А, В		i	i	P-No.	m		E .	(14)	RSM!	434	S. S.														1 1 1 1	: : : :	: : : :
Head   Proper   Proper   Head   Proper   Proper   Head   Proper   Proper   Head   Proper   Head   Proper   Head   Proper   Head   Hea	Mn-Cu-V		i	=	P-No.	Ψ.	Tu	e 8 mm and ess		345		8 × × × ×							2 4							1 1 1 1	: : : :	::::
P-No. 1   Tube	Mn-Cu-V		ŧ	=	P-No.	τ.	Ē	e .ver 18 mm to .8 mm incl.	i	317	462	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~	' '	_												1 1 1 1	: : : :	1 1 1 1
P-No. 1   Plate and shapes     P-No. 1   Plate, bar, and     P-No. 1   Plate, and parameter   P-No. 1	Mn-V		ŧ	Ħ	P-No.	Ħ.		υ.	:	345	448	S S S S S S S S S S S S S S S S S S S	*													1 1 1 1	1 1 1 1	1111
P-No.1 2 Plate and shapes 345 483 5m 161 161 161 161 161 161 161 161 161 16	∀		ŧ	į	P-No.	Ψ.		e and shapes .00 mm and ess	:	290	434	$S_m$														1 1 1 1	: : : :	::::
P-No.1 1 Plate, bar, and 317 448 S <sub>m</sub> 150 150 150 150 150 150 150 150 150 150	C, D		ŧ	÷	P-No.	Ε.		e and shapes 13 mm and ess	:	345	483	$S_m$					•	$Q^{-1}$	. ( 1	/						1 1 1 1	: : : :	1 1 1 1
	C, D		i	:	P-No.	Ξ.	Д.	e, bar, and hapes wer 63 mm to .00 mm incl.	:	317	448	S, S									XV				1	: : : :	: : : :	::::